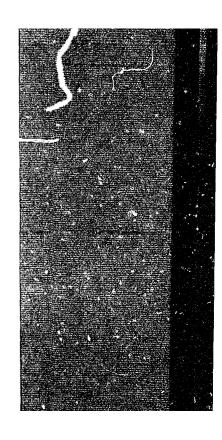
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15 November 1976

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ABSTRACT

The report describes the point design of a 3000-long ton (29,892 kilo-Newton) Surface Effect Ship (SES) that meets the near term requirements of the Office of Advanced Naval Vehicle Concepts Evaluation (ANVCE).

The point design is a weaponized testship, a concept that is a step toward a new class of fully combatant SES's and that is a logical progession from today's technology.

- (U) The SES point design is described in overall terms of General Description, Vehicle Performance, Maneuvering, Range, Payload, Weights, Volumes, Stability, Geometric-Form, and Ride Quality. Subsystems further described are Structures; Propulsion; Electrical; Command, Control, and Communications; Auxiliary; Outfitting and Furnishings; and Combat System. The report also includes sections addressing Logistic Considerations; Survivability and Vulnerability; and Technical Risk.
- (U) The near term point design SES 18 shown to be a cost-effective, minimum risk, and high performance means of satisfying ANVCE requirements.

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1/ INTRODUCTION

- (U) This report describes the point design of a near term (1980) Advanced Naval Vehicle Concept Evaluation (ANVCE) Study 3,080 LT (29,892.1 kN)

 Surface Effect Ship (SES). The point design has been developed in accordance with Modification PO0017 to Contract N00024-74-C-0924 for the SES Project Office (PMS-304).
- (U) The data in this report are for a weaponized test ship and were originally submitted in response to RFP N00024-76-5342(S). The SES was developed in accordance with 'Large Surface Effect Ship (LSES) Top Level Requirements (U)", Chief of Naval Operations, 28 May 1976, Rohr Document No. CDC-C-76-076 CONFIDENTIAL.
- (U) The design was documented in Rohr Industries, Inc., 'Technical Proposal for Design and Construction of a Large Surface Effect Ship," in five (5) volumes tonsisting of 34 books and 16 appendices, dated 19 July 1976, CONFIDENTIAL, as amended by "Rest and Final Proposal for the Design and Construction of a 3,000-Ton Surface Effect Ship, Volume I Management and Technical Change Summary," dated 12 October 1976 (with four (4) appendices). The near term SES design is not a combatant ship.
- (U) The near term SES design is presented In the format specified in the Office of Advanced Naval Vehicle Concept Evaluation (ANVCE) document WP-005A, "Point Design Description,' dated 13 August 1976. The terms ."3K", "near term ANVCE point design", and "1980 point design" SES are used synomously throughout the report to refer to the same design concept.

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- (U) The near term point design is described in English, as well as in SI (metric) units of measurement. The point design was developed with -English units as the primary standard of measurement. SX conversions shown in the text within parentheses conform to American National Standard Z210.1-1973, "Metric Practice Guide", 15 March 1973, American Society for Testing and Materials, which has been approved by the Department of Defense and its use stipulated by the ANVCE Project Office.
- (U) The near term SES performance data for range and speed were developed in accordance with the ANVCE design guidelines with the following exceptions:
 - *Lightship margins were not computed on the basis of a 15% reservation including service life but rather on a 7% reservation plus a fixed 25 LT (249.1 kN) service life allowance.
 - SWBS Croup-410, 440, 450, 460, 470, and 480, as well as 700 and military payload related variable loads, were not drawn from the near term "ANVCE Combat System Data Sheets for AAW, ASW, SSW (U)", Vol I and II, dated 30 June 1976, but were developed in accordance with 3KSES TLR specifications and government-furnished information received in the course of the performance of phase IIA 2000-ton SES program activities,
 - The tail pipe (trapped fuel) allowance was adjusted to reflect ANVCE guidelines (2% deep tank, 5% flat tanks). In addition, range and payload data were presented on the basis of the ANVCE definition of payload contained in "Definition of Terms" ANVCE WP-002 dated 2 April 1976. Appendix A, Subsection A.2.11.1 contains a more detailed discussion of these areas,
 - The marine fouling allowance was taken as that corresponding to a 1.0 mil (25.4 μ m) surface finish.

- (U) No engineering effort was expended to generate data for a near term SES point design report section or subsection where that information was not developed for the 3KSES in response to RFP N00024-76-5342(S).
- (U) In addition, no engineering effort was expended where near term point design data existed, or where ANVCE format or design/environmental standards required a <u>major</u> effort in reformating for compatibility with ANVCE standards. Data in those instances are furnished in the formats used to meet 3KSES development specifications with appropriate notations delineating the assumptions and criteria utilized.
- (U) This report contains the following major sections (subsections are delineated in more **detail** in the table of contents):

Section No,	<u>Content</u>
1/	Introduction
2/	Vehicle General Description
2.1	Principal Characteristics
2.2	Vehicle Performance
2.3	Ship Subsystem Descriptions
2.4	Survivability and Vulnerability
3/	Logistic Considerations
3.1	Reliability and Availability
3.2	Maintenance Concept8
3.3	Overhaul Concept
3.4	Supply Support Concept
3.5	Human Engineering
3.6	System Safety
4/	Technical Risk Assessment

. . .

. . .

(U)	Appendices_	
	A	Design Process
	В	Drawings and Diagrams
	C	Equipment Lists

- (U) Finally, the LSES TLR of 28 May 1976 defined ship displacements in the following terms:
 - o Full Load Displacement (FLD) would be approximately 3000 long tons (29,892.05 kN) and would characterize a ship complete and ready for service in every respect.
 - o Light Ship Displacement would be a complete and empty ship with all operating fluids less fuel (SWBS Groups 100 through 700 plus margins).
 - o **Mean** operating Displacement (MOD) would be **character**terized for two conditions:
 - o MOD-50; A complete and loaded ship ready for service in every respect with not more than 50% usable fuel.,
 - o MOD-10; A minimum loading condition for maximum speed operation in any sea state where the ship was complete and ready for service in every respect for service in every respect with not less than 10% usable fuel.
 - o Capacity Load Displacement would be a complete and loaded ship with all fuel tanks filled to maximum operating capacity and ready for service in every respect.

(U) A variety of performance and **design** data wexe developed in relation to these displacement definitions and for the near term SES point design they have been referenced in a number of the subsections that follow.

2/ VEHICLE GENERAL DESCRIPTION

(U) 2.1 PRINCIPAL CHARACTERISTICS

- (U) 2.1.1 SUMMARY -- The Near Term Point Design SES illustrated in Figure 2.1-1 is a weaponized testship, is designed for high speed operation in an open ocean environment, has an extended range capability, and carries a military payload. Primary missions are SES concept evaluation, demonstration of weapon system compatibility, and determination of potential military value. Characteristics are summarized in Table 2.1-1.
- (U) The following subsections describe the ANVCE near term point design SES in detail -- Section 2.2 outlines Vehicle Performance, Section 2.3 contains ship subsystem descriptions, and Section 2.4 addresses survivability and vulnerability.
- (U) The point design, in the on-cushion mode, operates on the captured air bubble principle to reduce hydrodynamic drag and achieve high speeds. In the off-cushion mode, it operates as a displacement hull. The ship is capable of maneuvering in both modes including turning, accelerating, decelerating, and backing, and can also hover in the on-cushion mode.

(U) The principal ship dimensions ore shown in Figure 2.1-2. The 266.25 feet (81.15 m) length overall and 108 feet (32.92 m) maximum beam satisfy the volumetric and performance requirements. The maximum beam permits transiting the Panama and Suez Canals. Effective cushion dimensions are 221 feet (67.36 m) length and 85 feet (25.91 m) beam. A cushion height of 18 feet (5.49 m) was selected to ease ship motions and structural loads in Sea State 6. The full load displacement is approximately 3,000 long tons (29,892.05 kN) including all contract margins and mission fuel load. Table 2.1-1 shows the principal characteristics of the design.

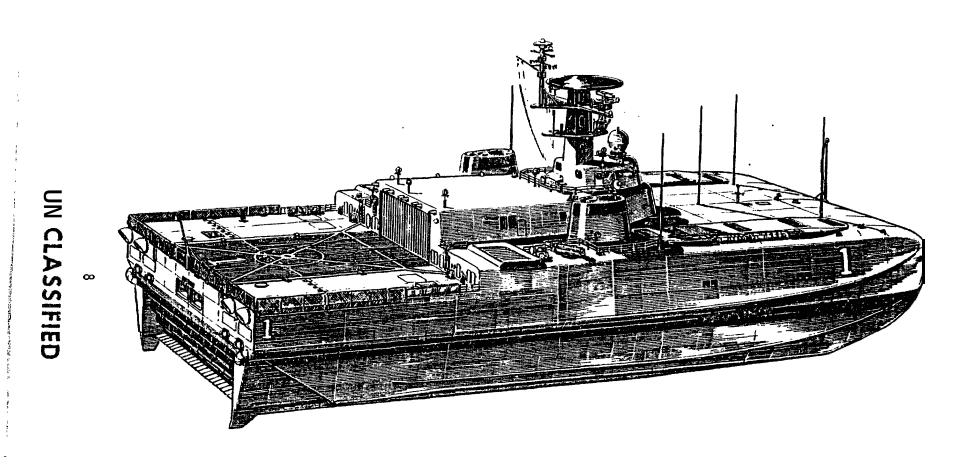


Figure 2.1-1 (U) · Near Term Point Design SES - A Step Toward the 100-Knot Navy (U)

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Table 2.1-1 (C). Principal Characteristics of the Near Term 3000 Ton Point Design SES (\mathbf{U})

OPERATION:	Weaponized testship with primary missions of SES concept validation, demonstration of weapon system compatibility and evaluation of military utility.
DIMENSIONS:	
a Length Overall (LOA) Ft. (m)	
Maximum Beam, Ft. (m)	108.00 (32.92)
Wet Deck Height (above baseline = ABL),	Ft. (a)
• Cushion Area, Ft. 2 (m²)	18,785.00 (1,745.13)
• Effective Cushion Length, Ft. (1)	
• Main Deck Height (ABL), Fe. (m)	40.00 (12.19)
. Sidehull Fence Depth (BBL), FT. (m) .	3.33 (1.02)
• Stabilizer Fin Depth (BBL), Ft. (m),	10.39 (3.17)
• Hullborne Design Waterline (ABL), Ft.	(m) · 20.33 (6.20)
• Maximum, Navigating Draft, Ft. (m)	30.72 (9.36)
POWER PLANTS :	
• Propulsion Engines	Four (4) General Electric (GE) LM-2500 or four (4) Turbo Marine FT-9A-2A
	Four (4) Aerojet Liquid Racket Co. (ALRC) Waterjet Pumps
• Lift Engines	Two (2) C.E. LM 2500's
• Lift Fans	Six (6) ALRC Centrifugal, Variable Geometry
CREW AND COMPLEMENT:	
a Vehicle . , , . ,	8 Officer, 8 CPO, 49 Enlisted
• Secondary Vehielaa (Helicopters/RPV's)	4 Officer, 1 CPO, 14 Enlisted
SYSTEMS:	
. Structures ,	All aluminum (5456), welded structure consisting of longitudinally stiffened plate supported by transverse web frames.
. Electrical ,	Independent 60 liz and 400 Hz subsystems, each powered by three (3) 375 kW Gas Turbine cenerator Sets interconnected by a ring bus.
■ Steering	Thrust vectoring, differential thrust, and thrust reversal with the outboard waterjet pumps only.
• Propulsion	Dual waterjet propulsors in each sidehull, driven by in-line gas turbines through separate reduction gear trains. Pump feed in each sidehull is with a single semi-flush inlet with variable roof ramp
• Lift	Three (3) centrifugal. variable geometry fans in each sidehull, driven by a single gas turbine through reduction gear.

Table 2.1-1 (C)(Cont'd)

	• •			_
SYSTEMS: (C	Cont'd.)			_
• 5	Seal. , . , , ,		dimensional, planing bow and • tatu	
a Sh	uip Integrated Control , ,	Closed loop lift systemans and/or of ouriliaz systems. c. integraced	wrol for steering, propulsion, and sucomatic control of ride (lift valve). Performance monitoring stric plant, and distribution ized ship damage control and scion and collision avoidance.	
• Ou	ntfit and Furnishings , , .	Hull comparts forming to Navability provis	y standards with generous habit-	
• Au;	kiliaries:		rieda	
	 Heating, Ventilating and Air Conditioning (HVAC) 		, axial flow fan, packaged air A/C) plants with dual-duct mixing	
	• Refrigeration , ,	Two (2) 400 Harefrigeration	r, powered centrifugal; packaged plants.	
	• Firemain end Auxiliary Seawator	Open loop, hori (0.10 m3/s) st	zontal system capable of 1600 gpm 125 ps1 (861.84 kPs).	
	• Scupper and Deck Drains , .	Standard gravi reinforced plas	ty drainage system utilizing glass tic (GPR) piping.	
	e Plumbing Drains (Soil and Weste)	Vacuum assiste to holding tank	ed collection discharged overboard or c.	
	. Main Drain		os and eductars for main machinery on and bilge water removal.	
	• Secondary Drain ,		ted eductora for miscellaneous ces not served by main drain system.	
	• Potable and Brash Water		oard system operated to minimize RP piping used extensively.	
	Gooling Water and Auxiliary Fresh Water Cooling		s (Freon and sea water cooled) are sed loop design meeting Navy standards	
	• Fuel Cil	Provides for fi fication of JP-	11ing, storage transfer and puri- 5 fuel for ship use.	
	● ∛❖ૠૹ♠∺□■ Fuel	Two (2) JP-5 fr storage throug service.	uel service tanks. filled from ships h filter coalescerr for helicopter	
	• Compressed Air , , ,	Low pressure si air from 3,000 provided.	ir from engine bleed and hieh pressure paig (20.68 mPa) compressor are	
	• Nitrogen		n is capable of supplying 70 to 48 to 20.68 MPa) of oil free	
	Fire Extinguishing		gh capacity AFFF, fixed flooding expansion foam.	
	• Hydraulic	Closed 3,000 p delivering 274	sig (20.68 MPa) system capable of $gpm (0.017 m^3/s)$.	



Table 2.1-1	(C) (COII	τα)		
e Auxiliaries (Cont'd.)				BAC O
. Replenishment at Sea (RAS)	fuel, pot	rea, port/starboa: able vater statio are provided.	rd alongside ms, and verti	KAS for cal flow
Anchoring		a 3,000 Lb. (13.3 iated cable winc		th anchor
. Mooring and Towing	Comprised chocks, an	of three (3) can d towing padeye	ostans, as wel s.	l as bits.
• Boat Handling and Scowage		i-man life rafts a iflatable rescue		
WEIGHTS :	LM	2500	FT-	. 9
• Full Load Displacement (FLD) (LT: kN)	3,000.0	(29,892.10)	3.000.0	(29,892.10)
Empty Weight (Lightship + Margins) (LT: kN)	1,661.2	(16,552.40)	1,699.3	(16,931.60)
• Fuel Weight (Capacity) (LT; kN)	1,838.9	(18,323.80)	1,838.9	(18,323.80)
• Usable Fuel at FLD (LT; kN)	1,179.9	(11,756.50)	1,141.6	(11,374.90)
• Unusable Fuei (1) at FLD (LT; kN)	64.6	(643.68)	64.6	(643.68)
• Other Load (LT;kN)	91.8	(914.70)	91.8	(914.70)
. Fuel Volume (Capacity) (Ft ³ ; m ³)	80,985.00	(2,293.2)	80,985.0	(2,293.00)
MOBILITY/PERFORMANCE SUMMARY:	I	M 2500	FT	- 9
Cushion Pressure (psf, kPa)	342.0	(16.38)	342.0	(16.38)
 Maximum Speed in Calm Water (kts; km/hr) at MCF and MOD-50 	66.0	(122.0)	92.0	(170.0)
• Maximum Speed at 3.94 Ft. (1.20 m) Signifi- cant Wave Height and MOD-50 (knots; km/hr)	64.0	(119.0)	85.0	(157.0)
• Hump Margin at 3.94 Ft. (1.20 m) Signifi- cant Wave Height, MOD-50 and MIP (%)	28%	(28%)	72%	(72%)
8 Best Range Speed, Calm Water (Kts; kM/Hr)	66.0	(122.01	90.0	(148 .0)
• Best Range Speed at 3.94 (1.20 m) Signifi- cant Wave Height (Kts; km/Hr)	64.0	(119.0)	76.0	(141.0)
. Time co Accelerate to Cruise Speed in Calm Water (Sec)	330,0	(330.0)	120.0	(120.0)
• Time to Accelerate to Max Speed in Calm Water (Sac)	330.0	(330.0)	260.0	(260.0)
Time Co Decelerate from Max Speed to 0 in Calm Water (Sac)	32.0	(32.0)	31.0	(31.0)
• Stopping Distance (Pt; km)	1,400.0	(0.43)	1.920.0	(0.59)
• Turn Radius at 40 Knots (20.58 m/s) spec (Ft: km)	4.009.0	(1.22)	3,200.0	(0.98)
• Range (nm; &oo	2,950.0	(5,472.0)	3.000.0	(5,565.0)
• Endurance (Hours)	42.0	(42.0)	39.0	(39.0)

⁽¹⁾ Per ANVCE Specification (2% deep tank, 5% flat tank.)

⁽²⁾ MIP applied in last minute of acceleration to avoid an asymptotic approach to waxinum speed.

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Table 2.1-1 (C) (Cont'd)

COMBA	SYSTEM:	Qty	System
	Armament	1	MK92/3 FCS
		1	STIR (Space and Weight)
		1	Harpoon FCS
			Torpedo Fire Control Panel MK309
		1	UWFCS (MK48 Torpedo) (Space and Weight)
		8	Vertical Missile Launchers
		a	Standard Missile (RIM-66B Mod, No Reloads)
		a	Harpoon Launchers
		a	Harpoon Missile (RGM-84-1, No Reloads)
		2	Torpedo Tube MK25/1 (Space and Weight)
		2	MK48 Torpedo (Space and Weight)
		2	Torpedo Tube MK32/5
		6	MK46/1 Torpedo (No Reloads)
		2	Close-In Weapons System MK16/0 (Space & Weight
	Underwater, Surface and Air		
	Serveillance and EW	1	Air Search Radar AN/APS-125
			Surface Search Radar AN/SFS-55
		1	DPEWS AN/SLQ-31(V-2) or AN/ SLQ-32(V-2)
			TACTASS AN/SQR-19
		7	Localization Sonar AN/AQS-13D
		1	CHF Telem Receiving Set AN/SKR-3A
		1	Sonar Receiving Set AN/UQR-1 (Mod)
		80	DIFAR Sonobuoy AN/SSO-53
		S 0	DICASS Sonobuoy AN/SSQ- 62
	Identification and Classification (IFF)	2	Interrogation Set AN/UPX-25(U)
		1	Transponder Set AN/UPX-28(U)
	Command and Control	1	computer AN/UYK-7 (V) (3)
. Command and Condy			Display Croup AN/UYA-4 (V) (4)
		1	Link T T
•	Secondary Sub-Vehicles		
	. Sub-Vehicles	2	Helicopters SH-3H or 1 XV-8B VSTOL
		10	Mini-RPV (Space)
	. Sub-Vehicle Armament	34	Torpedoes MK46/1

- (3) Growth for an additional AN/UYK-7 (V) Computer.
- (4) Includes three (3) OJ-194(V)3 PPI Consoles with growth for two (2) additional consoles.

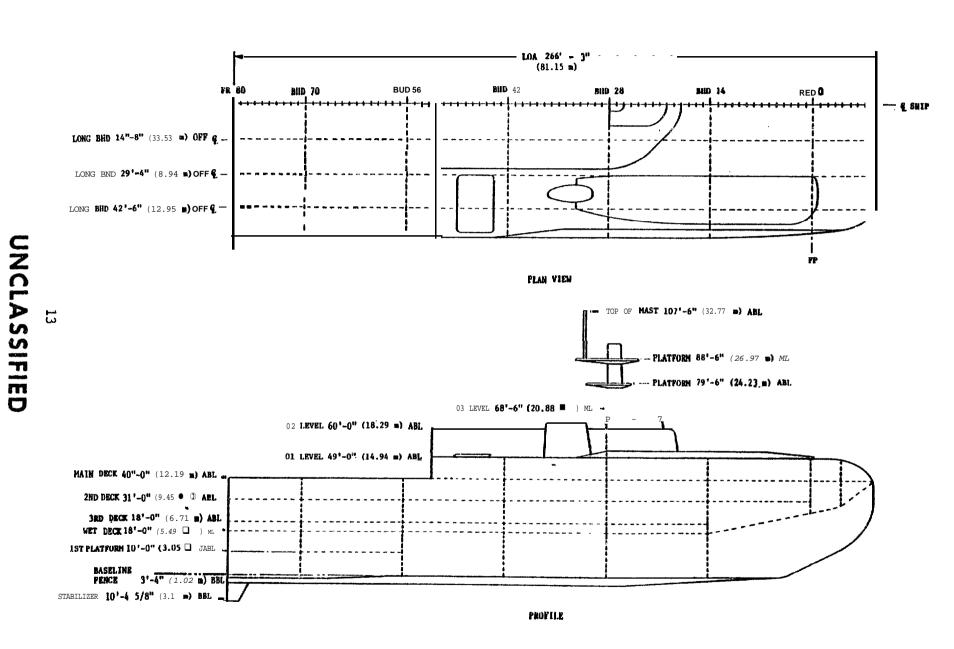


Figure 2.1-2 (U): Near Term Point Design SES Configuration (U)

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- (U) 2.1.2 GENERAL ARRANGEMENT DRAWINGS -- The general arrangement drawings of the ship are contained in Appendix B. Topside combat system locations are shown on the drawings. The drawings are:
 - Outboard Profile
 - Inboard Profile
 - 01 Level and Above
 - Main Deck
 - Second Deck
 - Third Deck
 - Wet Deck
 - Transverse Section
 - Sldehull Inboard Profile
 - Bow and Stern Views
 - Tank Arrangements and Tank Capacities
- (U) The drawings are grouped in Appenxlx B, Section B.1, for consistency of report format and the benefit of the reader. These drawings are completely up to date and definitive in those cases where minor discrepancies may be found in supporting drawings used elsewhere in this report.

- (U) 2.1.3 COMBAT SYSTEM DRAWINGS -- Weapons and sensor coverage on the near term SES are shown on drawings contained in Appendix B, Section B.2. The drawings illustrate coverage for:
 - Air Surveillance Radar (AN/APS-125)
 - Surface Search Radar (AN/SPS-55)
 - Collision Avoidance Radar
 - MK92 Fire Control System (CAS)
 - *STIR (MK54/0)
 - . MK16/0 Close-In-Weapon-System
 - AN/SLQ-31 (V2) EW System 1R Sensor
- (U) The drawings are grouped in Appendix B for consistency of report format and the benefit of the reader.

(U) 2.1.4 SHIP INTERFACES -- The near term SES is designed to functionally interface with other U. S. Navy ships, craft, shore commands and aircraft during operational deployment, and with Navy and other logistic facilities for support. The primary interface characteristics of the ship are:

*Vertical underway replenishment (VETREP) with the capability for rapid strike down.

- Underway alongside fuel and water replenishment (CONTREP)
- In-flight refuleing of helicopters (HIFR)
- · Capability of being towed
- Capability of receiving support services, including power, water, fuel and replenishment stores, when secured to a shore facility.
- *Capability for precision anchoring in depths not exceeding 40 fathoms (73.15 \mathbf{m})
- Mooring system to provide means for mooring alongside a pier or ship.
- Provide fuel and oil to helicopter and VTOL aircraft
- *Capability of maintaining visual and radio communication with other ships, aircraft, and shore facilities

- (U) 2.2 VEHICLE PERFORMANCE
- (U) 2.2.1 THRIJST, DRAG, AND POWER -- The predicted drag/displacement ratios for the near.term SES, as a function of ship speed and significant wave height at FLD, are shown in Figure 2.2.1-1, Performance is shown with the ride control system off, and with the ride control system operating at a level sufficient to meet or better the Rohr ride criteria shown in Figure 2.2.1-2. Comparable data at the MOD-50 condition are shown in Figure 2.2.1-3. In addition, a plot illustrating the speed dependent character of the drag components is presented in Figure 2.2.1-4. These data are based on analytic predictions which have been validated and enhanced by correlation with model test data. While no allowance was made for marine fouling, a 1.0 mil surface finish was assumed for all hydrodynamically wetted surfaces.
- (U) Figure 2.2.1-5 presents the propulsion system efficiency of the near term SES vs. speed and significant wave height. These data are based on the assumption that the propulsion power could be set at that level necessary to maintain a constant speed.
- (U) The transport efficiency of the near term SES as a function of speed and significant wave height is shown in Figure 2.2.1-5. In accordance with the definitions presented in ANVCE WP-002, dated April 2, 1976, transport efficiency was defined by:
 - Full Load Displacement (3000 LT; 29,892.1 kN) x Speed-(Independent Variable Total Power Required at Half Fuel (2400 LT; 23,913.6 kN) Condition
- (U) Figure 2.2.1-7 presents the maximum speed performance vs. significant wave height for the half fuel (MOD-50) condition. These predictions are based on the ride-control-off data, Figures 2.2.1-1 and 2.2.1-5 through 2.2.1-7 and the available 'thrust plotted in Figure 2.2.1-8.

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_FULL LOAD DISPLACEMENT (3000LT; 29,892 kN)
HEAD SEAS WITH WINDS

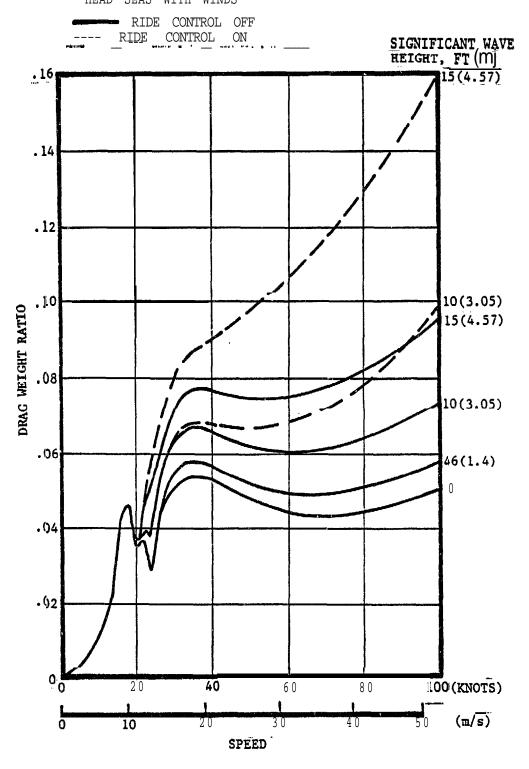


Figure 2.2.1-i (c): Near Term SES Drag/Weight Ratfo vs. Speed

and Significant Wave Height (U)

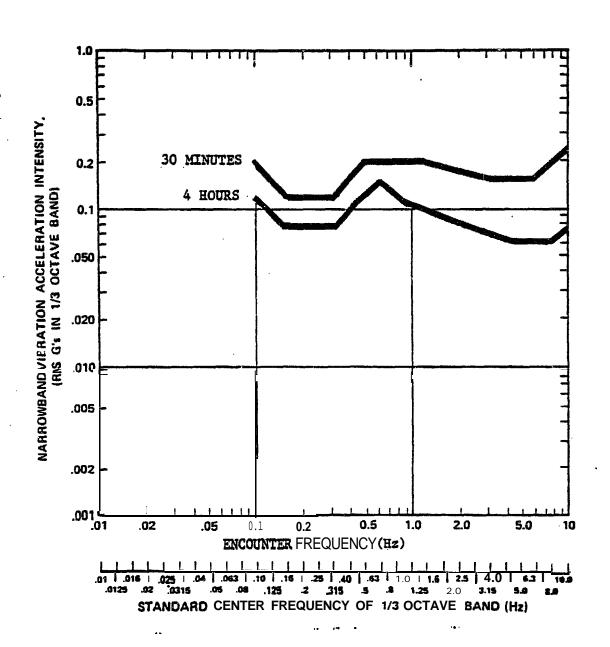


Figure 2.2.1-2 (U): Rohr SES Ride Criteria (U)

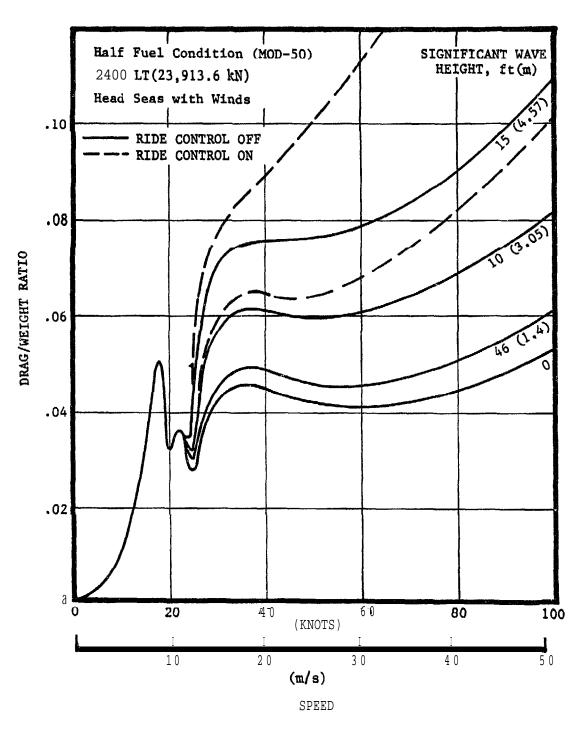


Figure 2.2.1-3 (C): 3KSES Drag/Weight Ratio Versus Speed and Significant Wave Height

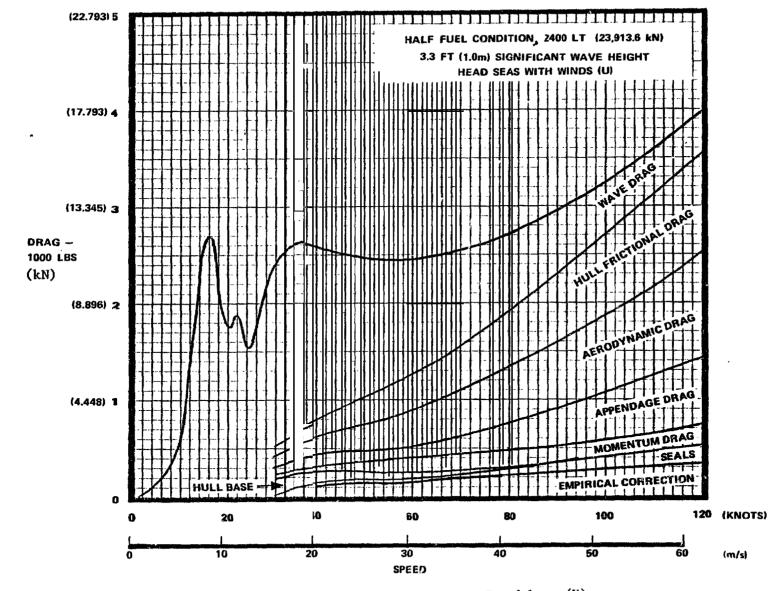


Figure 2.2.1-4 (C): Near Term SES Drag Breakdown (U)

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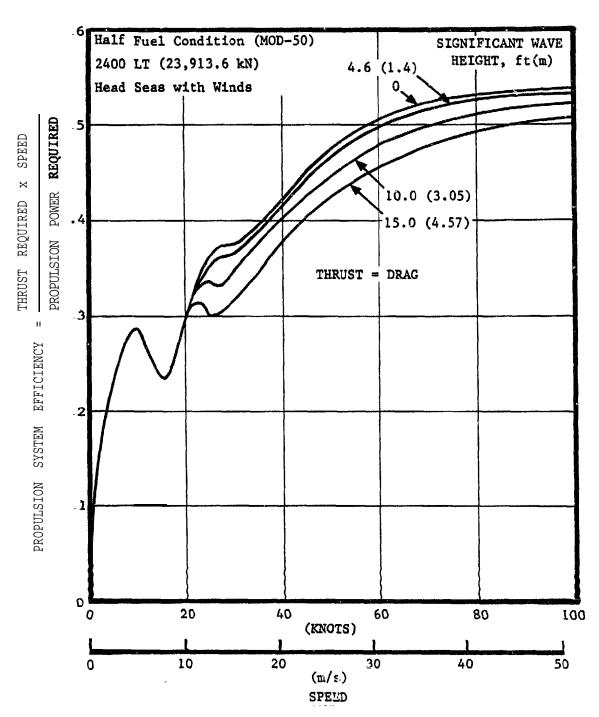


Figure 2.2.1-5 (C): Near Term SES Propulsive Efficiency Versus Speed and Significant Wave Height (U)

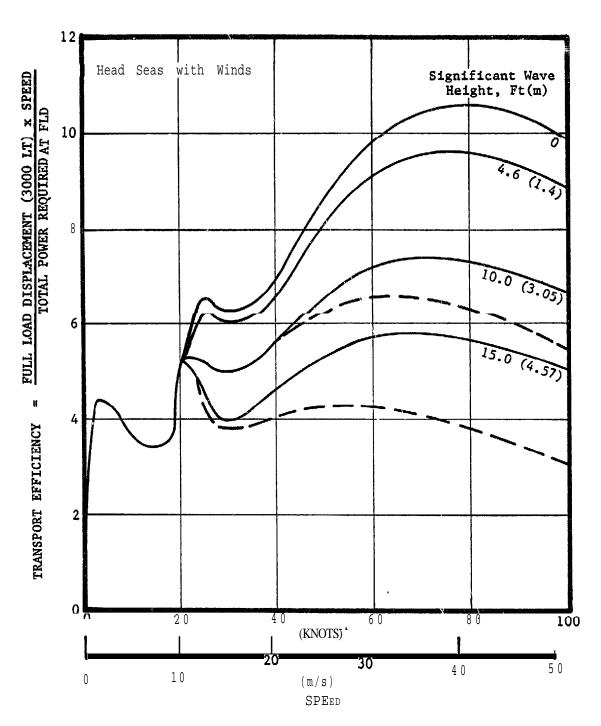


Figure 2.2.1-6 (U): Near Term SES Transport Efficiency Versus Speed (U)

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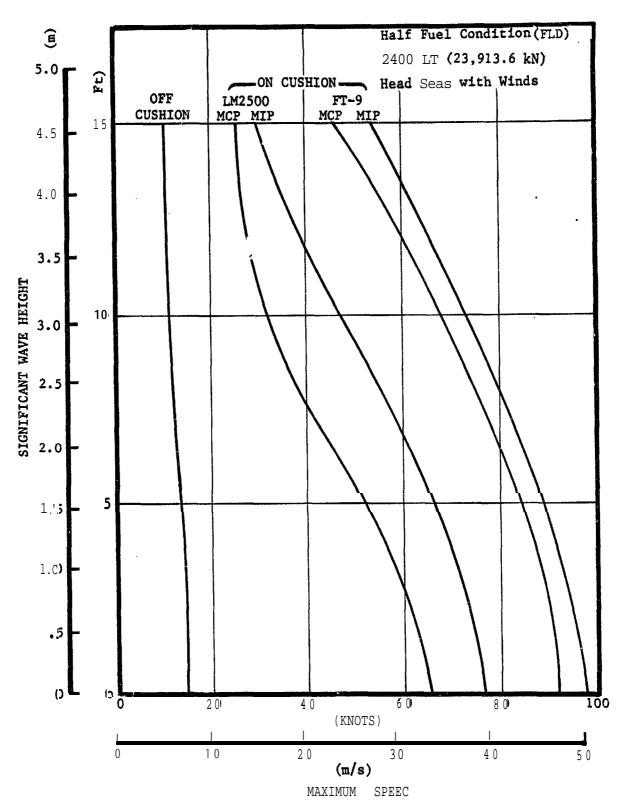


Figure 2.2.1-7 (C): Near Term SES Maximum Speed Versus Significant Wave Height (U)



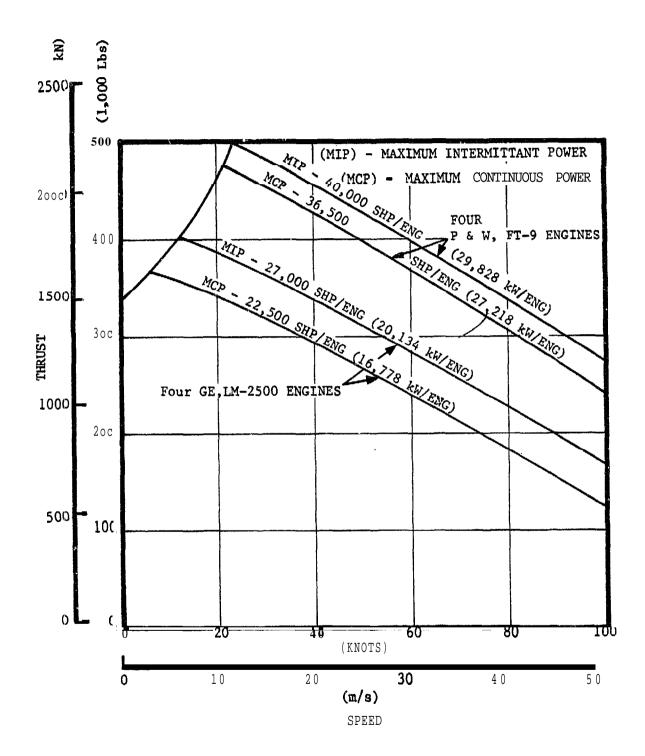


Figure 2.2.1-a (3): Near Term SES Available Thrust Versus Speed (U)

- (U) 2.2.2 MANEUVERING -- The steady state on-cushion turn performance of the near term SES configured with P&W FT-9 propulsion engines, in calm water and at an 83 percent fuel condition (2800 LT; 27,899.2 kN), is shown in Figures 2.2.2-1 and 2.2.2-2. After deceleration to a speed of 45 knots (23.15 m/s) for turns initiated at higher speeds, all four (4) waterjet pump nozzles are deflected an amount sufficient for the desired turn. The data shod the steady state turn performance characteristics after the initial lateral transient motions have decayed.
- (U) Figures 2.2.2-3 and 2.2.2-4 present the acceleration times from a standing start as a function of speed and **significant** wave height for LM-2500 and FT-9 engines, respectively. These maneuvers were computed on the basis that both the lift and propulsion engines are set at MCP and that the bow seal is partly retracted through hump transition. At low speeds, however, the power levels were limited to those imposed by cavitation limits of the **waterjet** inlets or pumps. The use of MIP during the last minute of the acceleration maneuver avoids an asymptotic approach to the maximum speed.
- (U) Figures 2.2.2-5 and 2.2.2-6 present the emergency deceleration performance as a function of speed and significant wave height for LM-2500 and FT-9 engines, respectively, as shown in Figures 2.2.2-7 and 2.2.2-8.

 Corresponding distances are shown in Figures 2.2.2-7 and 2.2.2-8.

 These maneuvers were accomplished by:
 - Engaging the thrust reversers,
 - Applying maximum continuous power to the outboard propulsion engines,
 - Reducing the inboard engine power to "idle", and
 @Retracting the stern seal by setting the stern seal fan variable geometry valves to "shut off."

(v) These procedures cause the ship to decelerate in a bow up attitude and avoid the possibility of undesirable pitch motions. Engagement of the thrust **reversers** requires 3.0 seconds. The remaining emergency stopping procedures are effected during this time interval.

2800LT (27,899.1 kN) DISPLACEMENT

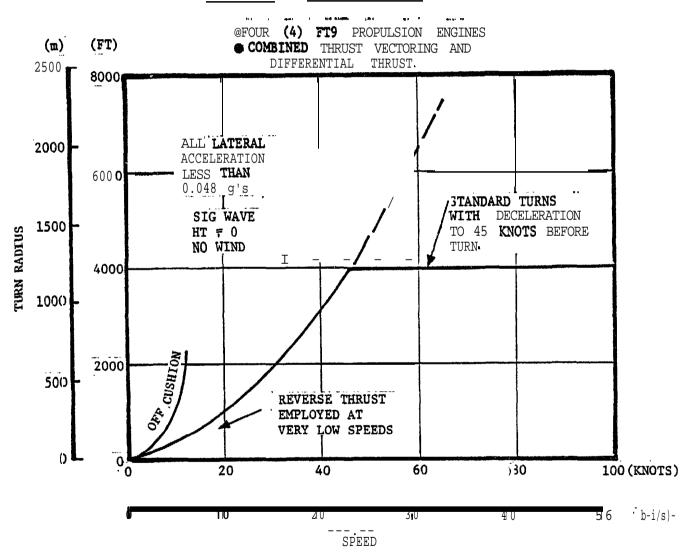


Figure 2.2.2-1 (U): Near Term SES Turn Radius Versus Speed (U)

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DISPLACEMENT = 2800 LT; (27,899.2 kN)
SIGNIFICANT WAVE HEIGHT = 0
NO WIND
COMBINED THRUST VECTORING
AND DIFFERENTIAL THRUST
ON CUSHION

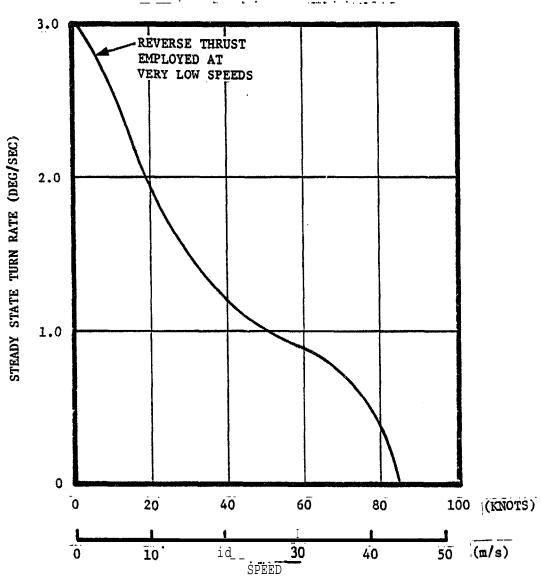


Figure 2.2.2-2 (C): Near Term Steady State Turn Rate Versus Speed (U)

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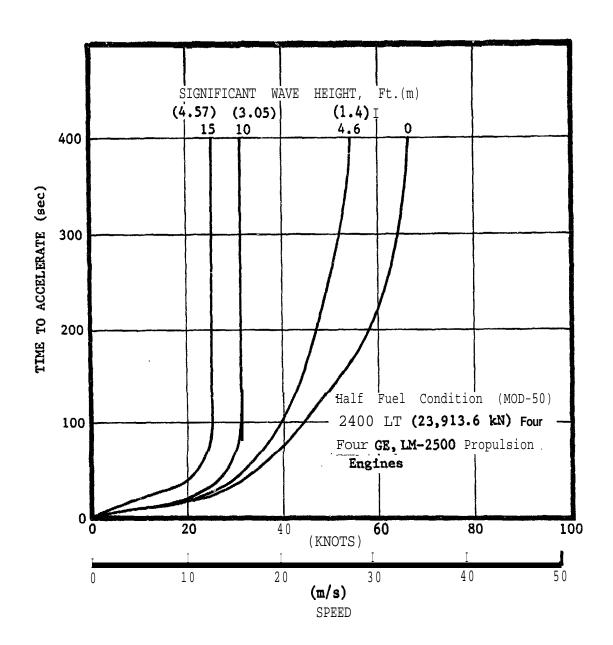


Figure 2.2.2-3 (C): Near Term SES Time to Accelerate Versus Speed (U)

William A Valletin Britania

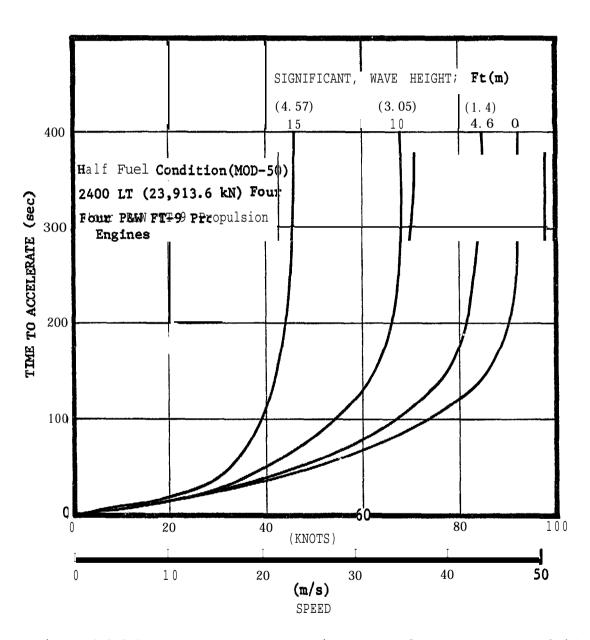


Figure 2.2.2-4 (C): Near Term SES Time to Accelerate Versus Speed (U)

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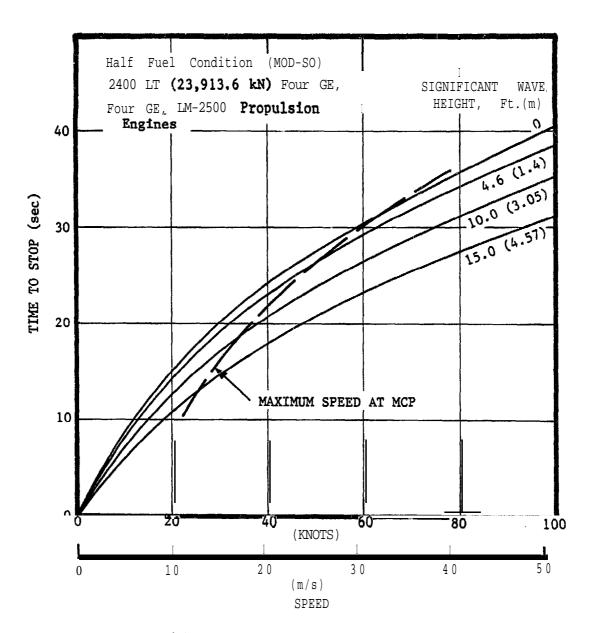


Figure 2.2.2-S (C): Near Term SES Time to Stop Versus Speed with ${LM2500}$ Engines (U)

Half Fuel Condition (MOD-50) 2400 LT (23,913.6 kN)_ Four P&W FT-9 Propulsion Engines

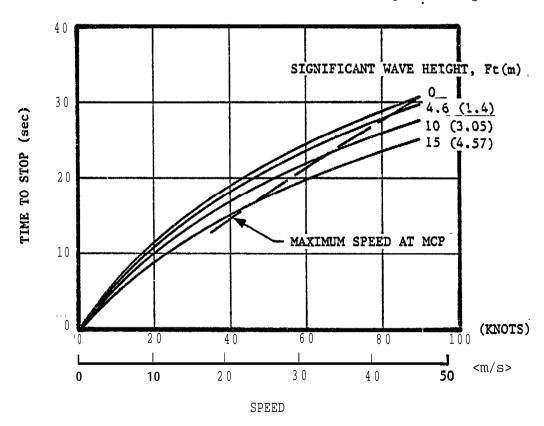


Figure 2.2.2-6 (C): Near Term SES Time to Stop Versus Speed with FT-9 Engines (U)

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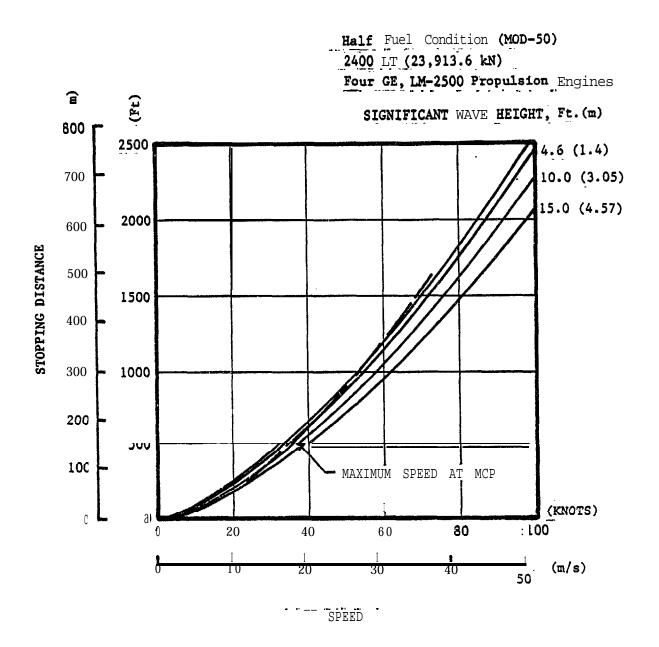


Figure 2.2.2-7 (C): Near Term SES Stopping Distance Versus $\bf Speed$ with $\bf LM2500$ Engines $\bf (U)$

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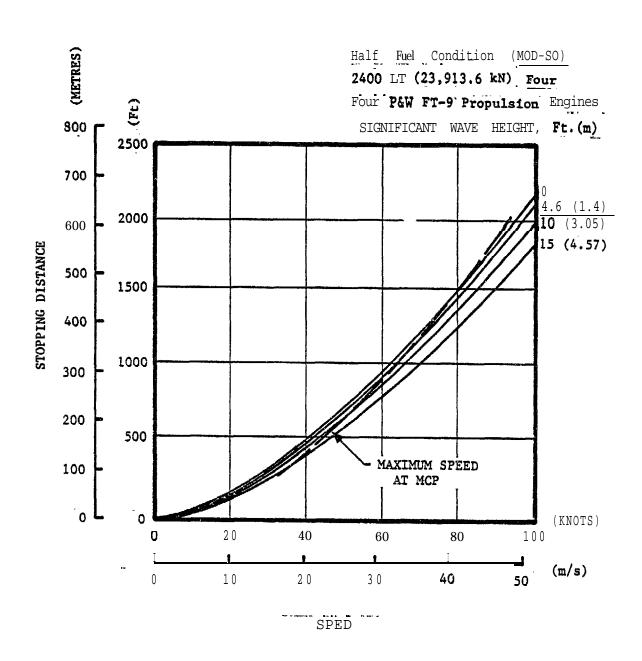


Figure 2.2.2-8 (C): Near Term SES Stopping Distance Versus Speed with FT-9 Engines (U)

- (U) 2.2.3 RANGE AND PAYLOAD The near term ANVCE SES exceeds the required range by about 10 percent with FT-9 engines and nearly attains the goal with LM2500 engines. Range is computed by integrating speed and fuel rate over the interval from full load displacement (FLD) to the near empty weight of lightship displacement plus unusable fuel.
- (U) The range and endurance characteristics are presented in Figures 2.2.3-l through 2.2.3-3, as influenced by speed, significant wave height and payload. The characteristics are shown with the ride control. system off and with the ride control system operating at a level sufficient to meet or better the ANVCE ride criteria. These data are based on the resistance data presented in Figures 2.2.1-1 and 2.2.1-3 and the propulsion system efficiencies reflected in the fuel consumption characteristics of FT-9 marine gas turbine engines presented in Figure 2.2.3-4.

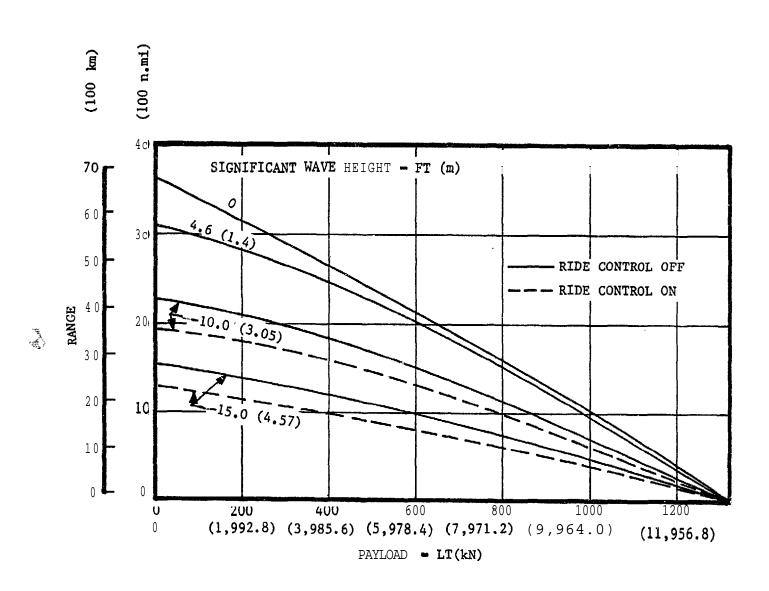


Figure 2.2.3-1(C) Range vs. Payload at Full Load Displacement for Head Seas with Winds (U)

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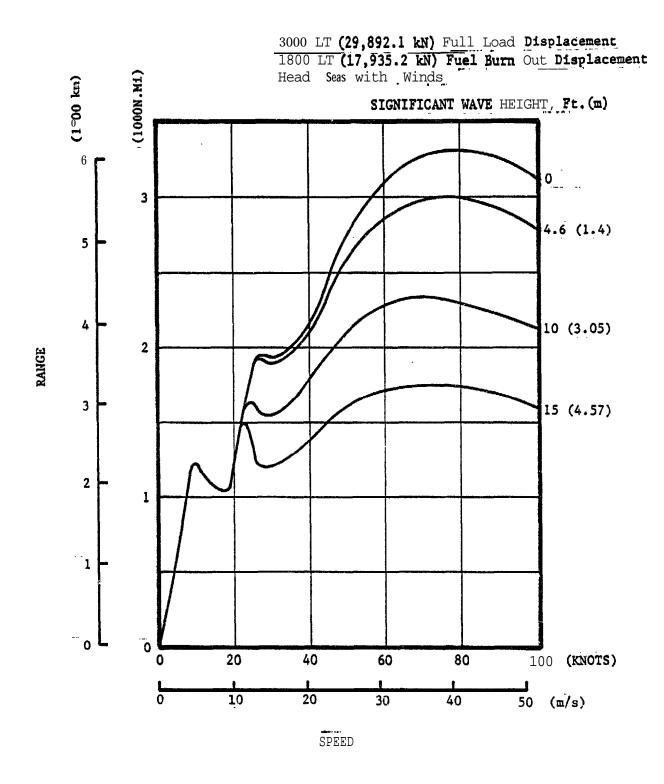


Figure 2.2.3-2 (C): 3KSES Range Versus Speed (U)

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300b LT (29.892.1 kN) Full Load Displacement

1800 LT (17,935.2 kN) Fuel Burn Out Displacement

Head Seas with Winds

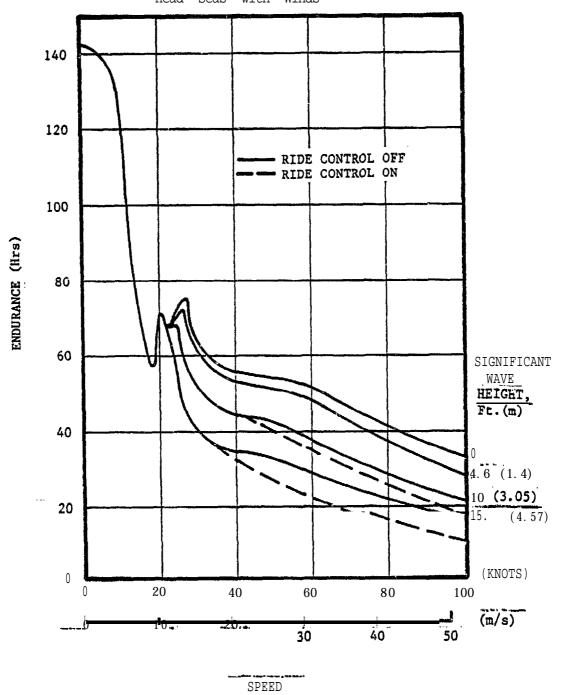


Figure 2.2.3-3 **(C):** 3KSES Endurance at Various Speeds and Significant Wave Heights (U)

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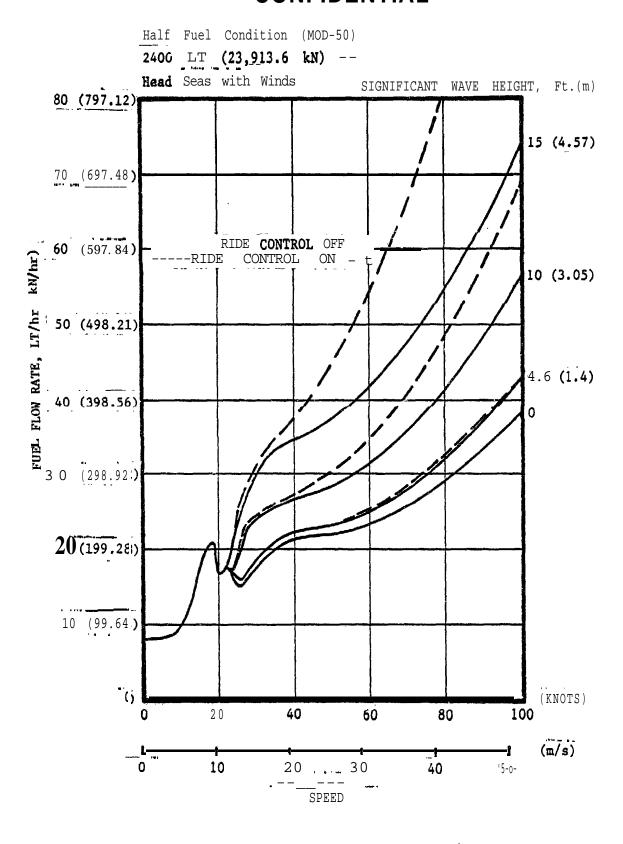


Figure 2.2.3-4 (C): 3KSES Fuel Consumption Versus Speed (U)

- (U) 2.2.4 WEIGHT AND VOLUME SUMMARY -- A summary of the light ship weight, variable load, contract margins and full load weight of the ANVCE near term SES is presented in Table 2.2.4-1. The ANVCE near term SES ship weights are identical to those of the Rohr 3KSES. The summary represents the results of parametric studies, design iterations, and trade-off investigations performed during the ANVCE near term SES design effort.
- (U) Table 2.2.4-2 is a summary of weights for a similar ship with FT-9 engines installed in place of the baselined LM 2500 engines for propulsion. The propulsion plant weight and contract margins were adjusted to reflect this substitution. The fuel weight was then reduced to arrive at a full load displacement of 3000 long tons (29,892.1 kN). The ship volume summary is presented in Table 2.2.4-3.
- (U) The design light ship, the total of SWBS groups 100 through 700, is. the displacement of the ship ready for sea in every respect, but excluding all variable load items such as crew, stores, ordnance, and fuel. Operating fluids such as lube oil, hydraulic fluid, and entrained water in the inlet and propulsor are included in the design light ship. The variable load items include the 125 man crew; provisions and effects, stores and spares for a 15-day mission; ordnance; both ship and aircraft fuel; and fresh water for the ship when operating at FLD.

TABLE 2.2.4-1 (U): WEIGHT SUMMARY WITH LM2500 ENGINES (U)

		WEIGHT								
SWBS GROUP		LONG TONS	SHORT TONS	METRIC TONS (1)	KILONEWTONS					
1.00:	HULL STRUCTUKE	805. 0	901.6	817. 9	8021. 0					
200 :	PROPULSION PLANT	190. 5	213. 3	193. 5	1897.3					
300 :	EI.ECTRICAL PLANT	61.8	69. 3	62.9	616.2					
400:	COMMAND 6 SURVEILLANCE	67. 0	75. 0	68. 0	667. 3					
500:	AUXILIARY SYSTEM	196.6	220.2	199.7	1958.8					
	567: Lift System	96.8	108	8. 4 98	.3 964.3					
600	OUTFIT AND FURNISHINGS	156.9	175. 7	159.4	1563.0					
700	ARMAMENT	51.4	57. 6	52. 3	512.6					
DESIG	N AND BUILDERS MARGIN	132.0	147.9	134. 2	1315. 6					
EMPTY	WEIGHT (LIGHT SHIP)	1661. 2	1860.6	1687. 9	3. 6552. 4					
F00:	LOADS:									
	Crews Provisions Stores Fresh Water Ordnance Main Vehicle Sub-Vehicle Sub-Vehicle Fuel	14. 7 5. 8 5. 8 18. 6 17. 2 6. 6 23.1 1246. 9	16. 6. 20. 19. 7. 25. 1396.	5 5.9 5 5.9 8 18.9 3 17.5 4 6.7 8 23.4	146. 7 57. 8 57. 8 185. 2 171. 9 66. 0 229. 7 12424.6					
1FULL	LOAD WEIGHT	3000.0	3360.0	3048.2	29892.1					

¹¹J_{Non-ST uni ts}

f i

TABLE 2.2.4-2.(U): WEIGHT SUMMARY WITH FT-9 ENGINES (U)

SWBS	WEIGHT							
GROUP	LONG TONS	SHORT TONS	METRIC TONS (1)	KILONEWTONS				
100: HULL STRUCTURE	805.0	901.6	817.9	8021.0				
200 : PROPULSION	226.0	253.1	229.6	2251.7				
300 : ELECTRICAL	61.8	69.2	62.8	615.6				
400: COMMAND & SURVEILLANCE	67.0	75.0	68.1	667.2				
500: AUXILIARIES	196.6	220.2	199.8	1959.0				
567: Lift System	96. 8	108	3.4 98.4	964.4				
600 OUTFIT AND FURNISHINGS	156.9	175.7	159.4	1563.1				
7 0 0 ARMAMENT	51.5	57.7	52.3	513.3				
DESIGN AND BUILDERS MARGIN	134.5	150.6	136.7	1339.8				
EMPTY WEIGHT (LIGHT SHIP)	1699.3	1903.2	1726.6	16931.6				
FOO: LOADS:								
Crews Provisions Stores Fresh Water Ordnance Main Vehicle Sub-Vehicle Sub-Vehicle Fuel	14.7 5.8 5.8 18.6 17.2 6.6 23.1 1208.6	16. 6. 20. 19. 7. 25.	5 5.9 5 5.9 8 18.9 3 17.5 4 6.7 9 23.5	146.7 57.8 57.8 185.2 171.9 66.0 229.7				
FULL LOAD WEIGHT	3000.0	3360.0	3048.1	29892.1				

⁽¹⁾ Non-ST Units

i

TABLE 2.2.4-3 (U): VOLUME SUMMARY (U)

	INTERNAL VOLUME (1)			
FUNCTION	CUBIC FEET	CUBIC METERS		
Main Propulsion (including main machinery box, uptakes, shafting)	119,034	3,371		
Lift System	109,881	3,112		
Personnel (including living, messing and all personnel support and storage)	104,454	2,958		
Auxiliary and Electrical (machinery spaces other than main propulsion and lift outside main machinery box)	100,962	2,859		
Payload (internal volume only)	150,955	4,275		
Other (including passageways, maintenance spaces and all other spaces not include d in above)	147,663	4,182		
TOTAL ENCLOSED VOLUME	732,949	20,758		

⁽¹⁾ Total enclosed volume does not include tanks and other innerbottom spaces below third deck, or **helo** landing and any weather decks.

- (U) 2.2.5 STABILITY -- The stability of the near term SES was addressed for both zero speed and underway conditions. The results show that the SES has adequate stability to meet the required operating ranges of speed, sea state and displacement.
- (U) 2.2.5.1 Stability at Zero Forward Speed -- The hullborne stability at zero speed of the near term 3KSES has been evaluated in accordance with the Navy criteria of acceptability (1). Two operating conditions that represent full,-load and minimum conditions were evaluated with:
 - a. A beam wind of 100 knots (185.2 km/hr), combined with rolling.
 - b. Topside icing
 - c. Crowding of personnel to one side
 - d. High speed turning for roll stability (per Navy criteria (1))
- (U) The intact stability at the MOD-10 condition with a beam wind of 100 knots (195.32 km/hr) combined with rolling produced the critical condition but with adequate stability as shown in Figures 2.2.5-1 and 2.2.5-2.
- (U) 2.2.5.1.1 <u>Static Stability in Hullborne Intact Condition</u> -- The static stability at zero speed was addressed by development of cross-curves of stability for a suitable range of ship displacement and for a range of heel angles from 0 through 90 degrees. The SES has a positive range of stability from 0 to 80 degrees as shown in Figure 2.2.5-3 and in Tables 2.2.5-1 and 2.2.5-2.
- (U) 2.2.5.1.2 <u>Stability in Damaged Condition</u> -- The fundamental adequacy of the SES with respect to reserve buoyancy and stability under conditions of hull damage in an open ocean environment has been addressed for the

^{(1) &}quot;Stability and Reserve Buoyancy of U.S. Naval Surface Ships", DDS079-1, dated 1 August 1975, Department of the Navy, Naval Ship Engineering Center,

- (U) full load and MOD-10 conditions per the Navy criteria and summarized for the following critical conditions:
 - a, Shell-to-shell flooding in compartments **TI** and III, with the longitudinal extent of damage equivalent to 15 percent **LOA.** This produced the least freeboard as indicated in Figures 2.2.5-4 and 2.2.5-5. The criteria of acceptability **is** satisfied.
 - b. Unsymmetrical flooding with penetration up to the centerline and with a longitudinal extent equivalent to 15 percent LOA was investigated throughout the length of the hull. The worst case was found to be with the damage in compartment IV and V. The maximum heel in this case was 7.48 degrees. Figures 2.2.5-6 and 2.2.5-7 show this condition, Requirements of the criteria were satisfied.
 - C. Unsymmetrical flooding with penetration to the first longitudinal bulkhead (not less than 10 percent maximum beam) and a longitudinal extend equivalent to 50 percent LOA was investigated throughout the length of the hull. The worst case was found to be for compartments III and IV and V and VI flooded. However, the requirements of the criteria were satisfied. Figures 2.2.5-8 and 2.2.5-9 depict this condition.
- (U) In summary, the ANVCE near term SES meets and exceeds the stability requirements at zero speed and the reserve buoyancy criteria for Large Surface Effect Ships of the U. S. Navy,

DATE OATE	188 AFF387	1 JUN 3 82321 OH		N T A C 1	\$ 7	A P I L T	T 7	5	1 5		•	ige 1
CC-PUTER OFFSET NO 115858001 CANG. NG. 1 N/A			3KSES			LMP 1 240.00 FEFT LOA 1 244.25 FEET			BEAM : 108.00 FEET NG. OF STATIONS : 51			
		•	BEAP W	ND COMPINED	WITH ROL	LIAG I P	Can optmatt	NG (P00-1)	O CGNBITIC	ts p	•	
			•	HV0005TA1		ENTSTICS	AT ZFRQ HEE	LANGLE				
LONGITE VERTICE VERTINGE TRANSV	# FREF-SURF ETACENTI ERSE . CD -VE : AFTW	(FG) (FG) (ACE (R (HPT) (TCB)		19.40 TONS 23.68 FT AF 27.51 FT AF 28.68 TGN-F 38.47 FT AF -4.61 FT CF -1.52 FT 18.31 FT AF	A Numer A Nume	- mêrls in	LORGITUMI LORGITUMI VERTICAL VERTICAL LORG. MET MOMENT TO ORAFT AT IMART AT	NAL CF CB C.G. F ACENTER ALTER TRI LCF	(LCG) = (LCF) = (LCF) = (HG) = (HG) = (HTI)= (HI = (HFI =	13: 1: 2: 4 91: 59:	3.08 FT AF 6.25 FT AF 7.04 FT AP 7.74 FT AP 0.74 FT AP 8.77 TON-F 8.56 FT AP F.83 FT AB	T F# V #Rul V #201 V #201 T/I*CM V #241
					SPEED -	100.00			·			
					- ,							
HEEL I	-15.440	-10.040	-5.000.	•.0	5.484	10.000	20.000	30-000	40.000	50.000	40.000	CERREES
CEASTI	14.750	14,708	16.307	18,444	18.255	14.723	31.905	4.424	-3.846	-12.457	-21-145	FEFT
ANIM I	-1.504	-4,-69	-3.459	-1.571	-3.441	-+-+12	-10.333	-12.359	-14.107	-14.434	-16.204	FEFT
62 . *	-32.413	-23.650	-13.639	-9.000	13.472	23.550	35.451	33-171	,25.115	23.440	18.533	FEFT
ATEM P	3,149	2.403	1.711	1-1+7	1.790	2.348	4.544	5.458	7.918	4,774	11,444	PCET
				D Y A		\$ T A 9	1111					
			PEA AS				0.4443 AND 044.5527 AND		DEGREES DEGREES			
				AATI Phi- RA-C		3.034 0.44 1.214	J NEGPEES					
					CRITER	IA SATISFI	ita					

HOTE & GZ VALUES INCLUDES CORRECTION TERM 0.05 COS(MEEL) TO ACCOUNT FOR UNKNOWN UNSYMMETRICAL MOMENT THE OYMAMIC STARILITY IS BASED ON MAXIMUM POSITIVE HEEL ANGLE OF 40.88 DEGREES

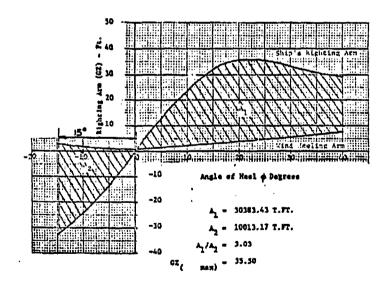


Figure 2.2.5-1 (U): 3KSES Hullborne Intact Stability (English Units) (U)

47

DATE COMPUTER OFFSET	: 10V 03-76	INTACT	TABILITY .	AHALYSIS	PAGE
93:10 :10 ·	I N/A		3KSES	LDP : 73.15 H LOA : 81.15 H	BEAM : 32.92 M NO. OF STATIONS : 51
	321	M WIND CONSINED WITH	MOLLING MEAN OFFRA	TING (MOD10) CONDITI	on
		HYDROGTATIC CILA	PACTERISTICS AT EURO H	EEL ANGLE	•
SEA NATER DISPIR LANGETHREAL CH VERTICAL CO MAINTHE FREE-SHE TRAINV. HETAGENT TRAINVERSE CH TRIN (*VE : AFT) DRAFT AT AP	(ICS) = (IIG) = FACE = ER (KIT) = (TCB) =	1901.90 CU-II 37.52 M AFT FP 8.39 M ANV NEGA 1330.16 KN-H 71.83 M ANV REGA 0.00 # FRM REVA -0.46 H 5.58 H ADV REGA	LOIGITU VERTICA VURTICA LOIG. M MONEST DRAFT A	L C.G. (KG) = LTACTHIER (XML) = TO ALTER TRIM (MII) = I LCF (H) =	17.52 A AFT FP 41.53 H AFT FP 3.67 H MIV RAUL 10.66 H AGV RAVE 27.55 H ANY RAVE 71.21 H-04/CH 1.76 H AGV RAVE 6.04 H AGV RAVE
		MINT PHETOLOGY	-vo- meels in intact c	ONDITION	
		HIND SPEED	- 165.32 KM/NR		
HEEL : -15.000	-10.000 -5.6	000 0.0 1.000	10.000 20.000	10.000 40.000	50.000 60.000 DEGREES
DPAFT: 4.494	5. 093 5. 5	5.780 5.5	76 I. 097 1. 629	1.349 - 1.172	-3.919 -6.460 H
TRIN 1 -2.684	-1 .972 - 1.1	76 -0.468 -1.0	49 -1.976 -3.150	- 2. 767 -4.300	-4. 400 -5.122 !!
- 9. 531	-7.111 -4.1	157 -0.002 4.10	7,180 10. 806	10,111 6.871	7. 266 5.649 N
13.AN 0. 972	0. 711 0.5	0.186 0.	111 0.731 1.233	1.113 2. 111	a. 979
			C STADILITY		
	AREA A1	92271.00 Kir-	M DETWEEN 0.0463 AND DETWEEN -14.5537 AND	40.0000 DEGREES	
		RATIO A1/A PHI+C	2 = 1.0343 0.4461 DEGREES		
		RA-C	• 0.3702 X DEPRITY = 1.025 METRIC	. Marca dan	
		HOLE I DON WALLE	DEMILI - (.025 AUTRIC	1000F/CU-FI	
	X	manuallianilianiliani			
	_		000000000000000000000000000000000000000	III SHIP'S RIGHTIN	NG ARM CURVE

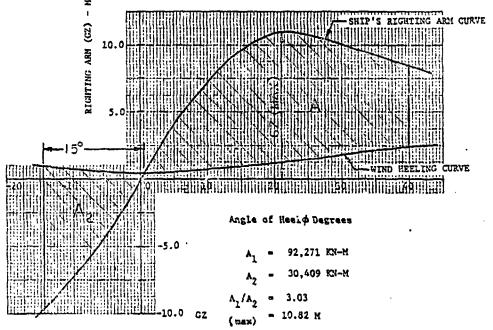


Figure 2.2.5-2 ($\tt U$): 3KSES Hullborne Intact Stability ($\tt SI$ Units) ($\tt U$)



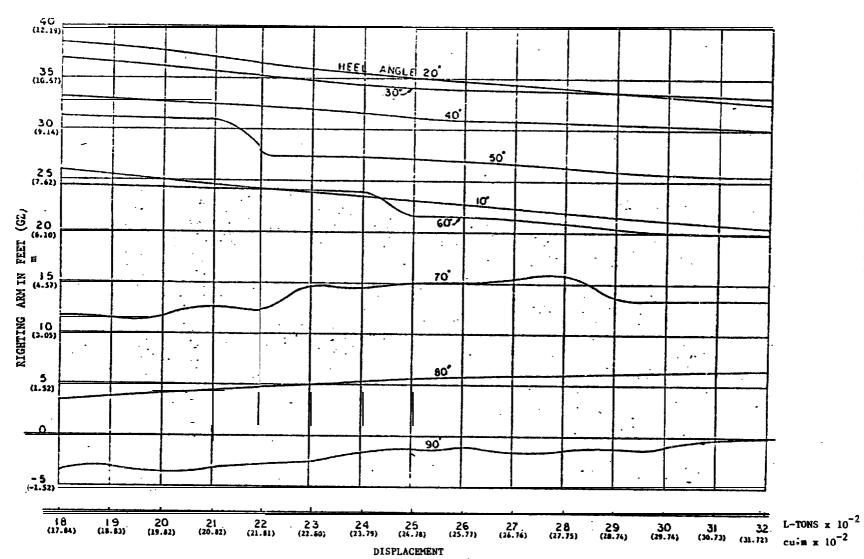


Figure 2.2.5-3 (U): Cross Curves of Stability in Intact Hullborne Condition (U)

DATE : JUN 28-76 CCPPUTER OFFSET NO :LSESOGO) CPWG. NO. .

:FPT:890-801-23

STABILITY CPOSS CHRVES

3KSES

LRP: 240.00 FEFT LOA: 266.25 FFFT

REAM : 108.00 FEET NC. OF STATIONS : 5]

PAGE t

	COMPUTATION IS GASE	COH ASSU	YED KC: 11	IINITIAL U	PRIGHT COM	DITION = 26	. 00 FT ABOV	Æ RRUL	
PHI : 0.0 DEGREES	DISPLACEMENT: RIGHTING ARM 1 LCR FROM FP:	1900.00 -0.00 129.57	2100.60 -0.81 136.39	2300.00 -0.01 130.95	2500.00 -0.61 131.43	2700.00 -0.01 131.85	5900.00 -0.01 132.33	3100.00 · -0.01 133.55	3300.00 L.TCNS -0.01 FEET 133.86 FFET
PHT : 10.00 DEGREES	DISPLACEMENT : PIGHTING AHM : LCB FROM EP :	1900.00 75.45 136.65	2100.00 24.67 136.96	2300.00 73.90 137.12	2500.00 23.14 137.73	2760.00 22.38 137.32	2900. 00 21. 64 137. 30	3100.00 2F. 92 137.16	3300.60 L.TONS 20.71 FFET 136.93 FFET
PHI : 20.00 DFGREFS	DISPLACEMENT : RIGHTING ARM : LCB FRCM FP :	1900.00 38.12 143.75	≥100.00 57.30 143.63	2300.00 ' 35.45 141.83	2500.00 35.15 141.29	2700.00 34.46 141.24	2500.00 33.74	3100.00 32.91 140.87	3300.00 L.TCNS 32.70 FEET 140.50 FEET
30.00 CFGREES	DISPLACEMENT : RIGHTING ARM : LCP FROM FP :	1900.00 36.33 147.55	2100.00 35.43 145.64	2300.00 35.16 145.61	2500.00 34.18 143.16	2700.00 32.96 143.02	2900. 00 33.70 142. 71	3100.00 33.41 142.29	33.10 FFET 141.74 FFFT
PH1 : 40.00 DEGREES	DISPLACEMENT : RIGHTING ARM : LCE FROM FP :	1900.00 33.14 150.92	2100.00 32.69 149.68	2360.60 37.31 140.32	2500.66 31.54 146.37	non. 00 30.24 144. 31	2500 . oo 30. 49 143. 36	3100.00 30.24 142.66	3300.00 L.TONS 29.95 FEET 141.82 FEET
PHI : SO.ON CFGREFS	DISPLACEMENT : RIGHTING ARM : LCS FROM FC :	31.15 161.20	2160.00 31.00 160.45	7360.00 77.41 149.03	2500. no 27.02 147.58	2760.00 26.46 145.54	2900.00 25 73 142.96	3100.00 25.60 142.18	3300.00 L.70NS 25.41 FFET 141.25 FFET
PHI : CO. OO CFGREES	DISPLACEMENT: RIGHTING APM: LCR FROM EM:	190 n.00 94.35 159.67	2160.00 74.28 150.41	23 00. no 24. 12 157. 13	2500.00 21.51 148.17	2700.00 21.21 146.78	2900.00 . 20.48 144.02	3100.00 19.88 141.63	3300.00 L.10N5 19.40 FEET 140.81 FFET
PHI : 70.00 DEGREES	DISPLACEMENT : RIGHT ING ARM : LCB FROM FP :	1900.00 11.42 140.17	2100.00 12.65 142.48	2300.00 14.77 149.08	2500.00 15.10 148.69	2900.00 15.41 149.20	2900.00 13.54 142.63	3100.00 13.40 141.06	3300.00 L.TGMS 13.43 FFET 140.46 FFET
PH1 : 20.00 CFGREES	CISPLACEMENT: PIGHTING ARM: LCB FROM FP:	1900.60 3.81 138.44	2100.00 4.54 136.79	2300.00 5.10 139.18	2500.00 5.55 139.51	2700.00 5.90 139.79	2900.00 6.19 140.04	3100.00 6.42 140.25	3300.00 L.TCNS 6.59 FFFT 140.12 FEET
PHI : 90.00 DEGREES .	DISPLACEMENT : RIGHTING ARM : LCH FROM FM :	1900.00 -2.58 141.04	2100.00 -5.14 137.30	2300.00 -3.51 137.P2	2500.00 -1.28 141.5P	2700. no -1.53 138.63	2960.00 -1.14 131.95	3100.00 - 0. 24 141.91	3300.00 L.TONS 0.03 FFET 142.00 FEET

Table 2.2.5-1 (U): Intact Range of Stability (English Units) (U)

UNCLASSIFIED

PHI

NO. OF	STATIONS : 51
1	
2.38	3270.60 CU-H
0.00	0 / 0 0 H
0.71	40.80 K
2.39	3270.60 CU-N
6.38	E / 9 6 M
1.81	41.74
2.39	3270.60 CU-M
0.05	9.81 M
2.94	42.32 M
	J270.60 E0-M I0.09 M 43.20 M
•36	3270.60 CU-H
9.22	9.13 H
3.48	43.23 H
2.30	3270.60 CU-H
7.80	7.74 H
3.34	43.05 H

CROSS CURVES OF STABILITY #HOV 03-76 COMPUTER OFFSET NO :LSESSOOT

3KSES

73.15 A 81.15 M LOA :

BEAM : 32.92 H

PAGE

	•	COMPUTATION IS BA	SED OH ASSU	MED KG IN	INITIAL UP	RIGHT COMD	ITION = 7.	D2 M ABOVE	RRI4L	
PHI : 0.0	មាលេខ នេះ	MEPLACEMENT : RIGHTING ARM :	1883.07 0.00	2081.29	2279.51 0.00	2477.72	2675.94 0.00	2974,16	3072.38 0.00	3270.60 CU-H
		LCB FROM FP :	39.49	39.74	39,91	40.06	40.19	40,33	40.71	40.80 M
PHI : 16.00	DEGREES	desplacionni :	1883.07	2081.29	2279.51	2477,72	2675.94	2874.16	3072,39	3270.60 CU-M
		RIGHTING ARM :	7.77	7.52	7.28	7.05	6.82	6.60	6.38	٤,96 👪
		LCB FROM FP :	41.66	41.75	41.79	41.83	41.86	41.85	41.81	41.74
PHI : 20.00	DECREES	DISPLACEMENT 1	1003.07	2081.29	2279.51	2477.72	2675.94	2874.16	3072.38	3270.60 CU-H
		RIGHTING ARES :	11.62	11.37	10.97	10.73	18.51	10.28	10.05	9.31 %
		LCD FROM FP :	43.82	43.84	43.23	43.07	43.05	43.01	42,94	42.32 M
PHI 4 30.00	DEGLEES	DISPLACEMENT :	1883.07	CE. 1862	2279.51	2477.72	2675.94	2874.16	3072.38	3270.60 EO-M
		RICETING ARM :	11.07	10.80	10.72	10.42	10.35	10.27	10,18	10.09 M
		LCB FROH FP :	44.97	44.39	44.38	43.64	43.59	43,50	43,37	43.20 E
PHI . 40.00	DEGREES	DISPLACEMENT :	1883.07	2081.29	2279.51	2477,72	2675.94	2874.16	♦□ ⊠₹ •36	3270.60 CU-N
		RIGHTING ARM :	10.10	9,96	9.82	9,61	9.40	9.29	9.22	9.13 8
		LCB FROM FP 1	46.00	45.62	45.21	44,61	43.99	43.7Ď	43.48	43.23 H
PHT : 50.00	DEGREES	DISPLACEMENT :	1683.07	.2081.29	2279.51	2477.72	2675.94	2874.16	3072, 38	3270.60 CU-N
		RIGHTING ARM :	9.51	9.45	8.35	8.24	8.07	7.84	7.80	7.74 H
•		LCa PROM PP :	49.13	16'Ob	45.42	44.98	44.36	43.57	43,34	43.05 11
PHI : 60.00	DEGREES	DISPLACEMENT :	1883.07	2081.29	2279.51	2477.72 •	2675.94	2874.1€	3072.38	3270.60 CU-N
	1011-104722	RIGHTING ARM I	7.43	7.40	7.35	6.56	6,46	6.24	6,06	6.04 M
		LCB FROM FP :	48.73	48.20	47.09	45.16	44.74	43.90	43,17	42.92 M
PHE : 70.00	DECREES	DISPLACEMENT :	1883.07	2081.29	2279.51	2477.72	2675.94	2874.16	3072,38	3270.60 CU-H
		RIGHTING ARM :	3.48	3.86	4.50	4.60	4.70	4.13	4.09	4.09 8
		LCB PROM FP :	42.72	43.43	45.44	45.32	45.48	43.29	43.00	42.81 M
FHI : 30.00	DEGIO:ES	DISPLACEMENT :	1883.07	2081.29	2279.51	2477.72	2675.94	2874.16	3072,38	3270.60 CU-!!
		RIGHTING ARM	1,10	1.38	1.55	1,65	1.80	1.89	1,96	2.01 H
		LCB PROM FP :	42.20	42.30	42.42	42.52	42.61	42.69	42,75	42.71 : I
PHI : 90.00	DEGREES	DISPLACEMENT :	1883.07	2081.29	2279.51	2477.72	2675.94	2874.16	3072.38	3270.60 CU-H
		RIGHTING ANN I	-0.91	-0.96	-0.77	-0.39	-0.47	-0.35	-0.07	0.01 M
		LCB FROM FP :	42.99	41.85	42.01	43.15	42.25	42.35	43,25	43.28 8

Table 2.2.5-2 (U): Intact Range of Stability (SI Units) (U)

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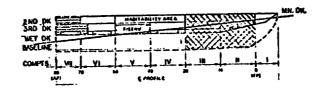
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DATE

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A N J L Y S I S

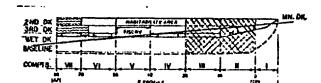


NOTE & MG VALUE INCLUIES ALLOWANCE FOR FREE-SURFACE FFFECT FOR SLACK TANKS

Figure 2.2.5-4 (U) Reserve Buoyancy with Shell-to-Shell Flooding (English Units) (U)

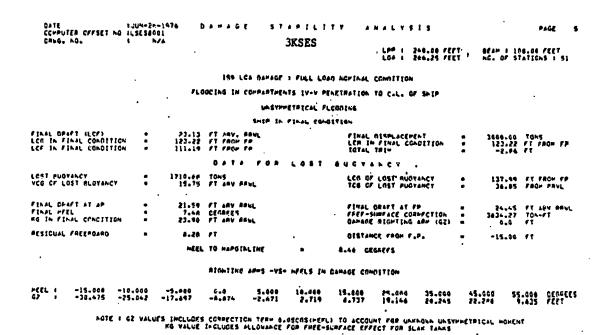
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PATE COMPUTER OFFSET NO	INOV 04-76	DAHAGE ST	ABILITY	ANALY61S		PAGE 11
DRIG. NO.	1 13/A		JKSES			
	,		•	LBP: 73.15 N LOA: 81.15 H		EAM 1 32.92 H O. OF STATIONS : 51
		enell-to-shell floodi	NG I FULL LOAD TRI	M FORMARD COMDITION		
		Compartment	s flooding II + II	II (FRAME 0-28)		
		Sylva	ETRICAL PLOODING			
		SHIP	IN FINAL COMULTION	ı		
PINAL DRAFT (LCF)	-	7 . 3 6 H ABV. RRHJ.	FI:	IAL DISPLACEMENT		2973.60 CU-H
LCG IN FUNL CONDITION		35,74 I I PROM PP		IN FINAL CONDITION	•	35.74 M PROH PP -6.68 M
LCF IN PINAL COMPLETION		16.35 M PROH PP	.107	AL TRI:1	-	-8.90 A
		D A T A PO	RLOSTDUOY	ANCY		
. LOST BUOYKICY	•	1624.36 CU-#	LC	OF LOST BUOYANCY	•	12.69 M FROM FP
VCG OF LOST BUOYANCY	•	6.57 HARV RRNL	KC	OF LOST BUOYANCE	•	-0.01 FROM RESTL
FI:JAL DRAFT AT AF	-	3.83 M ABV RRIL	FII	IAL DRAFT AT FP		10.47 M ABV RRIL
PINAL HEEL	-	0.00 DEGREES		E-SURPACE CORRECTION		964.78 H**4
KG IN PINAL CONDITION	•	7 . 2 2 H ABV RRIL	DVI	HAGE RIGHTING ARM (GE)	•	0.00
RESIDUAL PREEDOARD	•	0.92 N	DIS	TANCE FROM F.P.	•	-0.00
		HEEL TO HARGINLINZ	a 3.96	DEGREES .		



NOTE : KG VALUE INCLUDES ALLOWANCE FOR FREE-SURFACE EFFECT FOR SLACK TANKS

Figure 2.2.5-5 (U) Reserve Buoyancy with Shell-to-Shell Flooding (SI Units) (U)



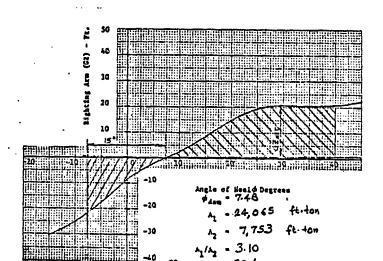


Figure 2.2.5-6 (U): Damaged Stability 15 Percent LOA Unsymmetrically Flooded (English Units) (U)

DATE COMPUTER OFFSET NO FLSESSON N/A

DANAGE STABILITY ANALYSIS

PAGE 5

3KSES

BEAM: 32.92. No. OF STATIONS: 51

15% LOA DAMAGE : FULL LOAD HOMINAL CONDITION

FLOODING IN COMPARTMENTS IVAV PENETRATION TO C.L. OF SHIP

UNSYMETRICAL FLOODING

SHIP IN FINAL CONDITION

FINAL BRAFT (LCF) -LTS IN FINAL CONDITION -LTF IN FINAL CONDITION -7.05. ABV REWL 37.56. FROM W 33.89. FROM FF

FINAL DISPLACEMENT LCB IN FINAL CONDITION TOTAL TRIN 2973.27 CU-37.56 u FROM +

DATA FOR LOST BUOYANCY

VOST DUOYAHOY # VCG OF LOST BUOYAHOY # 1695.44 CU-m 4.90 m ABV RRWL LCG or LOST DUOYANCY

42.06 m PROH FP 11.23 PROH RRWL

6.58 m ABV RRWL 7.48 DEGREES 1.28 9 ABV RRW1 FINAL OPAFT AT AP # FINAL HEEL # KG IN FINAL CONDITION #

FINAL DRAFT AT FP FREE-SURFACE CORRECTION = DAMAGE RIGHTING ARM (G2) =

7.45 • ABV RRWL 1097.85 m**4 0.00 m

RESIDUAL PREEROARD 2.52

DISTANCE FROM C.P.

-4.57 u

HEEL To MARGINLINE

RIGHTING ARMS -VS- HEELS IN DAMAGE CONDITION

-10.001 -7.633

-5.000 -5.304

-0.814

\$5.000 DEGREES 2.937 ::

HOTE : GZ VALUES INCLUDES CORRECTION THRM 0.05COS(HIEL) TO ACCOUNT FOR UNKNOWN UNSYMMETRICAL MOMENT KG VALUE INCLUDES ALLOWANCE FOR FREE-SURFACE EFFECT FOR SLAK TANKS SEA WATER DESSITY - 1.025 METRIC TONS/CU-H

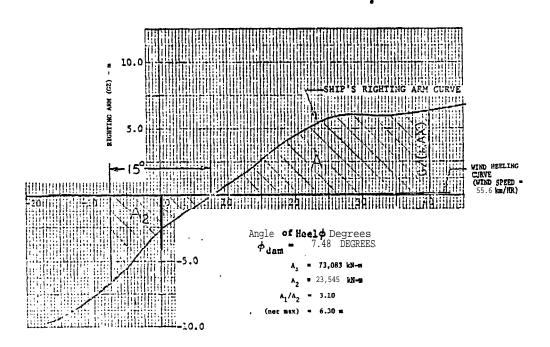


Figure 2.2.5-7 (U): Damaged Stability 15 Percent LOA Unsymmetrically Flooded (SI Units) (U)

```
DATE SUMMARY START NO IL TESSUAL DESG. AG. E A/A
                                                                  5 T & # I L I T V
                                                                     3KSES
                                                                                                LAP : 240.00 FFFT - PEAR : 108.00 FEET - LOA : 254.25 FFFT - LC. OF STATIONS :-51
                                      FLOCOING IN COMPARTMENTS ITT-IN-4-44 SENTINATION TO 24-33 LCCG. RMG.
                                                               UNEV-PETFICAL FLCCOING
                                                             SHIP IN FIRST CONDITION
                                                                                    ......
LEST MUNTINEY
WER OF LOST PROYAMEY
FIRAL MEEL
MG IN FIRAL CONDITION
RESIDUAL PRESSONA
                                                                                      DISTANCE FROM F.P.
                                                                                                                               240.00 /7
                                          NEEL TO PARSIALINE
                                                                                     6.75 CFABCFS
                                                 PIRHTIME APPS -VS- -FFLS IN PAMAGE COMPITION
PETL 1
                   TOTE I GZ VALUFS INCLUDES CORRECTION TOWN OLGEFOLD TO ACCOUNT FOR SLOK NAVOHA UNRYMMETRICAL MOMENT NO FORESTLORESE FEFECT FOR SLOK TANKS
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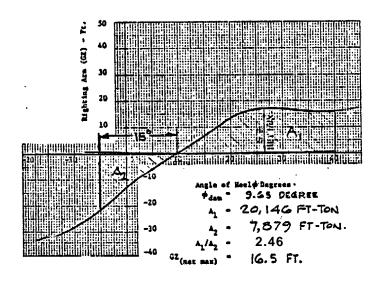


Figure 2.2.5-S (U): Damaged Stability 50 Percent LOA Unsymmetrically Flooded (English Units) (U)

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DRING, NO. : N/A

DAHAGE STABILITY ANALYSIS

PAGE 5

3KSES

LBP : 73.15 m

BEAM | 12.92 m NO. OF STATIONS | 51

50% LOA DAMAGE : FULL LOAD NOMINAL CONDITION

PLOCDING IN CONFARTMENTS III+IV+V+. T PENETRATION TO 8.94 LONG. BUD.

UNSTRUCTRICAL FLOODING

SHIP IN PINAL COMPITION

FINAL DÉAFT (LCF) = LCG IN FINAL CONDITION = LCF IN FINAL CONDITION =

6.95 abv. rrvl medied from FP medie b from be

FINAL DISPLACEMENT TOTAL TRIM 2973.27 CU-m 17.56 m FROM FP 0.15 M

DATA FOR LOST BUOTANCY

LOST BUOYANCY - VCG OF LOST BUOYANCY -

1654.13 CU-A 3.49. ABV RRWL

LCG.OF LOST BUOYANCY

45,49 m FROM PP 14.24 FROM RRVL

FINAL DRAFT AT AF FINAL REEL SQ IN FINAL CONDITION =

7.02. NIV RRWL 9.65 DEGREES 7.20 m ABV RRWL

FINAL DRAFT AT FP PRINT-BURFACE CORRECTION DAMAGE RIGHTING ARM (GI)

6.66 . ABV RRWL 1097.65 m-4 0.00 .

DISTANCE FROM F.P.

RESIDUAL FREEBOARD 2.16 m

HEEL TO MARGINLINA

· · 1.71 DEGREES

73.15 m

RIGHTING ARMS -VS- HEELS IN DANAGE CONDITION

-10.000 -8.552

-3.000 n.oon -6.366 -3.006

55,000 DECREES

The state of the s

NOTE: GI VALUES INCLUDES CORRECTION TERM 0.05008 (NIEL) TO ACCOUNT FOR UNKNOWN UNSYMMETRICAL MOMENT NG VALUE INCLUDES ALLOWANCE FOR FREE-SURFACE EFFECT FOR SLAK TANKS

SEA WATER DENSITY - 1.025 METRIC TONS/CU-M

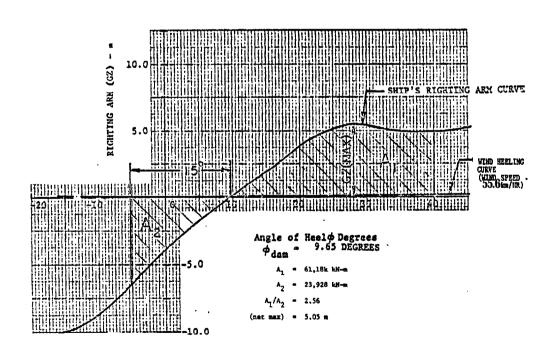


Figure 2.2.5-9 (U): Damaged Stability 50 Percent LOA Unsymmetrically Flooded (SI Units) (U)

57

- (U) 2.2.5.2 Static Stability Underway
- (U) 2.2.5.2.1 Off-Cushion Stability Underway -- Predicted off-cushion static pitch and roll stability characteristics for the ANVCE near term SES are presented in Figures 2.2.5-10 and 2.2.5-11, respectively. The ship has positive static stability with pitch and roll restoring gradients of approximately 99 x 10⁶ ft-lb/degree (134.23 x 10⁶ N·m degree) and 22 x 10⁶ ft-lb/degree (29.83 x 10⁶ N·m/degree), respectively. In the off-cushion mode, the SES is statically unstable in yaw but dynamically stable, thus providing satisfactory course keeping characteristics as influenced by the ride control system in a seaway.
- (U) 2.2.5.2.2. On-Cushion Static Stability Underway -- The predicted on cushion static stability data presented next shows that the ANVCE near term SES has positive stability in roll, pitch and yaw. Roll and pitch stability are shown at 40, 60 and 80 knots (20.58, 30.87 and 41.16 m/s); yaw stability data for 60 and 80 knots (30.87 and 41.16 m/s) only. The stability characteristics shown are for a nominal displacement of 2800 LT (27,899.2 RN) which approximates the Full Load Displacement condition.
- (U) The positive on-cushion pitch stability of the SES at 40, 60 and 80 knots (20.58, 30.87 and 31.16 m/s) is shown in Figure 2.2.5-12. Predictions are plotted with zero moment occurring at the nominal pitch trim attitude for each speed (the ship is trimmed at the pitch attitude for minimum drag consistent with non-broaching operation). Speed variation at a constant weight primarily alters the minimum-drag pitch attitude. These predictions were derived by Froude scaling hydrodynamic model test data without other correction. Positive static stability is indicated by the degree of negative gradient of the moments with their corresponding attitudes.
- (U) The average pitch restoring moment is approximately 18 x 10⁶ ft-lb/degree (24.40 x 10⁶ N·m/degree) for all speeds shown. The minimum gradient of about 8 x 10⁶ ft-lb/degree (10.85 N·m/degree) occurs on the curve for 40 knots (20.58 m/s).

- The yaw stability characteristics are shown in Figures 2.2.5-13 and 2.2.5-14 at speeds of 60 and 80 knots (30.87 and 41.16 M/S) for three (3) pitch attitudes (1, 0 and +1 degrees) and at two (2) angles of roll (0 and +2 degrees). Positive static yaw stability is shown for all conditions except the high speed, negative pitch case (80 knots (41.16 m/s) and -1 degrees. *rim). However, extrapolation of the dynamic stability indicates that the near term SES will be dynamically stable, in the directional sense, to bow down pitch angles as large as -2 degrees. In actual operation, a bow down trim attitude of this magnitude is difficult for the ship to achieve, and even more difficult to maintain. Strong pitch restoring moments ensure a rapid return to nominal attitudes even under failure mode conditions.
- and 80 knots (20.58, 30.87 and 41.16 m/s) is shown in Figures 2.2.5-15 through 2.2.5-19, respectively. **Predictions** are plotted for pitch attitudes of zero and plus and minus 1 degree and for yaw angles of zero, -2 and -4 degrees. The roll restoring moment gradients vary slightly with speed and ship attitude. The maximum gradient shown is approximately 4.3 x 10⁶ ft-lb/degree (5.83 x 10⁶ N•m/degree) (at 80 knots) (41.16 m/s); the minimum is about 2.6 x 10⁶ ft-lb/degree (3.53 x 10⁶ N•m/degree) which occurs at 40 knots (30.87 m/s) at a -4 degree yaw angle. The principal roll restoring moments are due to the **sidehull** design.
- (U) Two of the more significant features which contribute to the excellent stability characteristics of the ANVCE near term SES are the seal and **sidehull** designs, The Rohr advanced planing seals maintain their geometric integrity at all times, even in high sea states. The design precludes slope reversal in the pitch stability curve ("pitch clicks"), as exhibited

on the **100B** testcraft. The design also precludes catastrophic plow-in characteristics exhibited by other type seal designs at bow down attitudes.

- (U) The *nced pl: ing seal design increases the effective cushion length as a direct function of bow immersion; as the bow goes down, the effective cushion length boundary moves forward, providing additional pitch and roll restoring moments.
- (U) The design stiffness of the seals is a careful balance between stability requirements and ride quality. The Rohr design provides a degree of **stiffness** which maintains adequate roll and pitch stability while providing good ride qualities.
- (U) The sidehull forward sections contribute additional pitch and roll restoring moments at bow down attitudes in the same way as the advanced planing bow seal. This effect is obtained by designing the bow stem to match the bow seal contour. In addition, the stem angle minimizes destabilizing moments at bow down attitudes. The low (45 degree) deadrise angle of the sidehull design provides better pitch and roll stability than higher deadrise sidehull configurations.

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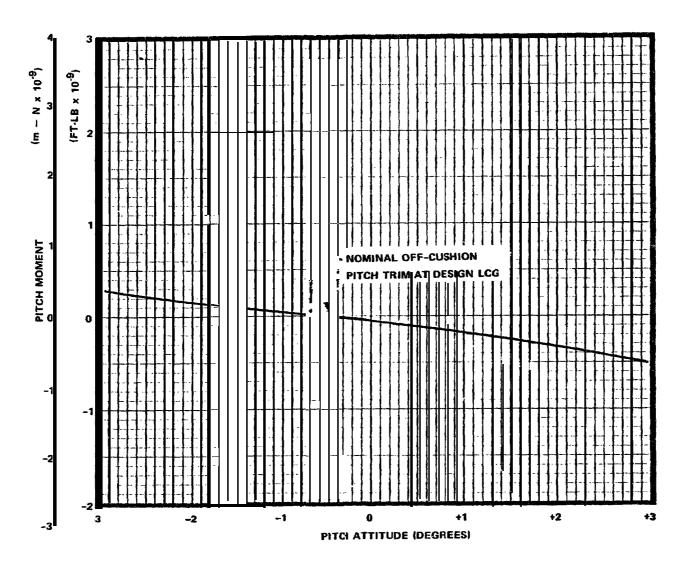


Figure 2.2.5-10 (U): 3KSES Static Pitch Stability, 83 Percent Fuel Condition (2800 LT; 27,899.2 kN), Off Cushion (U)

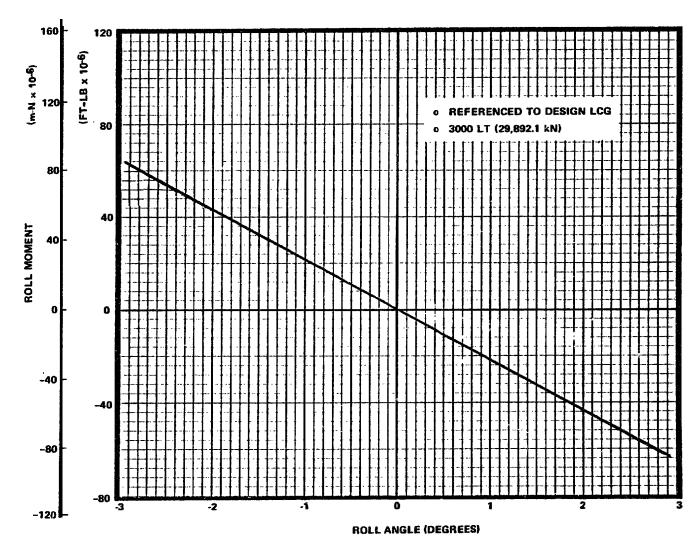


Figure 2.2.5-11 (U): 3KSES Static Roll Stability, 83 Percent Fuel Condition (2800 LT; 27,899.2 kN), Off Cushion (U)

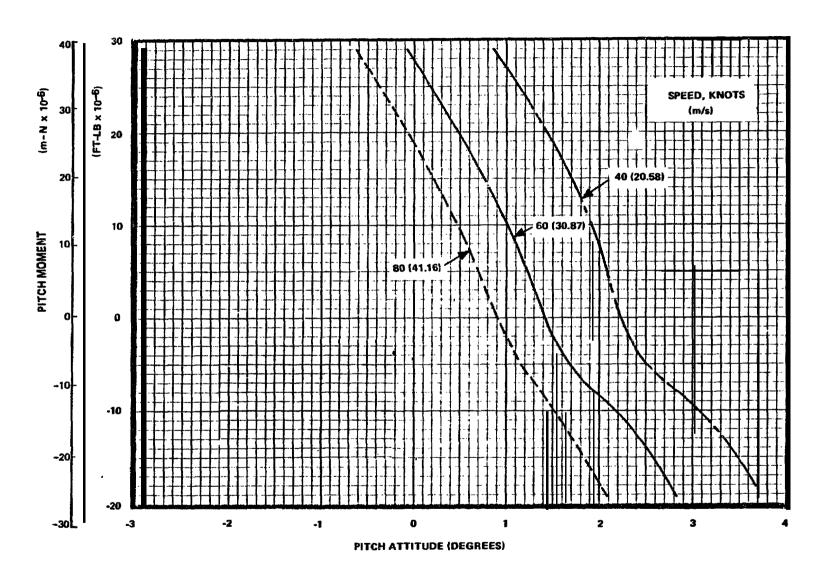


Figure 2.2.5-12 (U): 3KSES Static Pitch Stability, 83 Percent Fuel Condition (2800 LT; 27, 899.2 kN), on Cushion (U)

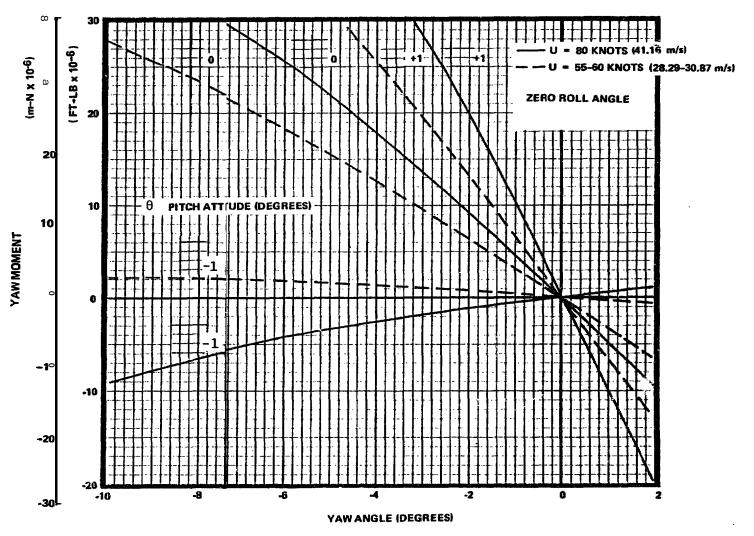


Figure 2.2.5-13 4: 3KSES Static Yaw Stability, 83 Percent Fuel Condition (2800 LT; 27,899.2 kN), on Cushion (U)

4, b',

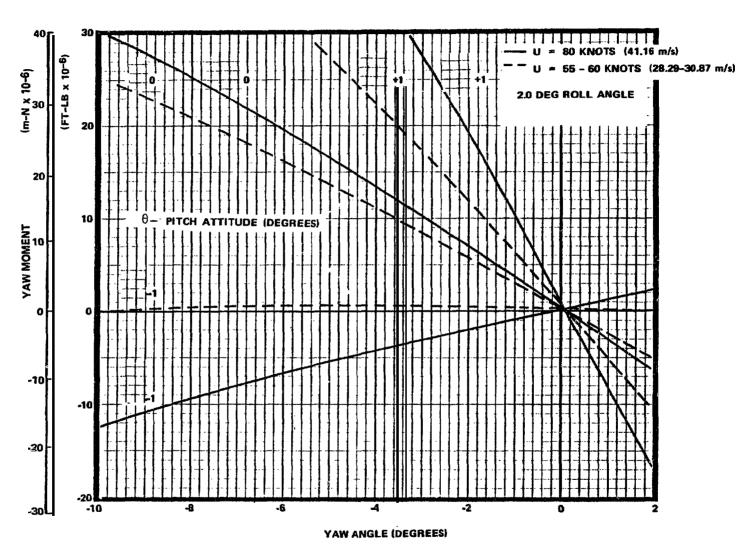


Figure 2.2.5-14 (U): 3KSES Static Roll Stability, 83 Percent Fuel Condition (2800 LT; 27,899.2 KN), On Cushion (U)

Figure 2.2.5-15 (U): 3KSES Static Yaw Stability, 83 Percent Fuel Condition (2800 LT; 27,899.2 kN), On Cushion (U)

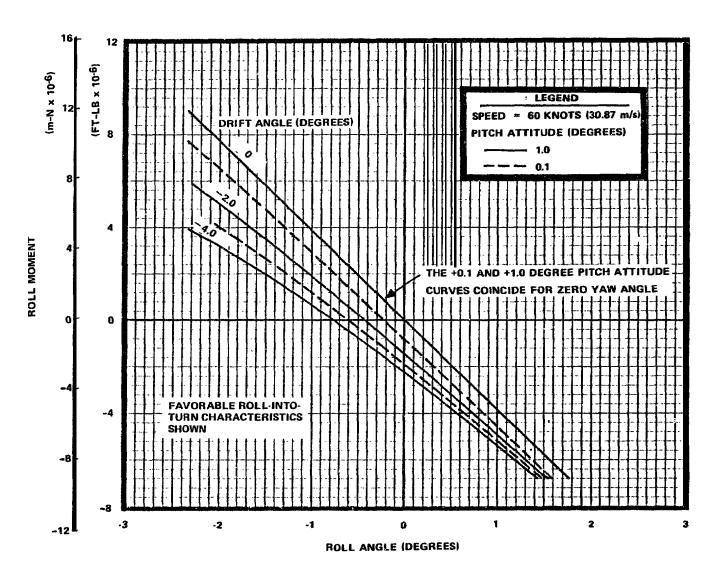


Figure 2.2.5-16 (U): 3KSES Static Roll Stability, 83 Percent Fuel Condition (2800 LT; 27,899.2 kN), On Cushion (U)

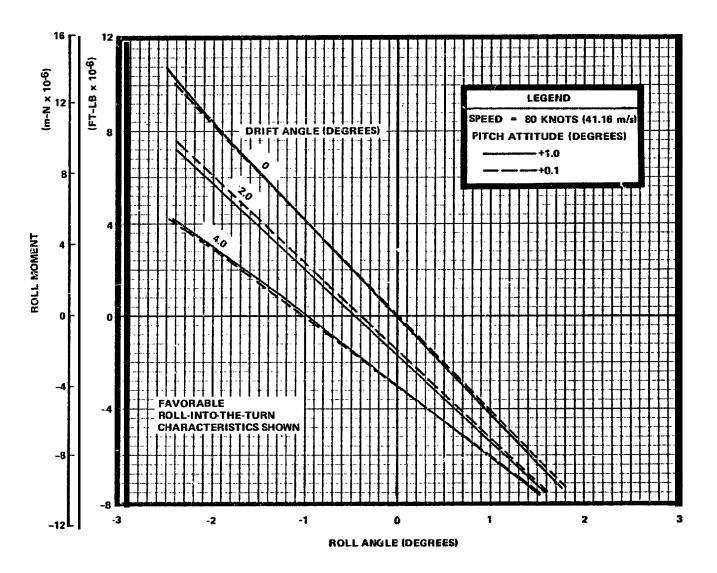


Figure 2.2.5-17 (U): 3KSES Static Roll Stability, 83 Percent Fuel Condition (2800 LT; 27,899.2 kN), On Cushion (U)

(U) 2.2.5.3 Dynamic Stability -- The predicted on-cushion dynamic stability characteristics for the near term SES are presented in Table 2.2.5-3. The values shown in the table are for the ship without ride control. Nominal pitch trim angles are included in the table for reference, At 40 knots (20.58 m/s), the yaw (directional) mode is overdamped for the nominal pitch trim angle; responses in yaw may then be characterized as a first order system with a time constant of about 4 seconds.

Table 2.2.5-3. (U) 3KSES Dynamic Stability Characteristics (U)

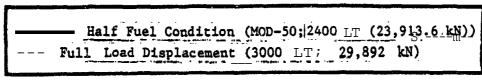
AXIS	SPF KNOTS		NOMINAL TRIM DEGREES	FREQUENCY HZ	DAMPING RATIO
ROLL	4 0	20.58	2.2	0.17	0.15
	6 0	30.87	1.4	0.18	0.14
	8 0	41.16	0.9	0.19	0.13
PITCH	4 0	20.58	2.2	0.19	0.22
	6 0	30.87	1.4	0.20	0.20
	8 0	41.16	0.9	0.21	0.16
YAW	4 0	20.58	2.2	*	*
	6 0	30.87	1.4	0.15	0.80
	8 0	41.16	0.9	0.07	0.45
HEAVE	4 0	20.58	2.2	0.65	0.28
	6 0	30.87	1.4	0.66	0.29
	8 0	41.16	0.9	0.68	0.31

*Overdamped, $\tau = 4.0$ seconds

- (U) Figure 2.2.5-18 presents the significant pitch angle deviations as a function of speed and significant wave height. These data are based on analytic modeling of the ship's vertical plane dynamic characteristics as influenced by the ride control system in a seaway.
- (U) Figures 2.2.5-19 through 2.2.5-21 present the significant roll angle deviations with speed, significant wave height, and seaway heading.

- (U) These data are based on hydrodynamic model test data collected in the DTNSRDC maneuvering basin.
- (U) Figure 2.2.5-22 presents the limit allowable drift angle as a function of speed. The angles are limited by propulsive control yaw moments available below hump and by operating boundaries (dictated by roll-yaw stability) above hump. The limit angles shown were derived from XR-1D testcraft model data and inlet broaching studies. (1)

(1) Barker, J., et al, "XR-1D Safety and Performance Prediction Report", Rohr Industries, Inc., Rept. No. RHR-75-266, 22 Aug. 1975 (Fig. 4-11 and 4-12).



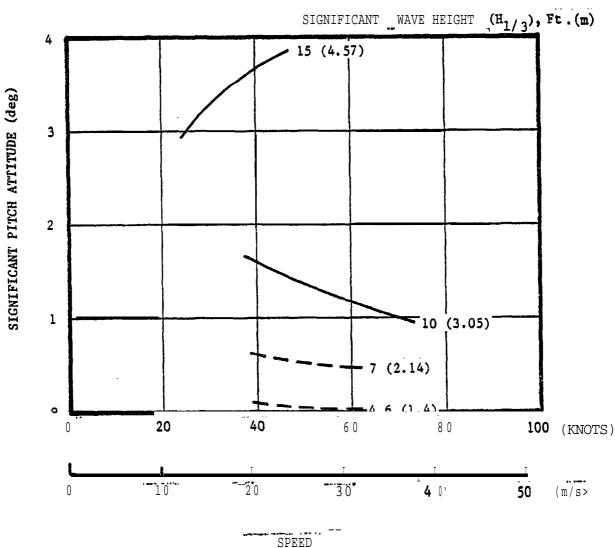


Figure 2.2.5-18 (U): 3KSES Pitch Deviation Versus Speed Head Seas (U)

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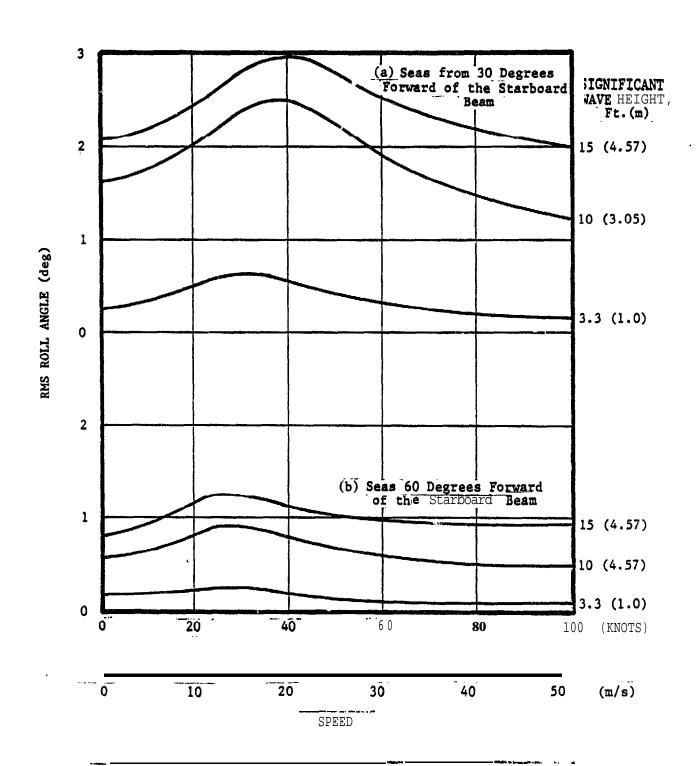


Figure 2.2.5-19 (U): 3KSES Roll Deviation Versus Speed (U)

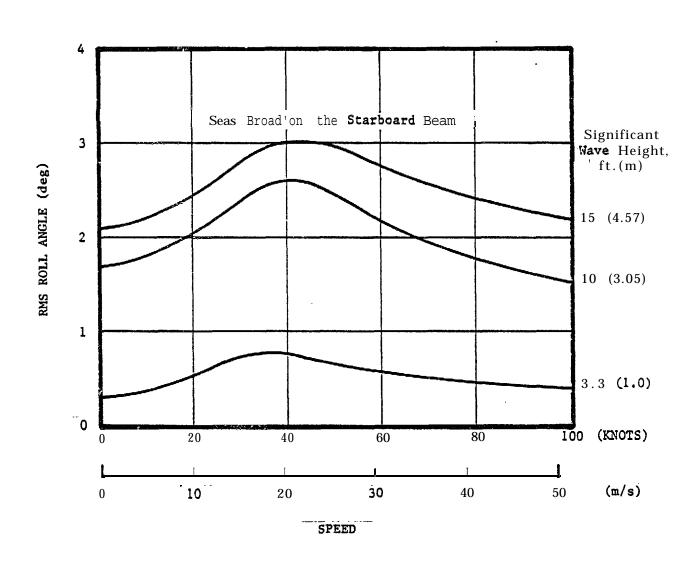


Figure 2.2.5-20 (U) : 3KSES Roll Deviation Versus Speed (U)

Seas from 30 Degrees Abaft the Starboard Beam

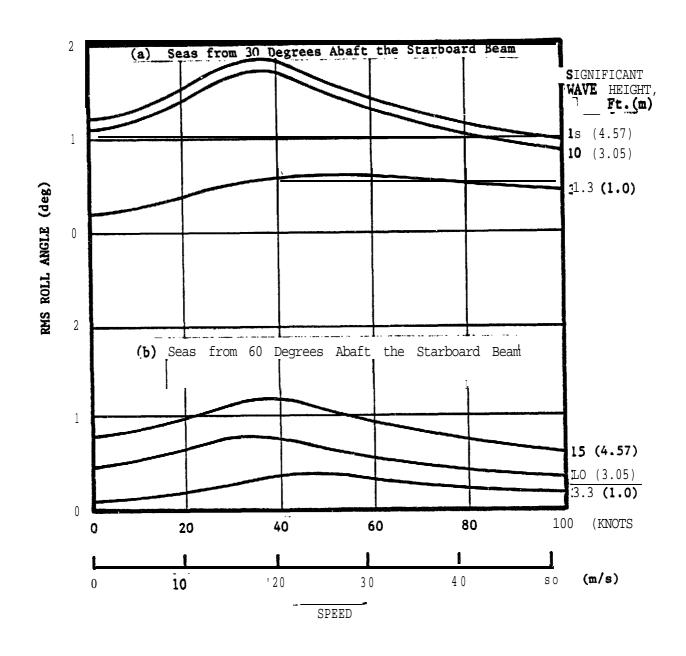


Figure 2.2.5-21(U): 3KSES Roll Deviation Versus Speed (U)

Full Load Displacement (3000 LT; 29,892.1 kN) Calm Water, On-Cushion

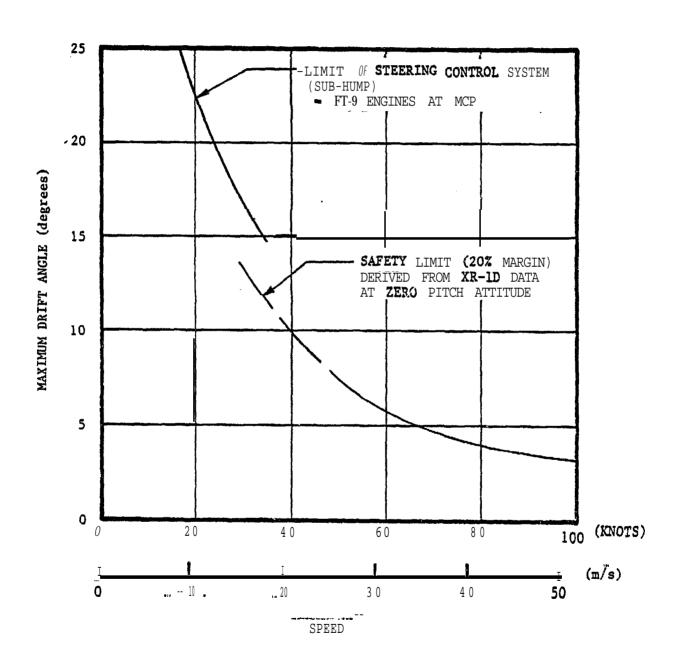


Figure 2.2.5-22 (U): Drift Angle Limit Versus Speed (U)

- (U) 2.2.6 GEOMETRIC FORM -- The geometric form of near term SES is described by the hull lines and the control surface drawings of this section.
- (U) 2.2.6.1 Hull Geometry -- The selection of the hull form is based on judicious compromises between overall hullborne and cushionborne performance; structural strength; manufacturing economy; volumetric requirements; combat suite; safety, survivability and efficiency of ship operations. The net result is shown in the lines drawing, Figure 2.2.6-1.
- (U) The **sidehull** geometry is based on the effects of **deadrise** and ventilation cutouts on the overall hydrostatic and hydrodynamic performance parameters, bow seal interface, **waterjet** inlet configuration and structural atrength requirements. Hydrodynamic drag considerations have influenced the choice of a slender body **sidehull** concept.
- (U) The full-length sidehulls enclose the sides of the bow seal, decreasing seal vulnerability to damage as compared with exposed bag and finger seal systems on partial-length **sidehulls.** The full-length **sidehull** vertical inner face also permits a simple bow seal/sidehull interface and allows the use of a two dimensional, modularized bow seal system.
- (U) 2.2.6.2 Principal Dimensions -- The principal dimensions, as related to the proportions and form characteristics of the sidehulls

and the centerbody, are based on the following considerations:

- Provision for the required cushion area in conjunction with space requirements for main propulsion machineries and waterjet inlets. The Panama Canal transit requirement established the maximum beam of 108 feet (32.918 m). The trace of the maximum beam follows 4 feet (1.219 m) above the upper chine and is canted inboard to the main deck and 01 level. ** nominal tumblehome at Station 10 is 3 feet 7 inches (1.092 m).
- The overall length of 266 feet 3 inches (81.153 **m)** was established from the maximization of performance parameters as related to cushion length to beam ratio, bow and stern seal geometry design, overall utility, and volumetric requirements.
- The wet deck height was selected at 18 feet (5.486 m) above baseline to minimize **wetdeck** slamming and cushion induced dynamic response, The wet deck is horizontal except forward of Station 4 where it ramps upward to minimize pitch induced slam loads and to provide a flat interface with the forward **seal** in its retracted position,
- The selection of main deck height at 40 feet 0 inches (12,192 m) above baseline was based on requirements of hull girder strength, reserve buoyancy in damage situations, and overall volumetric and space demands. The high main deck also provides a drier environment for engine air intakes and for helicopter operation, relative to lower main deck configurations that were evaluated.
- (U) 2.2.6.3 Control Surfaces -- The baseline design of the near term

 ANVCE SES incorporates two stern-mounted stabilizing fins, port and
 starboard, canted 28 degrees inboard from the bottom of the fence, as shown in
 Figures 2.2.6-1 and 2.2.6-2. Fin section geometry is shown in Figure
 2.2.6-3.

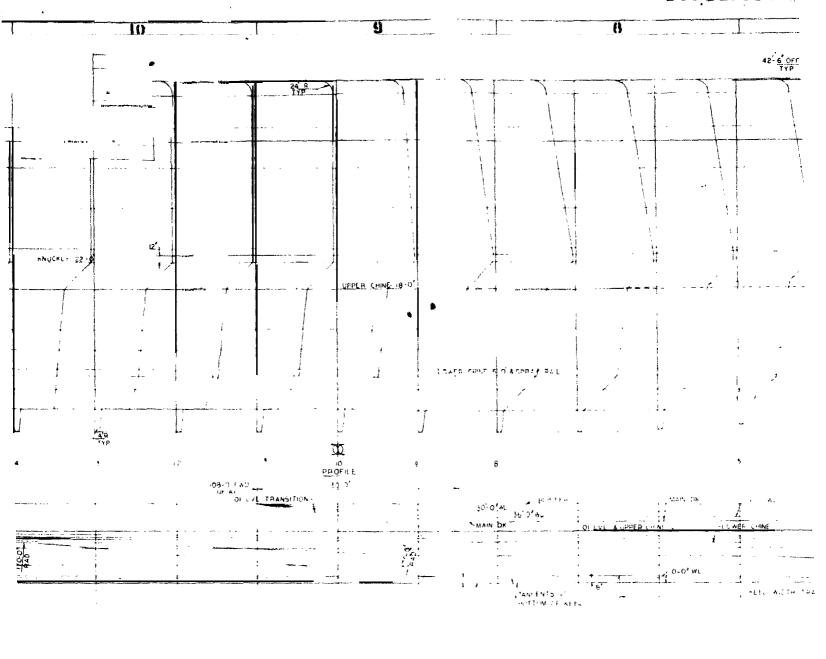
```
1. PRINCIPAL HYDROSTATIC PARAMETERS (OFF CUSHION)
    BLOCK COEFFICIENT (C,), FULL LOAD = 0.1662
    PRISMATIC COEFFICIENT (Pc), FULL LOAD = 0.8341
    NETTED SURFACE, FULL LOAD = 18,440 FT<sup>2</sup> (1713 m<sup>2</sup>)
TRANSVERSE KM, FULL LOAD = 273.81 FT (83.45 m)
    VERTICAL CENTER OF BUOYANCY (KB) = 10.62 FT (3.24 m)
TONS PER INCH IMMERSION (TPI) = 43.91 TONS/INCH (17.23 kN/m)
    LONGITUDINAL CENTER OF FLOTATION (LCF) = 137.11 FT (41.79 m)
      (FROM FP)
2. PRINCIPAL HYDRODYNAMIC PARAMETERS (ON CUSHION)
                                                 = 221 FT (67.36 m)
    CUSHION LENGTH
    CUSHION BEAM
                                                85 FT (25.91 m)
                                                = 18 FT (5.48 m)
    CUSHION HEIGHT
                                                = 118 FT (35.89 m)
    LONGITUDINAL CENTER OF GRAVITY
     (FWD OF TRANSOM)
                                               = 2.60
    CUSHION LENGTH/BEAM
                                               4.72
    CUSHION BEAM/HEIGHT
```

 $(\rho_{SU} = 1.025 \text{ METRIC TON/CU m})$

Figure 2.2.6-1 (U): ANVCE-JES Hull Lines Drawing LL802009 (Sheet 2 of 2)

UNCLASSIFIED 42'-0'W 36-0 W 30 - OWL 24 . O'WL 8 OM 9.0 12'- 0'm 9 - 0 W 6'-0" w MUD BASELINE APPENDAGE SEE DET 4-8 50 0 BL 75-0" WL 45 0 84 42-6' BL 30'-Q' BL 20<u>-0 B</u>L 10-0'BL UNCLASSIFIED 12

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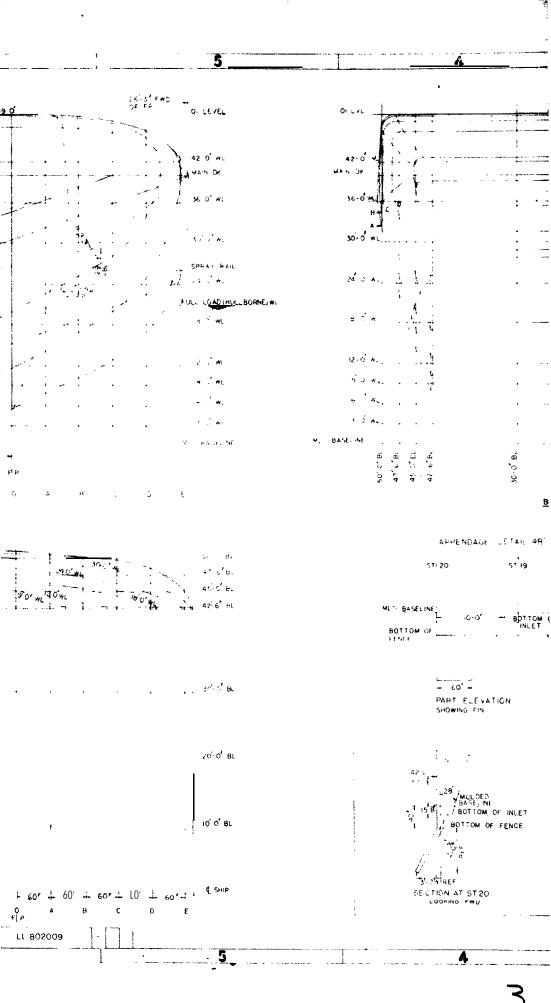


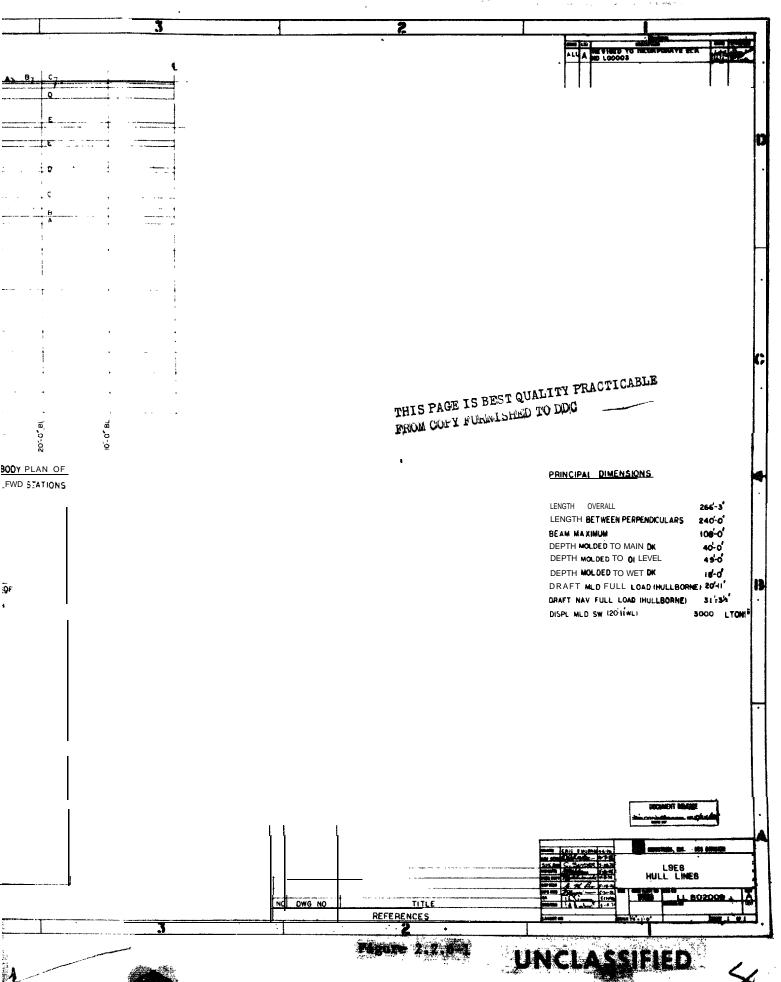
14 12 H 10 9. B 1 6

STATION SPACING 12-0'
PLAN

9 9 1 6

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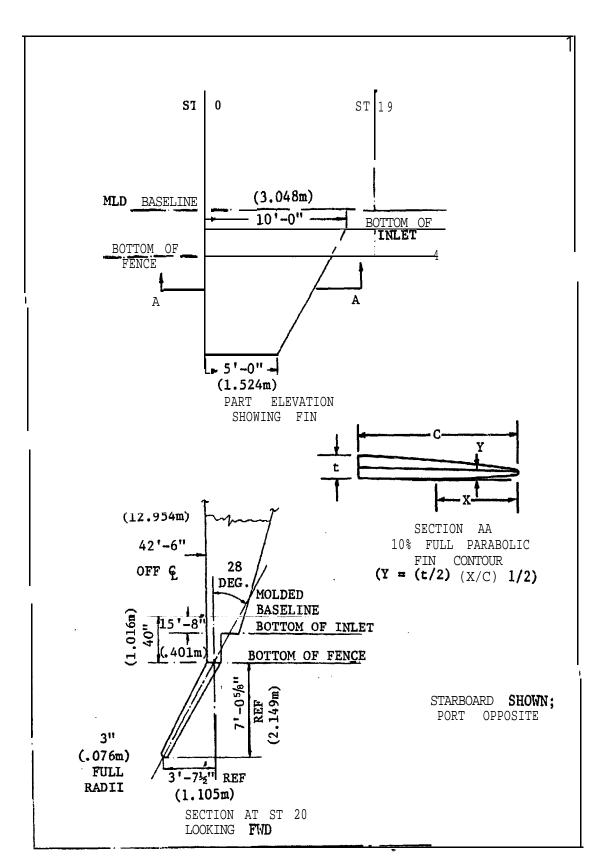


Figure 2.2.6-2 (U): Baseline Stabilizer Fin Geometry (U)

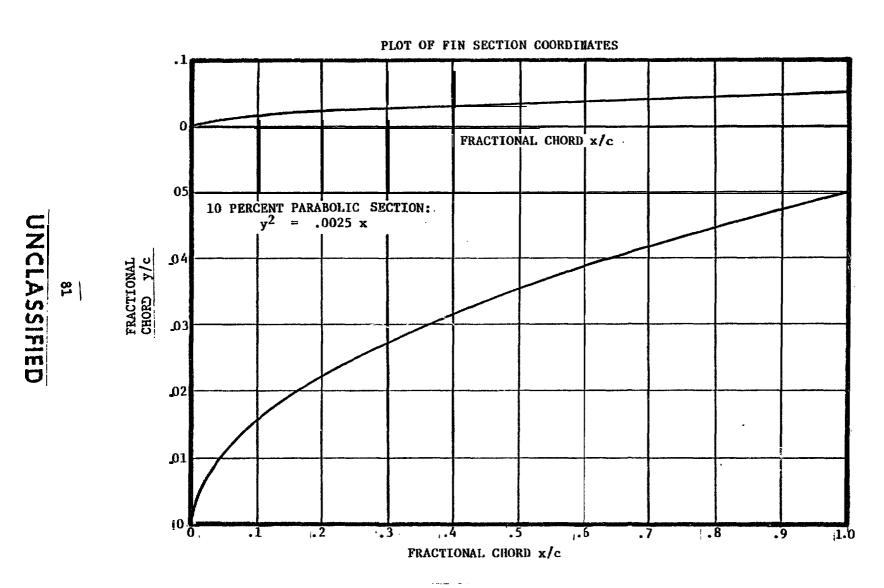


Figure 2.2.6-3 (U): Fin Section Geometry (U)

(U) 2.2.7 RIDE QUALITY

- 2.2.7.1 Near Term SES Ride Criteria Near term SES high speed operation in high sea states can result in vertical vibration modes not previously sustained by man over long periods of time. While considerable data exist on vibratory effects upon man, the heave acceleration environment of the near term SES centers in a portion of the frequency regime for which characterizing data are sparse.

 Certain near term SES resonances are predicted between 0.1 Hz and 5 Hz in the pricise range where human performance capability is most affected; primarily because the human body resonates at about 5 Hz., and because sailors may become seasick when ship motions traverse 0.1 to 0.5 Hz at energy levels above 0.06 g's (rms).
- (U) The primary purpose of developing a ride criteria is to establish the motion limits that can be tolerated by operations, maintenance and off-duty crew for specific mission durations. The importance of these criteria is to ensure a reasonable level of operating efficiency if craft motions are maintained at or below the limits.
- (U) The curves illustrated in Figure 2.2.7-1 were established from a comprehensive literature search by overlaying graphical data representing human performance decrement studies. The search encompassed hundreds of previous motion studies, experiments and simulations related to the adverse effects of vibratory environments on human performance. These data form the data base for the ride criteria, categorized by specific task type and correlated by rms g's versus the center frequencies of the one-third octave band. Although considerable vibration data and criteria exist above 1 Hz, very little is available to describe the effects on humans between 0.1 and 1 Hz. This influences the ride criteria since the predicted near term SES heave acceleration environment tends to center in this portion of frequency regime.

- (U) Although the data points cover vastly different conditions and show varying degrees of performance or motion sickness, trends were established for short term and long term conditions. Trend lines were compared with all other data points and with previously developed habitability criteria to establish firm ride criteria.
- (U) The present ride criteria represent 30 minute and 4 hour duration tolerance limits for adapted crews with ten to twenty percent expected performance decrement. In the frequency region of 0.1 to 0.5 Hz, ten percent of the crew could be expected to have some motion sickness. The actual task performance decrement of one of the ten percent displaying sickness might mean slower performance, increased errors or complete non-performance of assigned duties.
- (U) The identification of the kind and level of performance decrement expected must consider the specific tasks to be performed. The reduced tolerance between 1.0 and 10 Hz refers primarily to tracking tasks decrement. The operation of a decimal input device (with proper arm support and restraints) would suffer no performance decrement at motion levels near or even slightly above the ride criteria curves.
- (U) 2.2.7.2 Near Term SES Ride Quality -- Figures 2.2.7-2 through 2.2.7-5 present the frequency spectra of the heave acceleration levels at an **amidship** station with the ride control system both on and off. The power expenditure to control the ride is tabulated in Figures 2.2.7-4 and 2.2.7-5.
- (U) The influence of the ride control. system on the RMS heave acceleration levels at a speed of 60 knots (30.87 m/s) and 6.9 feet (2.1 m) significant wave height is illustrated in Figure 2.2.7-6. Then in Figure 2.2.7-7 through 2.2.7-13, the RMS vertical plane acceleration levels near the bow, amidship, and at the stern are plotted versus seaway heading for a number of speeds and significant wave heights. These data are based on hydrodynamic model test data collected in the David Taylor Naval Ship Research and Development Center (DTNSRDC) maneuvering basin.

8 3

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OPERATING CONDITIONS:
2800 LT (27,899.25 kN) DISPLACEMENT
70 KNOT (36.01 m/s) SHIP SPEED
SEA STATE 5

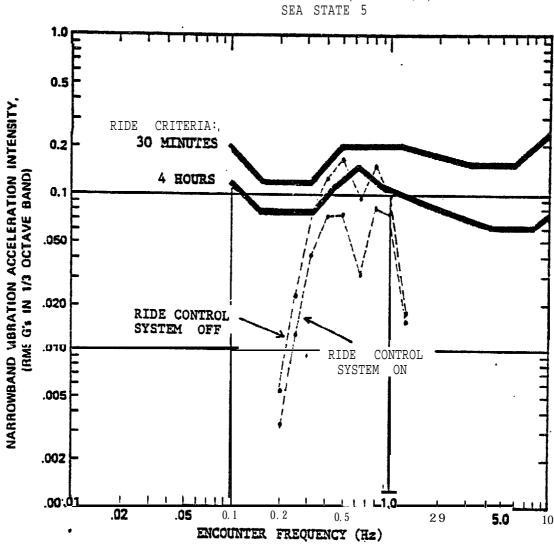


Figure 2.2.7-1 (U) The SES Controls Ship Vertical Motions (U)

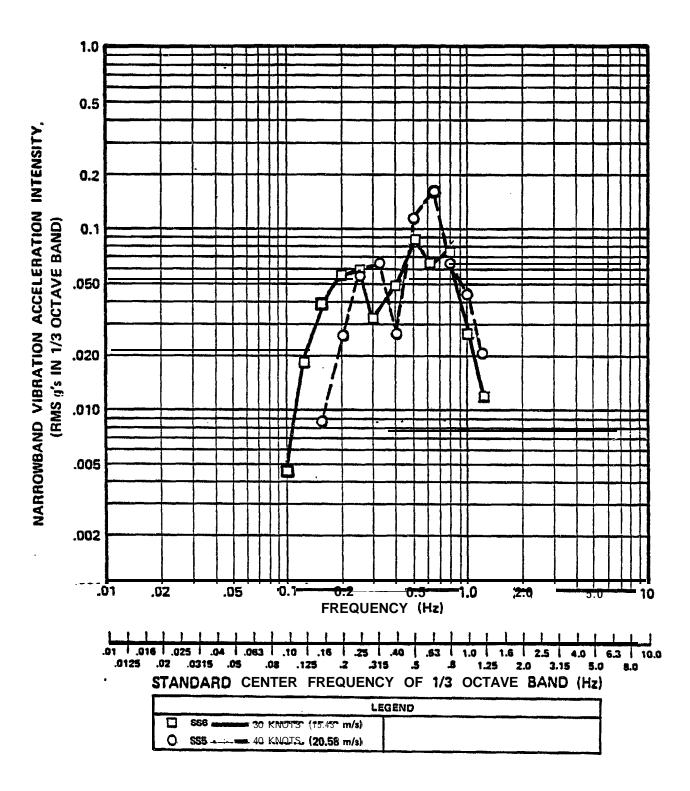


Figure 2.2.7-2 (U): 3KSES Half Fuel Condition (MOD-SO) 2400 LT (23,913.6 kN), Uncontrolled Ride (U)

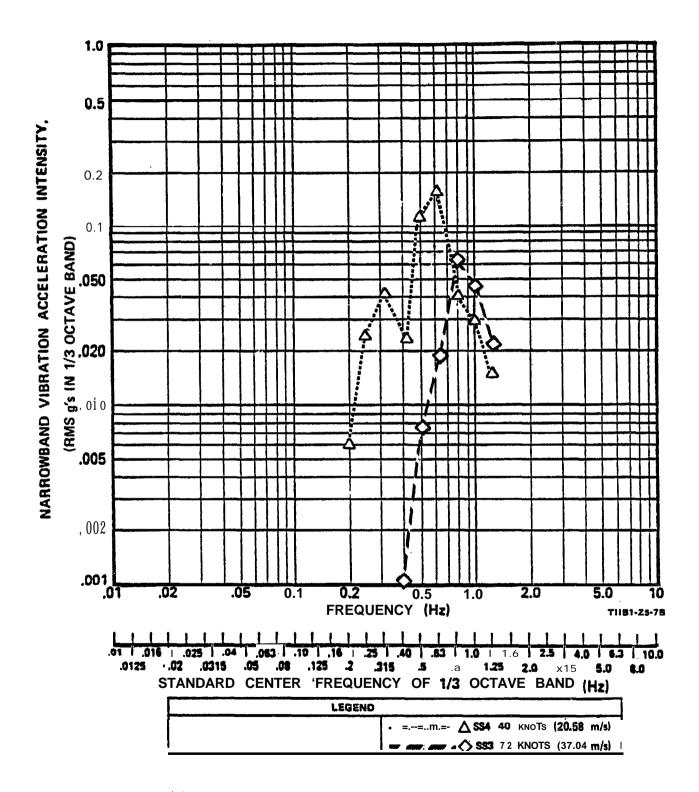


Figure 2.2.7-3 (U): 3KSES Full Load Displacement, 3000 LT (29,892.1 kN)
Uncontrolled Ride (U)

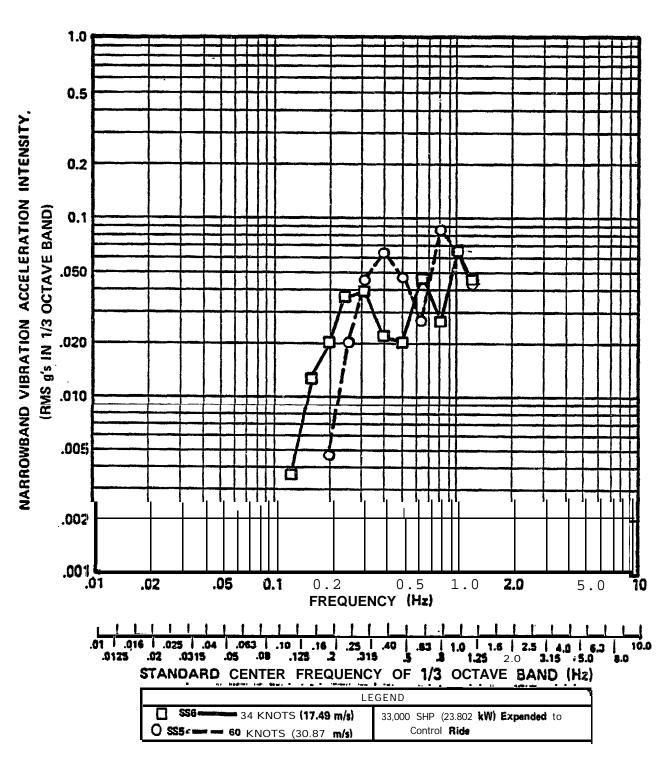


Figure 2.2.7-4 (U): 3KSES Half Fuel Condition (MOD-50) 2400 LT (23,913.6 kN), Controlled Ride (U)

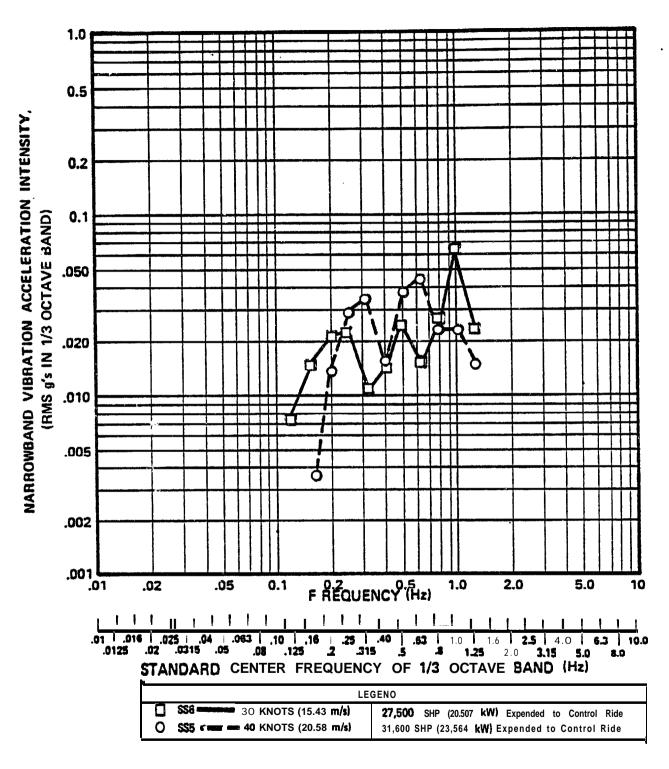


Figure 2.2.7-5 (U): 3KSES Half Fuel Condition (MOD-50) 2400 LT (23,913.5 kN), Controlled Ride (U)

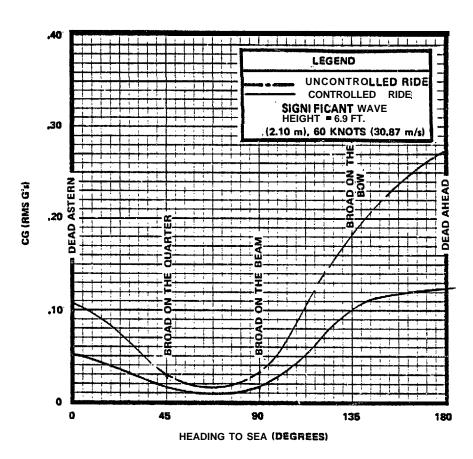


Figure 2.2.7-6 (U) 3KSES Variation in Vertical CG Acceleration with Heading at Full Load Displacement (U)

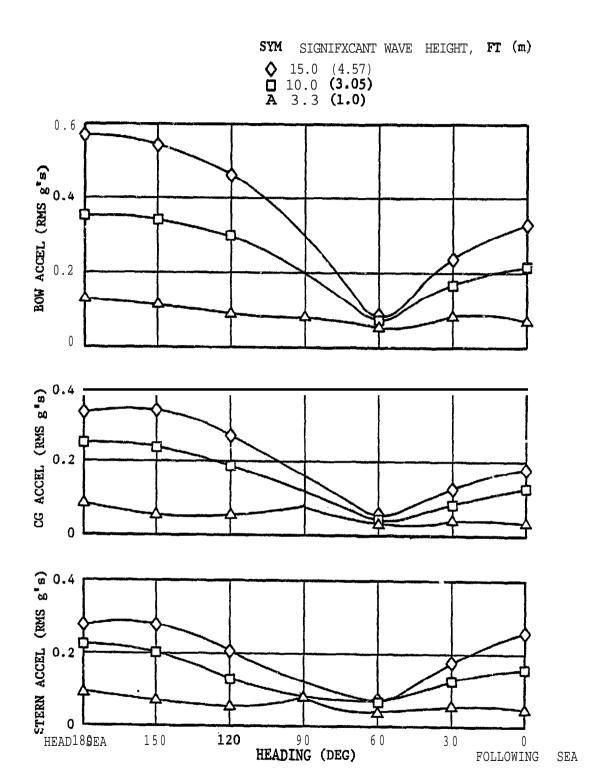


Figure 2.2.7-7 **(U):** 3KSES Variation of Bow, CG, and Stern Vertical Plane Accelerations with Heading at 60 Knots (30.86 m/s) (U)

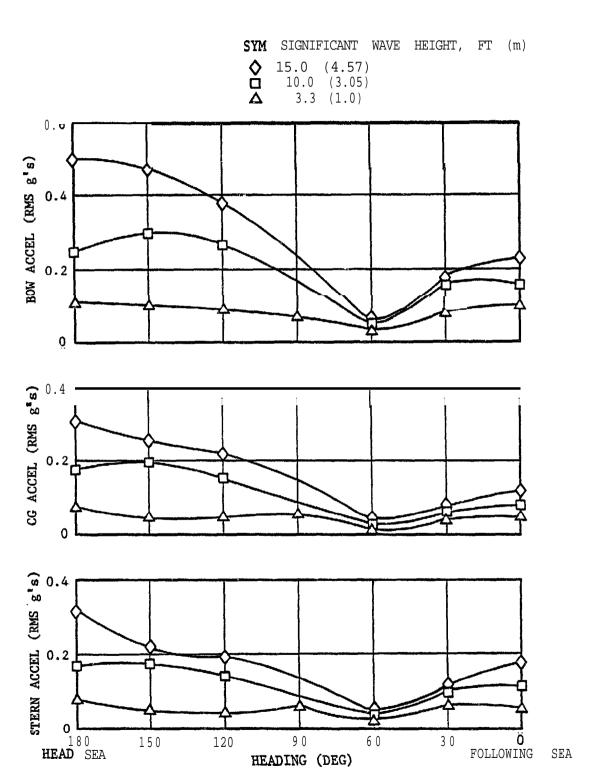


Figure 2.2.7-8 (U): 3KSES Variation of Bow, CG, and Stern Vertical Plane Accelerations with Heading at 50 Knots (25.72 m/s) (U)

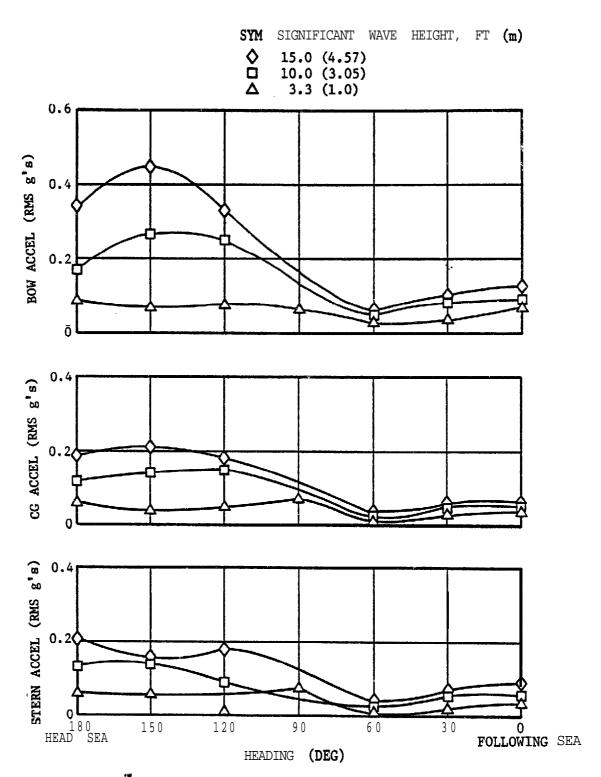


Figure 2.2.7-9 (U): **3KSES** Variation of Bow, CG and Stern Vertical Plane Accelerations with Heading at 40 Knots (20.58 m/s) **(U)**

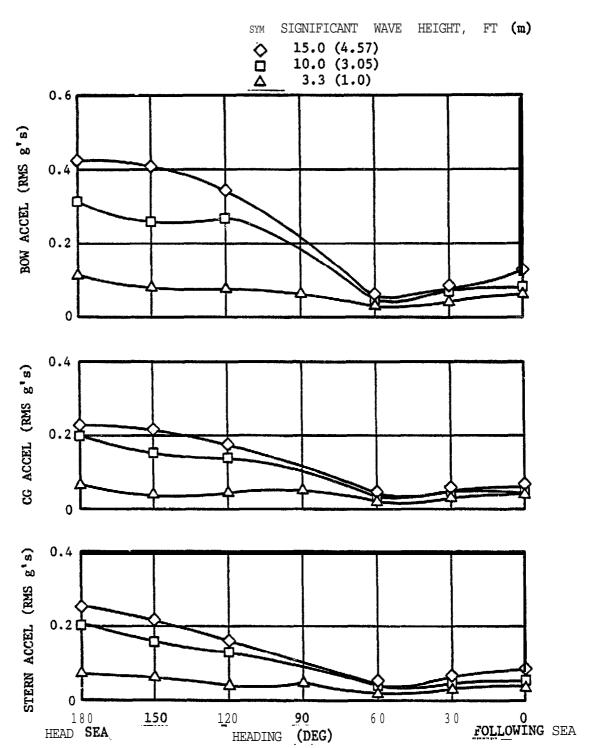


Figure 2.2.7-10 (U): 3KSES Variation of Bow, CG, and Stern Vertical Plane Accelerations with Heading at 35 Knots (18.01 m/s) (U)

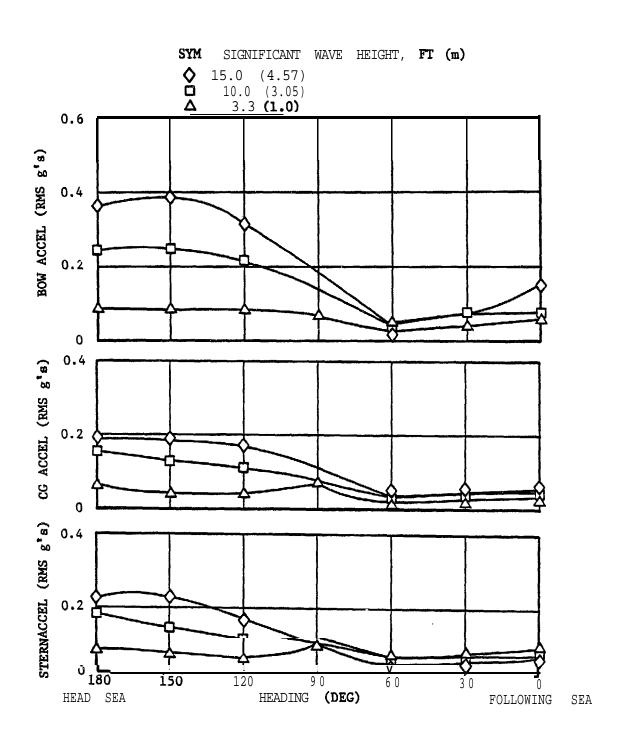


Figure 2.2.7-11 (U): 3KSES Variation of Bow, CC, and Stern Vertical Plane

Accelerations with Heading at 32 Knots (16.46 m/s) (U)

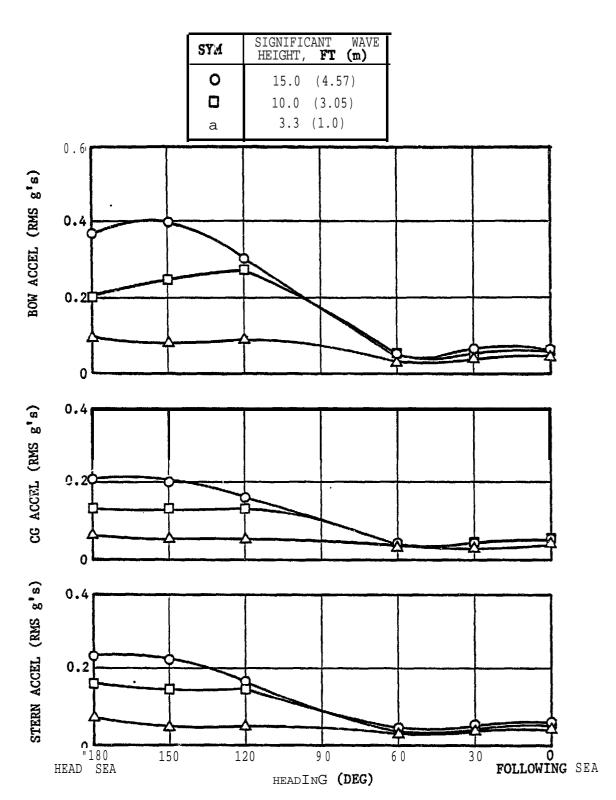


Figure 2.2.7-12 (U): 3KSES Variation of Bow, CG, and Stern Accelerations with Heading at 29 Knots (14.92 m/s) (U)

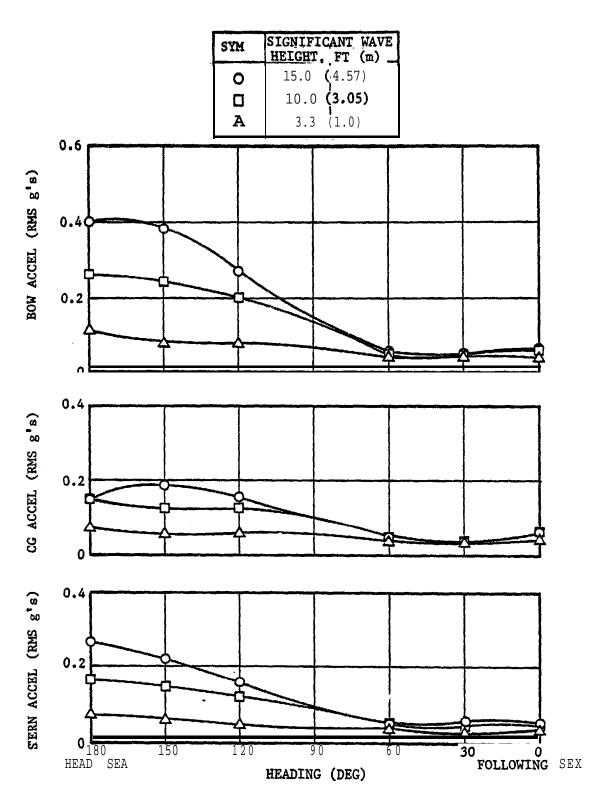


Figure 2.2.7-13 (U): 3KSES Variation of Bow, CG and Stern Accelerations with Heading at 26 Knots (13.38 m/s) (U)

- **(U)** 2.2.8
- MANNING
- (U) The anning presented herein delineates the minimum quantitative and qualitative personnel essential to the operation, maintenance and support of the near term SES under stated missions and configurations.

 These requirements are termed Organizational Manning and were developed in general accordance with the "Guide to the Preparation of Ship Manning" document, OPNAV 10P-23.
- (U) The developed manpower requirements are sufficient for performing all operational, maintenance, administration and support tasks required for the near term SES under the following Readiness Conditions: Special Condition I (Battle Readiness) for Anti-submarine operations, Anti-air operations, and Surface operations, Condition IV (Peacetime Cruising Readiness) and Condition V (In-Port Readiness). Table 2.2.8-1 displays the manning requirements in the prescribed format.

Table 2.2.8-1 (U): Manning (U)

VEHICLE			
7 2112 - 119	OFFICERS	CPO_	OTHER_ENLISTED
	Commanding Officer	BMC	1 DS1
	Executive Officer	ENC	1 DS2
	Operations Officer	WCS	3 1242
	First Lieutenent	HMC	I EN]
	Combat Systems Officer	ICC	1 EN2
	Electronics Material Officer	QHC	1 ENFN
	Engineer Officer	RMC	i eti
	Damage Control Assistant	SKC	1 ETH2
			1 ETR3
			1 EX2
			1 FTM1
			1 FTH3
			1 GH242
			1 Gt9f3
			3 GS1
			I GS3
			1 GSFN
			1 HT1
			1 HTPK
			1 102
			1 ICFN
			1 KS1
			1 H82 2 HS3
			1 051
			1 0\$2
			1 055N
			I OT1
			1 073
			1 57(2
			3 QHSN
			1 8142
			1 SH3
			1 SE3
			1 YN1
			1 nt3
			6 SN
			1 PN
	08	08	50
SECONDARY	VEHICLE		
	P-14		
	Halicopter Pilot	YDJC	1 ADJ1
	Helicopter Pilot		1 AMH1
	Helicopter Co-Pilot		1 AM3
	Helicopter Co-Pilot		1 ANSI
			1 AT1
			1 AT3
			1 AE1 1 AX2
			1 AX2 2 AV2
			2 AW3
			1 A02
			1 AN
	04	01	14
TOTAL COM			
ī	12	09	64



- 2.3 SHIP SUBSYSTEM DESCRIPTIONS
- 2.3.1 STRUCTURE
- (U) 2.3.1.1 Summary -- The twin, full cushion length sidehulls of the near term SES are designed to be aerodynamically and hydrodynamically clean, and to contribute to good stability, maneuverability and performance characteristics. The ship houses the required weapon suite within its three (3) major decks and provides an operational helicopter capability. The survivability and reliability of the structural system is designed for 20 year life across the expected profile.
- (U) The hull structure includes the shell plating, framing, structural bulk-heads, decks, superstructure, structural closures, mast and foundations.

 The functional requirements of the hull structural system are: (1) to provide a watertight envelope which houses all other subsystems, (2) to provide a structurally sound platform suitable to the performance goals of the craft, (3) to provide an envelope that can be conditioned for crew comfort and utility, and (4) to provide a platform for aircraft and weapon system operations.
- (U) The hull structural configuration is a compromise between overall hullborne and cushionborne performance, manufacturing economy, functional space requirements, combat suite, habitability, survivability and safety within the overall constraint of meeting mission requirements. It is designed to meet a specified 20 year life requirement while retaining a realistic balance between minimum weight, structural reliability and cost construction.
- (U) The near term SES hull is subjected to a wide variety of loading conditions, including impact loads, while operating at high speed. These loads would normally required a conservative, heavy structure; however, near term SES performance requirements dictate a more sophisticated and lightweight structure. For convenience, structural loads are subdivided into Primary and Local load categories. Combinations of these categories provide the basis for the development of structural design.

- (U) Primary loads are defined as those loads affecting the entire hull structural girder. These include overall hull bending, torsion and shear resulting from ship weight, and hull buoyancy distributions when the ship is off-cushion or from a wave impact when the ship is traveling at high speed on cushion.
- (U) Local loads are those applied over limited portions of the hull structure, such as loads resulting from hydrostatic or hydrodynamic pressure, desk burden, foundation and topside icing,
- (U) The hull bending, torsion, and shear that result from weight and buoyancy distributions when off cushion, and wave impact loads when transiting at high speed on cushion were investigated. The NASTRAN and multi-cell girder load distribution programs established internal loads for stress analysis. A plate/stiffener analysis computer program was used for the stress analysis of all major structural areas. Loads considered were those due to hull bending, torsion, pressures, drydocking and equipment.
- (U) Scantling design requires a delicate balance between structural weight and ease of fabrication, without sacrificing structural integrity. The scantlings were designed through the use of a computerized optimization program to vary frame, stiffener, and plate sizing with frame and stiffener spacing and provide comparisons of the resultant structural weight and the associated fabrication costs. A frame spacing of three feet with ten inch stiffener spacing was selected. In lightly loaded areas of the ship, such as superstructure, the frame and stiffener spacings were increased to provide light weight and faster ease of fabrication.
- (U) Hull structure optimization has **provided a** basis for optimum structural design of scantlings, **wetdeck** height, **wetdeck** ramp angle, full length side" lls, and keel length fences. The structure optimization has been instrumental in design decisions relating to the square bow near term SES.

- (U) The main hull girder is composed of a centerbody and two rigid sidehulls. The main, second, **third** and wetdecks, as well as seven (7) longitudinal bulkheads, comprise generally continuous longitudinal members which contribute to the section modulus over **the** entire length. All stiffeners on these members, as well as shell plating stiffeners, run longitudinally.
- (U) Bulkhead and deck penetrations are minimal, enhancing structural continuity,

 This result is a compromise between the location of structural bulkheads

 and the arrangement of machinery, equipment, and weapons systems. Minimizing

 the number of bulkhead and deck penetrations reduces the associated structural weight penalties which occur when primary load paths are interrupted

 and internal loads are redistributed through use of secondary load paths.

 Trusses are used to retain overall load carrying capability wherever large

 penetrations exist.
- (U) The hangar and pilot house structure located above the weather deck is assigned a secondary structural role and does not carry primary hull bending or hull torsion. As a consequence, the hangar is designed with a six foot (1.83 m) frame spacing and a 16 inch (0.41 m) stiffener spacing to provide adequate strength.
- welded to the deck plates. These members function as beam sections to span across openings between decks and form the vertical frame columns. These members are capable of reacting axial, shear, and moment loads in the plane of the frame. The sidehull and innerbottom frames are lightest when designed as an open truss configuration. These trusses react the locally applied hydrodynamic pressures and function integrally with the non trussed portion of the transverse frame. Stiffened webs are used in place of the trusses to accommodate tank boundaries, foundations or local load conditions. Reactions to the bow seal and stern seal loadings are concentrated at locally reinforced transverse frames at the wetdeck level.

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- (U) Transverse bulkheads are spaced at 42 feet (12.80 m) intervals, with the exception of the aft most compartment where a 30 feet (9.14m) spacing is used to accommodate the propulsion machinery. These bulkheads are all watertight. Vertically oriented tee members are spaced at 10 inches (254 mm) on center. The longitudinal bulkheads are sized to resist primary loads, flooding loads and drydocking loads. Stiffeners are arranged 10 inches (254 m) on center, nominally.
- (U) The all-welded aluminum hull structure is designed for ease of fabrication, for minimum weight, and to provide structural integrity under all loading conditions. Marine grade weldable aluminum alloys 5086 H116/117 and 5456 H116/117, are rated best overall for the primary hull structure because of mechanical, corrosion, manufacturing, and cost considerations. Of these two, 5456 H117 (H111 extrusions) was chosen because of its 19 percent greater strength-to-weight ratio compared to that for 5086-H117. The H117 temper is free of continuous grain boundary networks which would be susceptible to exfoliation or severe intergranular corrosion in a marine environment.
- (U) The basic ship structure would be fabricated in twenty (20) major structural assemblies including two (2) stabilizer fins (P&S) and the mast, Final assembly and erection would be accomplished outdoors in a building basin. All fabrication, subassembly, and assembly of the structures, from receipt of plate and extrusion until the assemblies are ready to be transported to the building basin for erection, would be performed indoors in a controlled environment. A 139,000 feet² (12,913.5 m²) Marine Assembly Facility would be required. Operations have been planned and sequenced to maximize down-hand and automatic welding such that no overhead welding is required prior to erection of the hull structure in the building basin, Overhead

- (U) welding required during erection would be less than **two** (2) percent of the total lineal footage of welding on the ship.
- (U) Erection of the hull in the building basin would proceed from the stern forward. This erection sequence was selected after reviewing outfitting density and erection sequences to determine that sequence which provides the longest **possible** span for the highest density area of the ship with respect to outfitting and system testing.
- (U) 2.3.1.2 Structural Drawings The **dra**. ;s that define the structural arrangement are contained in appendix B, Section B.3. They are:
 - o Main Deck Plating
 - o Longitudinal Bulkhead
 - o Transverse Bulkheads
 - o Transverse Frame
 - o Bow Plating and Framing
 - o Superstructure
 - o Structural Extrusions
 - o Plating Combinations
- (U) The drawings are grouped in appendix B for consistency of report format and the benefit of the reader.
- (U) 2.3.1.3 Key Structural Features -- Outstanding characteristics of the near term SES include the optimum choice of size and shape of the hull, seal interface, and structural layout of primary members. The design is characterized as being an exceptionally clean ship with smooth flowing lines.

- (U) The functional design of the ship provides minimum air turbulence for helicopter operations while the minimum motion characteristics of the ship enhance the ability of helicopters to take-off and land.
- (U) The physical constraints of the hull structure require that the craft have a beam of 108 feet (32.92 m) or less, a full load displacement of approximately 3000 tons (29892 kN.), and be capable of housing all required subsystems. Physical dimensions developed from parametric trade-offs established the following dimensions:
 - o Overall length of 266 feet 3 inches (81.15 m)
 - o Wet deck height of 18 feet (5.49 m)
 - o Wet deck ramp angle of 13.7 degrees
 - o Minimum main deck height of 40 feet above keel (12.19 m)
- (U) Internal geometry of the hull structure has been optimized to the following and are shown on figure 2.3.1-1:
 - O Stiffener spacing of 10 inches (0.25 m)
 - o Frame spacing of 3 feet (0.91 m)
 - o Transverse bulkheads spaced at 42 feet (12.80 m) intervals (aft bulkhead at 30 feet 9.14 m/ $_{T}$)
 - o Longitudinal bulkheads at approximately 14 feet (4.27 m) spacing
 - O Between deck height of 9 feet (2.74 m)
 - O Third deck height above keel 22 feet (6.71 m)
 - O Second deck height above keel 31 feet (9.45 m)
 - Main deck height above keel 40 feet (12.19 m)
 - 01 deck at 49 feet (14.54 m)
 - $_{0}$ 02 deck at 60 feet (control center deck) (18.29 m)

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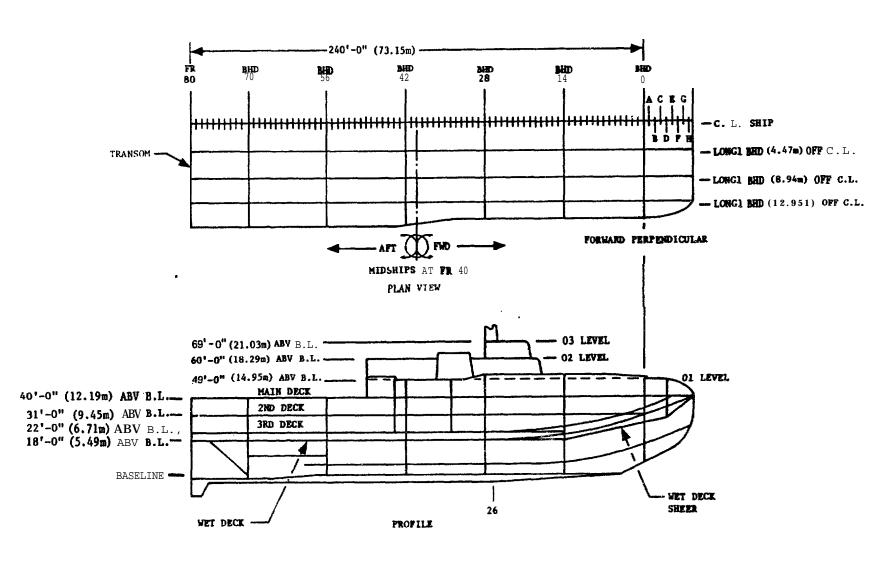


Figure 2.3.1-1 (U): Structural Configuration (U)

(U) 2.3.1.4 Structural Weight Breakdown -- The structural weight breakdown of the hull and superstructure is shown in Table 2.3.1-1:

Table 2.3.1-1 (U) Structural Weight Breakdown (U)

SWBS	LONG TONS	KILONEWTONS	%
110	419.03	4175.2	52.1
120	152.79	1522.4	19.0
130	518.60	1580.3	19.7
150	14.49	144.4	1.8
160	19.11	190.4	2.4
170	2.78	27.7	0.3
180	38.20	380.6	4.7
100	805.0	8021.0	100

(U) 2.3.1.5 Structure Risk Assessment -- The hull of the near term SES is designed to realistic worst case loading conditions which are forecast to occur within the ship lifetime. These structural loads were obtained from an extensive Rohr 2KSES/3KSES model testing and analytical loads development program. The structural materials are commercially produced aluminum alloys which have been utilized in existing Navy ships, such as the PHM and SES 100B. The baseline design configuration features conventional built-up plate-stiffener combinations, a conventional ship'framing system, and state-of-the-art welding and producibility details to minimize construction problems. Consequently, structure of the near term SES is producible, competitive with respect to cost; and represents an optimum design configuration for performance of the specified mission.

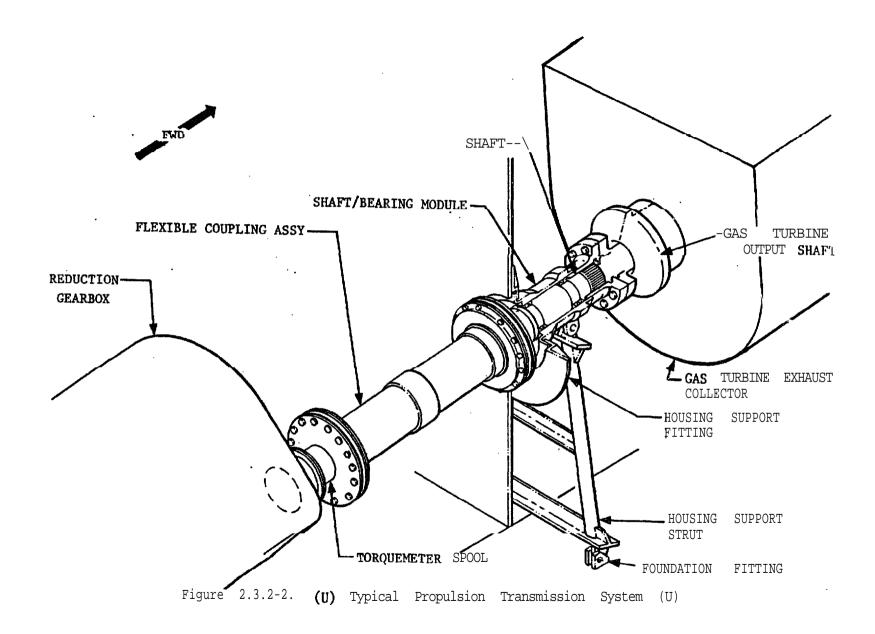
- (U) 2.3.2 PROPULSION -- The near term SES is powered by a waterjet propulsion plant, Its principle is the conversion of that mechanical energy supplied by the gas turbine-driven waterjet pumps into kinetic energy, by increasing the velocity of the seawater inducted at the waterjet seawater inlets and ejected through the waterjet pump exit nozzles. The general arrangement is shown on Figure 2.3.2-1.
- (U) The SWBS breakdown of the propulsion plant is:
 - Gas turbine system (234)
 - o Transmission system (242, 243, 244)
 - Waterjet propulsor system (247)
 - o Combustion air intake system (251)
 - Exhaust gas uptake system (259)
 - Lube oil system (262)

2.3.2.1 Summary Description

- (U) 2.3.2.1.1 Gas Turbine System -- A total of four (4) gas turbines, each driving a **waterjet** propulsor, are utilized in the near term SES propulsion plant. The four (4) turbines are arranged in pairs of two (2): one (1) pair is located on the starboard side of the ship and the other pair is located on the port side. Each gas turbine is operationally independent of the other .
- (U) The baseline propulsion gas turbine for the near term SES is the LM2500 gas turbine which is capable of delivering 22,500 continuous shaft horsepower (16,780 kW) and 27,000 intermittent shaft horsepower (20,130 kW). The alternate propulsion gas turbine is the FT9A-24 which delivers 36,500 continuous (27,220 kW) and 40,000 intermittent shaft horsepower (29,830 kW).

Figure 2.3.2-1 (U): Propulsion Plant General Arrangement, Starboard Side Only (U)

- (U) The FT9A-2A system requires more space than the LM2500 system due to its increased turbine length. The FT9A-2A engine is installed further aft, which shortens the drive shaft length. The propulsion plant is designed for future installation of the alternate FT9A-2A engine with a minimum impact on the propulsion plant or other ship systems.
- (U) The LM2500 marine gas turbine is derived from the TF39 military and CF-6 commercial turbo fan engines used respectively on the Lockheed C5A Galaxie and McDonnell Douglas DC-10 aircraft. The LM2500 gas generator consists of a variable vane 16-stage compressor; annular combustor; two-stage aircooled turbine and associated gearboxes; controls; and accessories. The power turbine has six stages and is a low-speed, low stress design. The LM25CJ engine, is presently in service on the DD963 class destroyers.
- (U) The FT9A-2A engines are identical with the FT9A-2 engines currently being developed by the Navy, Minimum interface hardware revisions are required for the SES application. The progenitor engines for the FT9A-2A are the FT4 marine and JT9D aircraft engines. The FT9A-2A gas generator has a low and high pressure compressor, each driven by a separate turbine and an annular combustor. The power turbine is derived from the FT4 power turbine now in service.
- (U) 2.3.2.1.2 Transmission System -- This system consists of the propulsion shafting, shaft flanges, shaft bearings with mounting structure, flexible couplings and torque meters. Each of the four transmission systems connect a propulsion gas turbine to a waterjet propulsor reduction gearbox input flange. The shaft, flanges, bearings, seals and bearing housing form the shaft/bearing module which is installed (or replaced) as a unit. Figure 2.3.2-2 illustrates the arrangement. The reduction gearbox is described next in the waterjet propulsor system description.



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- (U) 2.3.2.1.3 Waterjet Propulsor System -- The waterjet propulsor system consists of the integral reduction gearboxes and waterjet pumps, instrumentation, mounting links, steering sleeves with hydraulic actuators, waterjet pump inlet flex joints, thrust reversers with hydraulic actuators, transom flexible seals, waterjet pump priming systems, attached lube oil pumps with minor lube oil system components and piping, seawater inlets, seawater intake diffusers, bifurcated ducts and variable ramp roofs with hydraulic actuators, A nozzle closure valve and a thrust bearing are contained within each waterjet pump. A shaft brake is attached to each reduction gearbox.
- (U) Each reduction gearbox (four (4) total) contains necessary gearing to reduce the input speed and divide the power between the two (2) waterjet pump rotors which run at different speeds. The propulsor assembly gearbox details and gear train are shown in Figures 2.3.2-3 and 2.3.2-4.
- (U) 2.3.2.1.3.1 Waterjet Propulsor Assembly -- The waterjet propulsor is a two-stage; two-speed design based on the hydraulically similar PHM propulsor. The first stage is an inducer designed to produce a sufficiently high head rise at low suction (cavitating) conditions to permit the second stage impeller to operate at high rotation speeds without cavitation. The power split between the inducer and impeller is approximately 30:70. The inducer rotates at about 1/4 engine speed, the impeller at about double this. The propulsor assembly is shown in Figure 2.3.2-S.
- (U) 2.3.2.1.3.2 Waterjet Inlet -- Seawater for the four (4) waterjet propulsors is taken aboard through two semiflush inlets as shown in Figure 2.3.2-6. One inlet is located in each sidehull to serve the two waterjet pumps also located in each sidehull. Seawater for ship services is taken aboard through these inlets. The sidehulls are enlarged through fairings from their nominal cross-sections to accommodate the inlets. Waterjet inlet area is varied by continuous plate flexible ramp roofs, actuated by a hydraulic cylinder to control the seawater flow into the system. The water

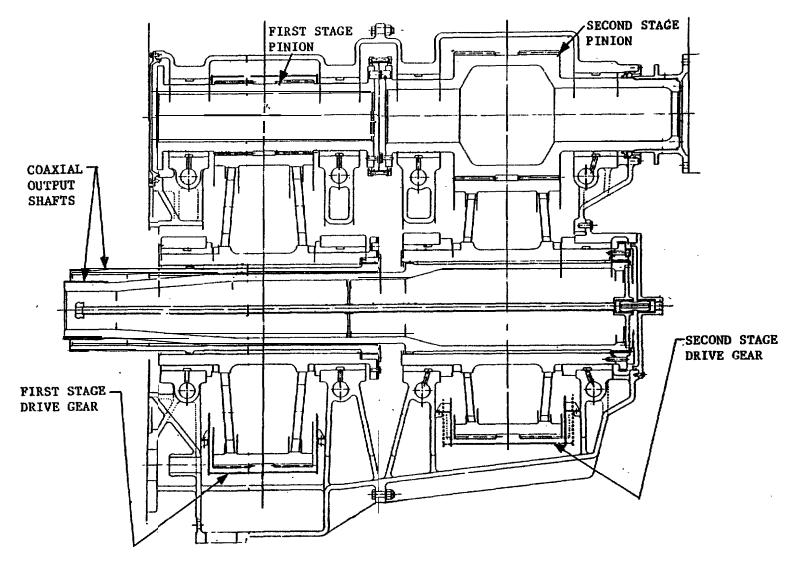


Figure 2.3.2-3. (U) Propulsor Assembly Gearbox Arrangement (U)

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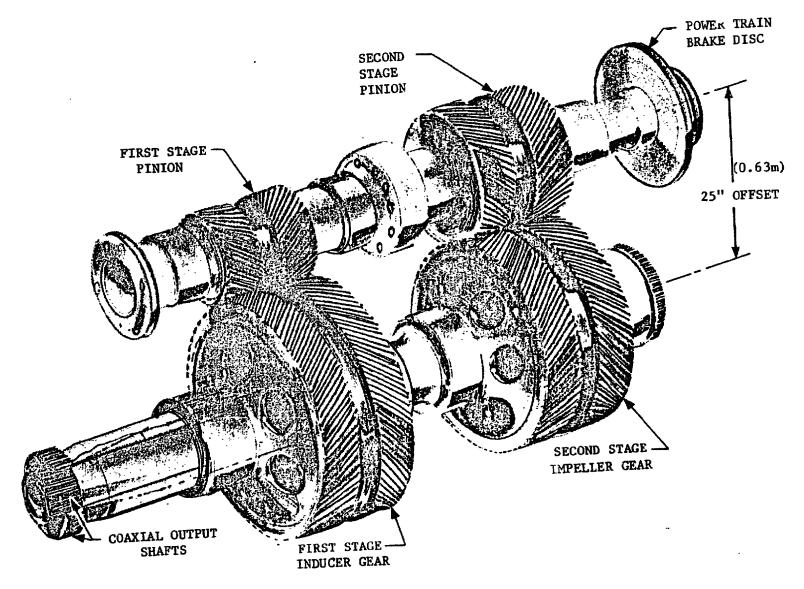


Figure 2.3.2-4 U): Propulsor Assembly Gear Train (U)

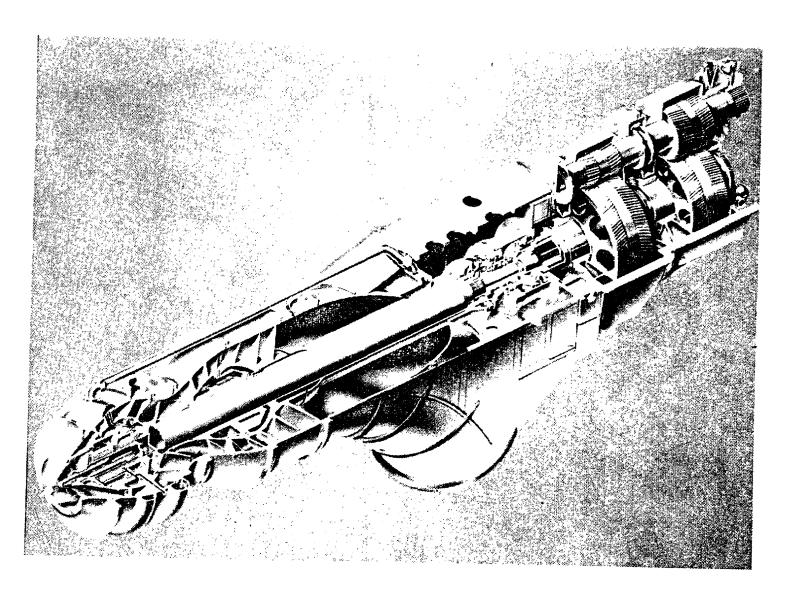


Figure 2.3.2-5 (U): Waterjet Propulsor Assembly (U)

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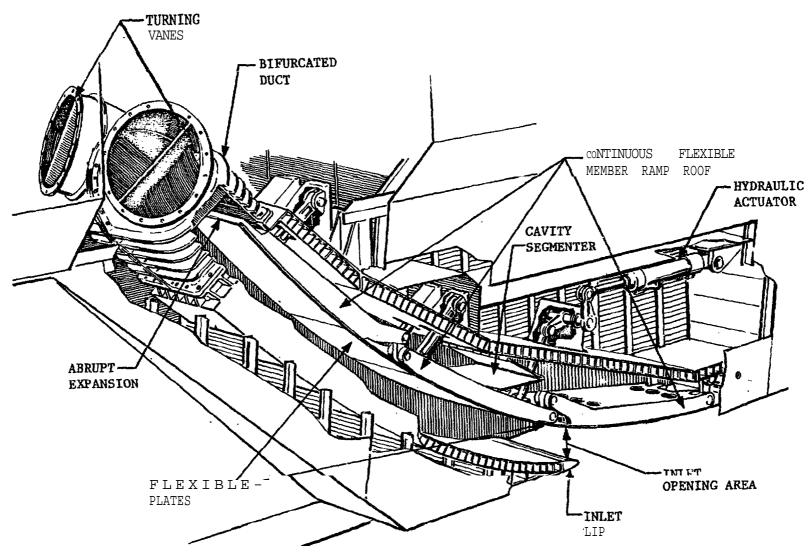


Figure 2.3.2-6 (U): Waterjet Intake Duct Ramp Roof (U)

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- (U) flow through each sidehull inlet passes into the diffuser section of the inlet duct and is distributed through a duct bifurcation to the two pumps. Water flows through the inlets by combination of pump action and ship forward motion, at a rate determined by the ship speed, inlet area settings and pump speed. The curved diffusers then turn and raise the water to the pumps through the bifurcated ducts. An abrupt expansion is used at the entrance to the bifurcated ducts. Each bifurcated duct is constant area, symmetrical and has integral turning vanes.
- (U) 2.3.2.1.3.3 Steering and Reverser System -- Each waterjet propulsor has an associated steering sleeve and the two outboard propulsors have thrust reversers. The discharge water from each pump's single fixed-area nozzle passes coaxially through a flexible seal at the transom, and subsequently through a swiveling steering sleeve mounted on the transom. The steering sleeve deflects the waterjet to generate side forces on the ship. Each sleeve is hydraulically actuated, utilizing the ship hydraulic system, and is instrumented to permit position monitoring.
- (U) The thrust reversers direct the waterjets in a forward direction. In operation, they are pivoted into the water streams by controllable position actuators. During reverse thrust operation, the high-velocity water is redirected forward, down, and slightly outboard to minimize spray and hazard to nearby objects. The thrust reversers are variable position to give full forward through full reverse thrust on the outboard waterjets.
- (U) 2.3.2.1.4 Combustion Air Intake System -- The internal configuration of the combustion air intake system and the location of the demister banks, acoustic panels, gas turbine plenums, air heating system, and external opening of the air inlet are shown in Figure 2.3.2-7. The features of the intake design which reduce salt spray are the coaming projecting above the 01 level; the vertical portion of the intake which requires the air to turn 90 degrees to enter the demister banks; and the drainage sump at the third deck level.

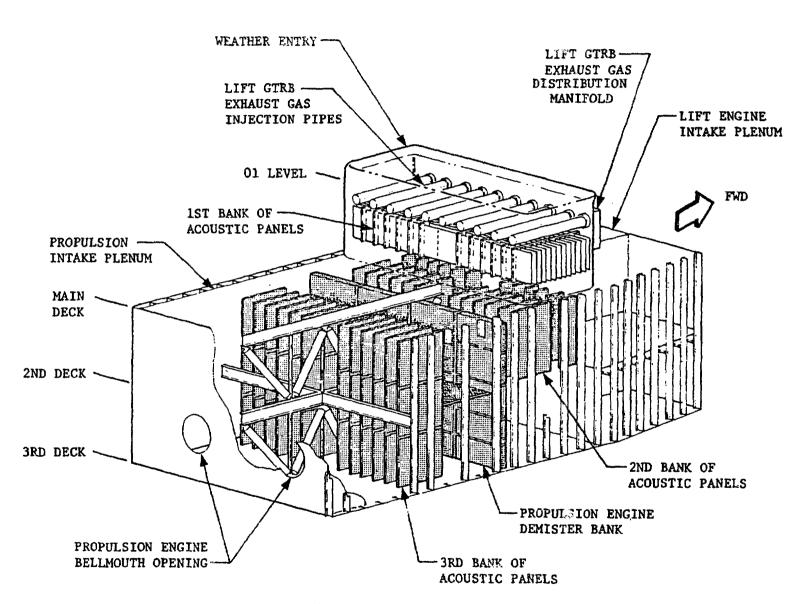


Figure 2.3.2-7. (U) Combustion Air Intake System Arrangement (U)

- (U) The intake contains sound supression panels in three locations to attenuate engine noise to acceptable levels. The panels in the section between the 01 level and main deck are comprised of six inch (0.15 m) spaced perforated panels, installed in the athwartship direction. Thin aluminum splitters between the panels form six inch (0.15 m) rectangular ducts.
- (U) The intake system is designed to accommodate the LM2500 or the FT9A-2A engine. The design considers all combination s of these engines for propulsion and lift. The only modification required when the FT9A-2A engines are employed is an increase of demiscer face areas.
- (U) Anti-icing, de-icing, and pre-heating of the intake system for the engines is accomplished by recirculation and mixing of lift engine exhaust gas at the weather inlet on each side of the ship, as is shown schematically in Figure 2.3.2-8. Each combustion air intake system supplies air to one lift engine, two propulsion engines, gas turbine generator(s), and the gas turbine cooling systems.
- (U) 2.3.2.1.5 Exhaust Gas Uptake System -- This system consists of the exhaust ducts (including supports and insulation) which are routed from the propulsion gas turbines to the transom, where the combustion products are exhausted; The design incorporates a water trap at the transom to provide stern wave protection, Each exhaust duct is acoustically treated to attenuate noise.
- (U) 2.3.2.1.6 Propulsion Lube Oil System -- This system provides lubrication for the bearings in the transmission system and for the waterjet propulsor assembly. The reduction gear system is of the dry sump type and the pump thrust bearing and seal module require most of the oil in the system. The reduction gear carries driven pressure and scavenge pumps. The lube oil system upstream of the mechanically driven pressure pump, and downstream

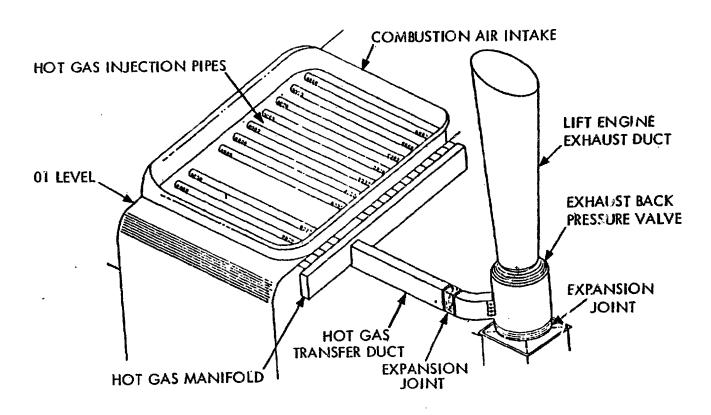


Figure 2.3.2-8 (U): Anti-Icing and Intake Air Heating System (U)

(U) of the mechanically driven scavenge pump, is defined as the propulsion lube oil system, Each propulsion drive train has its own dedicated propulsion lube oil system (four total). The lubricant used is 2190 TEP per MIL-L-17331.

(U) 2.3.2.2 Operation

- (U) 2.3.2.2.1 Start-Up -- The gas turbines are pneumatically started from the ship system. This system provides sufficient compressed air to start one gas turbine. The pneumatic (start air) system cross connects all propulsion and lift engines such that any one engine can start another by supplying bleed air from its compressor into the system. The start control sequence is automatic but manual start controls **provisions** are provided for back up. Each gas turbine engine can be started and ready to deliver power in approximately 90 seconds.
- (U) The waterjet propulsors are above the ship off-cushion waterline and thus require priming. Priming is accomplished in these successive steps: apply transmission brake to prevent rotation of the dry pump; shut nozzle closure; supply auxiliary water to rubber bearings; operate the air ejector that connects to both pump pairs; and when pumps are primed, water then covers the pump inducer centerline.
- (U) The brake is then released and the pump rotated enough to produce a static head of about 15 ft (4.57 m) $\rm H_2O$. When the nozzle closure is opened, the pump begins to deliver water, the ejector system and auxiliary water supply are shut off and the priming is completed. These features of the priming system are shown on Figure 2.3.2-9.
- (U) 2.3.2.2.2 Low Speed -- Low ship speed operation of the propulsion plant requires the ship to be *in* the off-cushion mode to reduce the possibility of broaching which could unprime the propulsors. Additionally, with the ship off-cushion, inlet head to the pump inducer is maximized to reduce suction specific speed. The steering sleeves and reverser may be configured to

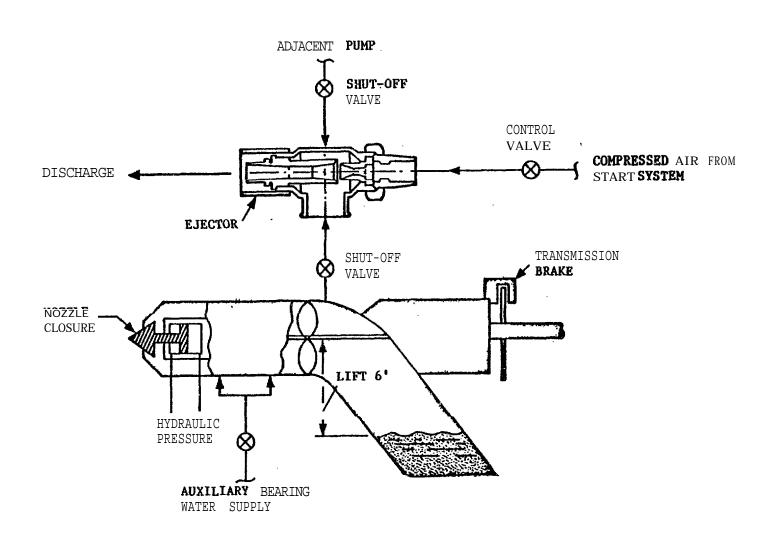


Figure 2.3.2-9 (U): Propulsor Priming System Schematic, Port and Starboard (U)

- (U) give low speed forward, astern, turning and sideways translation of the ship for **undocking** and maneuvering in a seaway. The power level is limited by the suction specific speed line of the propulsors to limit cavitation erosion of the pump components.
- (U) The seals are extended to a height/speed schedule when above 10 knots (5.14 m/s) ship speed to avoid reaching limit suction specific speed. The **waterjet** inlet is generally wide open in the low speed mode.
- (U) 2.3.2.2.3 Hump Transftion -- Hump transition requires the use of high power settings and, in the case of a heavily loaded ship and/or high sea state, may require use of the intermittent power level to produce the desired margin of thrust over drag. During transit, the waterjet inlet area is varied according to pump speed (power), ship speed and engine throttle setting. Suction specific speed limitations are not present at trans-hump speeds.
- (U) Ship heading control will require use of a combination of differential thrust, assymetrical throttle settings on the fan engines, and thrust vector control with the steering sleeves. The ships control system automatically determines the required combination and the mix of control forces that provides heading control with minimum fuel consumption.
- (U) 2.3.2.2.4 High Speed Cruise -- At cruise conditions, throttle settings for steady state conditions (and the associated inlet opening) are maintained by the propulsion control system.
- (U) 2.3.2.3 Machinery Characteristics -- The machinery characteristics are presented in the following tables:

Table 2.3.2-1:(U) Engine Characteristics (U)

	CHARACTERISTICS	
ITEM	LM 2500	FT9A-2A
Turbine Inlet Temperature - OF(OC) Air Flow - 1b/sec (N/Sec) Dry Weight - 1bs (kN) Compression Ratio at Max, RPM	2300 (1260) 146 (649) 10,405 (46.281) 15:1	2255 (1235) 250 (1112) 21,300 (94.742) 21:1
<pre>\$;FC = lb/HP-hr (kN/w-hr) Max. Power at Sea Level- HP @ 80"F (kW @ 27°C)</pre>	.381(2.28) 27,000 (20,134)	.397(2.36) 40,000 (29,828)
No. of Compressor Stages No. of Turbine Stages No. of Combustors Combustor Type Length-Inches (m) Diameter (Max) - Inches (m)	16 8 1 Annular 257 (6.528) 87 (2.210)	17 5 1 Annular 337 (8.560) 73 (1.854)

ITEM	CHARACTERISTICS '
<u>leduction</u> Gear	
Speed	4100 RPM
Power	40,000 SHP (29830 kW)
Weight (Dry)	9489 lb (42.21 kN)
Length	76 inch (1.93 m)
Width	59.50 inch (1.51 m)
Lubricant	2190 TEP per Mil-L-17331
Gears	Double-Helical 9310 steel
	one-piece pinion and shaft
Ratio	
First stage	4.359
Second Stage	2.0508
Bearings	Journal, Babbit lined
Casting	Cast Aluminum A356-T6
[ransmission]	
Length	138 inch (3.50 m)
Diameter	22 inch (.56 m)
Bearings	Fwd - Duplex ball thinwall
	Aft - Roller thinwall
Shaft and Flanges	4340 forgings
Flexible Coupling;	Double diaphragm = 1/2° misalignment
	capability
Torquemeter	Accurex Strain type

Table 2.3.2-3: (U) Waterjet Propulsor Characteristics (U)

ITEM	CHARACTERISTICS	
Enand	4100 DDM	
Speed Power	4100 RPM	
Weight (wet)	40000 SHP (29830 kW) 22571 1b (100.396 kN)	
Length	203 inch (5.16 m)	
Height	70 inch (1.78 m)	
Diameter	49.5 inch (1.26)	
Efficiency *	88.5%	
Headrise *	999 ft H ₂ O (304.5 m H ₂ O)	
Flow rate *	135,154 GP.M (8.527 m ³ /sec)	
Gross Thrust *	161,200 1bf (717.018kN)	
Nozzle Diameter	17.52 inch (.45 m)	
Speed Inducer *	940 RPM	
Speed Impeller *	1999 RPM	
Suction Specific	2,4250 at Inducer Centerline	
Speed Limit		

^{*}Values at RPM and Power quoted and total inlet head of 203 ft. $\rm H_2^{\,0}$ (61.9 m $\rm H_2^{\,0})$

Table 2.3.2-4: (U) Waterjet Inlet (U)

ITEM	CHARACTERISTIC
Width	48 inch (1.22 m)
Drop Fraction	0.5
Max Opening Area	14 ft ² (1.301 m ²)
Min Opening Area	$4 \text{ ft}^2 \text{ (.372 m}^2\text{)}$
Max Flow Rate	286,282 GPM (18.06 m ³ /s)
Variable Roof.	3 section continuous flexible plate
Bifurcation	Equal legs with -urning vanes

- (U) 2.3.2.4 Arrangements -- The drawings **and** sketches depicting the near term **SES propulsion** system **are** contained in Appendix B.4. They are:
 - 0 Propulsion System
 - 0 Waterjet Inlet
- (U) The drawings are grouped in Appendix B.4 for consistency of report format and benefit of the reader, A sketch of a section of the demister is shown as Figure 2.3.2-10.
- (U) 2.3.2.5 Propulsion System Weights -- Weights within the propulsion system SWBS 200 are shown in Table 2.3.2-5.

Table 2.3.2-5 (U). Propulsion System Weights (U)

		Weight		
SWBS	Subgroup	LT	kN	Percentage
234	Gas Turbines	21.56	214.8	11.3
241	Reduction Gears	16.07	160.11	8.4
242	Couplings	.71	7.07	.4
243	Shafting	1.19	11.86	.6
244	Shaft Bearings	.60	5.98	.3
247	Waterjet Propulsors	50.82	506.3	26.7
251	Combustion Air		,	
	System	20.12	200.47	10.6
252	Control System	.46	4.58	.2
259	Uptakes	14.21	141.58	7.5
261	Fuel Service System	.11	1.10	.1
262	Lubeoil System	4.60	45.83	2.4
298	Operating Fluids	59.56	593.43	31.3
299	Repair Parts	44	4.38	.2
200	Propulsion System	190.46	1897.65	100.0

(U) Table 2.3.2-6 shows the estimated functional weight percentage of major components within the propulsion system for an LM 2500 and FT9A-2A system.

Table 2.3.2-6 (U). Weight Percentage of LM2500 and FT9A-2A Propulsion System (U)

SUBSYSTEM	PERCENT OF SYSTEM		
DODOTOTEM	LM2500	FT9A-2A	
Engines	9.1	16.9	
Gearboxes	a.4	7.1	
Propulsors	12.6	10.6	
Comb Air System	10.6	11.2	
Comb Exh System	7.5	10.3	
Oper. Fluids	31.3	26.4	
Miscellaneous	19.9	17.7	

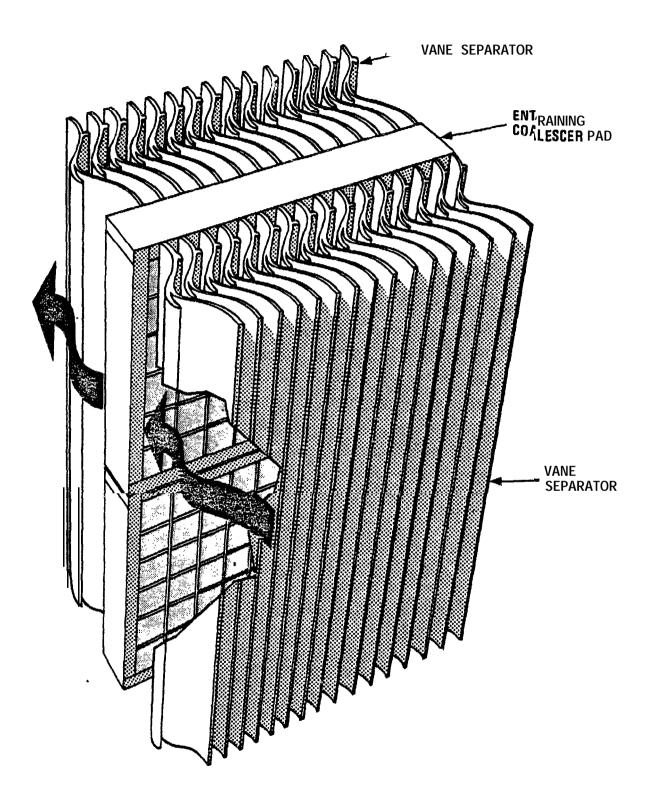


Figure 2.3.2-10 (U): Propulsion Inlet Water Demister (Coalescer) (U)

- 2.3.2.6 Propulsion System Technical Risk Summary
- Engines -- The GE LM2500 gas turbine engine is in production with a proven capability up to 21,500 SHP (16,033 kW), is low weight, and in use for other marine applications. The Pratt and Whitney FT9A-2A alternate engine is developmental and requires design and test effort; it will be in production after a few years; and component testing has begun, Current development efforts are limited to 33,000 SHP (24,608 kW).
- designed to transmit all anticipated alternating and continuous torques between the propulsion engine and the propulsor assembly without failure over a 20-year life span with specified overhaul of the life limited components; to have not more than 10 percent failures prior to the scheduled overhaul period of 5,000 hours minimum (10,000 hours goal) for the life limited components; to withstand a limit torque of 1,229,764 inch pounds (138.94 kN·m) without degradation of performance or failure; and to eliminate any critical speed (of any component) which is less than 125 percent of the system maximum operating speed.
- PHM pump now in operation. Comprehensive model tests have already been successfully conducted for the 40,000 SHP (29,828 kW) propulsar. The waterjet inlet has been extensively tested with models and with similar inlets of the operational SES-100A and XR-1 testcraft. The installation design of the waterjet propulsor assembly will withstand all anticipated input powers, thrusts and external loads due to ship accelerations and equipment malfunctions without failure for a 20-year design life and with specified overhaul. The waterjet seawater inlet duct system has been optimized to improve performance, cavitation characteristics, drag and structural weight on the basis of substantial analysis and model testing.

(U) The propulsion lube oil system, combustion air intake system, and exhaust gas uptake system are typical of present gas turbine ship installations.

All components are presently available and proven in service. For the combustion air inlet, anti-ice protection by exhaust gas mixing is the accepted method of General Electric, Pratt and Whitney, and Garrett.

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- (U) 2.3.3 ELECTRICAL SYSTEM
- (U) 2.3.3.1 Summary of Key Features -- Primary 450 Volts power for the near term SES is generated at 60 Hz and 400 Hz frequencies by six gas turbine generator (GTG) sets. Three (3) identical GTGs rated 375 kW 60 Hz and three identical GTGs rated 375 kW 400 Hz provide a total system capacity of 2250 kW. All six (6) GTGs are driven by Garrett ME 831-800 turbines.
- (U) The distribution system is arranged to provide an operational choice of ring-bus or split-plant operation. Six (6) ship service switch-boards are provided, three (3) for 400 Hz service and three (3) for 60 Hz service.
- (U) The lighting arrangement is based upon dividing the ship into four (4) lighting zones or "cubes". Three cubes comprise the internal illumination distribution system, while the fourth cube services the specialized needs of the helicopter hangar and landing lights. Lights throughout the ship are predominately of the fluorescent type and are energized by the 60 Hz system,



- (U) 2.3.3.1.1 Type of System -- A block diagram depicting the functional integration of the electrical system is shown in Figure 2.3.3-1. The power generation system provides all anticipated ship service primary and secondary electrical power with minimum weight, minimum development risk and maximum assurance of required performance, reliability, and flexibility. Both the 60 Hz and 400 Hz systems generate power at 450 V, 3 phase, ungrounded delta. Power quality meets or exceeds the requirements for Type II 400 Hz power and Type I 60 Hz power per MIL-STD-1399/103.
- (U) In addition to driving the 60 and 400 Hz generators, the ME 831-800 gas turbines provide bleed air for starting the propulsion and lift engines (LM2500 or FT9) and also provide a small amount of continuous bleed air for the ship's compressed air system.
- (U) The near term SES' operating loads are approximately 50% on the 400 Hz system and 50% on the 60 Hz system. The ship's 400 Hz operating loads are distributed evenly among the three 400 Hz switchboards, each of which serves consumers located nearest to the particular switchboard. Each switchboard is connected to the other switchboards by bus ties which form a ring bus arrangement.
- (U) Two of the three generating plants are generally connected to the ring bus arrangement for all operating modes, allowing the third unit to be in a standby mode. Generators may be added or deleted as the power demand dictates when operating with the ring bus system,
- (U) The 60 Hz power distribution system is similar to the 400 Hz system.
- (U) The lighting system provides adequate and reliable illumination in all areas of the ship, regardless of operating mode or condition. Special and detail lighting is provided for specific tasks. The lighting fixture arrangement is spaced to provide the prescribed levels of working

surface illumination, as well as uniform, shadow free illumination **for** all areas. The system provides the following illumination services throughout the ship:

- General white illumination in all spaces
- Detail illumination according to work task
- Low-level, red-band illumination for darkened ship
- Two levels of blue-band lighting in the Combat Information center
- Automatic and manual battery operated battle lanterns
- helicopter platform visual landing aid and VERTREP platform illumination for night operation
- Navigation and running lights
- (U) The lighting system utilizes 60 Hz power for economic reasons since a 400 Hz system offers no appreciable weight savings and would be appreciably higher in cost. The lighting system also provides power for numerous non-lighting loads wherever this arrangement yields weight savings.
- (U) The system utilizes the Navy concept of dividing the ship into vertical volumes, each approximately a cube, for optimum distribution. The ship is divided info four cubes. One cube is dedicated to the helicopter landing area and supporting lighting. The remaining three cubes are divided into the forward, middle and aft portions of the ship. The lighting distribution system is fed from the three 60 Hz switchboards. Each of the three ship cubes contains two transformer banks fed from different switchboards. One transformer bank in each cube receives two separate power sources via a two-way automatic bus transfer, for supplying power to all areas containing vital lighting. The other transformer bank in the cube receives power from one switchboard. Figure 2.3.3-2 illustrates this arrangement.

- (U) Lighting fixtures are designed to provide satisfactory illumination with optimum operational economy and minimum maintenance. Fluorescent lighting is used predominantly wherever feasible, owing to its superior lighting qualities and lower power consumption, Incandescent lighting is utilized only where a suitable fluorescent fixture is not available.
- (U) Standard Navy fixtures are normally used because of their proven qualities; in exceptional cases, other suitable fixtures may be selected where a functional advantage exists, or where an appreciable weight or cost saving can be achieved with no degradation in either service or reliability.

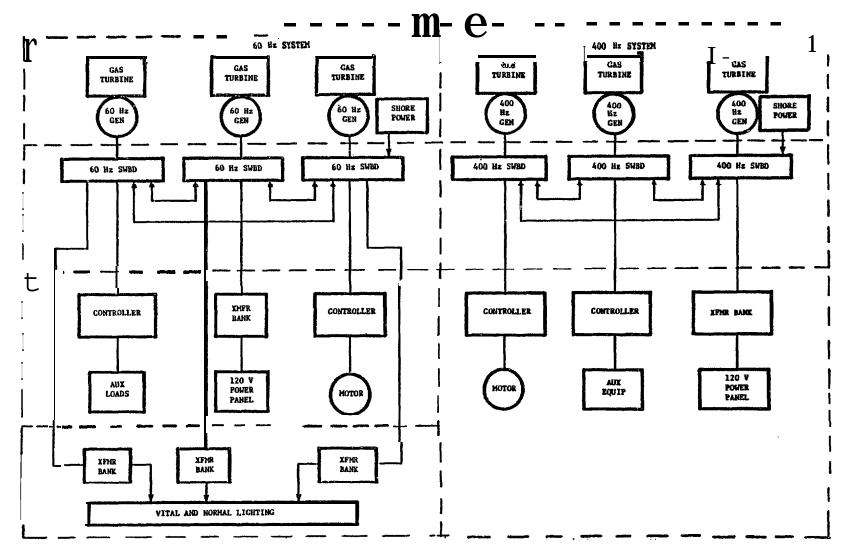


Figure 2.3.3-1 (U): Electrical Plant Functional Block Diagram Illustrating Independent 400 Hz and 60 Hz Systems and Ring Bus Arrangement Which Precludes Total Loss of Rower (U)

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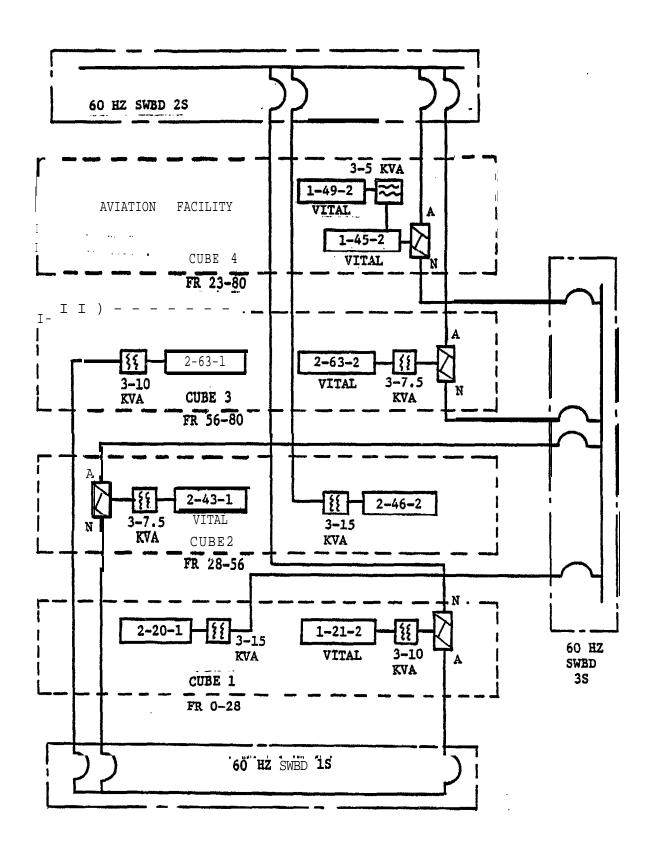


Figure 2.3.3-2 (U): Lighting System, One-Line Diagram (U)

- (U) 2.3.3.1.2 Type of Components -- The GTG prime movers are Garrett ME 831-800 turbines currently being qualified for the Navy PHM program. The 831 series engine has four million hours of operational experience in protected and unprotected environments.
- (U) The 400 Hz GTG system is very similar to the PHM patrol hydrofoil generator set: each employs an ME 831-80-0 turbine, gearboxes with identical basic castings, bearings, and primary gears (i.e., gears that drive the 400 Hz generator), and identical turbine auxiliaries. The 400 Hz generator is also very similar to the generator used on the PHM patrol hydrofoil generator set (same design and frame size), differing only by a slightly longer lamination stack to provide a higher power level output.
- (U) In addition to the 60 Hz GTG use of the same gas turbine as the 400 Hz GTG, the other major components (such as the fuel system, lube system, and governors) differ very little between the two power frequency systems.

 The gearboxes are fundamentally identical except for the output gears which provide shaft speeds of 1800 RPM for the 60 Hz generator and 8000 RPM for the 400 Hz generator.
- (U) Each GTG set comprises a gas turbine, reduction gear, generator, governor, fuel system, self-contained lube system, enclosure and control system. Figure 2.3.3-3 and 2.3.3-4 show the turbine prime mover major components, envelope, and weight for the 60 Hz and 400 Hz units, respectively,
- (U) Each GTG is equipped with an electrical starter operating from its own dedicated 24 V dc battery system. On starting, in-rush currents of 2000 amperes exist and voltage dips below 15 volts will occur. These 1 'ge voltage dips dictate the need for an individual, dedicated battery and charger/power supply for each of the GTG's. This arrangement also ensures very high starting reliability, positively guaranteeing "blackship" starts.

- (U) The ME 831-800 gas turbine engine provides bleed air from its compressor. This source of compressed air constitutes a cost and weight effective means for starting the lift or propulsion gas turbine engines. The maximum air bleed rate is 104 lbm (0.786 Kg/s) from each turbine, and the output of two turbines, cooled through a heat exchanger is required to start the LM2500 (or FT9 A-2A). The bleed air power drain is substantial during the engine starting cycle, so the off-line GTGs are available to perform this function without disturbance of normal power generation.
- (U) The 400 Hz and the 60 Hz distribution switchboards are identical in construction. Typical outline dimensions are shown in Figure 2.3.3-5, Local control devices and instrumentation for GTG's are provided within a control cabinet located on the GTG. Switchboards are of the free-standing, dead front type, constructed with aluminum framing and sheeting. Access space is provided at both front and rear of each switchboard. All devices for the remote control and monitoring of the switchboards are conveniently terminated at terminal boards in the rear of the switchboard to facilitate connection of the ship's cables. Reverse power protection for the generator sets is provided within the switchboards.
- (U) Circuit breakers mounted within the switchboard are of the proven reliable MIL-SPEC type. Molded case AQB Type circuit breakers are used within the distribution system to achieve reduced system weight and cost. The AQB Type bus tie and shore power circuit breakers are equipped with motor operated devices to enable remote operation. The generator circuit breakers are ACB Type.

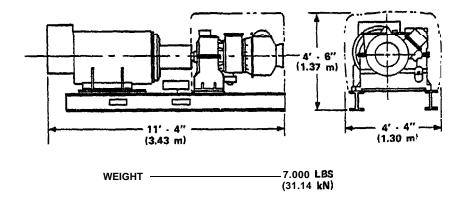


Figure 2.3.3-3 ($\bf U$) The 60 Hz Gas Turbine Generator Set is a Compact Unit ($\bf U$)

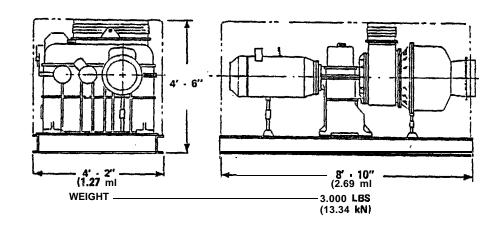
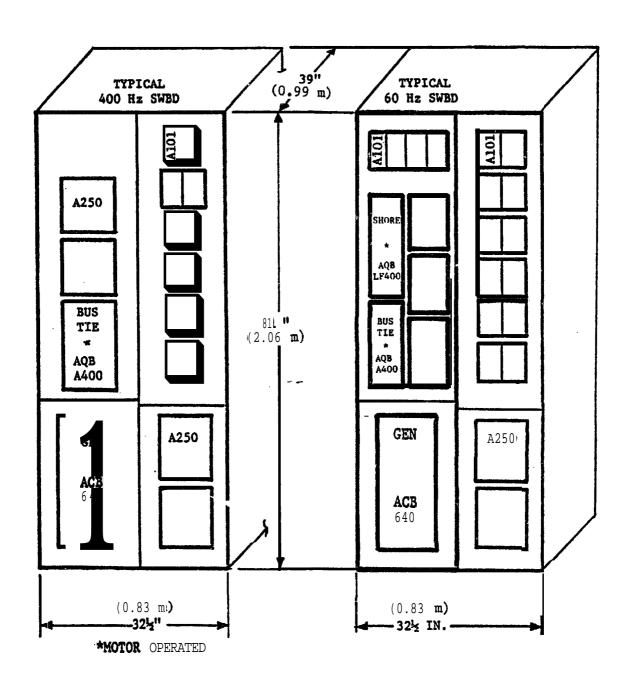


Figure 2.3.3-4 (U): The 400 Hz Gas Turbine Generator Set is Similar to the PHM GTG (U)



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Figure 2.3.3-5 (U): Switchboard Arrangement (Typical) (U)

- (U) 2.3.3.2 General Schematic -- An electrical system general schematic is shown in Figure 2.3.3-6, This shows the complete independence of the two primary power systems (60 Hz and 400 Hz) from each other. Each generator, both 60 Hz and 400 Hz, have an associated switchboard located in the same room with its generator, The locations of the generators have been made so that the three units for either system are dispersed one from the other, with two being low on the third deck and one high on the main deck.
- (U) Shore connections for both 400 Hz and the 60 Hz systems are made at connector receptacles located near the centerline on the 02 level. Inter locking is provided between the shore connection and the switch-board-mounted shore power circuit breaker to prohibit make-or-break of the shore connection under load.
- Each circuit breaker has been selected to provide adequate protection in the event of a fault. A sequenced opening of breakers will occur with the generator breaker operating last. Should distribution circuit breakers open, manual resetting of the breakers is required as a safety feature to ensure that the fault or overload is first removed. Selected breakers may be remotely opened for damage control purposes but manual reset is required. Large power consumers are fed directly from switchboards while smaller consumers are routed to power distribution panels located throughout the ship. Transformers are located in close proximity to distribution panels for loads requiring voltages other than generated voltage. Voltage and frequency monitors (VFM) are provided where required for protection of 400 Hz electronics.

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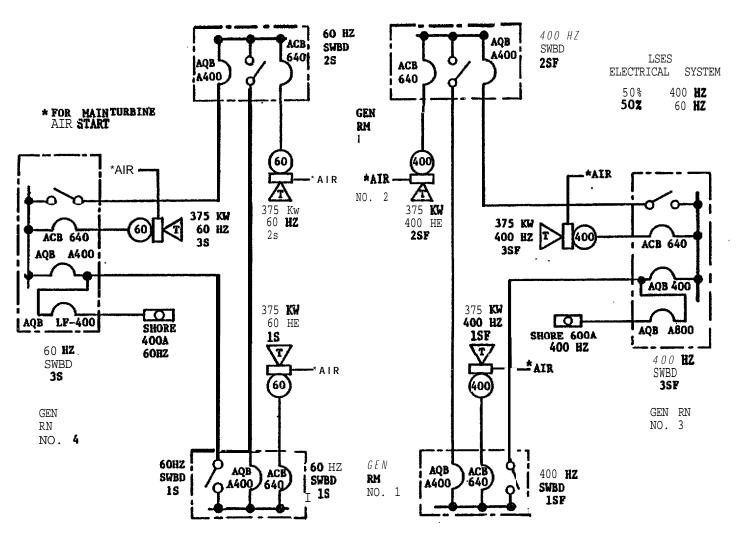


Figure 2.3.3-6 (u): Electrical System General Schematic, Illustrating Flexibility and Availability Provided by Multiple **GTGs** and Ring Bus Interconnection of Switchboards **(U)**

- W2.3.3.3 General Arrangement -- The six GTG sets are installed in four different rooms, separated from each other by at least two water-tight bulkheads GTG Rooms 1 and 2 are symmetrically arranged and located on the third deck at the outboard extremes of the ship, as illustrated in Figure 2.3.3-7. Figure 2.3.3-7 also shows the location of the two GTGs (one 60 Hz and one 400 Hz) within each room.
- (U) GTG Rooms 3 and 4 are located port and starboard on the main deck, just forward of the combustion air inlet plenum. Room 3 contains one 400 Hz GTG as shown in Figure 2.3.3-8. Room 4, on the port side, contains one 60 Hz GTG and is arranged similarly to Room 3. The starting/control battery, battery charger and switchboard for each of these GTGs are also located within the rooms.

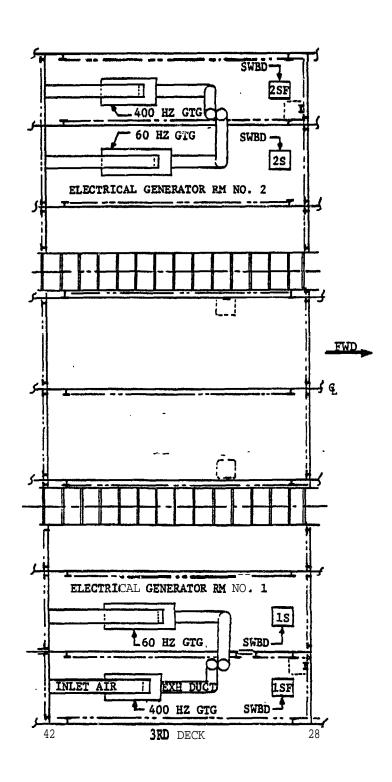


Figure 2.3.3-7 (U): Electrical Generator Rooms 1 & 2 (U)

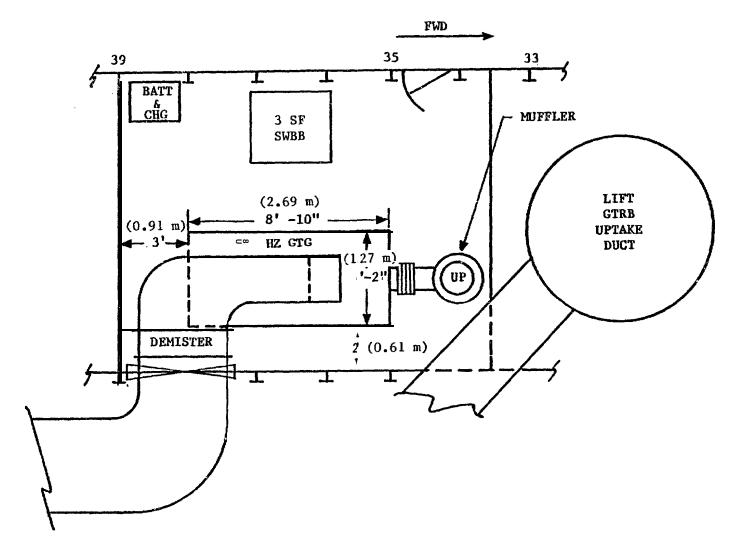


Figure 2.3.3-8 (U): Baseline GTG Room No. 3 (U)

- under the most demanding condition is approximately 500 kW of 60 Hz power and 500 kW of 400 Hz power. Therefore, the normal operating configuration requires two each of the 60 Hz and 400 Hz generators to be running, leaving one of each type in reserve. These off-line reserve generators are automatically started when required, and are thus functionally equivalent to conventional "emergency" generators.
- (U) The GTG sets and associated switchboards are arranged for remote control and monitoring and for limited local control. Automatic and manual controls are provided for remotely paralleling the three 60 Hz generators and for remotely paralleling the three 400 Hz generators. Both the 60 Hz and 400 Hz systems are equipped with voltage and frequency trim controls, load shedding, load sharing, malfunction shutdown, overload controls, and warning alarms.
- or a generator failure. These provisions include automatic start of an off-line generator and automatic paralleling with the system bus. In the event of failure of an on-line generator, an automatic load shedding scheme protects the remaining vital loads. Manual reset of breakers is required following load shedding as a safety precaution. Sustained generator overloads activate an automatic sequence to shed non-vital consumers, and to start up and parallel an off-line generator if necessary. Failure or malfunction of an operating generator also results in immediate automatic startup and parallel operation of an off-line generator. The system provides ample capacity for across line motor starting of the largest motors currently identified or anticipated for consumers.
- (U) Two power sources are supplied for all vital loads. The lighting, "Circle W" ventilation, electronics, fire pumps and ship's control receive normal power from one switchboard and an alternate supply from a different switchboard via a bus transfer device located near the using equipment. Other vital consumers are supplied from a different switchboard for each element of a vital equipment pair, to assure continuity of

service. Thus, in the event of a failure or casualty of the power supply to any vital load, all that systems' generators or switchboards (60 $\rm Hz$ or 400 $\rm Hz$) would have to fail to create a total loss of power.

- (U) A radial lighting distribution system is employed that consists of 450 V, 60 Hz feeders from the ship's switchboards reduced through suitable transformer banks to 120 V distribution panels within each cube. The distribution panels supply centrally located distribution boxes, which distribute power to individual local lighting circuits. Vital lighting in spaces, as defined under "Emergency Lighting System" in Section 331b of the GSS, is provided with a normal and alternate source of power through automatic bus transfer equipment, since this results in the simplest back-up arrangement.
- (U) Red low-level illumination is provided for standing lights, access routes and spaces requiring dark-adapted vision. Battery powered hand lanterns are provided throughout the ship to supply a limited amount of illumination in the event that other lighting sources fail.
- (U) 2.3.3.5 Electric Plant Weight Breakdown -- The following Table 2.3.3-1 shows the estimated percentage weights of the major equipments and components of the electrical system.

Table 2.3.3-1 (U): Electric Plant Weight Breakdown(U)

SUBSYSTEM	PERCENT OF SYSTEM	
Gas Turbine Generator Sets	20.0	
Turbine support equipment	14.0	
Switchgear, panels, fixtures	35.2	
Electrical Cable	20.5	
Miscellaneous	10.3	

⁽¹⁾ General Specification for Ships of the U. S. Navy

- high confidence that the requirements for electrical power will be completely met, regardless of operating condition, The associated trade-off studies provide assurance that the baseline system can be implemented with off-the-shelf equipment and at competitive prices. This system features six generators, of which only four are required to supply the maximum load. This offers advantages over other configurations which depend on a smaller number of larger generators. These advantages include:
 - A turbine or switchboard failure has less impact on total power generation capability.
 - Major components are smaller and easier to remove for depot repair or replacement.
 - Smaller GTG envelope and smaller exhaust piping allows greater installation arrangement flexibility.
 - Set enclosures are smaller and easier to remove in confined GTG rooms.
 - It is feasible to provide a reserve GTG for each power frequency.
- (U) The power distribution system is closely patterned after those standard Navy practices presented in GSS Sections 300 and 320 and in appropriate Design Data Sheets. Minor variations only have been made, including (1) the addition of a ring bus arrangement for added flexibility and reliability, and (2) substitution of disconnect switches for circuit breakers at one end of each bus tie cable to achieve weight and cost savings. Therefore, with high confidence, the system will provide satisfactory and reliable system performance.
- (U) The composite design of the electrical system and the definition of its individual components functional and physical characteristics is considered a source of low risk. This conclusion is based on the ready availability of the chosen hardware, as well as its statistically proven performance and the straightforward integration of the overall system.

- 2.3.4 COMMAND, CONTROL AND COMMUNICATIONS (c^3)
- (U) 2.3.4.1 Summary Description -- c^3 functions are accomplished by subsystems and equipments arranged and integrated to optimize the collection, evaluation, display, and dissemination of data and intelligence supporting command and control. The c^3 system includes equipments for:
 - Display

- Data processing
- Navigation and collision avoidance
- Interior communications
- Exterior communications
- (U) The ${\bf c}^3$ system interfaces with Combat System elements for underwater, surface and air surveillance, as well as Combat System fire control and weapons elements.
- (U) Worldwide navigation capability and continuous absolute and relative position as well as ship's speed, heading, drift angle and attitude, are provided by the navigation system. The navigation system includes the hardware and data processing necessary to receive and integrate signals from an inertial navigation system (SHIPS-G-5683; TYPE II), and from Omega (SRN-17) and satellite radio mavigation (AN/WRN-5: SATNAY).
- (U) The surrounding surface environment is monitored to provide the capability to sense and quantitatively measure potential collision situations. The collision avoidance subsystem displays the surface situation and computes trial evasive maneuvers so that the ship may safely avoid predicted areas of danger. Navigation aids, shoals, and other significant data are stored for display as a synthetic map along with radar derived data as an aid in coastline, harbor, river, and shoal area piloting.
- (U) 2.3.4.2 List of ${\bf c^3}$ Equipment -- The list of ${\bf c^3}$ equipment is contained in Appendix C. Interior Communications and Navigation Equipment are separately identified. The list itemizes equipment physical characteristics, weight and ship services requirements.

- (U) Collection, evaluation, display and dissemination of information relative to the friendly and enemy environment, and control of sensors and weapons is centralized in a combat information center (CIC). Equipment and operator stations are arranged on the basis of functional adjacency requirements to improve reaction time and permit positive control over weapons and sensors.
- (U) The CIC arrangement permits evaluation of the air, surface or subsurface environment from a centralized station. The CIC operators exercise control of all weapons, sensors and displays and keep the commanding officer apprised of the tactical situation.
- (U) Multiple path exterior communications are provided, and communications equipment is arranged functionally in a manner consistent with minimum manning. Transmitter and receiver groups are located in the transmitter room adjacent to the communication center. Remote control devices for transmitter and receiver groups are centrally located in the communication center.
- (U) 2.3.4.3 c^3 System Weights -- The following table 2.3.4-1 delineates the weights of major c^3 subsystems.

Table 2.3.4-1 (U): c^3 Subsystems Weights(U)

·	WEIGHT		
SUBSYSTEM	LT	kn	% OF TOTAL
Command & Control System	5.0	49.82	15.5
Navigation System	3.5	34.87	10.9
Interior Communi- cations	14.7	146.47	45.6
Exterior Communi- cations	9.0	89.68	28.0
System Total	32.2	320.84	100.0

- (U) 2.3.4.4 **c**³ System General Arrangements and Function Block Diagrams -- **c**³ system arrangements of the **ANVCE** Near Term Point Design SES are contained in Appendix B, Section B.S. Also included is an overall system block diagram encompassing only **c**³ e¹ ements but also elements of the ship's combat system **descri**ted in Section 2.3.7 of this report and a matrix of interior communications voice systems on the ship.
- (U) The drawings are grouped in the Appendix for consistency of report format and the benefit of the reader.
- (U) 2.3.4.5 c^3 Risk Assessment -- Only Navigation and IC Systems were evaluated in terms of risks. The remainder of c^3 systems are comprised of government furnished or government nomenclatured equipment with minimal risk to the near term SES design.
- (U) Since the Navigation and Collision Avoidance Systems (CAS) are comprised almost entirely of government nomenclatured equipments, there is low technical risk in its implementation. Modifications to the AN/APS-116 radar constitute the principal departure from nomenclatured equipment. There is low technical risk involved in developing the required NAVCAS computer programs. The CAS consists of the following elements:
 - a. CAS control and display
 - b. **AN/APS-116M** Collision Avoidance Radar Subsystem with its own dedicated control unit
 - c. CAS data processor and computer programs (AN/UYK-20(V))
 - d. CAS water depth sensor
 - e. CAS map data storage
 - f. Low light level television (space and weight)
 - g. Radar Beacon (space and weight)
 - h. Back-up search radar **AN/SPS-55** (part of Surface Surveillance, Section 4.4.6)

(U) Navigation and collision avoidance equipment was selected from Navy inventory items to meet the accuracy, reliability and special requirements of the near term SES. The interior communication system (IC) equipment group provides the means and methods for directing functions within the near term SES, other than for weapons control, by the transmission and reception of orders and the exchange of information by electrical and audible means. The IC equipment group also provides audio and television entertaiament. All IC system equipments are standard equipments and involve minimum risk.

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2.3.5 AUXILIARY SYSTEMS

- 2.3.5.1 Auxiliary Systems Less Lift System -- The **near** term SES **auxi-**liaries combine traditional shipbuilding experience with innovative design. The auxiliary systems were developed for operational requirements with performance, reliability and low weight as primary objectives.
- (U) 2.3.5.1.1 Climate Control System -- The Climate Control System consists of the compartment heating, ventilation, and air conditioning (HVAC) system; machinery space ventilation; and rhe ship's stores refrigeration system.
- (U) Heating, Ventilation and Air Conditioning (HVAC) -- The HVAC System provides conditioned air to various spaces and/or major equipment located throughout the ship. The system combines electrical resistance heating; meachnical fresh air supply and exhaust; and recirculating air conditioning. The system features 400 Hz electric motor powered packaged air conditioning plants and 400 Hz electric motor driven axial flow fans.
- Machinery Space Ventilation -- Thirteen air supply systems supply 100 percent summer cycle outside air to all auxiliary machinery rooms, electrical generator rooms, lift fan rooms, lift fan engine rooms, and main propulsion engine rooms. There are no duct preheaters for heating air in winter cycle.
- (U) Refrigeration System Two separate 400 Hz motor driven centrifugal, packaged type, refrigeration plants are provided for ship's stores refrigeration. Each refrigeration machine supplies freon to the cooling coils in the freezer and chiller spaces. One unit maintains the required temperatures for both spaces during normal operation with redundancy provided by the second machine; two refrigeration machines are used for pulldown.

- (U) 2.3.5.1.2 Seawater Systems -- The Seawater Systems consist of all seawater supply and drainage systems. These include fire main, sprinkling, auxiliary seawater, scuppers and deck drains, plumbing drains, and drainage systems.
- (U) Firemain and Auxiliary Seawater System -- The seawater services are furnished by a single combined firemain and auxiliary seawater system.

The system is arranged as a **firemain** for damage control considerations, and separated into fire and auxiliary service functions at the respective required pressures. Four each centrifugal Dumps are used. They are each capable of a delivery of 400 gpm $(.252 \text{mm}^{3}/\text{s})$ at 125 psig (0.862 kPa).

- (U) Scuppers and Deck Drains -- The scuppers and deck drains consist of all space deck drains at and above the second deck. Space deck drains (with GRP piping) from wet spaces and fan rooms are combined and directed overboard via scupper valves, The overboards are located on the third deck above the full load waterline to reduce drag.
- Which collect soil wastes from water closets and urinals, and waste drains from showers, lavatories, sinks, laundry, galley, and scullery. The drains are led to a vacuum collection tank from which wastes are either discharged overboard or directed to the collecting holding and transfer tank (CHT). Connections are also provided for discharge to shore receiving facilities.
- Drainage System -- The drainage system consists of a main and secondary drainage system which provides the drainage for the machinery spaces and other spaces on and below the third deck. The main drainage eductors of 500 gpm (0.317 mm³/s) capacity are provided for the propulsion engine rooms and the waterjet pump rooms. Eductor actuating water is provided by the fire main and auxiliary sea water

system. Discharge is overboard above the full-load off-cushion waterline,

- (U) 2.3.5.1.3 Fresh Water Systems -- Fresh water systems include **the dis**tilling plant, the potable water system, the turbine water wash systems, the cooling water system, and the auxiliary fresh water cooling system.
- Potable and Fresh Water System -- The potable water system is basically a standard shipboard system. The fresh (wash) water system operation is managed to reduce storage requirements. **GRP** piping is utilized to the maximum extent practical.
- Cooling Water System and Auxiliary Fresh Water Cooling

 Systems -- Two electronic cooling water systems are provided: the

 cooling water system (freon cooled) and the auxiliary fresh water

 cooling system (seawater cooled). Cooling water for electronic equipment is provided by a closed loop system in accordance with pertinent

 NAVSHIPS drawings.
- (U) 2.3.5.1.4 Fuels and Lubricants Systems -- The light weight fuel and lubricants system is a straightforward design featuring common functional manifolds and minimal connections. The system consists of the ship's fuel oil system and the aviation fuel system, where a common fuel (JP-5) is used for all ship fuel services.
- (U) Ship's Fuel Oil System -- The ship's fuel oil system consists of the fill, transfer and service.
 - Fuel Oil Fill -- Fueling of the ship is provided at port and starboard fill stations. The fueling stations utilize seven inch (0.18 m) probe receivers, each capable of 3000 gpm (1.89 mm³/s). There are 23 fuel tanks.
 - fuel Oil Transfer -- Provision is made for transferring fuel between tanks to shift the ship's center of gravity for optimum operating conditions. The 400 gpm (.252 mm³/s) pumps also transfer fuel to the fuel oil storage tanks,

- c. Fuel Oil Service The two deep fuel oil service tanks are located amidships, port and starboard, with their adjacent fuel centers including functional manifolds, pumping equipment, filter coalescers and proper distribution lines. The port and starboard fuel service subsystems have the capability of cross-feed (redundancy) in emergency conditions.
- (U) Aviation Fuel System -- The aviation fuel system consists of the helicopter fueling and service system. The system includes **two** service tanks that are filled from the ship's storage tanks through filter coalescers.
- (U) 2.3.5.1.5 **Air,** Gas and Miscellaneous Fluids -- The air, gas and miscellaneous fluids consists of compressed air systems, nitrogen systems, fire extinguishing systems, and hydraulic fluid systems.
- (U) 'Compressed Air Systems -- Both low pressure and high **pressure** air systems are provided.
 - O Low Pressure Air System -- The low pressure compressed air system is furnished by bleed air from the GTG and main propulsion engine gas turbine. The low pressure compressed air system **consits** of ship's service, control air, and starting air system.
 - o High Pressure Air System -- A high pressure air system is provided for charging MK-32 torpedo tubes. A nominal 3000 psig (20.68 **MPa)** compressor, dehydrator and air flasks are used for this particular launch activity.
- (U) Nitrogen System -- A Nitrogen System is provided for helicopter services. The nitrogencharging station in the helicopter hangar consists of five cylinders and a variable regulator capable of supplying 70 to 3000 psig (.048 to 20.68 MPa) of oil free nitrogen for helicopter tire inflation and other helicopter services.

- (U) Fire Extinguishing Systems -- The fire extinguishing systems on the ship consist of AFFF, Halon (FE 1301) fixed flooding systems, hi-expansion foam, and portable Halon extinguishers.
 - A high capacity AFFF proportioning system is provided **for** the helicopter hangar and landing area. A fixed sprinkling system is provided for the hangar and two hose stations are provided port and starboard on the landing platform.
 - Fixed flooding Halon systems which meet the requirements
 of NFPA No, 12A, are the primary fire extinguishing
 systems for propulsion engine rooms; lift fan engine
 rooms; auxiliary machinery rooms; waterjet pump room;
 and electrical generator rooms.
 - A high expansion (Hi-X) foam system is provided
 as a secondary (backup) system for the Halon fixed
 flooding systems, Port, starboard, and amidships proportioning units are supplied from the fire.main system.
- Hydraulic System -- A centralized hydraulic system provides hydraulic flow for the subsystems and incorporates a lightweight flexible power distribution system capable of rapid response. Operating mode evaluations indicate that **the main dydraulic** power can be delivered by two hydraulic pumps driven by the lift fan engines. These pumps will be augmented by electric motor-driven pumps of comparatively small flow rate during peak hydraulic load activities. The motor-operated pumps will also be used during off-cushion and **in-** port operations.
- (U) The maximum system hydraulic power available is 273.5 gpm (0.173 mm³/s) at 3000 psig (26.68 MPa); hydraulic fluid in accordance with MIL-H-83282 w-ill be used.

- (U) 2.3.5.1.6 Underway Replenishment System -- The Underway Replenish.

 ment System comprises the Replenishment-at-Sea System and the Ship

 Stores and Personnel and Equipment Handling.
- (U) Replenishment at Sea -- Replenishment at sea is provided for by VERTREP and alongside refueling. A combined VERTREP, HIFR, and Helicopter Landing area is provided on the main deck aft of the hangar.
- been simplified as much as possible by arrangement of magazines, storerooms and refrigerated spaces on the main and second deck for each of access. Handling on the main deck will be by hand pallet trucks, package truck, and manual means. Materials to be struck down to the second deck will be conveyed by a vertical conveyor, located starboard. A stores handling area is provided on the second deck. Heavy items will be handled by davit and monorail, The co-location of galley and refrigerated spaces eliminates need for a dimb-waiter.
- (U) 2.3.5.1.7 Mechanical Handling System -- The mechanical handling systems are the anchor handling, mooring and towing, boat handling, hangar door, and the helicopter securing-and traversing system.
- (U) Anchor Handling System -- The basic requirements are anchoring with a 70 knot (36.01 m/s) wind velocity, a 4 knot (2.06 m/s) current velocity, and in 40 fathoms (73.15 m) water depth,
- (U) A single anchor, of the **Danforth** Hi Tensile type, was selected on the basis of the recommended criteria.
- (U) Mooring and Towing. System --Three line handling capstans are provided to facilitate mooring alongside piers and other ships.
- (U) Boat Handling and Stowage -- Boat handling facilities consist of abandon ship equipment and an inflatable hard bottom boat for use

during helicopter plane guard operations and for man overboard recovery. Six MK-V inflatable CO₂ twenty five-man life rafts are provided in standard containers.

- (U) Hangar Door System -- Horizontally deployed hangar doors are used and consist of vertical hinged panels which travel on horizontal tracks. The doors are mounted under constant tension by spring loaded lower roller bearings that apply tension to upper roller bearings. Door operation is by an electric motor and gearbox drive.
- Helicopter Securing and Traversing -- A helicopter securing and traversing system will be installed, but its final design has not been made. Alternatives which will meet the design requirement with use of minimum space and weight include a present preference for a modified version of the prototype being developed by the Navy for use aboard aviation facility ships. The unit is modified to accommodate two (2) helicopters.
- (U) 2.3.5.1.8 Special Purpose Systems -- The Special Purpose Systems consist only of the Environmental Pollution Control System. The Environ-- mental Pollution. Control System is concerned primarily with the solid and liquid wastes produced by the ship. The primary item is the Collecting, Holding and Transfer (CHT) tank which collects all plumbing and fresh water drains. The holding tank is sized to accommodate one day's waste. A sewage pump is used to discharge waste from the vacuum collection tank to the CHT. This same pump (a standby pump is provided) is used to discharge the CHT to a shore connection. A sewage eductor is used outside of the contiguous zone.
- (U) Garbage is ground and flushed, **via** the vacuum collection tank, to the CHT, Solid trash is treated by compaction and **retained** aboard for disposal at a shore **facility**.
- (U) Contaminated oil drains (fuel, lube oil, **helo** defuel, stripping lines, etc.) are discharged into an oily water drain tank. They are pumped to share facilities by the waste oil drain pump.

(U) 2.3.5.1.9 Auxiliary Systems Percentage Weight Breakdown -- Table 2.3.5-1 shows the estimated percentages for major auxiliary subsystems less the lift system.

Table 2.3.5-1 (U): Auxiliary Systems Percentage Weight Breakdown (Less Lift System) (U)

SUBSYSTEM	% OF SYSTEM.
Heating, Ventilation, and Air Conditioning	13.4
Seawater	15.4
Fresh Water	9.1
Fuels and Lubricants	10.6
Handling and Storage Air, Gas, and Misc. Fluids	19.9
Mechanical Handling	19.0
Special Purpose System	11.3
Miscellaneous	1.3

- (U) 2.3.5.2 Lift System
- (U) 2.3.5.2.1 Air Distribution Summary Description -- Lift system air distribution consists of two sets of lift machinery and ride control equipment schematically shown in Figure 2.3.5.2-1. Each set of lift machinery is arranged in an in-line configuration, one set on each side of the ship. Power for each set of lift machinery is supplied by an LM 2500 gas turbine engine. The required power and speed is delivered to the lift fans via the lift power transmission system which consists of the reduction gear unit, shafting, and associated components. The lift fans draw air through inlets on the ship's deck, and discharge into separate and independent air distribution ducts. The forward fan on each side of the ship supplies air to the bow seal, the center fan supplies the cushion, and the aft fan discharges into the stern seal. Each fan duct is supplied with a shut-off valve to prevent back flow when the fan is not operating.
- (U) 2.3.5.2.2 Seals Summary Description -- The design for the bow and stern seals utilizes a series of flexibly connected fiberglass planers at the water interface. A nylon/elastomer pressure bag behind the planers provides the force to contour and support the planers. Entitled the Advanced Planing Seals, they are a new, improved concept in SES seals that combines excellent low drag performance with rugged, high wear resistance qualities. The excellent wear resistance of the planing seals is exemplified by high speed water impact erosion of the glass reinforced plastic (GRP) elements, orders of magnitude less than that occurring with the rubberized fabric material of the common bag and finger seal systems. The advanced planing seals also perform the normal and vital functions of containing the air in the cushion, contributing to ship ride quality, and providing pitch and roll restoring forces to the ship.

- (U) 2.3.5.2.3 General Arrangement -- The air distribution and seals are arranged as shown in Figure 2.3.5.2-2. Details of the air distribution and seals combined system are discussed in the next paragraph.
- (U) Power Units A total of two gas turbines, each driving three VG fans through a reduction gearbox, are utilized in the SES lift system. One LM 2500 is located on the starboard side of the ship and the other on the port side. Each gas turbine is independent of the other, and can deliver 22,500 (16.78 kW) continuous shaft horsepower and 27,000 (20.13 kW) intermittent shaft horsepower. The link mounting system is identical to that used for the propulsion plant LM 2500 gas turbines.
- the flange which connects the power transmission system begins at the flange which connects the power turbine to the reduction gearbox shaft. A disc type brake is mounted on the ;earbox at the input shaft. At the output side of the reduction gearbox, a torsionmeter is installed, Two diaphragm type flexible couplings are installed between the torsionmeter (gearbox output shaft) and the first lift fan, one at each end of the shaft. The driven power to the fan rotor is picked up through the integral fan couplers. The integral fan couplers are those sections of drive shaft within, and integral to, the fans which permit decoupling of any fan. Flange couplings are used at each end of the fan throughshaft to connect to the drive shaft. A length of shafting and two shaft bearing supports with associated couplings are situated between the fan couplers of the second and third fans. Seals are provided where the life shaft penetrates watertight bulkheads,
- (U) The lift reduction gearbox is a parallel shaft design with an overall reduction ratio of 1.93 to 1. The gearing is external double helical of involute form and is case hardened and ground to AGMA quality 12 or better. The gear case is an aluminum casting.
- (U) The power transmission system for each set of lift fans is designed to transmit a maximum of 27,000 bhp (20.13 kW) from the gas turbine

through the reduction gears to the lift fans. The system is designed to accommodate a maximum input shaft speed of 3,600 rpm (60 r/s) from the turbine and a **maximum** output shaft speed of 1,865 rpm (3108 r/s) from the reduction gear to the fans.

- (U) Lift Fans -- The Lift and Ride Control System uses a total of six lift fans. The fans are symmetrically located, three port and three starboard, with each group positioned in line on a common shaft.
- (U) All of the lift fans are identical except for assembly differences that depend on fan location. The fans are centrifugal type with an 86-inch (2.18m) diameter rotor, a housing, and variable geometry elements. They incorporate double axial inlet design, airfoil shaped radial blades, constant velocity volute housings, and a single circular discharge. The variable geometry fan elements (translating sleeve in each of the inlets) are included for modulating the air flow for ride control purposes. The fan is shown in Figure 2.3.5.2-3.
- (U) Included in the fan envelope are the self-contained rotor decoupler, rotor bearings and coaxial line shafting that permft independent decoupling of any fan while operating under any design load. The mechanisms additionally provide for remotely activated recoupling of fans from an at-rest condition.
- Lift Ducting -- The lift air is delivered through ducts to the bow seal, cushion, and stern seal. The bow and cushion air ducts are short, conical sections which act as diffusers to reduce high velocity losses. The stern seal air duct is long in comparison and employs turning vanes in most elbows to reduce pressure losses. The long stern seal duct was selected in favor of long shafting as a result of trade-off studies. Ducting sizes are optimized for the system operating conditions. Attention is given to the velocity of the air balanced against the space allocation for the ducts,

- (U) The four forward ducts have hydraulically operated butterfly type shut-off valves located near the fans. A louvered type shut-off valve is located over the stern seal in the long supply duct from the aft fan. These shut-off valves prevent back flow from the pressurized cushion if a fan is not operating for any reason. A pair of stern seal transfer ducts is also included in the lift system. These ducts provide stern seal to cushion air flow and contain throttling valves for stern seal to cushion pressure ratio regulation.
- Lift Air Intakes -- The lift fan inlets supply atmospheric air to the lift fans for pressurization and subsequent distribution into the cushion, bow seal and stern seal. Five openings are provided in the deck, port and starboard, to supply air for each group of three fans. The intake openings are positioned directly above the lift fan bellmouths. Four of the inlets are 12.2 feet (3.72 m) wide by 4.4 feet (1.34 m) long; the fifth inlet, which supplies the adjacent inlets of the mid and aft fans is 12.2 feet (3.72 m) wide by 8.9 feet (2.71 m) long,
- (U) A fairing is provided around the openings to eliminate the ship boundary layer air and surface water flowing on the deck from entering the fan air inlets. The inlet design incorporates aerodynamic turning vanes to direct the air flow downward into the fan rooms. The vanes permit recapture of about half the air velocity head across the deck. An electrical heating system is incorporated in the vanes to provide anti-icing capability. The vane walls of each flow passage are treated, to provide the necessary sound attentuation,
- Ride Control System -- The ride control system integrates the variable geometry lift fans and vent valves and their associated actuators with appropriate ship motion sensors and the controller electronics into an active system. The total active system modulates the seal and cushion airflows to reduce the ship's heave accelerations to an acceptable level. The primary ride control system uses the variable geometry fans to control airflow. Vent valves are provided to expand the flow range

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- (U) available in high seas and to provide maximum versatility for RCS development.
- (U) The variable geometry feature of the fans consists of hydraulically translated sleeves in the fan inlets, The sleeves have a maximum travel range of 18 inches (0.46 m). When fully closed, they reduce fan flow to less than 10 percent of design point conditions. The frequency response bandwidth of the sleeve actuating system is 0 to 2 Hz, The nominal maximum slew rate is 57 inches/second (1.45 m/s),
- (U) The cushion vent system consists of two identical valve and duct arrangements at the stern of the ship near the centerline. The ducts, as shown in Figure 2,3.5.2-4 extend from the two wet deck cushion openings just forward of the stern seal into the third deck area and out of the transom of the ship. Discharging the vented air aft generates some propulsive thrust. Hydraulically operated, fast response, louvered valves are positioned in the ducts at the third deck level. The valves, as pictorially shown in Figure 2.3.5.2-S are designed as a battery of modules for fast response, reliability, and ease of replacement, The cushion vent ducting and valves accommodate flow rates up to 60,000 cfs (28.3 m³/s). The valves are designed for a maximum pressure differential of 600 psi (4.14 MPa). Vent valve frequency response bandwidth is 2 Hz. Nominal maximum slew rate is 2.5 full strokes (close-open-close) per second.
- (U) Advanced Planing Bow Seal -- The advanced planing bow seal is illuscrated in Figure 2.3.5.2-6. Geometry of the seal is given in Figure 2.3.5.2-7. The seal consists of four main elements which are next described.
- (U) An elastomer pressure bag is attached to the bow at the 40 foot (12.19 m) waterline and normally extends aft in a continuous circular arc and connects to the wetdeck. The bag is configured of eight identical modules with elastomer end caps at the sidehull interfaces, The bag

The second secon

- (U) and end caps provide a flexible structure which contains the bow seal air while minimizing water ingress into the seal. The aft loop of the bow bag contains slotted openings of fixed width to provide controlled air flow between the seal and the cushion and to assure rapid water drainage.
- (U) The planer/stay portion of the seal consists of thirty-two (32) modules across the beam of the craft. These planer/stay modules are constructed of glass reinforced plastic (GRP) and are attached to the wetdeck at the 40 foot (12.19 m) waterline. The upper forward portion, or stay, has relatively low stiffness allowing it to conform to the curvature of the forward portion of the bow bag. Near the lowest portion of the bow bag loop, the stays widen and are joined together by flexible sealing strips to form a continuous fiberglass planer surface.
- (U) A 31-inch wide tapered GRP feather edge is attached to the trailing end of each bow seal planer module. This feather edge, having increased flexibility is used to attentuate the effective wave impact on the seal, assist in cushion sealing and improve the seakeeping capability of the craft,
- (U) Each planer is supported by a geometry strap and a retract strap. The strap provides mid span support and geometric control of the planer through the full range of sea states. The geometry strap normally carries a tension load due to the cushion pressure acting on the planers, but may be unloaded for a short duration when encountering high waves at a higher velocity.
- (U) A seal **retract** strap is attached to the retraction reel recessed inside the hull and extends down to an attachment at the aft edge **of** each planer, The straps provide for full retraction of the seals against the **wetdeck** for off-cushion operation and also for adjustments and trimming of the seals for minimum drag during hump transit, **partial**—cushion and full-cushion operation.

- (U) The 32 straps pass through slots in the wetdeck structure, and over sheaves, before attachment to the retraction reel drums. Provision is made at each drum for unlimited strap length adjustment. Locks at the retraction drive outputs prevent inadvertent seal extension by high loadings. The drive units allow for high and low speed seal retraction, low speed extending adjustment, and the rapid free-wheeling extention associated with the craft going on-cushion
- Advanced Planing Stern Seal -- The stern seal is illustrated in Figure 2.3.5.2-8 and the seal geometry is given in Figure 2.3.5.2-9. Planer/ stay elements are attached to the wetdeck and extend to waterline 0. These elements are similar to those of the bow seal, i.e., tapered trailing edge. I-beam reinforced planing section, and highly flexible stay section, with the exception being that the stays are the width of the planer module. Thirty-two modules span the beam of the craft. The full width stays protect the elastomer bag membrane from the erosive effects of direct high speed water impact. The planer attachments are identical to those of the bow seal.
- (U) A four-lobed bellows bag is attached between the wetdeck and planer to contain air pressure within the stern seal. The bellows bag is built in modular sections and is fabricated of the same nylon/elastomer material as the bow seal bag, Holes are located along the lower lobe of the bellows bag and sized to permit rapid drainage of water. The four-lobe bellows is optimum for seal spring rate requirements and for tensile loading in the membrane.
- (U) Convolute tension cables are connected between the wetdeck and the junctions of the lobes of the bellows bag in order to maintain the geometry of the bellows bag through the entire deflection of the seal. A set of lower geometry straps, connected between the wetdeck and the planer at the bellows bag/trailing edge junction, restrain the aft movement of the seal. Retract straps are attached to each planer near the planer's trailing edge and are connected to the retract system

- **(U)** reel in a manner similar to the bow seal system. The stern seal retraction system is similar to the bow system.
- (U) 2.3.5.2.4 Tabulation of Key Parameters -- The key parameters of the lift system are presented in Tables 2.3.5.2-1, Lift System Physical Parameters, Table 2.3.5.2-2, Lift System Point Design, and Table 2.3.5.2-3, Seals Design Parameters.

Table 2.3.5.2-1 (U): Lift System Physical Parameters (U)

		UNITS		VALUES	
		ENGLISII	(SI)	ENGLISH	(SI)
1.	Engine - LM2500, 2 Required		/7		
	Design Rotational Speed	RPM	(r/a) Kg	3500	(58.33)
	Maximum Continuous Power (MCP)	HP	(kW)	22500	(16785)
	Specific Fuel Consumption (SFC)	Lb/SHP-Hr	(kn/kw-Hr)	.40	(2.39)
	Maximum Intermittent Power (MIP)	ИР	(kW) //a/	27000	(20142)
	Volume	Ft ³	(m^3)	1064	(30.129)
	Basic Engine Weight	Lb	(kN)	10300	(45.817)
2.	Reduction Unit With Brake, 2 Req.				
	Power Capacity	UP	(kW)	27000	(20142)
	Gear Ratio			1.93	
	Gear Type: Single Reduction, Dougle	e Helical Invol			
	Volume	Ft ³	(m ³) Ky	114	(3.228)
	Weight Port	Lb	(kN)	4922	(21.894)
	Weight Starboard	Lb	(kN) /VJ	3762	(16.734)
3.	Lift Fans (ALRC) 6 Required				
	Type : Centrifugal, Dual Inlet, Cor	stant Velocity	Volute, Variable	Geometry, Deco	oupling Device
	Rotor Diameter	In	(m)	86	(2.184)
	Rotational Speed	RPM	(r/s)	1450	(24.17)
	Tip Velocity	FPS	(m/s)	5 4 4	(165.8)

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 $_{\text{\tiny T}}$ Table 2.3.5.2-1 (U): Lift System Physical Parameters Continued (U)

		UNITS		VALUES	
		ENGLISH	(SI)	ENGLISH	(SI)
3.	Lift Fans (ALRC) Continued		KPN		
	Design Head Rise	PSF	(k/Pa)	333	(15.9)
	Design Flow	CFS	(m ³ /s)	6039	(171.005)
	Design Efficiency, Fan Percent			8 4	
	Specific Speed, $N_S = \frac{NQ^{\frac{1}{2}}}{H^{\frac{3}{4}}}$			144	
	Exit Diameter	In	(m)	82.5	(2.096)
	Design Exit Velocity	FPS	(m/s)	163	(49.7)
	Maximum Rotational Speed	RPM	(r/s)	1865	(155.42)
	Maximum Flow (Approximate)	CFS	(m^3/s)	10000	(283.2)
	Maximum Power	UP	(kW)	9000	(6714)
4.	Transfer Shafting				
	Total Length (1) Per Ship	Ft	(m)	104	(31.699)
	Total Weight (2)	Lb	(kn)	8756	(38.948)
5.	Distribution Ducting				
	Total Length (3) Per Ship	Ft	(m)	532	(162.154)
	Total Weight (4)	Lb	(kN)	26631	(118,461)
6.	Fan Inlets				•
	Type: Flush Horizontal with Acoust	cic Turning Vanes	•		
	Velocity Ratio (IVR) at 80 Knots (Free Stream/Inlet	_	.70	
	Weight	Lb	(kn)	11910	(52.978)

1.

Table 2.3.5.2-1 (U): Lift System Physical Parameters, Continued (U)

- Pan internal shafting not included. (1) Total length from gear box interface to last fan.
- (2) Includes shafting flex couplings and bearing pedestals.
- (3) Includes ride control ducting.
- (4) Includes flex coupling and values.

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Table 2.3.5.2-2 (U): Lift System Point Design (U)

Ship Weight: LT (kN)

3000 (2.9892 E+04)

Wave Height H_{1/3}:

Ft (m)

4.6 (1.40)

Ship Speed:

Knots (m/s)

80 (41.2)

Pressures:

PSF (kPa)

Bow: 349 (16.71)

Cushion: 342 (16.38)

Stern: 376 (18 .00)

Total Flow Rate: CFS (m³/s) 37500 (I 061.88)

Lift System Efficiency: 76.1

Duct Losses: PSF (kPa)

Bow: 28 (1.34)

Cushion: 24 (1.15)

Stern: 39 (1.87)

Fan Parameters:

Speed: RPM 1535

Total Shaft Power HP (kW) 29743

Flow: CFS (m³/s)

Bow: 13146 (372.253)

Cushion: 13650 (385.525)

Stern: 10721 (303.585)

Engine Parameters (LM2500)

Speed: RPM 3266

Total Brake Power HP (kW) / 'V . 30661 (22873.1)

Total Fuel Flow Lbs/Hr (N/s) 13556 (16.750)

SFC $\frac{\text{Lbs}}{\text{RHP-Hr}} \left(\frac{\text{N}}{\text{kW-Hr}} \right)$, .442 (2.636)

Table 2.3.5.2-3 (U): Seals Design Load Parameters (U)

1. BOW SEAL LOADS

ELEMENT	DESIGN ⁽¹⁾ FACTOR (MINIMUM)	NOMINAL WORKING LOADS	MAXIMUM LOADS
Wet Deck Stay Attachment	2	2800 lbs/ft (114,000 N/m)	13,300 lbs/ft (202,000 N/m>
Fwd Wet Deck Bag Attachment	1.5	610 lbs/ft (8,900 N/m)	19,200 lbs/ft (280,000 N/m)
Aft Wet Deck Bag Attachment	1.5	380 lbs/ft (5,600 N/m)	13,200 1bs/ft (192,500 N/m)
Geometry Strap	2	20,990 1bs/strap (93,300 N/strap)	43,010 lbs/strap (142,000 N/strap)
Retract Strap	2	1,000 lbs/strap (4,450 N/strap	38,250 lbs/strap (171,500 N/strap)
Module-to-Module Joint	2.0	340 lbs/ft (5,000 N/m)	975 1.bs/ft (14,200 N/m)
Planer-to-Planer	2.0	1,000 lbs/ft (14,600 N/m)	5,000 lbs/ft (73,000 %/m)

⁽¹⁾ Maximum load multiplied by its respective design factor is the ultimate design load.

Table 2.3.5.2-3 (U): Seals Design Load Parameters (Continued) (U)

2. STERN SEAL LOADS

ELEMENT	DESIGN FACTOR (MINIMUM)	(1)	NOMINAL WORKING LOADS	MAXIMUM LOADS
Planer Attachment to Wet Deck	2		710 lbs/ft (10,500 N/m)	8,650 lbs/ft (126,000 N/m)
Geometry Strap			1,420 1bs/strap (6,320 N/strap)	
Wet Deck Attachment, Geometry Strap	1.5		1,440 lbs (6,400 N)	44,000 lbs (196,000 N)
Convolute Cable	2		8,426 lbs/cable (37,400 N/cable)	
Retract Strap	2		9,000 1bs/strap (40,000 N/strap)	39,550 1bs/strap (176,000 N/strap)
Stern Bag Wet Deck Attachment	1.5		830 lbs/ft (12,100 N/m)	13,200 lbs/ft (193,000 N/m)
Planer-to-Planer Joint	2.0		1,000 lbs/ft (14,600 N/m)	5,000 lbs/ft (73,000 N/m)

Maximum load multiplied by its respective design factor is the:
 ultimate design load.

Table 2.3.5.2-4 (U): Seal Materials Physical Properties (U)

Planer Materials Gloss Reinforced Fabric	UNITS English (SI)	VALUES (1) English (SI)
S-Type Class Epoxy Resin Sin Content Insile Strength Longitudinal Transverse Lexible Strength Longitudinal Transverse Lexible Modules Longitudinal Transverse	10 ³ psi 10 ⁴ Pa 10 ³ psi 10 ⁴ Pa 10 ⁶ psi 10 ⁷ pa	29 (29) 128 (610) 110 (525) 164 (795) 142 (680) 4.6 (21.0) 3.7 (17.7)
Pressure Bag Materials Elastomer Coated Fabric Presic Materials Nylon Fabric Neoprene Elastomer Presile S:rength Warp Fill	lbs/inch (N/m)	1805 316.00 1620 284,00
longation, Ultimate Warp Fill	7 7	53 53 72 72
Pressure Mag Materials — Elastomer Coated Fabric	UNITS English (SI)	VALUES English (SI)
Warp	oz/sq. yd (kg/m²) lbs/inch (N/m)	90 3.05 70 12.200

 $⁽¹⁾_{\text{Values}}$ for 0/90 degree eleven ply laminates

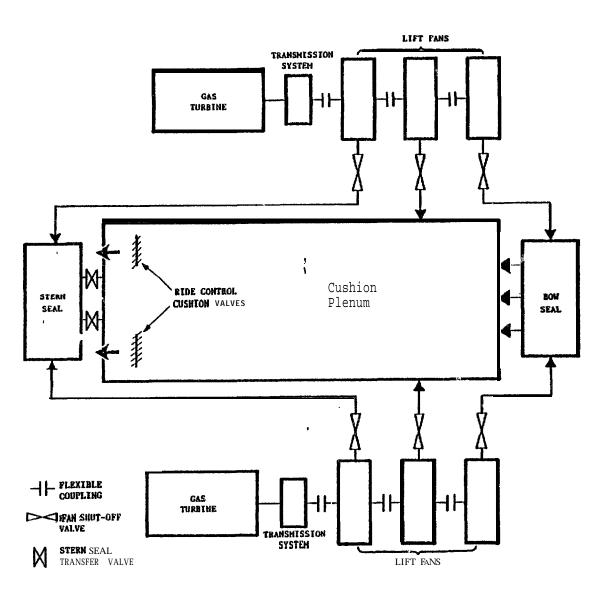
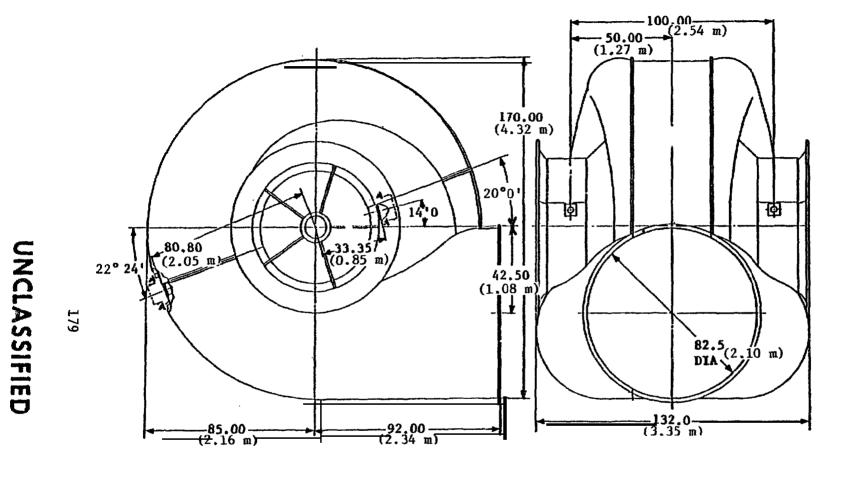


Figure 2.3.5.2-1 (U): Lift System Air Distribution Schematic (U)

Figure 2.3.5.2-2 (U): $_{\rm SES}$ Lift System Arrangement (U)

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Figure 2.3.5,2-3 (U): Lift Fan Envelope - 86 Inch (2.18 m) Diameter Rotor (U)

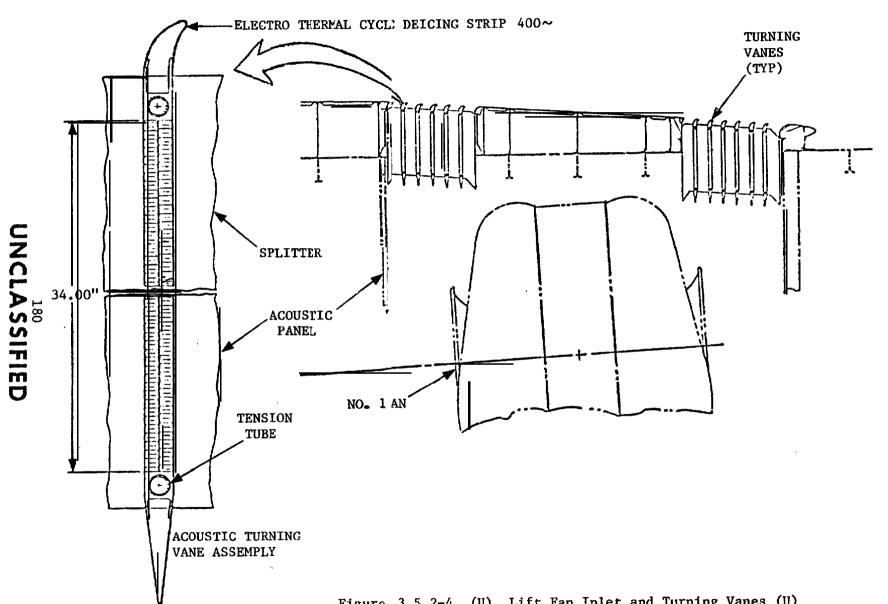


Figure .3.5.2-4 (U) Lift Fan Inlet and Turning Vanes (U)

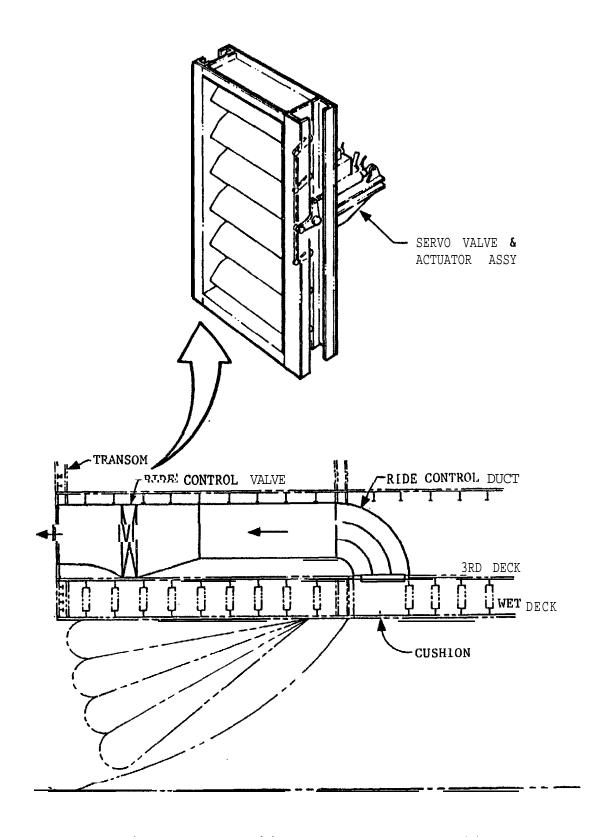


Figure 2.3.5.2-5 (U): Cushion Vent System (U)

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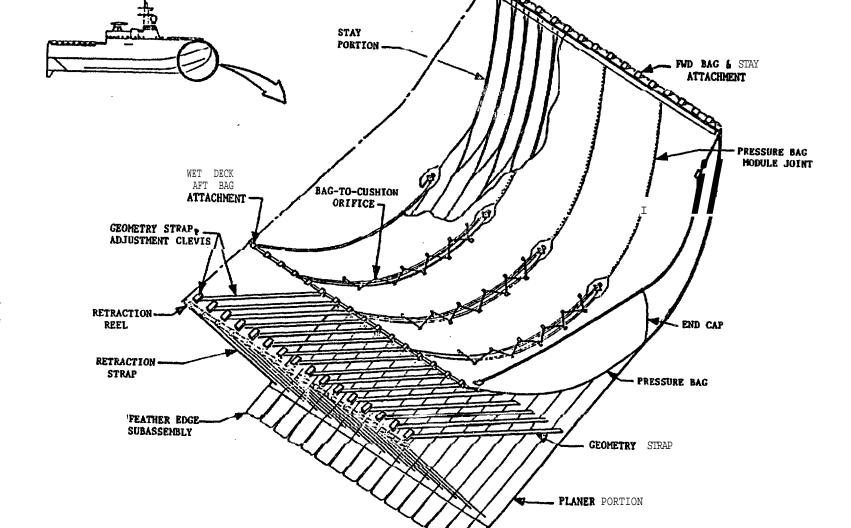


Figure 2.3.5.2-6 (U): Advanced Planing Bow Seal (U)

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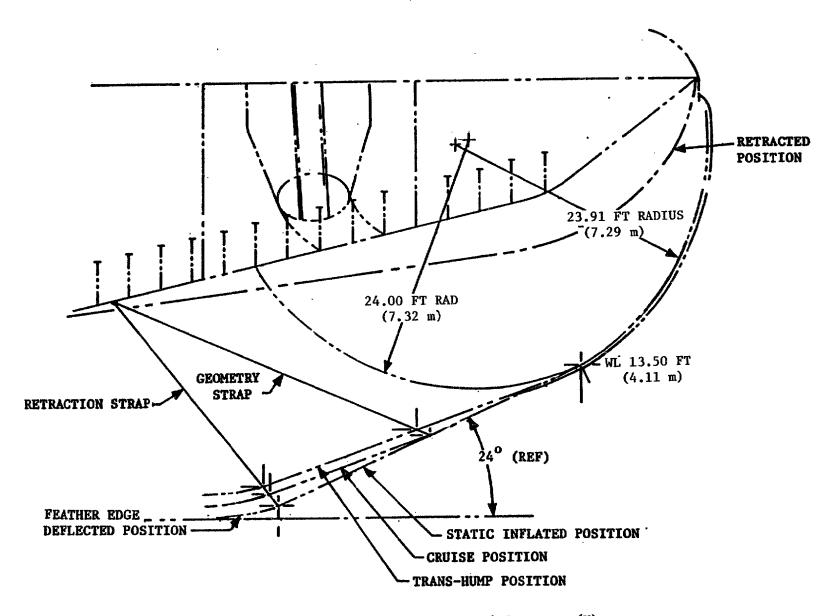


Figure 2-3.5.2-7 (U): Advanced Planing Bow Seal Geometry (U)

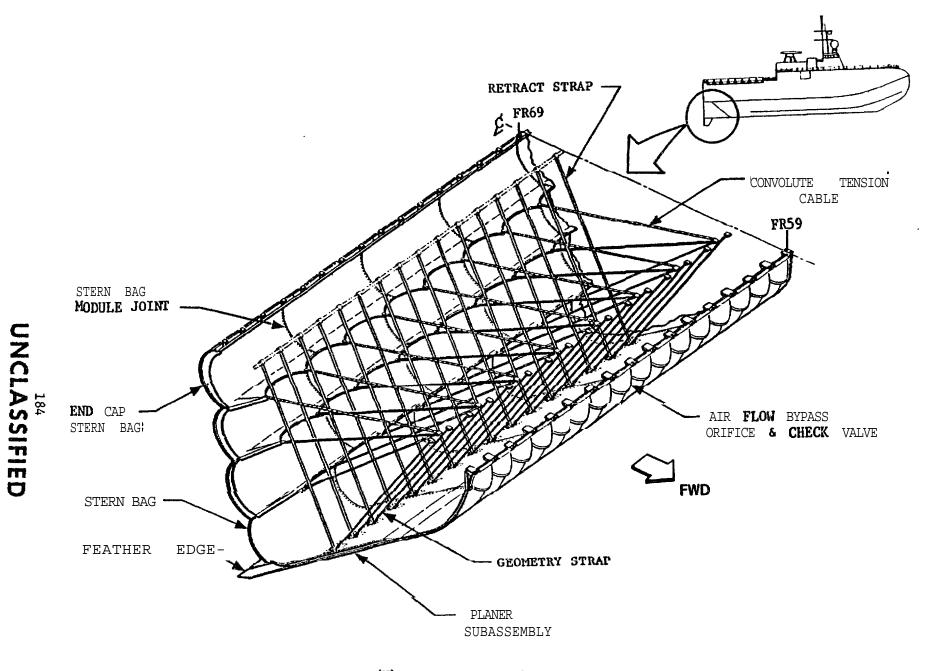
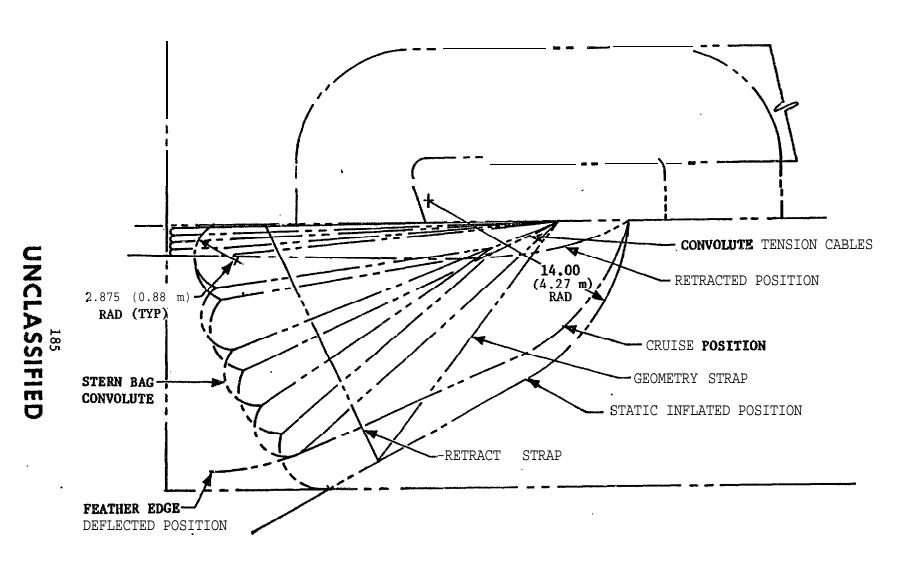


Figure 2.3.5.2-8 (U): Advanced Planing Stern Seal Assembly (U)



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Figure 2.3.5.2-9 (U): Advanced Planing Stern Seal Geometry

(U) 2.3.5.2.5 Lift System Weight Breakdown -- Table **2.3.5.2-5** shows the weight of each major lift system subsystem and each subsystem's percentage of the Lift System total.

Table 2.3.5.2-5 (U). Lift System Weight Breakdown (U)

	WEIGHT			
SUBSYSTEM	LT	kn	% of TCTAL	
Engines	10.5	104.62	10.9	
Fans	20.5	204.26	21.2	
Reduction Gear and Shafting	8.2	81.70	8.5	
Seals	31.1	309.88	32.2	
Ducting	7.4	73.73	7.7	
Valving	4.5	44.84	4.7	
Intakes	8.8	87.68	9.1	
Uptakes	1.9	18.93	2.0	
Support System	3.7	36.87	3.8	
LIFT SYSTEM TOTAL	96.6	962.52	100.0	

- (U) 2.3.5.2.6 Lift System **Technical** Risk -- The total lift system risk depends upon the individual **component** risks. Considering the diversification of functions and the number of components included in the lift system (lift air machinery, ride control elements, seals), the overall risk is subjective and is based on the relative importance of each function as follows:
 - Lift Gas Turbine Engine System -- The **LM2500** gas turbine engine is a production unit and is in use in other marine applications. The integration of this power unit into the lift system is well within present technological capabilities.
 - Power Transmission System -- Reduction gear design employs proven technology and similar gearbox designs have been utilized for marine applications. The transmission system arrangement and component selection are proven and within the present state-of-the-art. There is no apparent development risk for this system.
 - Variable Geometry Fan -- The variable geometry fan concept has been proven feasible by test at a number of scaled sizes. Especially significant is the use of 1/4-scale ALRC lift fans with VG on the XR-1 testcraft. The lift fan design must be verified in terms of full-size ship requirements and integrative ramifications.
 - Duct Configuration -- The analysis of the lift system duct configurations predicts the pressure losses with a high degree of confidence. The construction uses proven marine/aircraft concepts.
 - Lift Air Inlet -- The analysis supporting the lift inlet design is **base** on existing aerodynamic flow concepts. The materials and the shaping of the turning vanes are well within the current technology of the marine/aircraft industry.
 - Ride Control Valves -- The ride control valves are a type similar to that used successfully in the 100A program. Proven

- (U) off-the-shelf type components are used throughout the system.

 To further improve reliability, the mechanism is a simple straight-forward linkage design similar to aircraft linkage systems that are presently in use.
 - Advanced Planing Seals -- The success of the advanced planing seals in the most recent model tests lends a high degree of confidence in the design. The analysis of loads for the full-scale configuration, along with the design objectives and materials selections, indicate that all considerations are well within the present state-of-the-art.

(U) 2.3.6 OUTFITTING AND FURNISHINGS

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- (U) 2.3.6.1 Key Features of 0 & F System -- Outfit and furnishings (O&F is composed of a number of subsystems whose functional requirements include providing (1) habitable living and functional working spaces for the ship's crew, (2) safety features and fittings such as rails and lifelines, (3) ease of access to the working and living spaces, (4) protection against abrasion or galvanic corrosion for the hull structure (5) insulation to provide passive thermal, fire and acoustic protection and (6) storage and service spaces as required for the ship and its crew to perform their mission., All O&F subsystems conform to General Specifications for Ships of the U.S. Navy and OPNAVINST 9330.7A (proposed).
- (U) 2.3.6.1.1 Habitability -- Crew living spaces are compartmented with a maximum of 12 men to a compartment. CPO living spaces are compartmented with a maximum of 5 men to a compartment. Officers staterooms are double occupancy except that the Commanding Officer and Executive Officer each have single, separate staterooms.
- (U) **Messing** areas are located within a convenient distance of respective crew living spaces. Cross-traffic has been avoided. The galley is centrally located to serve the crew from one side and the CPO and commissioned officers from the other, again, eliminating cross-traffic.
- (U) Recreation areas are also located within a convenient distance of the respective crew living spaces. The habitability spaces are all located on the second deck and the watch stations are readily accessible for all hands.
- (U) 2.3.6.1.2 Stowage -- Dry provisions, chill storage and freeze storage are located next to the galley. The vertical conveyor is located within a few steps of the galley and each storage area. Supply Department storerooms and spare parts storerooms were located in areas of the ship convenient to users (e.g., repair shops).

- (U) Deck gear lockers are located near each mooring and towing station.

 This provides convenience for stowing deck gear and facilitates keeping the decks clear at all times.
- (U) 2.3.6.2 Passive Fire Protection -- A fire protection system is necessary as an element of damage control and must incorporate within the system both active and passive means. The **active** fire protection system is described in **2.3.5.** The passive fire protection system is designed to protect the primary structure until the active system is brought into play.
- (U) For the design of the fire protection system, the ship spaces were grouped into two major classifications: Group 1, liquid fuel fire hazard spaces; and Group 2, solid combustible fire hazard spaces. In addition to fire protection for these spaces, passive fire protection is provided for the torpedo and small arms magazines.
- (U) 2.3.6.2.1 Group 1 Liquid Fuel Fire Hazard Spaces -- Group 1 consists of all engine rooms, auxiliary machinery spaces, gas turbine generator rooms and the helicopter hangar,
- (U) Passive fire protection for bulkheads and overhead structures for all Group 1 spaces are provided by a ceramic fibrous felt sandwich panel.
- of one inch (254 mm) thick refractory fiber felt of four (4) 1b_f/ft³ (191.46 N/m³) density (Carborundum Fiberfrax felt or equivalent) between 0.010 inch (.25 mm) corrosion resistant steel (CRES) half-hard front face sheet and 0.012 inch (.30 mm) aluminum, marine grade, back face sheet.

 Ceramic tubular spacers 0.5 inch (12.7 mm) outer diameter x 0.156 inch (3.97 mm) inner diameter with #6 CRES screws and nuts are employed on a lo-inch (254 mm) grid pattern to maintain the panel thickness and hold the face sheets together.

- (U) Close-out members of the panel are 0,010 inch (.25 mm)

 CRES half-hard channels with 1/2 inch (12.7 mm) flanges seam half-hard channels with 1/2 inch (12.7 mm) flanges seam welded to the front face sheet and riveted to the back face sheet.
- Panel Attachment The panels are attached to the structure by screw attachment with #6 CRES screws to 0.06 inch x 0.5 inch x 1.0 inch (1.52 mm x 12.7 mm x 25.4 mm) aluminum rectangular tubing "furring strips", The furring strips are attached to the structure by adhesive honding with an adhesive modified with a fire retardant. The panels are spaces from the primary structure with a 1/2 inch (12.7 mm) air gap.
- sealed from vapor penetration as well as heat penetration by sandwiching the panel ends between two strips of refractory fiber felt which are compressed between a 0.060 inch (1.52 mm) aluminum strip at the back of the joint and a 0.030 inch (.76 mm) CRES strip at the front or fire threat side of the joint. Corner joints are similarly sealed with 0.060 inch (1.52 mm) aluminum angles and 0.030 inch (.76 mm) CRES angles which are used as corner trim.

 Wicking would be prevented by inserting the paneis in aluminum channels which are adhesive bonded to the deck with a fire retarded adhesive. A silicone sealant would then be used to seal the panel in the channel.
- (U) Decks -- The decks in Group 1 areas are protected with a 0.25 inch (6.35 mm) thick ceramic fiber moist felt insulation (Refractory Products Company WRP-X-AQ or equivalent). It is a moldable fibrous ceramic felt in an inorganic colloidal silica binder and has a density of 15 lb/ft (717.99 N/m³). The felt is packed in plastic bags during

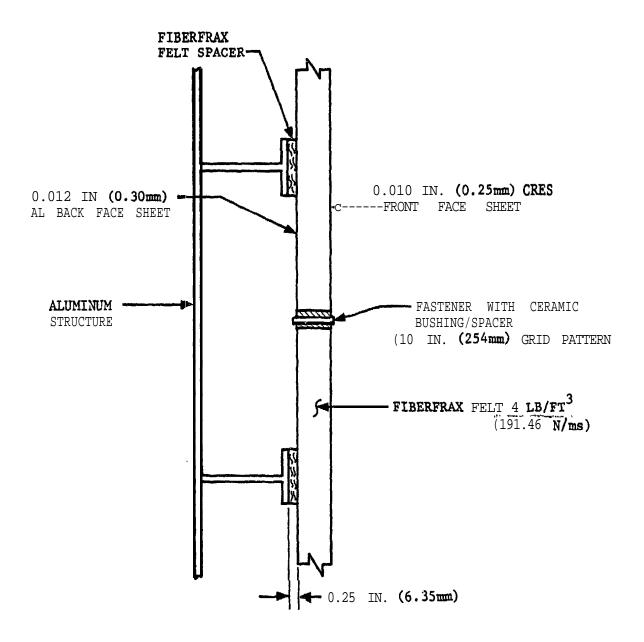
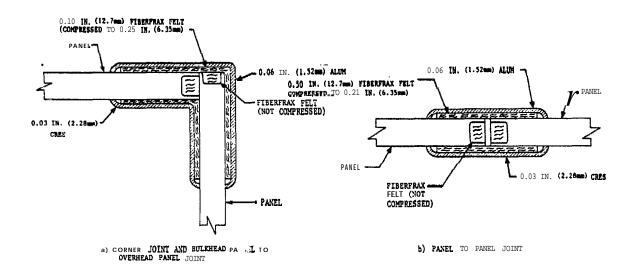
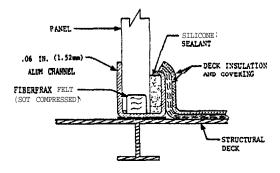


Figure 2.3.6-1 (U): Insulation Panel Design (U)



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c) BULKHEAD PANEL TO DECK JOINT

Figure $\bf 2.3.6-2$ (U): Panel Joints Designed to Prevent Vapor Leakage In a $\bf JP-5$ Fuel Fire (U)

- (U) shipment and storage to prevent drying. After adhesive bonding the felt to the deck, it is allowed to air dry and harden. The felt is bonded to the deck with an air-setting ceramic cement (Carborundum QF-180 or equivalent), which has a layer thickness of 0,010 inch (.25 mm). After air drying, the felt is faced with fiberglass cloth impregnated with a fire retarded epoxy resin. The cloth is an 1800 plain weave with a weight of 10 oz/yd² (3.33 N/m²). The epoxy resin is room temperature curing (Shell Epon 934 modified with fire retardant agents or equivalent).
- (U) Stanchions, Penetrations and Ladders Stanchions are protected by wrapping with 0.750 inch (19 mm) thickness of the moldable fiber moist felt insulation. The moist felt is bonded to the stanchion with ceramic cement. The moist felt is overlapped 1.5 inch (38.1 mm) to prevent a direct path to the protected member. All penetrations are sealed to prevent passage of vapors. Where the penetrating member is exposed to a fire hazard, it would be protected from structural collapse with moist felt insulation and/or intumescent paint. Ladders would be fabricated from corrosion resistant steel.
- (U) 2.3.6.2.2 Group 2 -- Solid Combustible Fire Hazard Spaces -- Group 2 consists of all electro c spaces, living spaces and command centers.
 - Passive fire protection for bulkhead and overhead structure for all Group 2 spaces is provided by a refractory fiber felt sandwich panel similar to the panels used for Group 1 spaces but with a panel thickness of 0.5 inch (12.7 mm).
- (U) The decks are protected with deck covering underlay material and tile or carpeting in these spaces, Stanchions, penetrations and ladders would be treated as described for Group 1.

- (u) 2.3.6.2.3 Magazines -- The passive protection for the magazines is one inch (25.4 mm) thick lightweight glass thermal insulation on the interior surfaces of the compartments and 0.5 inch (12.2 mm) thick fire protection panels on the exterior surfaces of the compartments.
- (U) 2.3.6.3 Estimated Percentage Weight Breakdown -- Table 2.3.6-1 shows the estimated weight percentage of the major components of the O&F System.

Table 2.3.6-1 (U). Estimated Weight Percentage of Major Components of the ${\bf O\&F}$ System (U)

SUBSYSTEM	ò	OF	SYSTEM
Ship Fittings		2.	. 6
Hull Compartmentation		11.	. 5
Presserve and Coverings		18.	6
Hull Insulation		42.	9
Furnishings		24.	. 0
Miscellaneous		0	. 4

- (U) 2.3.6.4 **O&F** Arrangement Drawings -- Arrangements of **O&F** subsystems are shown in the drawings contained in Appendix B, Subsection B.l. The Hull Insulation, Sheathing and Deck Covering System for the near term SES is shown on drawings contained in Appendix B, Subsection **B.6.**
- (U) 2.3.6.5 Outfit and Furnishings Risk Assessment -- The fittings, furnishings, coatings, and outfit items used on the ANVCE near term SES possess proven shipboard capability, and are not peculiar to the SES. Passive fire protection system concepts have been proven by an extensive test program. Consequently, the risk involved is considered minimal and is no greater than that of the outfit and furnishings subsystems of conventional Navy Surface Ships.

- (U) 2.3.7 COMBAT SYSTEM -- The combat systems of the ANVCE Near Term Point Design SES consist of systems that provide a capability in demonstrating the military value of an SES that performs anti-submarine warfare (ASW), anti-air warfare (AAW) and surface warefare (SUW) naval missions. These equipments are listed in Appendix C which contain6 weight, volume, geometry, and service requirements for each item,
- (U) The combat system6 comprise subsystem6 for underwater, air and surface surveillance. The subsystems consist of surface and air search **radars**, passive ESM systems, towed and dipping sonar devices, and dispensed **sono-** buoys. Target identification and classification is accomplished by an IFF system.
- (U) Fire control systems are provided for surface-to-&r, surface-to-surface, and underwater weapons. Surface-to-air and point defense weapons consist of vertially launched Standard missiles, and dual Close-In-Weapon System (CIWS) installations (space and weight). The anti-shipping weapons are Harpoon and MK48 torpedo. The ASV self-defense and offensive weapon is the MK46/1 torpedo. Weapons and sonobuoy delivery for offensive ASW operations is accomplished by helicopter (SH-3H). Space reservations have been made for future applications of mini-RPV's for SUW target localization and weapon terminal guidance, as well a6 for relaying sonobuoy field telemetry data.
- (U) 2.3.7.1 Surveillance Air surveillance is provided by the air search radar AN/APS-125 and DPEWS AN/SLQ-31(V2) or AN/SLQ-32(V2) systems.

 A backup capability is provided by the MK92/3FCS.
- (U) Surface Surveillance is accomplished by the surface search radar AN/SPS-55, with a backup capability furnished by the collision avoidance and navigation system.

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- (U) Underwater surveillance is provided by the TACTASS AN/SQR-19 and AN/AQS-13D sonars. DIFAR An/SSQ-53 and DICASS AN/SQ-62 sonobuoys are dispensed by helicopter (SH-3H). Sonobuoy data link is via UHF telemetry receiving sets AN/SKR-3A.
- (U) 2.3.7.2 Armament -- Armament includes surface-to-air missiles, surface-to--surface missiles, missile launching systems and air drop and over-the-side launched torpedoes. Small arms and pyrotechnic devices, hi-rate gun munitions handling, and stowage facilities are also provided. Armament missile systems are controlled by the fire control system elements of Command and Surveillance. Torpedoes are controlled by underwater fire control elements.
- (U) Armament provides the ship with weapons and a means for delivery of those weapons to counter air, surface, and subsurface threats with provisions for the following:
 - o Eight environmentally sealed and protected Harpoon missiles carried in four lightweight launchers, each holding two cannister launched missiles.
 - Eight environmentally sealed and protected SM-1 missiles carried in four lightweight vertical launchers, each holding two cannister launched missiles.
 - o Two MK-32 triple torpedo tubes for the over-the-side launch of MK-46 torpedoes.
 - o Miscellaneous ordnance and small arms.
 - o Space and weight reservations for two $MK-^{5}$ MOD V torpedo tubes for ship launch of two MK-48 torpedoes.
 - ${\bf o}$ Space and weight reservations for two MK-15 MOD 0 Close-In Weapon Systems (CIWS)

- (U) 2.3.7.3 List of Combat System Equipment -- The list of combat system equipment (non-variable load items) is contained in Appendix ${\bf C}$. The list itemizes equipment physical characteristics, weight, and ship services requirements.
- (U) 2.3.7.4 Combat System and Military Payload Weights -- Table 2.3.7-1 presents the weights of major components within the combat system and includes variable load elements. Table 2.3.7-2 shows military payload weights (C³ + Combat System) In accordance with ANVCE WP-002 definitions.

Table 2.3.7-1 (U) COMBAT SYSTEM WEIGHTS (U)

		WEIGHT			
SWBS	TITLE	LT	kN	% OF TOTAL	
450 & 460	Surveillance Sys	20.4	203.26	23.9	
470	ECM	1.3	12.95	1.5	
480	Fire Control	1-2.2	121.56	14.3	
711	Guns	11.4	113.59	13.3	
721	Launch Devices	10.6	105.62	12.4	
751	Torpedo Tubes	27.5	274.01	32.2	
761	Small Arms	0.4	3.99	.5	
782	Aircraft Weapon Handling	0.5	4.98	.6	
783	Aircraft Weapons Stowage	1,1	10,96	1.3	
COMBAT SYS	TEM TOTAL	85.4	850.92	100.0	

Table 2.3.7-2 (U) MILITARY PAYLOAD WEIGHTS (U)

			WEIGHT		
SWBS	TITLE:	LT	kn _	% TOTAL	
400	C&S Less Nav & Internal	49.5	493.82	2:	
	Communications				
700	Armament	52.3	520.92	29.1	
F21-27	Ordnance	47.7	475.08	26.5	
F42	Helicopter JP-5	30.5	303.70	16.9	
MILITARY	PAYLOAD TOTAL	180.00	1793.52	100.00	

- (U) 2.3.7.5 Combat System General Arrangements -- The arrangements of the near term SES Combat Systems are shown in drawings contained in Appendix B, Section B.1 and B.7. The coverage of the weapons and sensors are shown on the figures contained in Appendix B, Section B.2. The armament system functional block diagram is contained in Appendix B, Section B.7.
- (U) 2.3.7.6 Combat System Risk Assessment -- The specified combat weapons and sensors suite is entirely Government-defined and has the minimal risk associated wit; well-funded development and selections from off-the-shelf equipment items.

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(U) 2.4 SURVIVABILITY AND VULNERABILITY

- (U) 2.4.1 SIGNATURE CHARACTERISTICS
- (U) 2.4.1.1 Radar Cross Section (.3-18GH_Z) Radar cross section not provided. Data not available.
- (U) 2.4.1.2 Microwave Signature **Microwave** signature not provided. Data not available
- (U) 2.4.1.3 Infrared Signature -- Infrared signature not provided. Data not available.
- (U) 2.4.1.4 Visibility -- Visibility not provided. Data not available.
- (C) 2.4.1.5 Acoustic Signature -- The airborne radiated noise signature comes primarily from the engine combustion air inlets, propulsion exhausts and lift fan air inlets. The total signature at a distance of one (1) metre would be approximately 100 dB re 20µPa in the 250 Hz band.

 Including spreading and absorption, a 45 dB sound pressure level in the 250 Hz band will be reached at approximately 500 metres.
- (U) Target strength, \mathbf{dB} at a one (1) yard (0.9144 m) reference distance $(\mathbf{dB} \text{ re 1 yd.})$, is shown in Table 2.4.1-1 and the underwater radiated noise signature $(\mathbf{dB} \text{ re 1} \mu \mathbf{Pa})$ is shown in Table 2.4.1-2.
- (C) The near term point design SES probably has a distinctive line spectra at approximately 500 Hz. This relates to the blade passage frequency of the lift fans. The acoustic signature will probably show directionality abeam and abaft the waterjets.
- (C)Airborne radiated noise signature may be reduced by treating the combustion inlet, propulsion exhaust, and fan inlets with additional splitters.

 Underwater radiated noise signature may be reduced by suitably treating

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TABLE 2.4.1-1 (C): Estimated Target Strength (dB) (U)

SHIP ASPECT CONDITION	ВЕАМ	STERN/BOW
On Cushion	15	2
Off Cushion	20	10

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- (C) the engine and fan mountings. This will reduce distinct spectral lines, but will virtually do nothing to reduce the overall level in any given 1/3 octave band, since most of the energy in any band results from the impingement of the waterjet stream on the ocean's surface.
- (U) 2.4.2 . HARDNESS -- Hardness features are not furnished. The near term SES is not designed to warship hardness standards nor does it feature armor protection.

TABLE 2.4.1-2 (C): Estimated Underwater Radiated Noise Signature (dB re 1μ Pa) (U)

	SHIP SPEED				
INTENSITY .	10 knots (5.14 m/s)	50 knots (25.72 m/s)	80 knots (41.14 m/s)	120 knots (61.74 m/s)	
Itntensity of Highest Line (0-100 Hz)	180	174	168	160	
Intensity of Highest Line (≥ 100 Hz)	170	164	158	1.50	
Intensity of 1/3 Octave Band 2kHz	180	175	170	160	



3 / LOGISTIC CONSIDERATIONS

- (U) The principal logistic elements contributing to the near term SES design baseline are maintenance planning, supply support, ship manning, training, technical publications, and support system requirements, Interdependently and interacting with other requirements, these elements affect ship sizing, light ship weight, variable load weight, and inherent design capabilities for performing selected missions. The overall approach to logistics will support the near term SES design, construction and fleet use.
- (U) The support system provides the logistic support resources required to maintain the ship in an operational readiness condition capable of meeting the availability requirement of the missions. The logistic support resources include personnel and training, initial and back up inventory of spares and repair parts, industrial support facilities (intermediate and depot support levels) and common/peculiar support equipment (intermediate and depot repair shops). These logistics elements are displayed in the support system block diagram, Figure 3-1. The support 'system is compatible to the maximum degree possible with U. S. Navy and other existing logistics support activities.

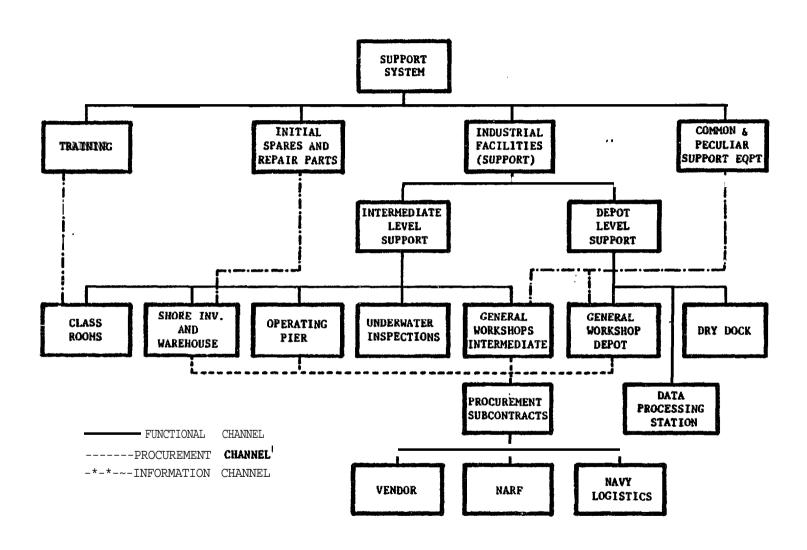


Figure 3-1 (U): Support System Block Diagram (U)

(U) 3.1 RELIABILITY AND AVAILABILITY

Since Rohr Marine, Incorporated, is not conducting the Supportability/
Availability analyses (ANVCE WP-008) (1), availability block diagrams are
not a part of this report; however, subsystem availability predictions
applicable to the near term SES are a part of this report. In addition,
MTBF and MTTR data for major components of SES Subsystems are listed in
Paragraph 3.1.3 along with a utilization factor.

- (U) 3.1.1 SES UTILIZATION (Not Provided)
- (U) 3.1.2 SES SYSTEM AND SUBSYSTEM -- The predicted availability for the near term SES is shown in Table 3.1-1. The predictions are based on a ten-day mission. Availability is defined as the ratio of mission uptime to total at sea time scheduled for the mission. The predicted availability for the near term SES is high due to:
 - High reliability and maintainability of the stay-stiffened planing bow and stern seals (e.g., all seal components can be replaced without dry docking)
 - Redundancy in the Lift and Propulsion Systems
 - High availability of the Electrical, Auxiliary, and Command and Surveillance Systems
 - A Design approach that emphasizes RMA

The second secon

(U) The availability predictions listed in Table 3.1-1 are relative to a mature design. Furthermore, these predictions are for a ship maintained in accordance with the Maintenance Concept outlined in Paragraph 3.2.

The combat functions have not been considered in computing these predicted availabilities.

⁽¹⁾ This is **the** understanding derived from the 20 September 1976 meeting at PMS-304. Rohr Marine, Inc., was to receive a "questionnaire/list of required data" from NAVSEC. In the interim, the information submitted in this report presents data used in the Rohr Marine, Inc. **RMA** analysis.

Table 3.1-1 (U). SES Availability Prediction for the Near Term SES (U)

SUBSYSTEM	AVAILABILITY PREDICTION
Hull Structures Propulsion Plant' Electric Plant Command and Surveillance Auxiliaries Systems Lift System Overall Ship	0.9990 0.9627 0.9990 0.9828 0.9874 0.9768

- vided in this subsection are for only mission essential equipments. Not listed are equipment in the combat or C3 systems with the exception of those functions required for maneuvering and navigation. The equipment R&M data is listed by subsystem. The following definitions apply to the data lists:
 - EQUIPMENT Major equipment group of function
 - MTBF Mean Time Between Failures, hours
 - MTTR Mean Time to Repair or Restore (the times listed include a 50 percent allowance for conditions at sea)
 - UTIL Utilization Factor. That portion of time that the item is in use during the mission, hours.
 - NR Non-Repairable at sea.

(U) 3.1.3.1 Propulsion Plant R/M Data

EQUIPMENT	MTBF	MTTR	UTIL
Combustion Air Supply Heating (NR)	11,500	5. 0	0.5<
Combustion Air Supply	244, 000	1. 5	1.0
Combustion Air Supply (NR)	81, 300	5. 0	1.0
Gas Turbine	6, 450	3. 0	1.0
Gas Turbine (NR)	10, 500	24. 0	1.0
GTRB Lube Oil Cooler	90, 000	4. 5	1.0
GTRB Luve Oil Filter - Supply	60, 000	4.5	1.0
GTRB Lube Oil Filter - Scavenge	60, 000	4.5	1.0
Flex Coupling (NR)	72, 780	4. 0	1.0
Tongue Meter	10,000	1. 5	1.0
Shafting and Bearings (NR)	11, 600	6. 0	1.0
Thrust Reverser	6, 150	6. 0	0.1 <
Propulsor (NR)	6, 700	8. 0	1.0
Waterjet - Steering	6,150	6. 0	1.0
Exhaust Duct	62, 000	4. 5	1.0
Exhaust Duct (NR)	300, 000	15. 0	1.0
GTRB Cooling Blower	18, 250	2. 25	1.0
Lube Oil Pump - Pressure	21, 800	3. 0	1.0
Att. Lube Oil Pump - Press.	34, 500	4. 5	1.0
Lube Oil Pump - Scavenge	21, 800	3. 0	1.0
Att. Lube Oil Pump - Scavenge	34, 500	4.5	1.0
Lube Oil Filter/Separator	30, 000	4. 5	1.0
Lube Oil Control Manifold	46, 730	3. 0	1.0
Lube Oil Cooler	45, 000	4. 5	1.0
Vacuum Pump	18, 250	2. 25	1.0
Inlet Sensors and Control	5, 000	1. 5	1.0
Inlet Ramp Actuator	6, 100	3. 7	1.0
Inlet - Miscellaneous	91,000	3. 0	1.0
Inlet (NR)	45, 000	15. 0	1.0
Propulsion System - Misc.	10,000	3. 0	1.0
Sensors for System Control	10,000	1. 5	1.0

(U) 3.1.3.2 Electric Plant R/M Data

EQUIPMENT	MTBF	MTTR	UTIL
Engine Air Supply	70,000	1.5	1.0
Gas Turbine Generator 60 Hz	8,330	3.0	.1.0
Gas Turbine Generator 60 Hz (NR)	12,500	8.0	1.0
Gas Turbine Lube Gil Cooler	90,000	4.5	1.0
Exhaust Duct (NR)	26,000	5.8	1.0
60 Hz Switchboard	645,000	1.5	1.0
60 Hz Power Panel	173,580	1.5	1.0
60 Hz Transformer	1,000,000	1.5	1.0
400 Hz Switchboard	645,000	1.5	1.0
400 Hz Power Panel	173,580	1.5	1.0
400 Hz Transformer	1,000,000	1.5	1.0
Cooling Fan	18,250	2.25	1.0
Lighting Vital Spaces (each light)	1,000,000	1.5	1.0
Gas Turbine Generator 400 Hz	10,000	3.c	1.0
Gas Turtine Generator 400 Hz (NR)	15,000	8.0	1.0
28 VDC Rectifier	.36,000	1.5	1.0
28 VDC Distribution Box	28,930	1.5	1.0
28 VDC Power Panel	43,395	1.5	1.0

(U) 3.1.3.3 Navigation and Collision Avoidance System R/M Data

EQUIPMENT	MTBF	MTTR	UTIL
Anti-Clutter Collision Avoidance Radar AN/APS-116	1,100	4.5	1.0
Anti-Clutter Collision Avoidance Radar AN/APS-116 (NR)	10,000	3.0	1.0
Surface Search Radar SPS-55	1,300	2.7	1.0
Surface Search Radar SPS-55 (NR)	12,000	1.8	1.0
Collision Avoidance Computer AN/UYK-20	2,000	0.4	1.0
Navigation Computer AN/UYK-20	2,000	0.4	1.0
Navigation Data Switchboard	2,000	1.5	1.0
SAT-NAV	500	0.75	1.0
OMEGA	1,500	1.5	1.0
Inertial Nav	5,600	1.0	1.0
Gyro (Types I and II)	5,600	1.0	1.0
Depth Sounder AN/UQN-4	5,700	0.75	0.5
Doppler Speed Sensor	1,000	1.5	1.0
Interior Communications	20,000	1.5	1.0
HF Transceiver	. 1,100	0.5	1.0
UHF Transceiver	2,100	0.5	1.0
VHF Transceiver	2,500	9.5	0.1
VHF Antenna (NR)	25,000	2.0	1.0
Transfer Switchboard	16,000	0.7	1.0

(v) 3.1.3.4 c^3 Ship Controls R/M Data

EQUIPMENT	MTBF	MTTR	UTIL
Wheei	5,000	1.5	1.0
Autopilot	1,666	1.0	1.0
Propulsion Power Lever Actuator	5,000	1.0	1.0
Lift Throttle	5,000	1.0	1.0
Lift Control (Ship's Control Console and Propulsion Control Console)	5,000	1.0	1.0
Autopilot Control Display Unit	5,000	1.0	1.0
Navigation - Collision Avoidance Display	4,000	1.0	1.0
Central Processing Unit	2,000	1.0	1.0
Fire Protection Controls	1,000	1.0	1.0
Electric System Control	1,000	1.0	1.0
Fuel Management Control	1,000	1.0	1.0
Auxiliaries Control	1,000	1.0	1.0
Pbwer Supply	90,000	1.5	1.0
PPI Display	4,000	3.0	1.0
PPI Display (NR)	4,000	3.0	1.0
Commanding Officer Communications Console	50,000	1.0	1.0
Ship's · Control Console - Monitoring'	10,000	1.0	1.0
Propulsion Control Console • Monitoring	10,000	1.0	1.0

EQUIPMENT	MTBF	MTTR	UTIL
Air Conditioning Unit	12,650	4.0	0.5
Recirculating Fan	13,880	3.0	1.0
Mixing Box	38,080	3.0	1.0
Supply Fan	18,250	3.0	1.0
Exhaust Fan	18,250	3.0	1.0
Supply Fan - Machinery Space'	18,250	3.0	1.0
Exhaust Fan - Machinery Space	18,250	3.0	1.0
Fire Pump (NR)	10,500	8.0	1.0
Distiller	2,000	4.0	1.0
Distiller (NR)	6,000	9.8	1.0
Pump - Potable Water	15,150	3.0	0.5
Pump - F.W. Transfer	15,150	6.0	1.0
Pump - Coolant, Electronic	15,150	4.5	1.0
Demineralizer	32,860	4.4	1.0
Heat Exchanger	90,000	4.5	1.0
Valve, Temp. Control	11,700	3.0	1.0
Pump, Fuel Transfer	3,760	4.5	0.3
Pump, Fuel Service	3,760	4.5	1.0
Pump, Fuel Trim	3,760	4.5	0.5
F.O. Filter	60,000	4.5	1.0
Manifold, Fuel	10,000	3.0	0.75
Heat Exchanger .	90,000	4.5	1.0
Mass Flow Multiplier	50,000	4.5	0.1
Valve, Motor Operated	20,000	3.0	0.05
Regulator, Air	46,730	3.0	0.05
Air, Receiver	16,700	4.5	1.0
Hydraulic Pump - Att.	15,000	'3.0	1.0
Hydraulic Pump - Motor Drive	10,500	3.0	1.0
Filter, Hydraulic	60,000	4.5	1.0
Cooler, Hydraulic	90,000	4.5	1.0
Control, Hydraulic System	50,000	4.5	1.0

(U) 3.1.3.5 Auxiliaries R/M Data (Continued)

<u>EQUIPMENT</u>	MTBF	MTTR	UTIL
Regulator, Hydraulic Resevoir	23, 365	3. 0	1.0
Anchor Windlass (NR)	3, 350	5. 0	0.05
Capstan Mooring (NR)	3, 350	5.0	0. 05
Fuel, Probe-Receiver	50, 000	3. 0	0.10
Pollution Control System	10, 000	3. 0	1.0
Hangar Door - Actuation	10,000	4. 5	0.05
Hangar Door - Manual	10, 000	4. 5	0.05
Sensors for System Control	10,000	4. 5	1.0
Auxiliaries - Misc.	5,000	3. 0	1.0
Fire Detection	2.000	1. 5	1.0

(U) 3.1.3.6 Lift System R/M Data

EQUIPMENT_	MTBF	MTTR	UTIL
Gas Turbine	6,450	3.0	1.0
Gas Turbine (NR)	10,500	24.0	1.0
GTRB Cooling Blower	18,250	2.25	1.0
GTRB Lube Oil Cooler	90,000	4.5	1.0
GTRB Lube Oil Filter - Supply	60,000	4.5	1.0
GTRB Lube Oil Filter - Scavenge	60,000	4.5	1.0
Torsionmeter	10,000	1.5	1.0
Exhaust Duct (NE)	26,000	5.8	1.0
Reduction Gear (NR)	188,000	8.0	1.0
Pump - Lube Oil Pressure	21,800	3.0	1.0
Pump - Lube Oil Pressure - Attached	34,500	4.5	1.0
Pump - Lube Oil Scavenge	21,800	3.0	1.0
Pump - Lube Oil Scavenge - Attached	34,500	4.5	1.0
Filter Separator - Lube 0il	30,000	4.5	1.0
Control Manifold - Lube Oil	46,730	3.0	1.0
Lube Oil Cooler	90,000	4.5	1.0
Vacuum. Pump	18,250	2.25	1.0
Shafting & Bearings (NR)	41,000	6.0	1.0
Demister	23,200	1.5	1.0
Lift Fan	28,190	3.0	1.0
Lift Fan (NR)	8,000	18.0	1.0
Shut Off Control Valve	5,900	4.5	1.0
Control - Ride Control Valves	10,000	3.0	0.1 4
Ride Control Valve	5,900	4.5	0.1 ∠
Bow Seal (NR)	6,965.	4.3	1.0
Stern Seal (NR)	6,965	4.7	1.0
Bow Seal Retract	9,120	4.5	0.054
Control - Bow Seal Ret.	5,000	1.5	0.05 4
Stern Seal Retract	9,120	4.5	0.05
Control - Stern Seal Ret.	5,000	1.5	0.05 AI
Misc. Valves 6 Piping	10,000	1.5	1.0
Sensors for System Control	10,000	1.5	1.0
Transfer Valves	5,900	4.5	1.0

(U) 3.2 MAINTENANCE CONCEPTS

In consonance with the 3KSES TLR, the maintenance concept for meeting the objectives and availability goal is to: (1) perform the preventive/corrective maintenance on critical equipment onboard; (2) accomplish the emergency repair of non-critical equipments with helo provided (VERTREP) augmentation from the intermediate level support resources; and (3) defer/schedule all non-essenti: equipments/components maintenance for in-port availabilities. For design purposes, particular emphasis was to be given to: (1) maximization of the use Of existing and projected Navy equipments to permit use of standard maintenance procedures and supply support, (2) use of performance/condition monitoring for detecting incipient failures for critical equipments, and (3) provisions for equipment accessibility to support a component/module replacement strategy. The replacement strategy includes scheduled replacement, replacement on condition, and replacement at failure depending on the subsystem/equipment criticality. Therefore, the maintenance concept in support of near term SES availability and mission is based on a number of objectives and constraints.

- (U) The maintenance objectives of the near term SES are:
 - Support the SES in the achievement of assigned test and demonstration missions while assuring safety of ship and personnel, and meeting availability requirements,
 - Use the inherent maintenance capability of operator personnel.
 - o Minimize shipboard maintenance manning.
 - o Minimize "at sea" repair to the vital and critical equipments and components.
 - o Minimize ship carried weight of logistic resources,
 - o Use the most cost-effective distribution of effort between shipboard and off-ship maintenance.
 - O Use **helicopte** service (V'ERTREP) to provide logistics resources not carried on board, i.e., personnel skills, special tools and test equipment, spares, etc.

- Use the replace and restore concept to the maximum, vice piecepart repair.
- Provide adequate accessibility for servicing to minimize secondary removals/replacements.
- Maximize the use of existing and projected Navy equipments to permit use of standard maintenance procedures and supply support. Navy rotatable pool stocks will be used as applicable.
- Achieve incremental subsystem overhaul by maintenance actions and scheduled replacement of subsystems accessories and related auxiliaries consistent with the major item replacement cycle.
- (U) The maintenance constraints placed on the near term SES are:
 - Accomplish both preventive and corrective maintenance actions, to the maximum extent possible, while in port.
 - o In view of the perennial need to minimize shin weight, a single item weight limitation of 160 Lbs (711.72 N), will relegate a few "potentially repairable at sea" maintenance tasks, on critical equipments, to a non-repairable at sea category.
 - $_{
 m O}$ At sea maintenance shall be limited to that required, consistent with ship speeds and sea states.

- (U) 3.2.1 INTERMEDIATE LEVEL SUPPORT -- Intermediate Maintenance
 Activities (IMA's) will accomplish PM not within the capacity of the ship's
 crew. The IMA will provide condition monitoring services not otherwise
 within the capabilities of the monitoring equipment aboard ship, during
 upkeep or Maintenance Availabilities'of the ship. Intermediate-level
 maintenance for the near term SES will include support from shore based and afloat
 Intermediate Maintenance Activities.
- (U) 3.2.1.1 Shore Based Intermediate Facilities -- The shore based intermediate level support will provide the following types of facilities to meet the operating, maintenance, training and supply support requirement of the near term SES:
 - Operating pier -- will provide for safe and efficient mooring of the ship for servicing, maintenance and/or testing. The mooring provisions will be designed to be specifically compatible with the ship. An unobstructed access and a sufficient depth of water are required. Bits and chocks will provide the capability of withstanding wind loading up to 100 knots (51.44 m/s). Compatible dockside fittings for fueling/defueling, fresh water, compressed air, 60 and 400 Hz electrical power and telephone connections to the SES will be provided. Crane services for loading/unloading equipments/components and materials, gangways and/or ramps for personnel access/ egress will be provided. Emptying the collecting/holding tanks (CRT) of sewage and other liquid wastes will be accomplished by appropriate sludge barges and receiving vehicles. Solid trash (compaction) disposal will be provided.
 - Intermediate level maintenance ships/capability such as:
 - (1) Ship fitters/welding/pipefitters
 - (2) Mechanical pumps/auxiliary machinery
 - (3) Electrical generators/switchboards

- (4) Electronics/test instrumentation
- (5) Underwater inspection/repair support shop including photographic service
- (6) Operational computer program maintenance
- Training classroom8
- Supply warehousing and storerooms
- The administrative space, personnel, furnishing and equipments necessary to coordinate the logistics resource support, including the planning and scheduling of the resupply services to support the SES while at sea. Support shall be provided for logistics resources not carried on board (i.e., personnel skills, special tools and test equipment, spares, etc.).
- (U) 3.2.1.2 Afloat Intermediate Facilities -- The afloat intermediate level activity will provide the following types of maintenance shop/capability support:
 - Ship fitters/welding/pipefitters
 - Mechanical pumps/auxiliary machinery
 - Electrical generators/switchboards
 - Electronics/test instrumentation •
 - Underwater inspection/repair support shop, including photographic service.
- (U) 3.2.2 DEPOT LEVEL SUPPORT -- Depot-level support maintenance of the near term SES will include the following:
 - Preserving the underwater body, and maintaining sea-connected tanks, valves, pipes, and fittings.

- o Performing repairs requiring heavy lift capability and special tools and test equipment (examples: bow/stern seals, radar antennas, gas turbines, waterjet propulsor, electrical generators).
- Removing, installing, and testing certain equipments identified as stock rotating spare items (example: main propulsion and lift gas turbines),
- Stocking and repairing designated stock rotating spares items at selected depot maintenance activities.
- (U) **The** depot level support will provide a dry dock and the necessary work shops for systems/equipments for overhaul and/or repair beyond the capability of the intermediate maintenance activity. The depot level support will provide general workshop and **drydock** facilities and services.
- (U) 3.2.2.1 General Workshops -- Depot level maintenance shops/capability will provide maintenance for the waterjet propulsor, lift fans, seals, structural/welding, computers, consoles and related electronics, and gas turbine generators. A Naval Rework Facility is required for gas turbine maintenance. Overhaul points for communications, sensors, computers, displays, and related electronics must be designated. A facility, either Government or Contractor, is required for the maintenance of operational computer programs.
- (U) 3.2.2.2 Drydock -- A safe and efficient facility capable of dry-docking the ship. Cranes, temporary pdwer, compressed air, fresh water, salt water, firemain, sewage collection and disposal shall be available at the dock site.

(U) 3.3 OVERHAUL CONCEPT

Regular overhauls, as now understood, are to be eliminated by intensive use of the upkeep periods as maintenance availabilities. The near term SES will employ the concept of progressive overhaul. Equipment replacement and alteration will be accomplished progressively during relatively frequent maintenance availability periods of short duration. Dry-docking will be accomplished, primarily to provide for major emergency repairs and/or ship alterations. The ship system will be designed to be capable of incremental overhaul of its subsystems and subsystem accessories and related auxiliaries. Operational usage and schedule replacement will be consistent with the major item replacement schedule.

(n) 3	.3.1	SCHEDULING (Not provided)
(U)	3.3.2	PIPELINE REQUIREMENTS (Not provided)
(U)	3.3.3	SHIPYARD OVERHAUL FACILITIES (Not used; see Section 3.2.2)
(U)	3.3.4	LAND-BASED TEST FACILITIES (Not provided)

(U) 3.3.5 MAINTENANCE **PROGRAM** INTERFACE (Not provided)

(U) 3.4 SUPPLY SUPPORT CONCEPT

The sizing of near term SES storerooms, commissary system and other supply spaces are constrained by design requirements and indirectly by the maintenance concept and the personnel requirements. Within the permissible volumetric and weight limits, the near term SES design provides the necessary supply support capability. The design supports the requirements for 10 and 15-day missions in accordance with LSES TLR mission profiles. A salient design consideration is the frequency of underway replenishment, including helicopter (VERTREP) delivery of required logistics resources, and the requirement for underway refueling.

- (U) The supply support concept provides material support for the assigned miss ions. The support includes initial outfitting of provisions, medical supplies and spares and repair parts as well as replenishment,
- (U) The supply concept provides for the fitting out of the SES.

 It provides for the material to be assembled in a shore warehouse prior to loading on board. The 1 pading of spares and repair parts and equipage will be kept to a minimum consistent with space and weight constraints, necessary to support availability and mission requirements. Tailored loads of support for specific missions will be utilized,
- (U) Replenishment for all repaitables, consumables, and spares will be provided through underway Replenishment Groups at sea and through normal resupply methods in port. Supply requirements, for at sea emergencies, will be accomplished by the utilization of helicopter deliveries. Maximum utilization of in-port delivery for all repairables, consumables, and spares, will be a program objective.

(U)	3.4.1	MODIFIC.	ATIONS provided	TO MOBILE	LOGISTIC	SUPPORT	FORCE SHIPS	
(U)	3.4.2	UNIQUE 3.2.2)		FACILITIES	(Not used	l; see	Sections 3.2.1	and
(U) 3	3.4.3	UNIQUE	REPLE	NISHMENT	TECHNIQUES	(Not	provided).	
(U)	3.4.4	UNIQUE	SUPPLY	SUPPORT	PROCEDURES	(Not	provided)	

(U) 3.5 HUMAN ENGINEERING

The human engineering effort on the near term SES encompassed three basic areas: (1) design of major work stations; (2) design of maintenance access; and (3) evaluation of the design in terms meeting habitability criteria requirements.

- (U) 3.5.1 DESIGN OF WORK STATIONS The pilot house, chart room, Central Operating Station (COS) and CIC were analyzed with regard to man-machine interfaces and functional adjacencies between operating personnel. The pilot house and COS are equipped with integrated display-control consoles to allow minimum manning at these stations. These work stations have also been designed for seated operations to ensure a high level of operability during operation in all sea states and at all ship speeds.
- (U) 3.5.2 MAINTENANCE ACCESS -- All ship spaces and equipments within these spaces were analyzed to determine if sufficient space had been allocated for both corrective and preventative maintenance. The analysis revealed that all subsystems and equipments can be installed, serviced and removed with a minimum of effort. Particular attention was given to the waterjet propulsors, propulsion gas turbines, lift gas turbines and the lift fans with regard to maintenance removal requirements.
- (U) 3.5.3 HABITABILITY -- The point design was compared with the habitability criteria stated in OPNAVINST 9330.78. The space allocations for crew living areas meet or exceed those requirements for berthing, sanitation spaces, recreation spaces, galley and messing. The crew living areas occupy a single deck to minimize the need for crew members to move about the ship whether on- or off-duty. Therefore, these design features are provided: (1) all off-duty crew spaces are on the same level to minimize the use of ladders; (2) within the constraints of crew/CPO officer off-duty separation, the best possible adjacency between mess rooms, recreation rooms, and berthing

- (U) spaces, has been provided; (3) sanitary spaces are located within the **living** spaces,
- (U) Noise/thermal separation between living and machinery spaces ensures a more comfortable off-duty crew environment. Off-duty spaces are separated from machinery spaces/engine rooms by passageways, storerooms or other effective noise and thermal barriers. This lessens ship weight allocation for insulation material while improving habitability.
- (U) The ride quality to be expected by the crew at higher sea states, in combination with higher ship speeds, was examined. Large amplitude vertical accelerations can exceed human tolerance levels to a point where human performance can be affected. To ensure that human performance is not degraded, Human Factors developed ride criteria limits that were used to verify the adequacy of the ride control system in limiting vertical accelerations within the operational sea state and speed envelope.

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(U) 3.6 SYSTEM SAFETY

The primary intent of safety program requirements is the elimination or control of hazards inherent in the design and in the operation of the SES in its environment. Particular attention was given to safe ship survival in any singular hardware malfunction of the lift, propulsion, steering, reversing, or **sidehull** damage from foreign objects, regardless of sea state, speed, displacement, or maneuver at the time of the casualty, Safe survival of personnel functional capabilities and preclusion from injury during and operational or maintenance phase is eugally important.

- (U) 3.6.1 PRELIMINARY HAZARDS ANALYSIS -- The entire ship design was reviewed for gross hazard failure modes. Design characteristics singular to the SES were given emphasis. A major effort was placed on hull arrangement and structure, lift system, seals, waterjet pump, fire protection integration, operation and maintenance hazards, selected control aspects, and outfit and furnishings.
- (U) In most instances, operational ambient failure conditions were defined to aid the designer in eliminating a hazard, rather than relying on preventative manned reaction and cautionary training. Where ship critical or catastrophic conditions were predictable, equipment failure modes and personnel hazards were examined as possible sequences or parallel events. Closely examined were seals, the lift system machinery, fire detection, and extinguishment integration, selection of passive fire protection materials, and personnel protective gear for use while underway,
- (U) 3.6.2 SAFETY TRADE-OFF STUDIES -- System Safety Personnel investigated or supported all safety critical trade-off studies. Selected subjects of design safety trade-offs, applicable to the 1980 (near term) point design, follow.
- (U) 3.6.3 FIRE PROTECTION -- System Safety actively participated in the design of the fire protection system. Safety required that a

- (U) bulkhead be capable of sustaining a fuel fire for 30 minutes on the insulated side, with the non-insulated side not to exceed 400°F (204°C). This is achieved by a passive fire protection design with a stainless steel face shield backed-up by varying thicknesses of fiberfrax.
- (U) Auto/remote/local Halon 1301 fixed flooding with interconnect capability, and HI-X foam generators for back-up are used in fire suppression. Critical electronic spaces are similarly protected except without Hi-X foam. All other ship spaces are protected with standard Navy fire extinguishing systems. For example, the Helicopter hangar and flight deck are protected by AFFF foam sprinkling and hose reels and the torpedo room and missile stations with seawater sprinkling,
- (U) 3.6.4 LIFT SYSTEM AND SEALS -- The ride control and lift system design and development has been under continuous surveillance for its safety impact. Maintenance access and interlocking alarms into the fan compartments have been provided, The near term SES incorporates guide vanes on 0.5 Ft. (0.15m) centers at the fan inlet that preclude a personnel falling hazard,
- (U) 3.6.5 RIDE CONTROL -- Safety features for maintaining control of body motions underway include: safety/shoulder harness restraining devices for seated positions, arm restraints for console operators, padded barriers and railings for walking and standing functions, non-skid deck surfaces, and head protective gear. Non-critical maintenance activities are minimal while underway in high sea states to minimize personnel injury from random vertical accelerations.
- (U) 3.6.6 HELICOPTER CONTROL STATION -- The helicopter control station was located at the port side of the Helicopter Hangar at the 01 level. This provides greater visibility and safety in the event of a helicopter crash or platform fire. In the event of a hangar fire, emergency egress to the 01 level is readily available.

- (U) 3.6.7 DECK EDGE CURVATURE -- The deck edge curvature provided for aerodynamic purposes has a two foot (0.61m) radius. This maximizes deck hand safety compared to larger radii, particularly during installation or removal of portable life lines and stanchions from the deck edge perimeter and the replenishment of stores and ordnance while underway.
- (U) 3.6.8 ANTENNA LOCATION -- The AN/APA-171 antenna is located on the main mast, approximately 30 feet (9.14m) above the weather deck, This lowers the personnel RADHAZ level on the helicopter platform and avoids radiating energy into the Pilot House, compared to possible alternate locations farther aft and closer to the 01 deck.

4 / TECHNICAL RISK ASSESSMENT

- (U) One of the design objectives has been to incorporate standard practices and parts to the maximum degree. Equipments developed and available from existing Government inventory have been preferred over new equipments to be developed. Qualification by extension of existing designs has been used to the extent practicable in lieu of development of new items.
- (U) The ship configuration is a viable concept and can be developed with minor or acceptable levels of risk. Furthermore, the near term ANVCE SES has been configured to accept further design alternatives which may enhance ship performance, utility and reliability. The overall technical risk is assessed as follows:
- (U) Hull Structure -- The hull is designed to realistic worst case loading conditions forecast to occur within the ship lifetime. The materials are commercially produced aluminum alloys which have been utilized in existing Navy ships, such as the PHM and SES-100B. The baseline configuration is conventional with state-of-the-art details to minimize construction risk. The hull as presently configured is producible, cost effective, and adequate to perform the specified mission.

- (U) Propulsion System -- The proven LM 2500 and the developmental FT9A-2A alternate engines were chosen on the basis of proven capability and advanced developmental status, respectively. The transmission design features high state-of-the-art reliability and performance. The waterjet propulsor and inlet design has been optimized on the basis of extensive analysis and sub-scale tests. All other components are typical of PHM, SES-100A, and XR-ID practice; are presently available and proven in service.
- (U) Electrical System -- The baseline system design can be implemented with off-the-she&? equipment. The design is low risk, cost effective, and will provide satisfactory and reliable performance with high confidence.
- (U) Command, Control, and Communication (c^3) -- The c^3 systems are comprised almost entirely of Government Nomenclatured Equipments with attendant low risk in their use, The only potential risk is \mathbf{HF} communications during on and off cushion ship operations that would effect the antenna ground plane. The risk associated with other c^3 equipment is low or well within the state-of-the-art and absorbed by substantial, funded ongoing programs.
- (U) Lift System -- The LM 2500 gas turbine is a production unit used in other marine applications. Lift fan development is based on extensive **subscale** testing. The other elements of the air distribution system are typical of present gas turbine ship installations and within the present **state** of-the-art. The advanced bow and stern planing seals have proved highly successful in sub-scale tests. While there are no historical research or performance data on this particular **SES** application, **full** scale loads analysis and materials selection indicates **that all** considerations are within the state-of-the-art.

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- (U) Outfit and Furnishings (O&F) -- Nearly every item in the O&F system is a proven shipboard item not peculiar to the SES. The risk is equivalent to that of O&F on conventional Navy ships. Passive thermal/fire and acoustic protection systems are based on extensive testing and material evaluations, The risks associated with their application will be minimal.
- (U) Combat System -- The risk is that associated with ongoing Government development of the combat system equipments. The interface design risk is low.

APPENDIXA

DESIGN PROCESS

- (U) The various near term ANVCE Point Designs will be arrived at from different technology bases. Different standards, criteria and assumptions are used because of the different program offices and other Navy organizations involved. For example, structural safety factors between different vehicles are not the same, weight margins are frequently different and different ambient conditions may be assumed in quoting engine performance.
- (U) The near term SES point design concept outlined in this report adheres, wherever practicable, for consistency to information provided in such ANVCE documents as:

ANVCE Primary Documentation

- o WP-010 dated 27 August 1976 "Environmental Conditions"
- o WP-008 dated 20 August 1976 "Supportability/Availability"
- o WP-007 dated 30 July 1976 "Point Design Guidance
- o WP-005A dated 13 August 1976 "Point Design Description"
- o UP-002 dated 2 April 1976 "Definition of Terms'

ANVCE Supplementary Documentation

- "Design Standards for Surface Point Designs, Revision A",
 ANVCE Memorandum 90-76, dated 10 August 1976
- (U) WP-005A was used as the basis for the data developed in this report and was assumed as having precedence over other stated documentation requirements in cases of conflict. As a further aid to making proper evaluation of the near term SES point design presented in this report, this Appendix provides a basis for the insight needed into the design approach, criteria, philosophy and trade studies used in arriving at the design. This Appendix collects in summary form those pieces of information needed to identify the source of data and the design process used.

(U) A.1 APPROACH

For a basic vehicle configuration and the major subsystems, several methods of establishing characteristics exist. They may be classified into three groups:

- o Scaling -- projection of characteristics based on ratioing up or down from a chosen vehicle
- o Modification -- development of characteristics based on small changes to an existing vehicle
- o Synthesis -- development of characteristics **based** on design data, parametric analysis and theoretical investigations
- (U) The approach primarily used for the Rohr version of the ANVCE near term SES Point Design is modification to the Rohr 3KSES proposed design. This proposed design is, in turn, based upon scaling of appropriate model and testcraft data, as well as upon synthesis as just defined. The specific approaches in each disciplinary area are next identified and presented in concise form.

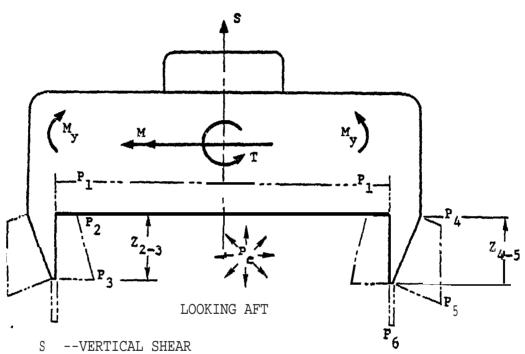
(U) A.2 DESIGN CRITERIA

-Those pertinent design criteria, standards and assumptions used in the Point Design are provided in the following areas: hull structure, propulsion, electrical plant, command and surveillance, auxiliary systems, lift system, outfit and furnishings, armament, loads, weight margins, and vehicle. Tabular forms and references are used as appropriate in the sections that follow for each **of** these areas.

- (U) A.2.1 HULL STRUCTURE -- Loads were developed that correspond to a number of operational definitions. The selected loading conditions are the result for a ship operating over a 20-year life anywhere within its operational envelope.
- (U) The following load cases are considered "operational" and the required safety factors used when applying these loads are 1.30 on the minimum yield strength and 1.80 on the ultimate strength:
 - o Load Case 1 -- Cushionborne, Operational This case is based on on-cushion operation anywhere within the operational envelope. There are no heading or speed restrictions.
 - o Load Case 2 -- Hullborne, Low Sea State This case represents hullborne operation (entirely off-cushion) in sea states 5 and below. There are no heading or speed restrictions.
 - o Load Case 3 -- Partial Cushion, High Sea State This case is for partial-cushion operation (not entirely off-cushion) in sea states 6 and above. There are no heading or speed restrictions.
- (U) The following load cases were considered as emergencies due to system(s) failures, Because the ship is in an emergency mode, operational maneuvers to alleviate loads and motions would be deemed appropriate, The

- (U) safety factors used for the following two conditions are 1.0 based upon the minimum yield strength and 1.50 based upon the ultimate strength:
 - o Load Case 4 -- **Hullborne/Lift** System Failure in High Sea States → This condition is for a lift system failure in sea states 6 and above. Headings within 45 degrees of head seas are not considered, but there is no restriction on speed.
 - o Load Case 5 -- Hullborne/Lift and Propulsion Failure in
 High Sea States → This condition is for lift and propulsion system failures in sea states 6 and above. Speed
 is considered to be zero, but there is no restriction
 upon heading.
- (U) A final load condition considered was for ship damage with subsequent flooding. The safety factor applied was 1.20 on the minimum ultimate strength. No safety factor is used for yield strength since the ship would already have suffered structural damage; therefore, local yielding was permissible.
 - o Load Case 6 → Damaged Ship → This condition is for the ship suffering maximum damage (two compartments flooded). Still water bending moments are considered along with hydrostatic loads due to flooding to the "V-Lines".
- (U) The 3KSES hull structure is designed to the predicted maximum once per lifetime loads that the ship will experience in a twenty-year life. These loads are not considered singly since those sea and weather conditions which produce the most severe loads, such as longitudinal bending, also produce other associated loads, such as shear, torsion and those due to hydrodynamic pressure forces. Figure A.2.1-1 presents the load nomenclature and definitions used in the description of the structural load cases which follow. Figure A.2.1-2 presents the maximum cushionborne bending moment and the shear, hydrodynamic pressures and vertical accelerations associated with

- (U) Load Case 1. The loads resulting from the conditions of Load Cases 2, 3 and 4 are presented in Figures A.2.:-3, A.2.1-4, and A.2.1-5, respective-1y. The loads resulting from Load Case 5 were found to be significantly less than those of Load Case 4, and are not presented. The many possible damage conditions of Load Case 6 are too numerous and complex to discuss in this document, However, the hydrostatic heads associated with flooding to the V-Lines were the loads which determined the scantlings of many structural elements.
- 1 cads with the safety factors specified in the load cases, the hull structure was originally designed to somewhat higher loads but lower required factors of safety. These high loads were the result of off-cushion ship operation in high sea states (sea states 6 through 9). Model testing and analysis demonstrated the advantage of partial—cushion operation in those high sea states, and this mode has been adopted for operation in that portion of the operational envelope. With completely off-cushion operation in sea states 6 and above due to an emergency, such operational maneuvers as implied by the speed and heading restraints of Load Cases 4 and 5 are deemed appropriate. The near term SES hull structure adequately withstands the developed loads and adopted factors of safety.
- (U) Fatigue Considerations -- A well established fatigue life (FATLF) computer program, along with accelerated time and fatigue testing of full scale welded panels, was used for verification of the endurance capabilities of the ship structure. Fatigue life of those test panels was increased significantly through the use of special fabrication and welding techniques. Basic joint design, along with controlled and scheduled welding and an in-service failure prevention plan, should assure a safe operational lifetime.



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- BENDING MOMENT (POSITIVE MOMENT INDICATES UPPER DECKS IN COMPRESSION)

 ${f M}_{{f Y}}$ - TRANSVERSE BENDING MOMENT (POSITIVE MOMENT INDICATES UPPER DECKS IN COMPRESSION)

Figure A.2.1-1 ($\tt U$): Loads Nomenclature and Reference System ($\tt U$)

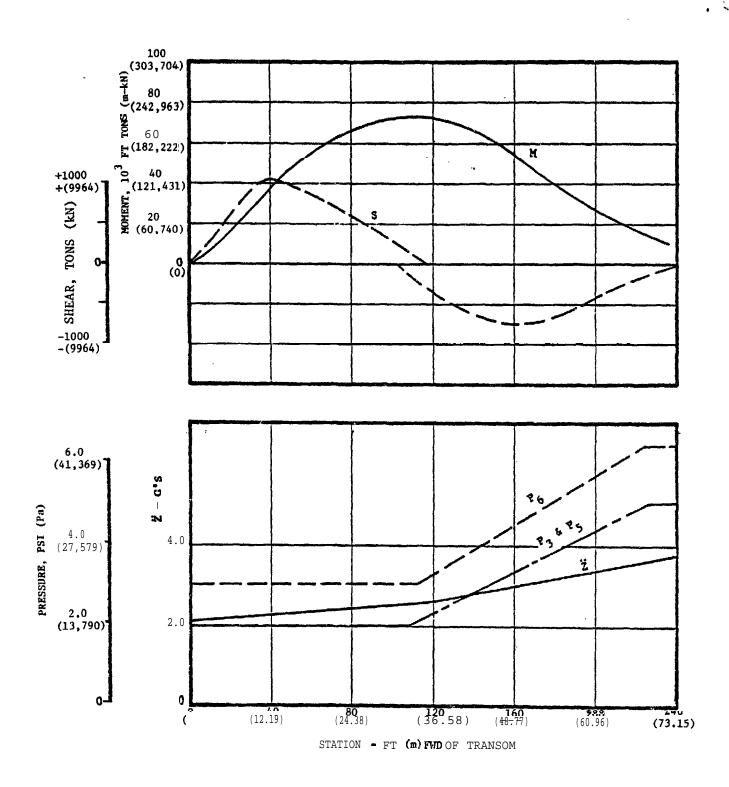


Figure A.2.1-2 (U): Shear Loads, Bending Moments, Hydrodynamic Pressures, and Vertical Accelerations Corresponding to Load

Case 1 (U)

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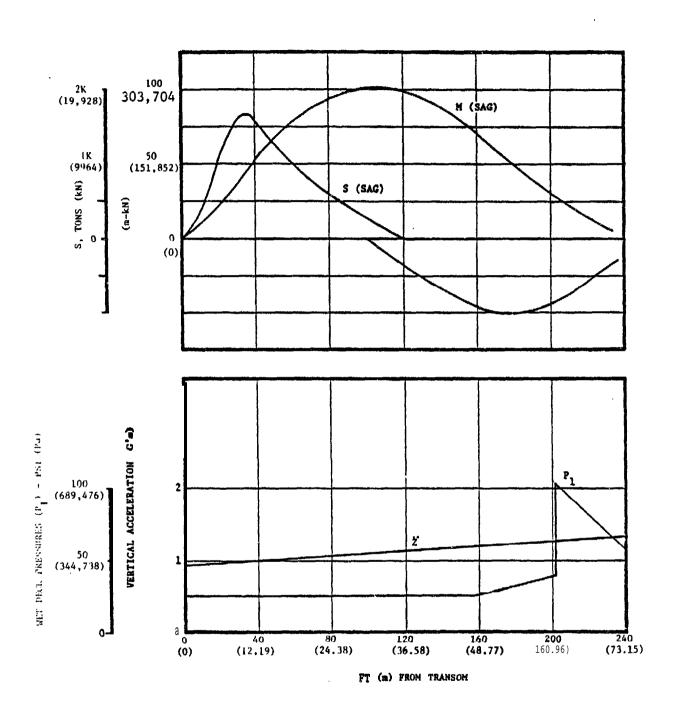
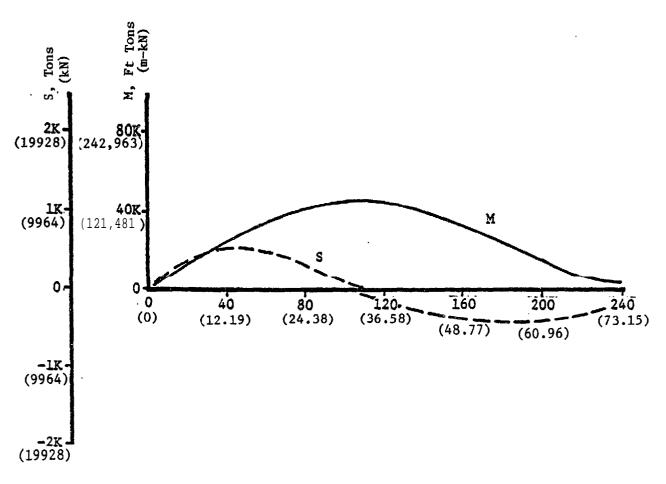


Figure A.2.1-3 (U): Shear Loads, Bending Moments, Hydrodynamic Pressures, and Vertical Accelerations Corresponding to Load Case 2 (U)



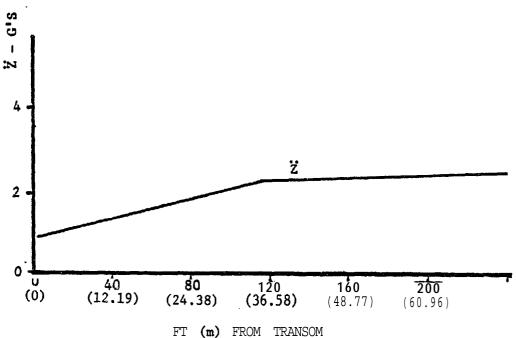


Figure A.2.1-4 (U): Shear Loads, Bending Moments, and Vertical Accelerations Corresponding to Load Case 3 (U)

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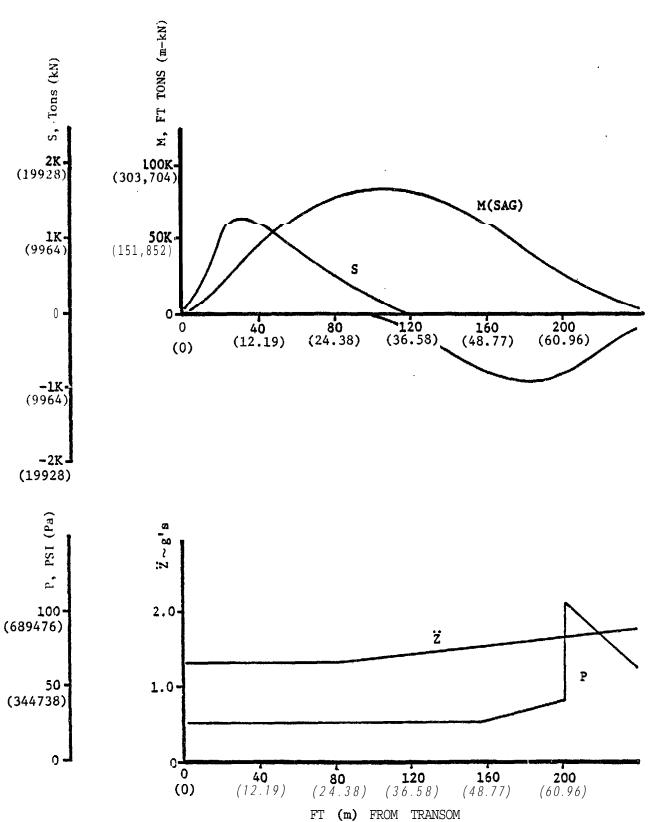


Figure A.2.1-5 (U): Shear Loads, Bending Moments, Hydrodynamic Pressures, and Vertical Accelerations Corresponding to Load Case 4 (U)

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- (U) A.2.2 PROPULSION -- General design criteria for the near term SES propulsion system includes maximization of performance, reliability, maintainability and simplicity. The propulsion machinery spaces are designed to accommodate either G.E. IM2500 or P&W FT9A-2A gas turbine engines. Specific design criteria applied to the point design are:
 - All machinery accessible for maintenance off-cushion without dry-docking, No corrosive air/water interfaces.
 - Short, straight drive shafts with no alignment and vibration problems. Flexible couplings to absorb dynamic misalignments.
 - Overspeed gas turbine engine control for protection against propulsion inlet air ingestion without complete engine shutdown.
 - Non-redundant link mounted propulsion components. The link mounted propulsor has less deflection than a gun mount. This simplified alignment, steering, and reversing interface and reduces vibration problems.
 - Low loss combusion air inlet system designed for 2.9 inches (7.36 cm) H₂O loss for LM250O and 4.9 Inches (0.12 m) H₂O loss from FT9A-2A installation. Sufficient interval flow area to install a three stage inertial coalescer—interid moisture separator operating at a face velocity of 17.5 ft/sec (5.33 m/sec). The total salt ingestion goal is 0.00136 ppm with a projected water wash interval of 450 hours. Capability to withstand a 4 foot (1.22 m) wave of green water on the O1 level without demister flooding and resultant breakthrough. Sufficient volume forward of the engine bellmouths to reduce pre-swirl and counter swirl to less than 5 and 12 degrees, respectively, and to keep distortion below 10 percent,

- (U) o Low loss propulsion exhausts the design criteria for sizing the exhausts is based on obtaining maximum net thrust to the ship with low weight, back pressure, fuel consumption and jet thrust within a maximum limit of 6 inches (0.15 m) $\rm H_2O$.
 - Acoustically treated intake and exhaust to meet Navy Category E requirements on the flight deck.
 - o Anti-icing system designed to provide protection to $-20^{\circ}F$ (-28.9%).
 - Engine cowling designed to limit potential personnal contact areas to 125°F (51.7°C).
 - Minimum number of moving parts in the flexible ramp roof variable area waterjet inlet. Smooth roof contour at any opening position. Vented roof cavities and pressure balanced to reduce structural loads and weight. Symmetrically configured bifurcated duct for low velocity distortion,
 - Propulsion inlet designed to provide cavitation free operation to ship speeds exceeding the maximum ANVCE specified speed.

- (U) A.2.3 ELECTRIC PLANT -- The near term SES Electric Plant design has been guided and controlled by a set of design criteria, standards, and a system design philosophy, collectively oriented toward the design of an uncomplicated and **flexible** system featuring minimum weight, cost, and fuel consumption. The current design highlights the following:
 - Adequate generated power, measured by operating margins, off-line reserves, and power quality
 - Weight and envelope minimization
 - Environmental compatibility
 - Minimal technical risk
 - Interface compatibility with ship structure
 - Adequate RMA and Safety considerations
 - Use of proven components where practicable
 - Use of standard Navy design precepts for the power distribution system
- (U) The system design philosophy emphasizes the criticality of a continuous source of electrical power, with judicious minimization of system weight, envelope size, and cost of components and installation. Every effort is made to strike a proper balance between innovative and traditional design. Modernization to include superior materials or components is encouraged, particularly where significant benefits accrue in reduced life-cycle costs, enhanced safety, or performance improvements. A number of standards were incorporated in the design methodology to ensure suitability for Navy use and compatability with the anticipated marine environment. Among these were:

MILITARY SPECIFICATIONS AND STANDARDS

MIL-E-917 Electric Power Equipment, Basic Requirements (Naval Shipboard Use)

MIL-STD-454 Standard General Requirements for Electronic Equipment

MIL-STD-1399/	Interface Standard for Shipboard Systems Section 103 Electric Power, Alternating Current
MIL-S-16036	Switchgear, Power, Naval Shipboard
NIL-C-17361	Circuit Breakers, Air, Electric, Insulated Enclosure (Shipboard Use)
MIL-C-17587	Circuit Breakers, Air, Electric, Open Frame Removable Assembly (Shipboard Use)
MIL-C-17588	Circuit Breakers (Automatic - ALB) and Switch, Toggle (Circuit Breaker, Non-Automatic - NLB), Air, Insulated Enclosure, 125 Volts and Below, AC or DC, Naval Shipboard
MIL-G-3124	Generator, Alternating Current, 60-Cycle (Naval Shipboard Use)
MIL-G-21480	Generator System, 400 Hz AC, Aircraft
MIL-G-22077	Generator Sets, Gas Turbine, Direct-and Alternating-Current, Naval Shipboard Use
0902-001-5000	General Specifications for Ships of the U. S. Navy (GSS); Naval Sea Systems Command (NAVSEA)
DDS-300-2	Design Data Sheet, AC Fault Current Calculations
DDS-311-3	Design Data Sheet, Ship Service Electric Power System, Application and Coordination of Protective Devices
DDS-304-2	Electrical Cables, Rating and Characteristics
DDS-311-Z	Design Data Sheet, Voltage Regulation for AC Ship Service Electric Power Systems

- (U) A.2.4 COMMAND AND SURVEILLANCE -- The Combat System, including command and surveillance, armament and navigation and collision avoidance, was dictated by the 28 May 1976, LSES Top Level Requirements Document.

 Equipment lists were provided by the U. S. Navy.
- (U) The Ship Control System design was based on utilization of existing, approved equipments, such as FFG control system components and the AN/UYK-20 computer, to provide integrated control of ship operations, damage control, and auxiliaries and ship plant monitoring by a minimum of crew members.

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- (U) A.2.5 AUXILIARY SYSTEMS
- (U) A.2.5.1 AIR CONDITIONING -- Requirements for the near term SES air conditioning system are:
 - o The decentralization of the air conditioning system by dividing the load in small serviced areas, and by using the fan **rooms** to accommodate the cooling/heating/fan integral units.
 - o Replacement of the chilled water system by straight air and hot-cold mixing boxes, and selection of lightweight foam type reinforced materials for **ducting**.
 - o Existence of state-of-the-art components already qualified by commercial and/or military requirements and in actual operation.

The results foreseen are:

- o Pseudo redundancy, since failure of one unit will bring only fractional failure to the subsystem.
- o Weight savings inherent to aircraft components.
- o Energy savings by proper management and more efficient equipment.
- o Reliable system by the use of qualified components.
- (U) A.2.5.2 LUBE SYSTEM -- A number of subsystems on board require lubrication. The prime thermal drivers for propulsion and lift and the electric power generating units will be self-contained; others like propulsion gearing, power transmission, waterjet pump and lift gearing, fans and power transmission will require dedicated lube subsystems.

- (U) The concept of a single centralized lube system versus multiple, dedicated systems was analyzed on the basis of: weight, cross contamination, cooling requirements, length of lines and bulkhead penetrations involved, reliability, and redundancy. Decision was made to employ the multiple, dedicated systems conceptual approach.
- (U) The standard way of using cooling seawater should be accepted only if it does not demand extra loading on the seawater subsystem, as for propulsion gear-pump units where water is available from the waterjet pump (second-stage cavity). The lift system should employ air as cooling media, and the location of the heat exchanger (oil to air) should be established (inlet or outlet of fans).
- (U) Pre- and post-operation lube oil circulation should be provided, as well as standby lubrication to assist main lube pump in low speed operation. The aeration of lube oil should be considered and the quality of lube oil should be closely controlled. High holding capacity for particulate containination and dewatering (vacuum plus coalescers) filters is inherent in the use of advanced practices and state-of-the-art components. A closed lube system should be compared with alternate schemes.
- (U) Short coupled lines should be used as exemplified by advanced systems used in other industries (petrochemical), and the clustering of fittings and components should be replaced by functional manifolds. The material for transmission lines should be compatible with that for gears and bearings, and should reflect low weight, fatigue strength compatibility and ease of handling. The lines should be supported by resilient mounts.
- (U) The results foreseen from this approach are enhanced system functioning, weight savings, energy savings (by using cooling media already available), and improved reliability by use of qualified components, practices in other industries, and application of naval operation experience.

- (U) A.2.5.3 SEAWATER SYSTEM -- An integrated seawater system serves firemain, seawater service and sprinkling functions with an appreciable weight reduction. Additional weight savings were effected by installation of an open horizontal loop, i.e., elimination of the wet weight cross connection.
- (U) Installation of GRP piping for seawater, auxiliary system and wet **firemain** removes corrosion problems and effects weight savings of approximately one-third, compared to that for an equivalent copper-nickel system. Components to be used are readily available and qualified for marine use.
- (U) A.2.5.4 POTABLE AND FRESH WATER SYSTEMS -- Generation of potable and fresh water from seawater requires selection of the desalination process, i.e., reverse osmosis versus one of the several types of distilling processes. The inability of presently available reverse osmosis units to meet the salinity requirements of the general ship specifications prohibited ics use.
- (U) The trade-off of potable and fresh water systems involved investigation of components and configurations possessing potential weight savings. This led to the selection of vacuum-assisted water closets and low water demand showers. The resulting weight reduction is due to the reduced quantity of water collected and stored via the drainage system and the reduced pumping capacity requirement. Further weight reduction was obtained with GRP piping,
- (U) A three-distiller configuration to reduce the stored potable water tonnage was investigated. Each unit was capable of supplying the ship's
 daily demand, and the tank tonnage was reduced by one-third of the required
 40 GPM (25.24 mm³/s) per accommodation. The fresh water storage tonnage
 was reduced by restricting the utilization of fresh water (demineralized)
 to gas turbine engine washing and to make-up water for the auxiliary fresh
 water electronic cooling system.

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- (U) The selection of electrical power (in lieu of gas or steam) was made .upon its ready availability. The 400 Hz supply was selected for all pump motors, on the basis of weight savings over 60 Hz types,
- (U) A.2.5.5 DISTILLING PLANT -- The selection of the type and capacity of distilling plant(s) requires the consideration of ease of maintenance and operation, quantity and form of energy available, and the fresh water requirements. The three basic types of distillers for naval ships are vapor compression, submerged tube, and flash. Each was evaluated in the trade process of optimal design selection.
- (U) A.2.5.6 FUEL SYSTEM -- The fuel system performs the following functions: provides fuel of proper quality to all the thermal drivers for propulsion, lift, and electric power generation provides CG location management by using fuel transfer as a means of trimming; provides storage and service of fuel for the aircraft on board.
- (U) Designated tanks are established for: trimming and storage, storage, service for on board equipment, and service for aircraft. The need of interconnecting tanks for functional operation dictates the use of multiple controls and a well planned distribution system that provides redundancy. Fluid lines with mechanically assembled joints of well known reliability are used in sections which may need to be removed and replaced; otherwise, butt weld connections are used. Proliferation of connections is avoided by use of functional manifolds. Due to high flow conditions, valves must have defined times for the close-to-open or opento-close cycles to avoid hammering. Lines should be supported by resilient mounts to avoid premature fatigue and undue noise or vibration coupling, Underway fueling should be in agreement with naval practices.
- (U) The quality of the fuel should be closely controlled by use of high capacity filters for particulate contamination and water removal in lines between storage tanks and service tanks, and between service tanks and thermal driver units or aircraft.

- (U) The results foreseen are: weight savings, by a judicious selection of componnets and materials, and reliability by the use of **redundancy** and qualified components.
- (U) A.2.5.7 COMPRESSED AIR SYSTEM -- The compressed air system permits propulsion and lift turbine starting. It supplies air for actuation of back pressure valves, exhaust gas transfer valves and propulsion engine exhaust doors, and for miscellaneous uses as required. Weight reduction of the compressed air system was acheived by starting the GTG's by electric battery power. Several tons of high pressure charged air bottles were thereby eliminated. Practically all of the compressed air system components would be selected from available and qualified light. weight components.
- (U) A.2.5.8 FIRE EXTINGUISHING SYSTEM -- A trade-off study was made to provide the design criteria and rationale for selection of the best flooding extinguishing agent. CO2 and Halon 1301 extinguishing systems were compared, and a Halon 1301 system was found to require less weight and to discharge a much shorter time as shown here:

Agent	Compart- ment	Quantity of Agent	Total Discharge Time		Toxicity Class
co ₂	5000 Ft ³ (850 m ³)	250 Lb (1.11 kN)	90 Sec	825 Lb	
Halon 1301	5000 Ft ³ (850 m ³)	141 Lb (627 N)	10 Sec	263 Lb	6

- (U) A.2.5.9 HYDRAULIC SYSTEM -- The choice of hydraulically-powered actuators/motors in lieu of either electrical or pneumatic equipment included weight, performance, cost, compatibility of design, installation and environment factors in each application. A trade-off study indicated a weight saving of several tons by employing hydraulically-powered -equipment. The studies resulted in selection of the following system features:
 - o Hydraulic Fluid: MIL-H-83282 was selected due to its ability to be operated at fluid temperatures up to $400^{\circ}F$ (204°C); it is a synthesized hydrocarbon fluid that is interchangeable with MIL-H-5606.
 - O System Pressure: 3000 psi (20.68 **kPa)** is recommended as the system pressure; it is the most widely used high pressure, and consequently, a great variety of qualified components are marketed from which to choose.
 - o Optimum Fluid Temperature: A fluid system temperature between 100 to $130^{\circ}F$ ($54^{\circ}C$) is recommended for stable fluid operation.
 - O Pump Selection: Variable displacement constant pressure pumps of aircraft type were selected for lightweight and input horsepower economics proportionate to flow rate.
 - o Reservoir: Pressurized reservoirs (bootstrap type) are substantially lighter and require less stored fluid (fluid weight alone is reduced by 1600 lbs (7117 N) minimum). These reservoirs are sized to deliver the required pump inlet pressure and maintain the entire return system pressurized which avoids external contamination.
 - o Rigid Tubing: CRES 304 is the selected material; it is readily available in the required diameters, relatively easy to bend and weld, and is appreciably less costly than tubing made from 21Cr-6Ni-9Mg. Welding was selected in preference to the use of fittings in the interest of

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- minimizing leakage. Welding is also preferred to brazing on the basis of fabrication and inspection considerations.
- o Flexible Tubing: Flexible tubing is Teflon-lined to avoid static charges and dirt contamination associated with rubber (which also "sluffs off" particles which can dam servo valves).
- o Cleanliness and Filtration: Hydraulic fluid cleanliness must be **enforced**, fluid must be purchased to Class 1, components must be clean to Class 2 prior to installation, and the entire system must be maintained at Class 3 by adequate on-board filtration.
- (U) A.2.5.10 POLLUTION CONTROL -- The pollution control systems are for wastewater and oil. Wastewater includes sewage (human body wastes, blackwater, soil lines) and sanitary (or gray) water, which includes shower, laundry and galley water). The selection of a marine sanitation device includes evaluating the regulations, technical and operational factors, installation, and maintenance, and the cost of the system.
- (U) A weight trade-off analysis for the marine sanitation device was made on the basis of a one-day operational period to disclose a weight saving through use of a no-discharge type compared to a flow-through type. A waste oil tank, sized for a 15-day mission provides storage of collected waste oil from machinery and equipment drains for subsequent disposal at a shore facility, thereby conforming to the zero oil discharge regulation.

- (U) A.2.6 LIFT SYSTEM
- (U) A.2.6.1 AIR MANAGEMENT -- The design criteria applied to the development of the near term SES lift system air management concept were:
 - o The total nominal cushion flow rate at low sea states, to be 37,500 CFS (1061.88 m^3/s).
 - o The cushion pressure for a 3KSES point design would be 342 PSF (16.38 **kPa)**.
 - o Approximately 1/3 equal parts of total air supply to be delivered to the bow seal, cushion and stern seal.
 - o A system efficiency shall be at least 75 percent. The system efficiency is defined as

$$\frac{(P_c \times Q)/550}{BHP} \times 100$$

where P_c = cushion pressure at rated design

Q = total cushion flow

BHP = prime mover horsepower output

- O All machinery should be capable of withstanding the following ship acceleration levels in g's: 6 up, 4 down, 2 forward, 3 aft and 0.5 thwartships.
- o The lift system should have a minimum availability factor of 0.9285. Availability is defined as the ratio of mission uptime to total planned mission length.
- O There would be no requirement for blast resistance.
- o Machinery spaces'would be acoustically treated to meet categories A through H.

- O Aircraft structural design practices would be applied to the design of machinery components in a marine environment with the goal of producing high strength-to-weight ratio components with a correspondingly high reliability.
- Mechanical vibration requirements for all ship machinery and equipment would be in accordance with Section .073c of the GSS.
- (U) In support of these criteria, thirty-three separate component specifications were developed to govern the lift system design.

- (U) A.2.6.2 SEALS SYSTEM
- (U) **A.2.6.2.1** Seals Design -- The **seals** design was developed within requirements which include :
 - o sealing of the cushion **with** a minimum drag and minimal leakage of cushion air;
 - o design for a minor influence upon ship pitching motion in the absence of ride control;
 - o in concert with ride control devices, aids in reducing bow and CG accelerations to a level compatible with ride quality requirements; and
 - o exhibits lateral compliance while operating in waves other than those dead ahead or astern.
- (U) The seals are of modular design with the flexible seal material modules separated by tear inhibiting attachment fittings to reduce seal vulnerability. They are designed to minimize water ingress into the pneumatic bags and to provide for the rapid drainage of water that enters the bag. Standardization was emphasized in all portions of the design. Seal system weight was minimized with total design weight less than the following:

	Maximum Acceptable		<u>Target</u>	
Bow Seal, lb (N)	33,000	(147,000)	25,000 (111,000)	
Stern Seal, lb (N)	32,000	(142,000)	25,000 (111,000)	

(U) Attachment fittings were designed to minimize weight, be simple to remove and replace, to minimize structural fatigue of the flexible seal pressure bag material, to resist the effects of the marine environment, and (between hard structure and fabric) to be designed such that rubbing and impacting between the two structures is minimized to reduce wear. Further constraints included requirements that any seal system operational failure mode would not result in an unsafe ship operating condition and that retraction would be provided for off and partial cushion operation.

(U) The result is seal systems that feature ease of maintenance, repair and replacement with simple tooling and procedures in **drydock**, at dockside, 'and at sea. Non-elastomeric surfaces were employed on the planing bow and stern seals at the seal water interface to minimize hydrodynamic drag and **maximize** seal service life. The major seal system components were designed to MTBF characteristics of:

<u>Seal System Component</u>	Minimum Acceptable	Target Service Life
Planing Surface at	400 Operating	100 hours at
Seal/Water Interface	Hours	80 knots (41.16 m/s)
Bag and Upper Loop	I.000 Operating	2000 Operating
Seal Structures	Hours	Hours

- (U) A.2.6.2.2 Seal Materials -- Tear strength of the coated fabric pressure bag material was specified as a minimum of 300 pounds (1,333 N) with a target of 500 pounds (2,220 N), for tear propagation in the fill direction. (Tear strength is considered to be the controlling factor in the selection of the pressure bag material.) Tensile strength of the pressure bag material has a required minimum of 1000 pounds per inch (175,000 N/m) in the warp direction and 800 pounds per inch (140,000 N/m) in the fill direction. The pressure bag material is required to possess good environmental resistance, consistent with the seal system design specifications. The weight of the pressure bag material is minimized, consistent with the other requirements, with a maximum weight of 100 oz sq yd (33.25 N/m²).
- (U) The pressure bag material requirements included surviving 10⁶ cycles at 20 percent of ultimate tensile strength in the warp direction (R=0.2); the goai was 10⁶ cycles at 30 percent of ultimate tensile strength (R=0.2). Seams in the pressure bag material must meet the requirements for the coated fabric. The seams must also be relatively flexible and stiffness discontinuities in the joint minimized.
- (U) **Flexural** fatigue strength of the glass reinforced plastic (GRP) planer material shall be a minimum of 90,000 psi (6.20 x 108 Pa) in the

- (U) longitudinal direction and 85,000 psi $(5.85 \times 10^8 \text{ Pa})$ in the transverse direction. Target values are 135,000 psi $(9.30 \times 10^8 \text{ Pa})$ in the longitudinal direction and 105,000 psi $(7.25 \times 10^8 \text{ Pa})$ in the transverse direction.
- (U) Maximum acceptable decrease in **flexural** fatigue strength of the planer material after aging in hot water shall be 18 percent. The target **value** is 12 percent. Tensile strength of the planer material shall be a minimum of 70,000 psi $(4.83 \times 10^8 \text{ Pa})$ in the longitudinal direction and 60,000 psi $(4.14 \times 10^8 \text{ Pa})$ in the transverse direction. The corresponding target values are 107,000 psi $(7.38 \times 10^8 \text{ Pa})$ and 90,000 psi $(6.20 \times 10^8 \text{ Pa})$. Tensile modules of the planer material shall be a minimum of $3.7 \times 10^6 \text{ psi}$ $(2.5 \times 10^{10} \text{ Pa})$ in the longitudinal direction and $3.4 \times 10^6 \text{ psi}$ $(2.3 \times 10^{10} \text{ Pa})$ in the transverse direction. The corresponding target values are $5.0 \times 10^6 \text{ psi}$ $(3.4 \times 10^{10} \text{ Pa})$ and $4.2 \times 10^6 \text{ psi}$ $(2.9 \times 10^{10} \text{ Pa})$.

A.2.7 OUTFIT AND FURNISHINGS

(U) 'A.2.7.1 HABITABILITY -- The habitability standards should conform or exceed General Specifications for Ships of the U. S. Navy and OPNAVINST 9330.74 (proposed). Crew accommodations are as follows (based on manning requirements of the TLR):

	~ERTHING BREAKDOWN				
	TLR	TLR	TLR	TOTAL	ACTUAL
	CREW	GROWTH	RESERVATIONS	BERTHS PROVIDED	CREW
Enlisted	85	2	8	95	63
CPO	10	2	1	13	9
Officer	15	0	2	17	12
TOTAL	110	4	11	125	84

(U) A.2.7.2 PASSIVE FIRE PROTECTION -- The design philosophy for treatment of spaces in Group 1 implies prevention of primary aluminum structure from reaching 400° F (204° C) for a period of 15 minutes. This conservative approach in an active system design results in detection and extinguishment of a fire in within 5 minutes maximum.

The Fiberfrax panel system was selected for its superior performance relative to other lightweight systems considered. The methodology used to establish the insulation thickness is described in the following steps:

- a. A computerized thermal analysis established the relationship between felt insulation thickness and temperature of the structure under fire conditions.
- b. A full-scale JP-5 fuel fire *test* was conducted and the temperature distribution of the front face sheet of the insulation panels was monitored throughout the test.

- (U) c. The temperature/time profile obtained during the tests was used as an input to thermal analysis, and temperature/ time curves were obtained for several insulation thicknesses (see Figure A.2.7.2-1 and A.2.7.2-2).
 - d. From the curves of Figures A.2.7.2-1 and A.2.7.2-2, plots were made of insulation thickness versus time for the structure to reach 400° F $(204^{\circ}$ C) (See Figure A-2.7.2-3).
- (U) The passive fire protection material for Group 2 spaces was also selected on the basis of smoke and toxic properties in a fire environment. The concern stems from the direct threat to personnel and from restricted visibility along escape routes. The very low smoke and toxic gas emission properties of Fiberfrax made this material attractive for application in Group 2 spaces.
- (U) The design approach to treatment of spaces in Group 2 is based on a modification of the fire loading concept described in the Society of Naval Architects and Marine Engineers (SNAME) Aluminum Fire Protection Guidelines. The fire loading of a space is a measure of the quantity of combustibles per unit deck area. It is expressed as pounds of wood per square foot with combustibles other than wood related to wood with a heat capacity of 8000 BTU/lb (1.86 x 10 J/kg), The methodology used to establish the amount of protection (insulation thickness) is described in the following steps:
 - a. Full scale fire tests were conducted with fire loadings of 12.5, 10, 7.5, 5 and 2.5 lbs mass of wood per square foot (61.0, 48.8, 36.6, 24.4 and 12.2 kilograms of wood per square meter).
 - b. The temperature/time profiles of the front face of the insulation panels during the tests were used as input to the thermal analysis computer program. Figure A.2.7.2-4 shows the temperature/time profiles for the various fire loadings.

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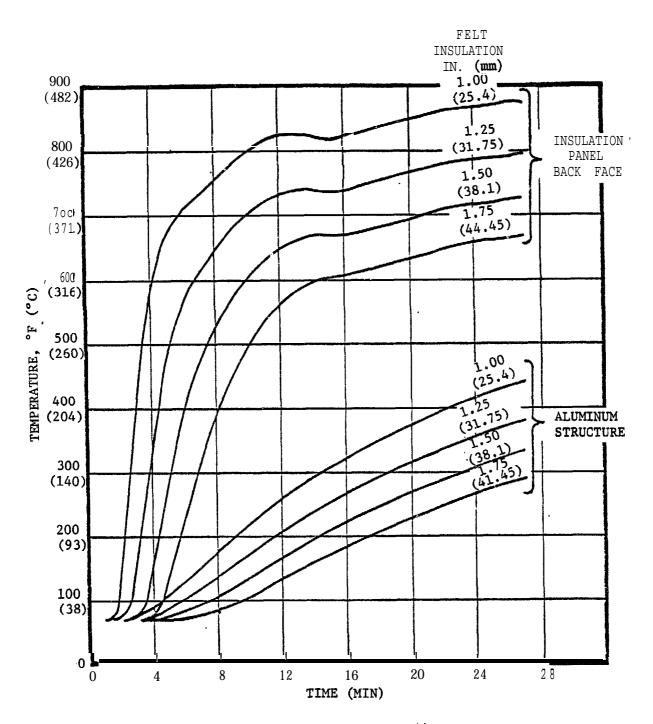


Figure A.2.7.2-1 (U): Temperature/Time Curves of Back Face of Insulation Panel and Aluminum Structure for Various Insulation Thicknesses in a JP-5 Fuel Fire (Structure Not Insulated on Far Side) (U)

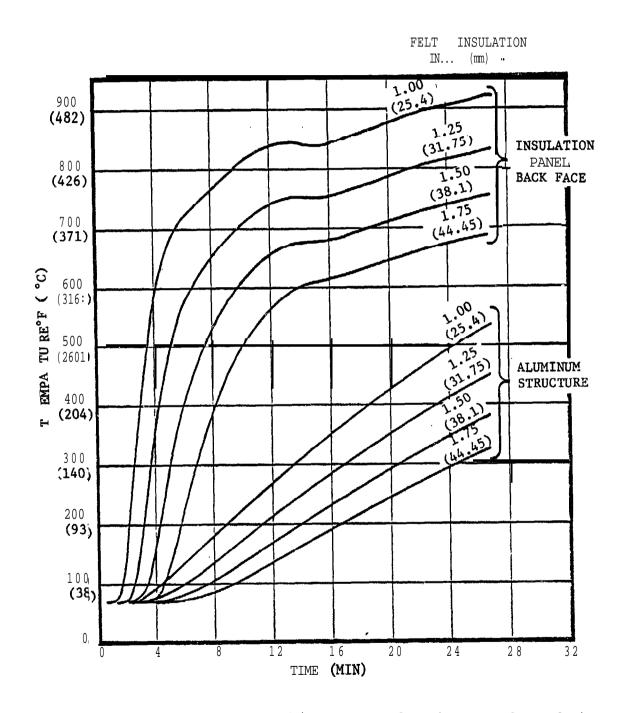


Figure A.2.7.2-2 (U): Temperature/Time Curves of Back Face of Insulation Panel and Aluminum Structure for Various Insulation Thicknesses in a JP-5 Fuel Fire (Structure Insulated on Far Side) (U)

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Figure A.2.7.2-3 (U): Insulation Thickness Vs Time for Aluminum Structure to Reach 400 Degrees I? in JP-5 Fuel Fire (U)

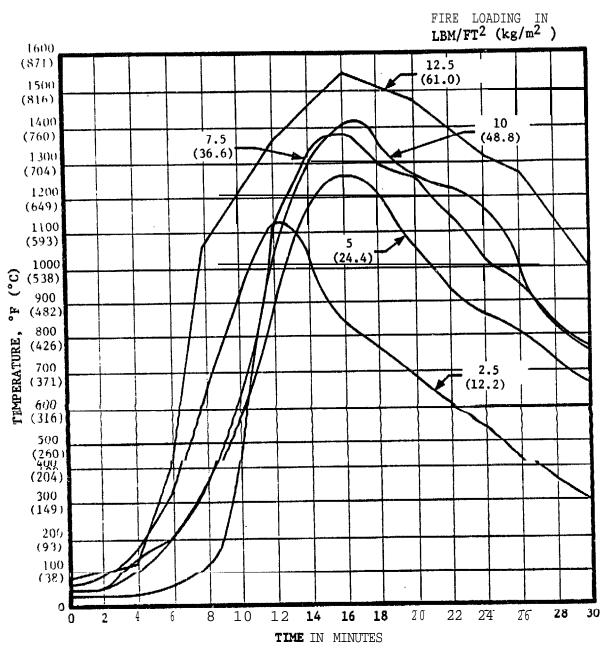
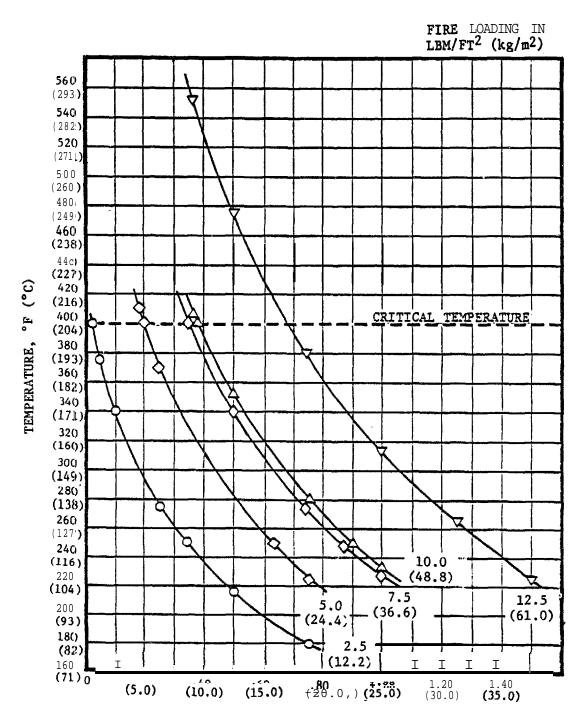


Figure A.2.7.2-4 (U): Temperature/Time Profiles on Front Face of Insulation Panels in Wood Crib Fires with Various Fire Loadings (U)

- C. The program output the temperature/time envelope of the aluminum structure for various amounts of insulation thickness. The maximum temperature of the structure with a given insulation thickness for each fire loading is plotted in Figures A.2.7.2-5 and A.2.7.2-6.
- d. From the curves of Figures A.2.7.2-5 and A.2.7.2-6, plots were made of insulation thickness versus fire loading for one-side and two-side insulated structures (see Figure A.2.7.2-7).
- e. The insulation thickness was selected from these curves.

 (Panel thicknesses in increments of 0.25 in. (6.35 mm) were selected for practical manufacturing and ready material availability.)
- (U) The primary need in protecting magazines is to provide cooling when there is an adjacent fire hazard. Water sprinkling is the most efficient means to cool these spaces. Likewise, glass thermal insulation can be used more efficiently than refractory fibrous insulation in these spaces to prevent heat from entering,



INSULATION THICKNESS, INCHES (mm)

Figure A.2.7.2-5 (U): Maximum Temperature of Aluminum Structure Versus Insulation Thickness for Various Fire Loadings in Solid Combustibles Fires (Structure Insulated on Far Side) (U)

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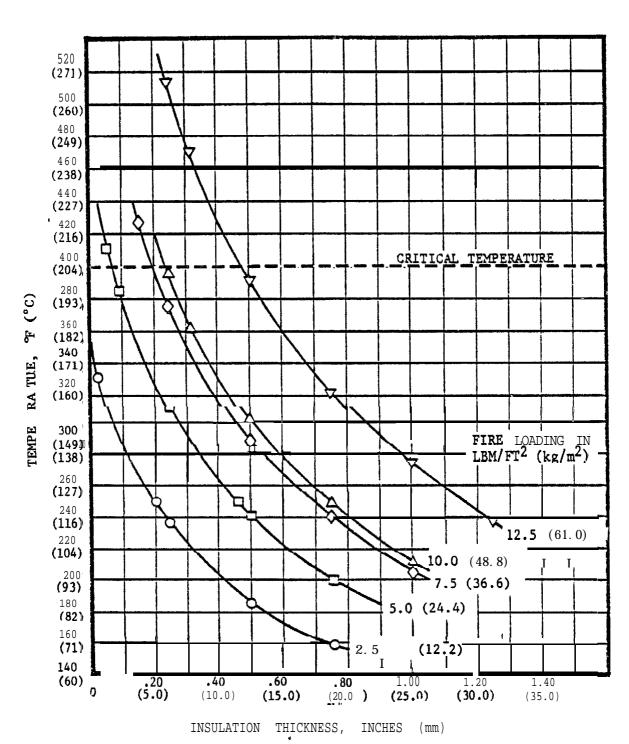


Figure A.2.7.2-6 (U): Maximum Temperature of Aluminum Structure Versus Insulation Thickness for Various Fire Loadings in Solid Combustibles Fires (Structure Not Insulated on Far Side) (U)

Figure A.2.7.2-7 (U): Felt Insulation Thickness Versus Fire Loading to Prevent Aluminum Structure from Exceeding 400 Degrees F (U)

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- (U) A.2.8 ARMAMENT All topside sensors and armament were required to have as great an unobstructed coverage envelope as practicable. The order of precedence for coverage in descending order for the near term SES is:
 - o TACAN
 - o EW System AN/SLQ-31 or -32(V2)
 - o Air Search Radar (AN/APS-125)
 - o Surface Search (AN/SPS-55)
 - o Collision Avoidance Radar (CAB)
 - o FCS MK92
 - o STIR
 - 0 CIWS (MK15/0)
- (U) Harpoon launchers are fixed and should be facing forward. Both Harpoon and the vertical launchers for the Standard missile were to be located to result in minimal plume ingestion problems for the main combustion air intakes. Torpedo launchers were to be located to facilitate 0 45 deg options for firing MK46 torpedoes and a fixed athwartship axis for MK48 torpedoes (based on NAVORD studies). Appendix B contains diagrams outlining the coverage of the major near term SES weapons and sensors.
- (U) The near term SES would only have a reload capability at sea for the MK46 torpedoes used in helicopter related ASW operations. As specified in the 3KSES TLR, no reload-at-sea capabilities would be provided for vertical missiles, MK48 torpedoes, surface launched MK46 torpedoes or Harpoon missiles.

A.2.9 LOADS

(U) .The weight allowances for variable load items were derived from Naval Ships Technical Manual dated 1 March 1974, Chapter 9290, Paragraph 173.1, titled "Detailed Description of Conditions of Loading for Surface Ships." Paragraph 173.1(a) covers weight allocations for crew and effects as follows:.

	Pounds (Newtons)	Per Man
Officers (commissioned or warrant)	400 (1779)	
Chief Petty Officers	330 (1468)	
Other Enlisted Personnel	230 (1023)	

(U) The 3KSES TLR used as the near term SES requirement specifies a ship personnel complement of 17 officers, 13 chief petty officers and 95 enlisted men. The weight allowances then are:

		We:	ight
Personnel	Qty	Lbs.	(kN)
Officers	17	6800	(30.24)
Non-Corns	13	4290	(19.08)
Enlisted	9 5	21850	(97.19)
TOTAL	125	32940	(146.5)

- (U) This 146.5 kN total corresponds to 14.71 long tons (F10).
- (U) Paragraphs 173.1(c) and (d) of the referenced Technical Manual cover weight allocations for provisions, personnel stores, and general stores as follows:

<u>Provisions</u>	Pounds	(Newtons)	Per	Man	Per	Day
Dry		3.20	(14.2	3)		
Freeze		1.11	(4.9	4)		
Chill		1.65	(7.3	4)		
Clothing and Small St	tores	0.07	(0.3	1)		
Ship's Store		0.80	(3.5	6)		
General Stores		1.06	(4.7	2)		
Clothing and Small St Ship's Store	tores	0.07	(0.3 (3.5	1)		

(U) The ship provisions, personnel stores, and general stores using those provisioning allowances for a X-day mission follow:

Provi	sions and Personnel	Stores:	
	pounds/man/day	30.38 N/man/day 15 days	
102.45 x 125	pounds/man men	455.7 N/man 125 men	
12,806 (56.96 kN)	pounds ÷ 2240 =	5.71 long tons (56.96 kN)	(F31)

<u>General Stores</u>	
1.06 pounds/man/day	4.72 N/man/day
x 15 days	15 days
15.9 days	70.73 N/man
x 125	125 men
1,988 pounds+2240 =	0.89 long tons (F32)
(8.84 kN)	(8.84 kN)

(U) The 3KSES TLR requires support for two SH-3H helicopters for a 15-day mission at the rate of 45 flight hours per month. The SH-3H has a nominal fuel consumption rate of 215 gallons per hour (0.0027 m^3/hr) and JP-5 weighs 6.8 lbs per gallon (6654 N/m^3). Therefore,

215	gallons/hour	$(.0027 \text{ m}^3/\text{hr})$
x 6.8	pounds/gallon	(6654 N/m³)
1,462	pounds/hour	(1.80 N/hr)
x 45	hours/month	45 hours/month
65,790	pounds/month/helicopter	(292.6 kN)

(U) 65,790 lb. ÷ 2 (15-day mission) = 32,895 pounds/helicopter

x 2 helicopters

65,790 pounds

(292.6 kN)

65,790 lb.+ 2240 pounds/ton = 29.3 long tons (292.6 kN)

helicopter fuel allocation

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A.2.10 WEIGHT MARGINS

(U) **The** near-term ship weight margins were allocated in the same manner as for the 3KSES. The weight margin policy for the 3KSES was outlined in Navy letter **PMS304-20** SER 2091; dated 12 June 1975. The weight margins were applied as in the cited letter with the exception of M13 (preliminary design and advanced development margin) which was depleted because the appropriate design phase has been completed for the 3KSES. The following Table **A.2.10.1** outlines the margins as applied.

TABLE A.2.10-1 (U): Near Term SES Weight Margins (U)

SWBS No.	DESCRIPTION	ALLOCATION
M11 M12 M13	Detail design margin Building Margin Preliminary Design and Advanced Development	2.5% LS ⁽¹⁾ 1.0% LS
M21	Margin Contract design margin	3.0% LS 2.0% LS
M22 M23	Contract Mod. margin GFM Margin	1.0% LS 0.5% LS
M25	Service life Margin	25 LT (249.10 kN)

(1) LS = Light Ship

A.2.11 VEHICLE

(U) A.2.11.1 Payload Weight Breakdown — The vehicle weight summaries shown in Table A.2.11-1 and A.2.11-2 detail the near term ship as defined in ANVCE WP-002, "Definition of Terms", dated 2 April 1976, Section III. The contract margins are included in the vehicle empty weights. These weight breakdowns support range and payload performance projections in Section 2.2.3.

TABLE A.2.11-1 (U): Vehicle Weight Summary (FT9A-2A) (U)

SYMBOL	TITLE	LONG TONS	SHORT TONS	METRIC TONS	KILO NEWTONS
₩ <mark>E</mark>	Empty Weight Less fixed payload items	•	1748.3	1586.0	15,554
₩c	Ship's Complement and Effects & Store	21.4 es	24.0	21.7	213.2
W _P	Payload	177.2	198.4	180.0	1,765
W _F	Liquids	1240.5	1389.3	1260.4	12,360
W	Vehicle Weight	3000.0	3360.0	3048.2	29,892
		<u></u>			

TARLE A.2.11-2 (U): Vehicle Weight Summary (FT9A-2A) (U)

SYMBOL	TITLE	LONG CONS	SHORT TONS	METRIC TONS	KILO NEWTONS
w _E ¹	Empty Weight less fixed payload items	1599.0	1790.9	1624.7	15,932
w _C	Ship's Complement and effects & stor	21.4 ks	24.0	21.7	213.: 2
$W_{\mathbf{p}}$	Payload	177.2	198.5	180.0	1,765
W _F	Liquids	1202.4	1346.7	1221.7	11,981
lW	Vehicle Weight	3000.0	3360.0	3048.2	29,892

- (U) A.2.11.2 STABILITY AND RESERVE **BUOYANCY** The near term SES must survive, with margin, the operational hazards of the open ocean, as specified in the criteria of:
 - o Goldberg, L. L., Tucker, R. G., "Current Status of Stability and Buoyancy Criteria Used by the U. S. Navy for Advanced Marine Vehicles?, Naval Engineers Journal, October 1975.
 - Sarchin, T. H., Goldberg, L. L., "Stability and Buoyancy Criteria for U. S. Naval Surface Ships", Transactions of the SNAME⁽¹⁾, Volume 70, 1962.
- (U) The freeboard and internal subdivision of the near term SES must be selected to satisfy the qualification of the criteria for reserve buoyancy and stability in terms of:
 - o Hullborne intact stability
 - o Reserve buoyancy under conditions of hull damage
 - o Damaged stability
- (U) Analysis has demonstrated that the near term SES design would meet the Navy criteria established for Large **SES's** as set forth **in** the cited references for displacements in excess of 3000 tons.

⁽¹⁾ Society of Naval Architects and Marine Engines

(U) A.2.12

MANNING

The Rohr selection criteria for crew size and composition, and some of the design alternatives by which the near term SES crew size and composition was developed, are the result of a Rohr-developed methodology that utilized the LSES TLR and the "Guide to the Preparation of Ship Manning" (OPNAV 10P-23), to impact design and trade-off studies.

- (U) A.2.12.1 MANNING CRITERIA In arriving at the crew size and composition, the following criteria were used:
 - o The design of the near term SES supports demonstration of the feasibility of platform performance, including that at high speed, and combat capability.
 - o The near term SES was not designed to meet **existing** Navy standards for wartime use, but incorporates salient features of a combatant ship for evaluation purposes.
 - Required operational capabilities (ROC) as defined in the LSES TLR of 28 May 1976 were employed to identify requirements for manned stations and their location, control equipment, and manned station layouts and support systems. The ROC are projected for performance of military value demonstration and combat system compatibility.
 - The Projected Operational Environment in Fleet Operations was derived from the TLR to delineate specific capabilities which the fully-ready LSES should achieve as goals as follows:
 - 1) At sea in peace time, Readiness Condition IV and for Battle Readiness, Special Condition I.
 - 2) A capability to perform anti-air, anti-submarine, and surface warfare on a non-simultaneous basis.
 - 3) A capability to meet emergency contingencies.
 - 4) A capability to perform maintenance for which the crew is assigned responsibility.

- (U) A.2.12.2 READINESS CONDITION Conditions of readiness for the purpose of determining operational, maintenance, administrative and support capabilities required to support the SES are as follows:
 - Special Condition I Battle Readiness -- These conditions for the near term SES are:
 - 1) Condition I ASW: Anti-submarine operations
 - 2) Condition I AAW: Anti-air operations
 - 3) Condition I SUW: Surface operations
 - All required personnel continuously alert. All required operational systems manned and operating. No maintenance expected except that associated with critical and vital equipment repair. Reduced readiness requires changes from the Required Operational Capabilities approved by the Chief of Naval Operations for Special Condition I.
 - condition IV: Peacetime Cruising Readiness Operational systems are normally manned only to the extent necessary for effective **ship** control, propulsion and security.

 Accomplishment of all underway maintenance, support and administration functions is expected. Maximum advantage is taken of **training** opportunities. Expected endurance at Condition **IV** is 15 days.
 - Condition V: In-Port Readiness Systems are manned to the extent necessary for effective operation. Watch stations are assigned as required to provide adequate security.

 Personnel on board are adequate to meet anticipated in-port emergencies and perform in-port functions as prescribed by unit ROC. Accomplishment of all required maintenance, support and administrative functions is expected. Maximum advantage is taken of training opportunities and (subject to the foregoing requirements), the crew has a maximum opportunity for rest, leave, and liberty.

- (U) A.2.12.3 MANNING ALTERNATIVES The following design alternatives were instrumental in the development of the crew size and composition:
 - The near term SES Ship's crew was developed within the implied requirements of the combat system concept for a weaponized test ship, rather than for a lead ship of its class.
 - The administrative office requirements were combined into two adjoining offices to provide adequate working areas rather than individual office spaces per man. The combined offices are:
 - 1) Operations Office, Weapons Office, Engineering Office -The combination of these offices into the department
 office provides working space for the department heads,
 ship's 3-M coordinator, and engineering personnel.
 - 2) Data Bank/Technical Library Within the data bank there are work areas for research and equipment, for making copies of stored data and for conference purposes.

This central administrative complex is interconnected by arches providing access to each function performed. The design is economical in terms of equipment, furniture, space, and manpower utilization, and results in weight reduction.

- The central (rather than remote) control concept has resulted in more efficient utilization of manpower. Ship automation features include full control of ship's steering, propulsion, auxiliaries and damage control from the pilot-house and Central Operating Station (COS). The minimum required watch positions for the SES are:
 - 1) OOD/Ship Control Officer
 - 2) JOOD/Asst. Ship Control Officer
 - 3) Lookout/Signalman
 - 4) Propulsion Control Console Operator
 - 5) Damage Control/Auxiliary Console Operator

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- (U) A.2.12.4 OPERATIONAL MANNING **REQUIREMENTS** Operational manning is the sum of quantitative and qualitative naval manpower needs to man essential operating stations during a specified condition of readiness. The operational manning requirement for a condition of readiness is expressed in terms of the related condition watch organization. The minimum essential **operational stations** developed for the near term SES are:
 - o Special Readiness Condition I, manned on a one-section basis, requires 79 operational stations.
 - Readiness Condition IV, manned on a three-section basis, requires 5 operational stations (15 personnel). The minimum number of personnel required for Readiness Condition IV is 54 (duty and watch).
 - o Readiness Condition V, manned on a one-in-three watch rotation within each of six duty section basis, requires two operational stations (36 personnel).
 - Flight quarters, manned on a one-section basis, requires 37 operational stations.
- (U) A.2.12.5 ORGANIZATIONAL MANNING The organizational manning requirements developed for the near term SES are:

	Officers	CPO's	Other Enlisted	Total
Crew Helo Det.	0 8 0 4	9 8	<u> </u>	1 0 6 6
TOTAL	1 2	0 9	6 4	85

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(U) A.2.13 PERFORMANCE

The proposed 3000 LT (29,892 **kN)** near term SES includes all of the fuel (for both ship and helicopters), sensors and weapons specified in the 28 May 1976 3RSES TLR. The basis upon which the near **term** SES design performance was developed compared with 3RSES TLR requirements is:

Design Parameter	3KSES TLR LT (kN)	ANVCE Near Tern SES LT (kN)
Full Load Displacement (LT; kN) Mean Operating Displacement (LT; kN)	3000 (29,892) 1 920⁽¹⁾ (19,131)	
Wind Speeds	Pierson Moskowit (no attitude gra	-
T 'ail Pipe (Trapped Fuel) Allowance (LT; kN)	46 (458)	64.6 (644)
Marine Fouling Allowances	1 mil Surface Fi	nish
Ambient Temperatures • air water	80°F (26.67°C) 59°F (15°C)	

Mean Operating Displacement at 10% fuel load (LM2500 propulsion)

⁽²⁾ Mean Operating Displacement at 50% fuel load (LM2500 propulsion)

- (U) Detailed comparisons between performance of the **3KSES** design and the 28 May 1976 TLR requirements regarding speed, hump margin, acceleration and deceleration, turning, range, and operational Sea State performance are outlined in the following sections:
- (I) A.2.13.1 SPEED The predicted speed capability and requirements are shown in Table A.2.13-1. All speed requirements at a full load displacement of 3000 LT are met, including the requirement of 30 knots (15.4 m/s) with a significant wave height of 15 ft (4.57 m), FT9A-2A engines at maximum continuous power (MCP), and for headings up to 60 deg or more from a head seas condition. Off-cushion, the SES provides operational speed capability approaching the goals.
- (U) A.2.13.2 HUMP THRUST MARGIN Comparison between the predicted and required hump thrust margins for the **3KSES** are shown in Table A.2.13—2. The near term SES betters the deceleration goals specified with either the LM 2500 or FT9A—2A configuration. The acceleration goal is met with the LM 2500 configuration; however, the FT9A—2A configuration requires almost twice the acceleration goal time interval.
- (U) **A.2.13-4** TURNING -- The TLR specified that the **3KSES** must meet the following:
 - o On and off-cushion, ahead and astern, control of heading for docking, **undocking** and low speed maneuvering in a seaway.
 - o Maximum Tactical Diameter of $4500~{\rm Et}~(1.37~{\rm km})$ at speeds below hump speed.
 - o Maximum Tactical Diameter of 15,000 ft (4.57 km) when entering a turn at maximum speed. (The SES is not required to maintain constant speed in turns above hump speed).

		Significant Wave Height	Requirement	Knots (m/s)	Predicted Capability
Operation	Configuration	Ft.(m)	Coal	Min,	Knots (m/s)
Ahead On-Cushion	Mean Operating Displacement LM2500 at MCP	3.3 (1.0) 15.0 (4.57)	100 (51.4) 80 (41.2) 40 (20.6)	70 (36.0) 30 (15.4)	76 (39.1) 70 (36.0) 31 (15.9)
	Full Load Displacement FT9 at MCP	0 3.3 (1.0) 15.0 (4.57)'	100 (51.4) 80 (41.2) 40 (20.6)	70 (36.0) 30 (15.4)	81 (41.7) 73 (37.6) 30 (15.4)
Ahead Off-Cushion	Full Load Displacement Engine at MCP	0	15 (7.7)		14 (7.2) LM2500 15 (7.7) FT9
	in the second se	15.0 (4.57)	10 (5.1)		9 (4.6) LM2500 9 (4.6) FT9
Astern Off-Cushion	Full Load Displacement Engines at MCP	0	10 (5.1)		5 (2.6) LM2500 5 (2.6) FT9
		10.0 (3.05)	5 (2.6)		4 (2.0) LM2500 4 (2.0) FT9

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Table A.2.13-2 (C). Hump Thrust Margln (U)

		Significant Wave Height		Requirement-%		Predicted Capability
Operation	Configuration	Ft.	_	Goal	Min.	%
Ahead On- Cushion	Mean Operating Displacement LM2500 at MIP (1920 LT; 19,131 kN)	3.3	(1.0)	15	10	8 6
	Full Load Displacement FT9 at MIP (3000 LT; 29,892 kN)	3.3	(1.0)	30	20	2 1

Table A.2.13-3 (C). Acceleration/Deceleration (U)

		Sea	Require	ment	Predicted (apability
Operation	Configuration	State	Goal	Min.	LM2500	FT9
Ahead On-Cushion Acceleration to Full Speed	Full Load Displacement	0	180 Sec.		92* Sec.	330 Sec.
Ahead On-Cushion Deceleration from Full Speed	Full Load Displacement	0	3000 Ft. (914.4 m)	.	735 Ft. (224.0 m)	2955 Ft. (900.7 m)

^{*}This value is seemingly low only because the LM2500 configuration is limited to sub-hump operations at FLD.

- (U) Figure A.2.13-1 depicts several turns attainable with the FT9A-2A configured near term SES at full load displacement. The figure shows that the SES can better the TLR's on- and off-cushion turn requirements.
- (U) A.2.13.5 RANGE -- The predicted range performance characteristics are compared with the TLR range requirements. The range capability of both the LM2500 and the FT9A-2A configuration is computed on the basis of an average speed greater than 63 knot8 (31 m/s); the LM2500 configuration nearly attrins its goal, bettering the requirement by more than 15 percent. The FT9A-2A configuration better8 the requirement by about 10 percent. Range performance is shown in Table A.2.13-4.
- between ship operating mode and the operational envelope are shown in Figure8 A.2.13-2 and A.2.13-3. These figure8 define the operating envelope8 in term8 of the operating mode, speed and sea state. Figure A.2.13-2 define8 operating envelopes which are based on the Navy's Top Level Requirement8 (TLR) of 28 May 1976 while Figure A.2.13-3 defines the envelopes which are TLR goals. The on-cushion envelopes define the operation with respect to the propuision system gas turbine engine. The baseline engine is the LM2500 and the alternate is the FT9A-2A.

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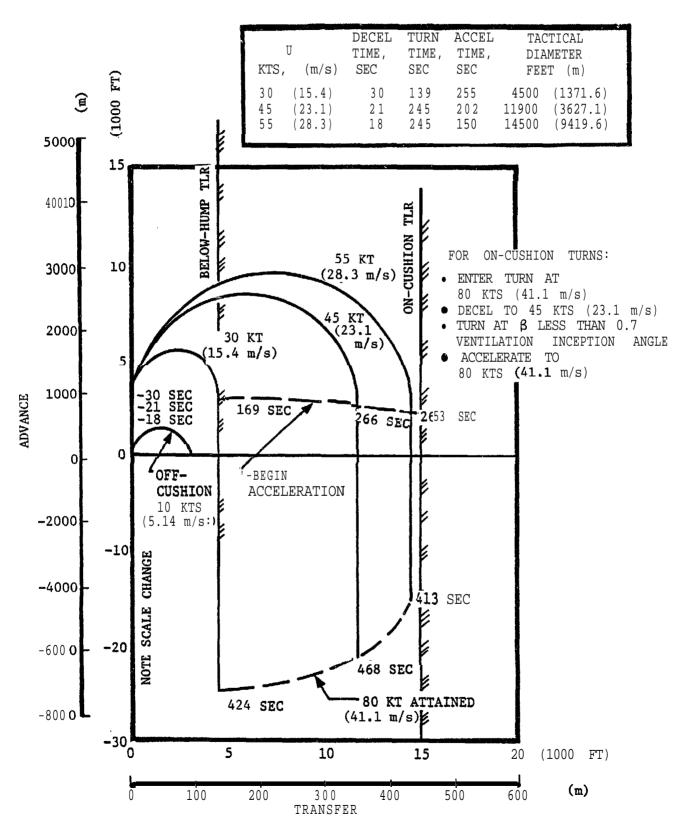


Figure A.2.13-1 (U): 3KSES Turning Capability (U)

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Table A.2.13-4 (C). Range (U)

		Significant Wave Height	Requirement NM (km)		Predicted Capability	
Operation	Configuration	Ft. (m)	Goal	Min.	NM (Km)	
Ahead On- Cushion	Full Load Displacement LM2500 Average Speed . 60 Kts (31 m/s)	3.3 (1.0) (No Wind)	3000 (5556)	2500 (4630)	2960 (5482)	
	Full Load Displacement FT 9 - Speed 60 Kts (31 m/s)	3.3 (1.0) (No Wild)	3500 (6482)	3000 (5556)	3295 (6102)	

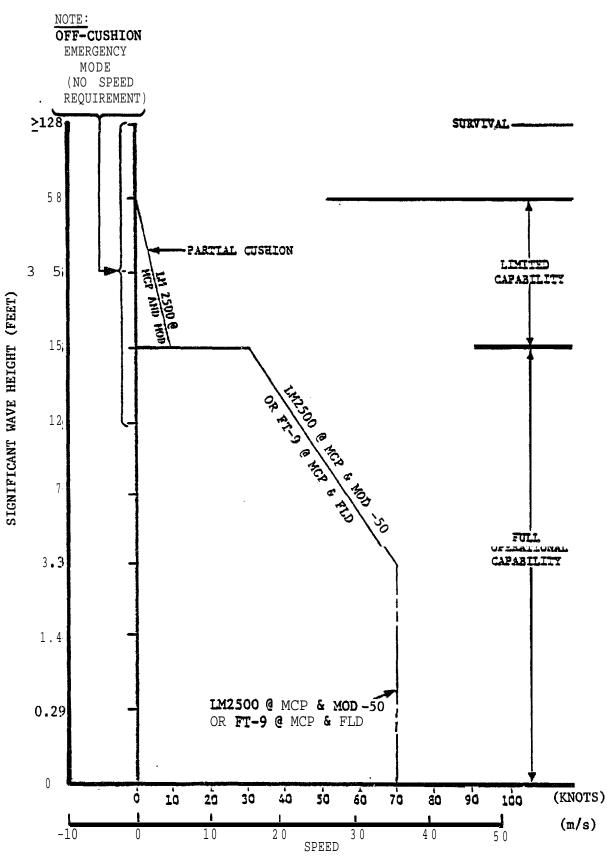


Figure A.2.13-2 (C): 3KSES Operational Envelope - Requirements (U)

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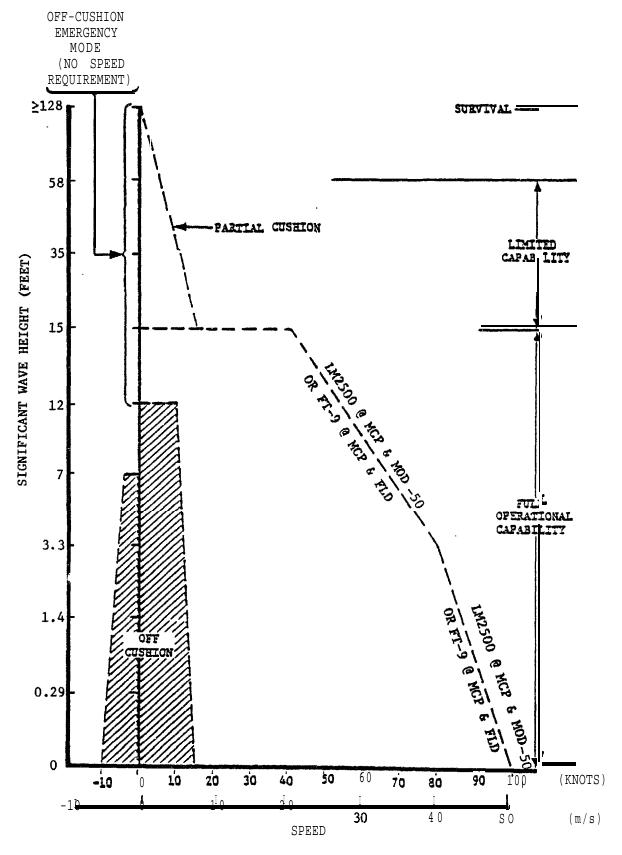


Figure A.2.13-3 (C): 3KSES Operational Envelope - Goals (U)

A.3 DESIGN PHILOSPHY

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- (U) The overriding philosophy of the near term point design **SES** design for the capability of demonstrating that the SES concept can fulfill a role as an operational fleet unit. Every design decision has supported this philosophy. The result is a balanced design in which no single feature is dominant. All subsystems and their components **were** accorded careful development and engineered to meet the specified Top Level Requirements **(TLR)**.
- (U) The ANVCE near term SES is a cost effective design, inhabited and operated by sailors, which provides superior performance, seaworthiness, and survivability in high sea states. The design philosophy is manifest in the ship's performance and subsystems design.
- (U) The SES meets or betters Top Level Requirements for speed, range, and hump margin in all sea states at a full load displacement of 3,000 LT (29,892.1 kN). The available range margin can be traded off against producibility, weight or increased payload. It betters all requirements for turns, Translation and rotation maneuvers are easily made at zero and low forward speeds for docking, harbor operations, and certain tactical situations. It comes to a full stop from maximum speed in 1000 yards (914.4 m).
- (U) The ride quality is much better than required for crew comfort and performance of precision tasks. The superior ride quality is maintained over the entire operational envelope and has been proven at sea. A destroyer (DD-963) cruising at 10 knots, sea state 5, meets the established 4 hour limits. The near term point design SES operating at 60 knots, sea state 5, easily meets and can exceed the same 4 hour ride criteria.
- (U) The design is inherently stable. It is safely operable well beyond tha limits of the operational requirements. It is **functional** in sea state 6.

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- (U) It is designed to survive hurricane conditions. Extensive digital computer simulation and 3,800 hours of tow tank testing confirmed the design as stable and safe.
- (U) It is a habitable and highly maintainable ship due to careful attention given to functional space arrangements and by designing the ship with 9 foot (2.74 m) deck heights to assure adequate head room in all spaces where activity is required. Duty stations and living spaces are located away from noise and vibration producing machinery. All living spaces and messing areas are located for best ride quality and with least noise.
- (U) The lift and ride control system is unique and effective. It is a proven system. The ride control system (RCS) attenuates vertical motions to levels within ride criteria limits.
- (U) It utilizes an advanced planing seal concept which easily meets the trans-oceanic requirements of long life and high reliability. The seals are a marked advance in the state-of-the-art.
- (U) The propulsion system is designed for operational use. It is a simple, proven system sized for growth. It is a symmetrical system port and starboard that is easily aligned and maintained,
- (U) The near term point design SES incorporates an integrated ship control system which enables five (5) men to operate the ship in complete safety. It is designed for centralized operation, operational simplicity, full exploitation of the SES potential, and fail safe operation. Reliability and safety are fully integrated into the design.

(U) A.4 TRADE-OFF STUDIES

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Many design variations were considered during the development of the near term SES point design. These required various trade-off studies in the general areas of ship configuration, subsystems, and performance.

- (U) A.4.1 CONFIGURATION TRADE-OFFS -- The near term SES is a full langth sidehull ship with an effective length-to-beam (L/B) ratio of 2.60. The choice of full length sidehulls over partial length sidehulls was the result of trade studies that included consideration of parameters such as drag, static and dynamics stability, sea worthiness, seal design, maneuverability and structural weight fraction.
- (U) The selected seals design resulted from trade-offs that considered the application of a two-dimensional, planing type seal, or a bag-and-finger type seal. Factors evaluated in the definition of the seal baseline included design simplicity, durability, response characteristics, high speed drag, performance and off-cushion drag penalties.
- (U) Lateral directional stability at high yaw angles is provided by fixed ventral fins. The specification of these devices and their related fences are the results of trade-off studies considering various geometries and evaluating their drag, waterjet inlet broaching, and maneuvering performance.
- (U) The configuration also includes semi-flush waterjet inlets and related ventilation cutouts. T; 3 location and geometry of the inlets and ventilation cutouts are the result of trade studies involving drag, weight, propulsion efficiency, and machinery location considerations.
- (U) A number of trade-offs were made to determine the impacts of variation in bulkhead spacing, frame and stiffner spacing and number c decks within the hull. The considerations were optimization of the structural weight fraction while providing sufficient enclosed volume to accommodate the required ship company, machinery fuel, and specified military payload.

(U) A.4.2 **KEY** SUBSYSTEMS TRADEOFFS

- (U) A.4.2.1 Main Propulsion System -- Trade-offs for the propulsion machinery subsystem emphasized criteria which resulted in a design that provides optimum performance, low development risk, minimum complexity, high reliability, maximum protection from environmental elements, good habitability and replacement of most major components without drydock of the ship. The primary tradeoff was between waterjet propulsors and partially submerged, supercavitating propellers, Waterjets were chosen because the produce much lower noise and vibration levels, are less susceptible to damage by floating debris, have less complex transmission systems, can be maintained without drydock (except for some elements of the waterjet inlet), and can be acquired at lower cost and with less developmental risk.
- (U) The propulsion system utilizes four LM2500 gas turbines identical to those in service on the DD963 ships, The LM2500 engine has low fuel consumption, adequate power, long life and high reliability. However, all components of the propulsion subsystem are sized to accommodate the higher rated FT9A engines, with but minor modifications.
- (U) Other major tradeoffs were in the propulsion machinery arrangement, combustion air system, and waterjet inlet. All propulsion components, except the waterjet inlet, are located above the wet deck to obtain good maintainability and minimize complexity. Use of seemingly available space in the sidehulls resulted in poor installations with disadvantages outweighing the marginal advantages in performance, Similarly, the combustion air system was generously sized to minimize engine power losses and maximize accessability, salt removal, and noise suppression.
- (U) The selected waterjet inlet is a flush, variable roof arrangement that provides superior cavitation and recovery performance, simplicity and low drag.

- (U) A.4.2.2 Lift System -- An intensive parametric tradeoff study of both axial, mixed flow and centrifugal fans resulted in the selection of dual inlet, single discharge, constant velocity volute, centrifugal fans because of their low weight, compact geometry, and favorable performance properties. A further tradeoff resulted in consideration and rejection of two circulation control designs when compared to the fan concept. Circulation control was found relatively complex and not as advanced as the technology for fans; a proven 1/4 scale fan model was in operational use. The selection of the lift prime mover was based upon the fan power requirements which matched the proven LM2500 with no other GT available in the power range for comparison.
- (U) Trade-offs of various fan locations and their attendant shafting and ducting complexity were performed. The result was a design featuring simple inline shafting, minimum ducting length and minimum use of duct elbows. The inline shafting employs proven marine helical gear sets (single reduction) over more complex planetary gears.
- (U) Ducting trade-offs are closely related to those for the power transmission, With fan locations and air delivery points established, further trade-offs determined the minimum weight ducting configuration, with no common plenum or duct:plenum, no duct air spliting, and use of round ducts. The fair-weather intake design resulted from trade-off studies concerning free stream pressure recovery, noise control, fabrication techniques, base drag, weight, and water ingestion.

- (U) The location of the ride control ducting and valving was determined through trade-offs involving ship's available space, thrust augmentation, and weight impact.
- (U) Major trade-offs for the seals were in seal geometry, modularization, and selection of detail hardware and materials. The seals geometry trade-offs compared planing seals with bag-and-finger seals. The planing seal geometries included both two- and three-dimensional (curved bow planform)

seals in concert with the square bow/full length sidehull tradeoff.

- (U) The planing seal was selected for its demonstrated lower drag forces, improved wear resistance and the durability of glass reinforced plastic planar elements. The two-dimensional plan 3 bow seal was selected along with the square bow/full length sidehull because together they offered a more simplified seal design, modularization of components, and improved seal maintainability and reliability.
- (U) Modularization trade-offs were performed to optimize seal maintenance, to minimize loads and, to assure high performance in a seaway. Components included were number and type of restraints (straps and cables) and quantity of planers and bag modules,
- (U) Significant hardware and seal mater trade-offs included comparisons of (1) straps and cables, (2) mater s for planers, pressure bags, restraints, attachments, and modular oints and (3) planer-to-strap transition attachments. Key criteria in these trade-offs were weight, reliability, maintainability, ease of fabrication, and methods of design verification.
- (U) A.4.2.3 Trade-offs optimized the electrical power generation and conversion subsystem design. The weight was reduced by almost 50% by increasing the use of 400 Hz power. **The** power requirements were adjusted through judicious **selection** of user equipment so that 400 Hz and **60Hz** power **consumptions** were equal. The weight savings resulted from the extensive use of 400 Hz generators and motors, which weigh less than one-eighth as much as their 60 Hz counterparts.
- (U) Direct generation of 400 Hz power by generators powered by aircraft-derivative turbines (in lieu of 60 to 400 Hz converters) was a principal factor in this accomplishment. While impressive weight savings at reasonable dollar cost were made, further conversion to 400 Hz usage

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would result in sharply increased costs, owing to the need for special equipment development.

- (U) Auxiliaries -- Weight trade-offs were made of 12 air conditioning systems and equipment items as a result of this study. A decentralized system was selected. This system divides the load into smaller serviced areas, each using packaged air conditioning units.
- (U) Single centralized and multiple dedicated lube systems were analyzed on the basis of weight, cross contamination, cooling requirements, length of lines, bulkhead penetrations, reliability, and redundancy. A multiple dedicated system was selected.
- (U) The trade-offs for the potable and fresh water systems involved investigation of components and configurations possessing potential weight savings. Vacuum-assisted water closets and low water demand showers were selected. Weight was reduced through reduced quantities of collected and stored water via the drainage system and the reduced pumping capacity requirement. Further weight reduction was obtained by selecting GRP piping.
- (U) A trade-off study was made to provide the design criteria and rationale for the most advantageous total flooding extinguishing agent. ${\tt CO}^2$ and ${\tt Halon}$ 1301 extinguishing systems were compared, and a ${\tt Halon}$ 1301 system was preferred over a ${\tt CO}_2$ system for its lower weight and shorter discharge time.
- (U) Hydraulically-powered actuators, motors, and **pumps** were compared to electrical and pneumatic equipment on the basis of weight, cost, compatibility installation **requirements** and operating environment. Trade-off comparisons indicated a weight saving of several tons by employing hydraulically-powered equipment. In some instances, the electric motor-driven actuators appeared so bulky and cumbersome as to be essentially impractical. In the case of high-per-

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- (U) formance servo-driven devices such as the ride control valves, low inertia servo-motors with power ratings not readily obtainable would be required.
- (U) Weight trade-offs for marine sanitation on the basis of a one-day operational period disclosed weight savings by utilizing a no-discharge type compared to a flow-through type.
- (U) A.4.2.5 Outfittings and Furnishings -- Trade-offs were made for the insulation and protection (fire, thermal and acoustical) of the aluminum structure. A rigid panel placed outside the frames was compared to a flexible blanket placed against the structure. The rigid panel design was selected because of:
 - Ease of installation
 - Reusability of panels after removal for inspection of structure
 - Ease of modular panel replacement
 - Elimination of separate sheathing and false ceilings
 - Resistance to deterioration during normal shipboard use
 - Efficient thermal protection of structure through utilization of an air gap between the panel and structure and a reflective surface facing the fire threat and
 - ullet Elimination of insulating against fire for the cabling and piping systems.
- (U) Contrariwise, the advantages of flexible blanket design are lower cost, slightly lower weight, increased space, and elimination of the hazard of fire penetration behind the insulation panel. However, the develop-

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ment of an effective and practical seal for panel joints to prevent fire penetration offset the advantages of flexible blanket design.

- (U) The large amount of insulated and sheathed cabling and piping external to the flexible blanket design, coupled with the relatively close frame spacing of 3 feet, further minimized the increased space advantage of the flexible blanket design and imposed a weight and cost disadvantage.
- (U) **A.4.3** PERFORMANCE TRADEOFFS -- Maximum performance of the selected design configuration was optimized with respect to:
 - 1) Speed (at minimum drag) with maximum continuous power.
 - 2) Thrust margin at hump speed with maximum intermittent power.
 - 3) Range.
 - (U) Optimization of each of these performance factors involved selecting a best operating policy (i.e., the determination of operating trim and draft), lift system airflow settings, and seal adjustments within the adjustment latitude and constraints of the design. While this selection could be an n-dimensional optimization process of great complexity, only a limited number of major effects need be considered in practice. The key trade-offs are:
 - 1) Trim and Draft for Least Drag Ship operating attitude for minimum drag is determined by comparing tank test data with analytically-derived relationships. The resulting policy is checked against system constraints to assure that the desired attitudes can be achieved with the available adjustments.

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- 2) <u>Lift System Optimization</u> Airflow distribution, pressure ratios and seal settings are optimized with empirical data in conjunction with analytical characterizations of the lift system. Policies for least drag and least total fuel rate are developed,
- 3) Optimum Cruise Speed There is a speed at which range is maximized for each vehicle weight between zero and 100 percent fuel. This speed is found recursively by a performance computer program that includes appropriate representations of drag, lift system and propulsion system characteristics.

APPENDIX B: DRAWINGS AND **DIAGRAMS**

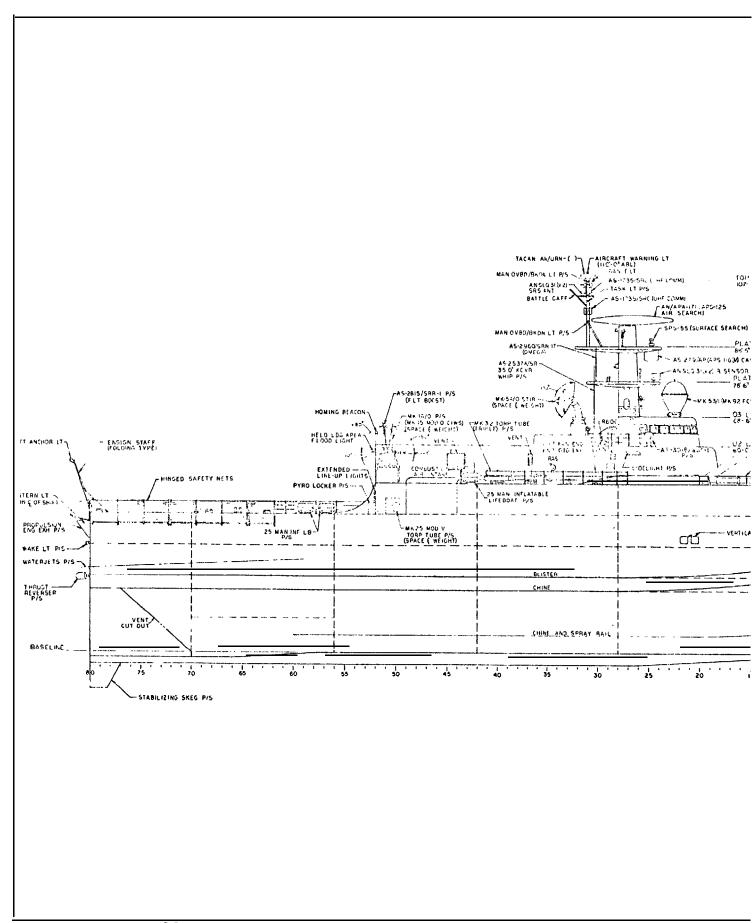
(U) B.1 GENERAL ARRANGEMENT DRAWINGS

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(U) This section of Appendix B contains the general arrangement drawings for the near term ANVCE SES Point Design. These drawings are as follows:

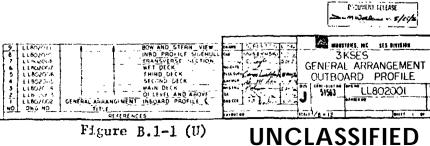
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B.1-2	Inboard Profile	LL802002
B.1-3	01 Level and Above	LL802003
B.1-4	Main Deck	LL802004
B.1-5	Second Deck	LL802005
B.1-6	Third Deck	LL802006
B.1-7	Wet Deck	LL802007
B.1-8	Traverse Section	LL802008
B.1-9	Inboard Profile	LL802010
B.1-10	Bow and Stern Views	LL802011
B.1-11	Tank Arrangement	LL801001
	and Tank Capacities	



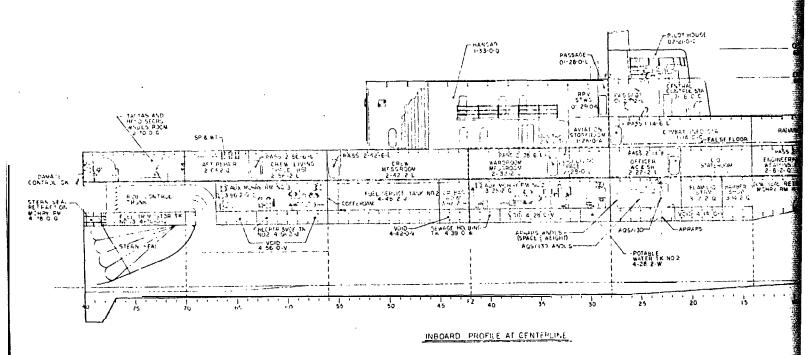


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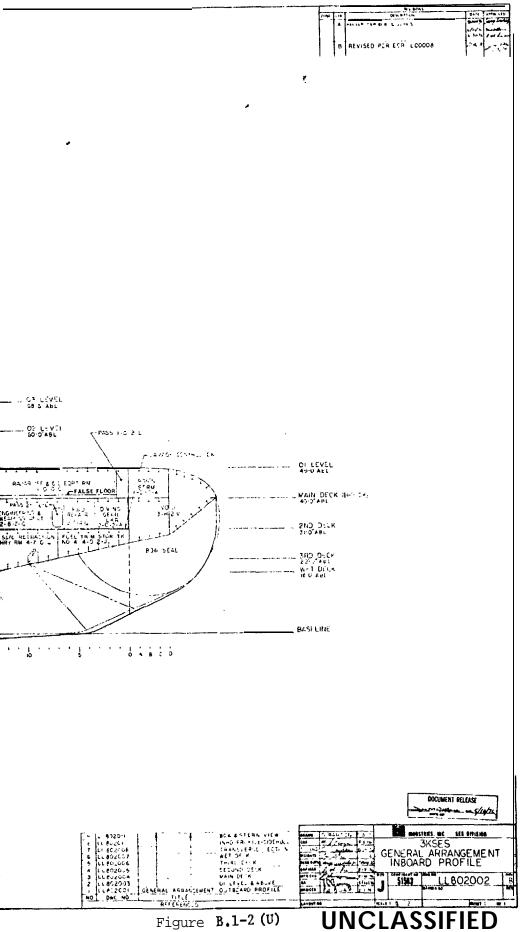
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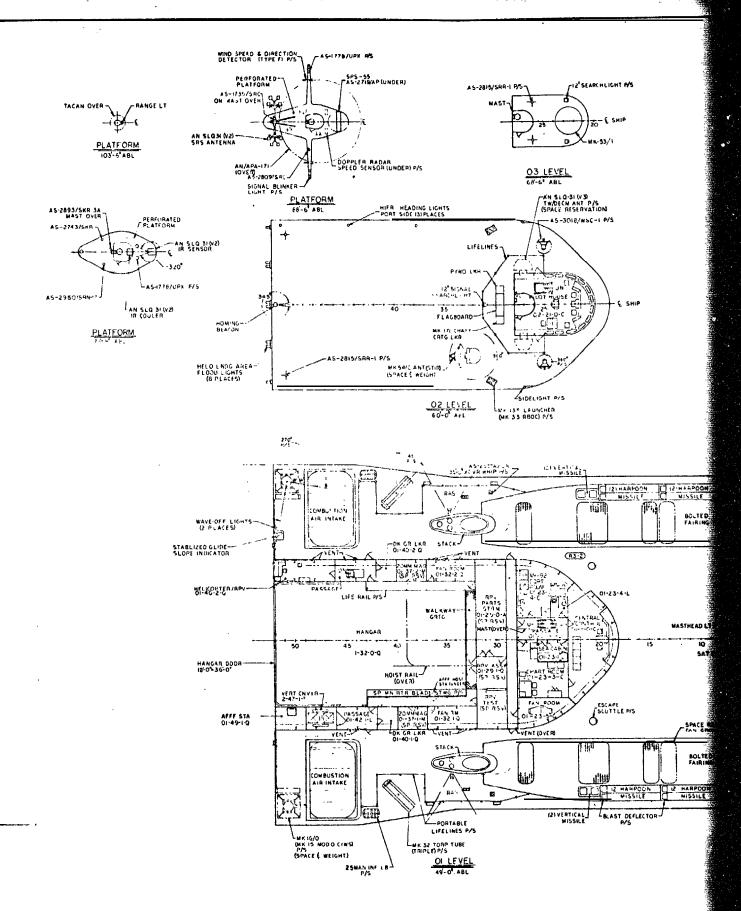


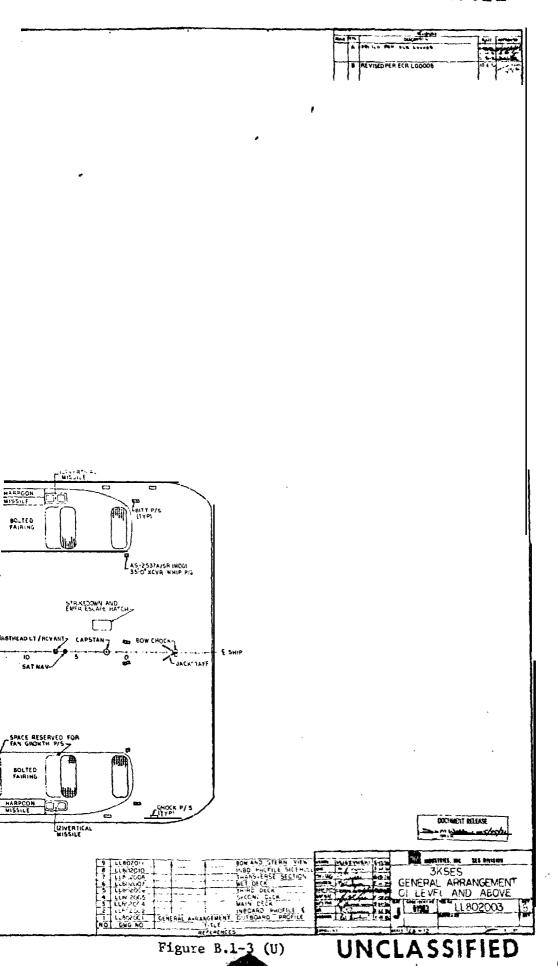




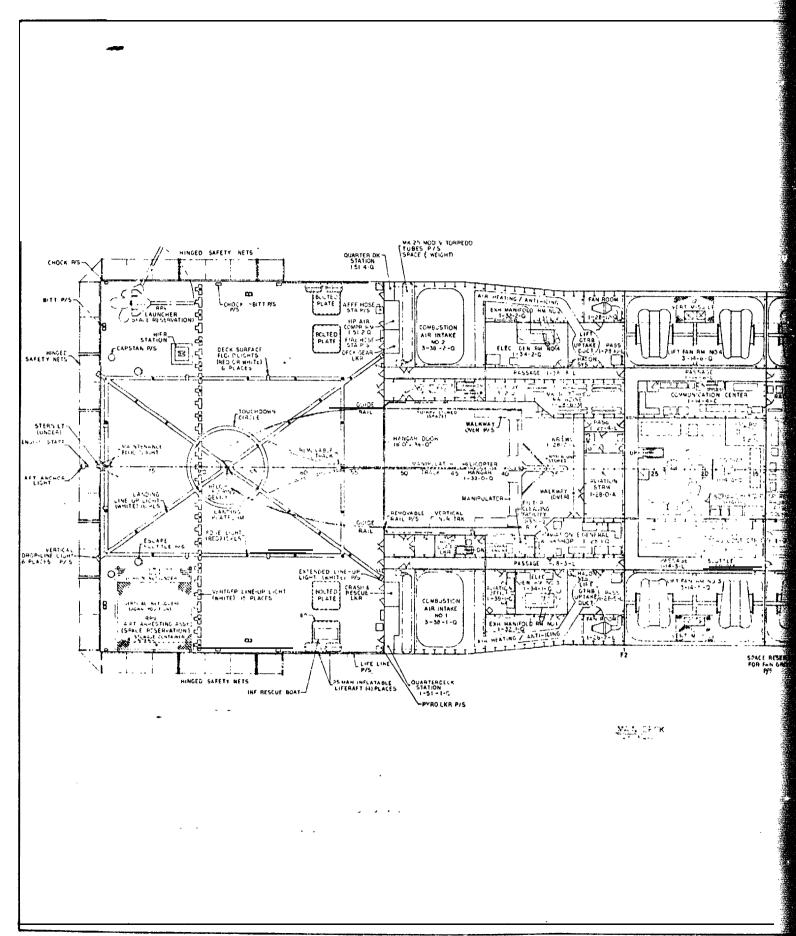
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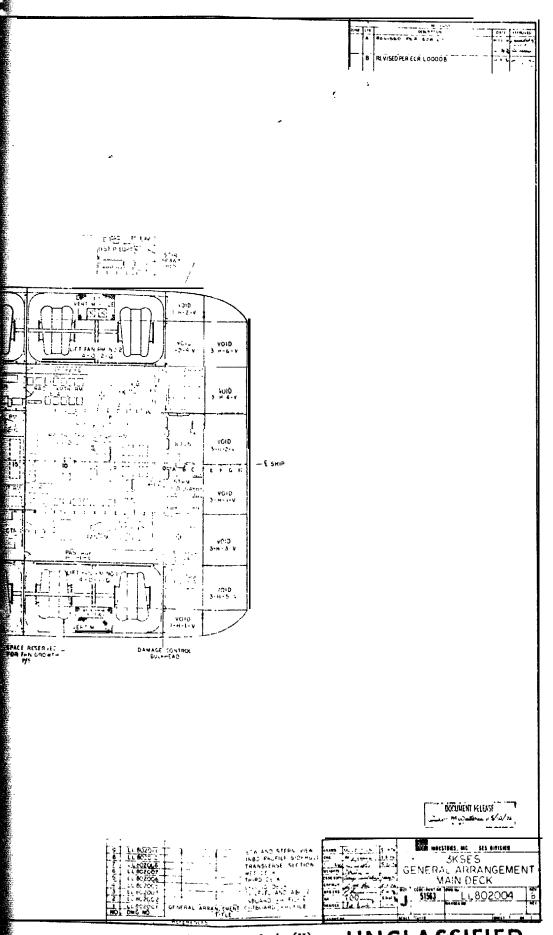
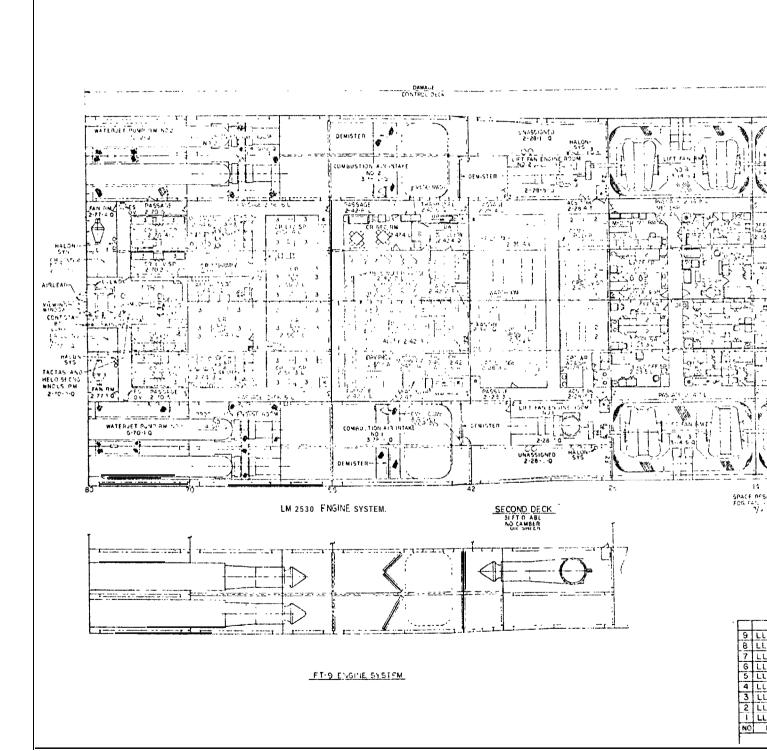
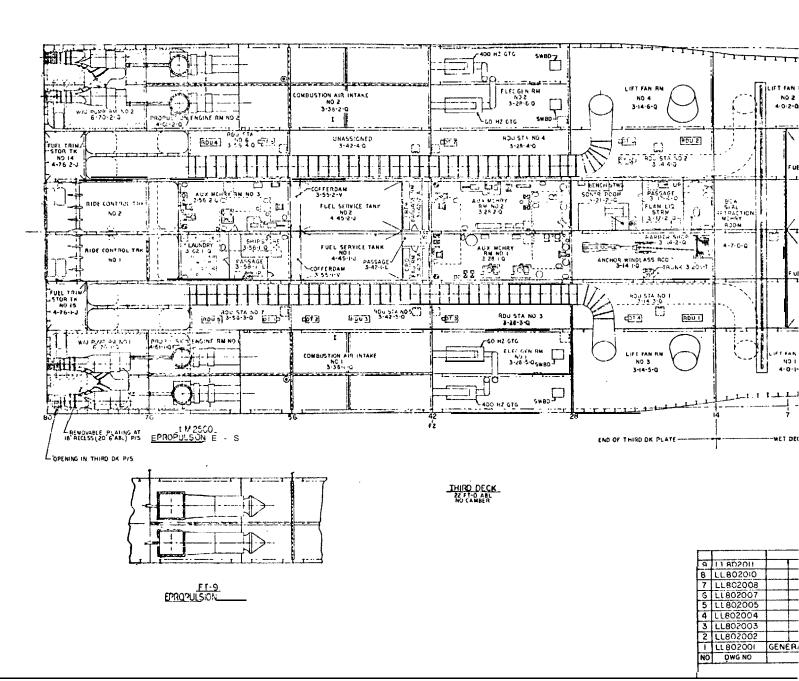


Figure B.1-4 (U)

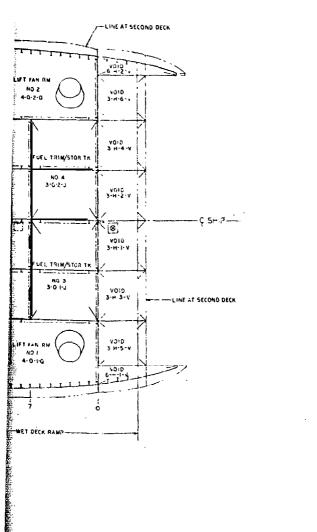


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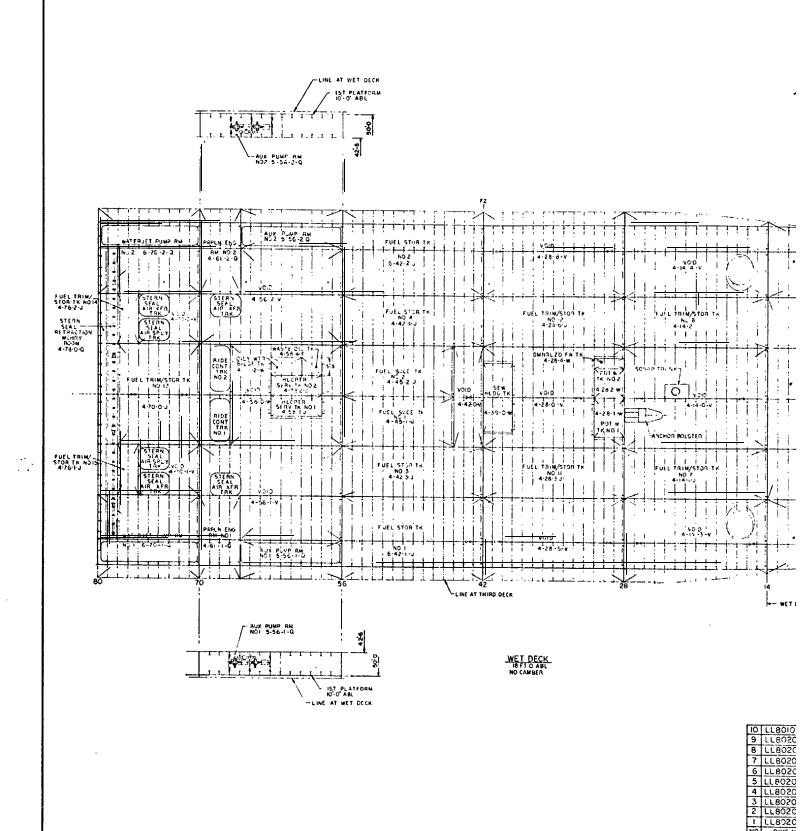




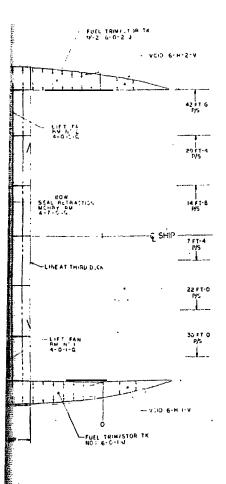


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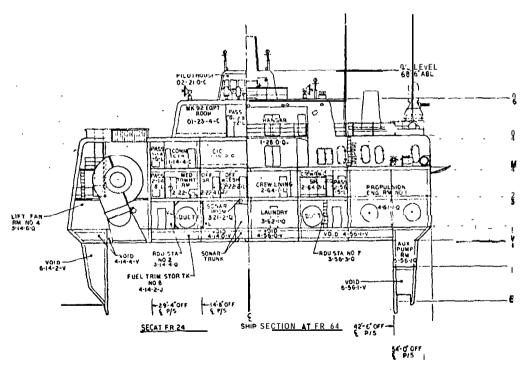
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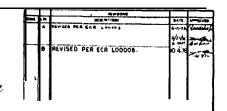
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TRANSVERSE SECTION



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IRD DECK 12-0" ABL VET DECK 6-0" ABL

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ASELINE

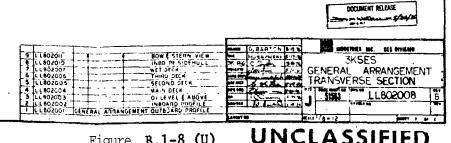
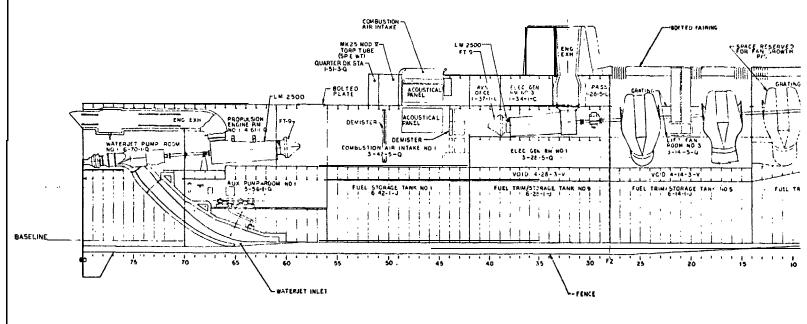


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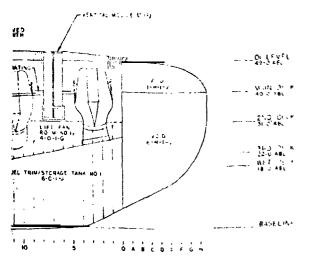


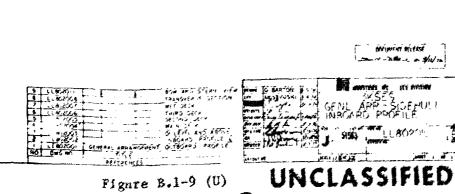
SIDEHULL INBOARD PROFIL STARBOARD SIDE LOOKING TO PORT.

BELOW SRD DK SECTION PLANE IS APPROXIMATELY 44-0" OFF & SHIP ABOVE SRD ON SECTION PLANE VAR ACCORDING TO EQUIPMENT SHOWN

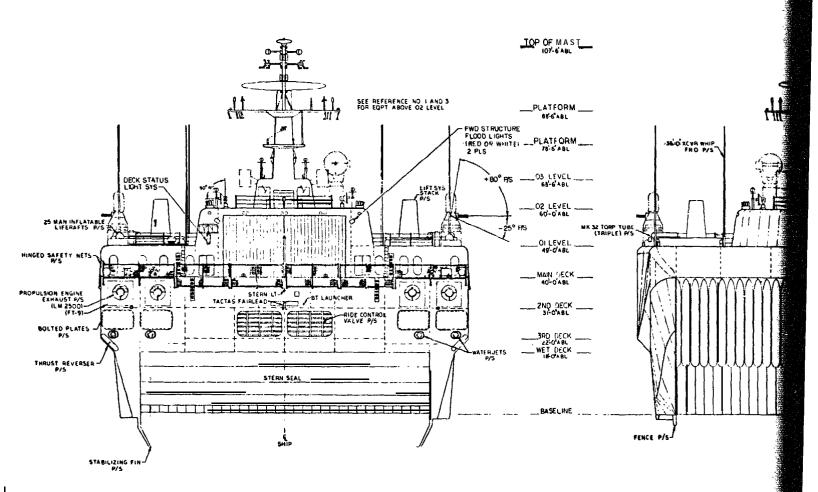




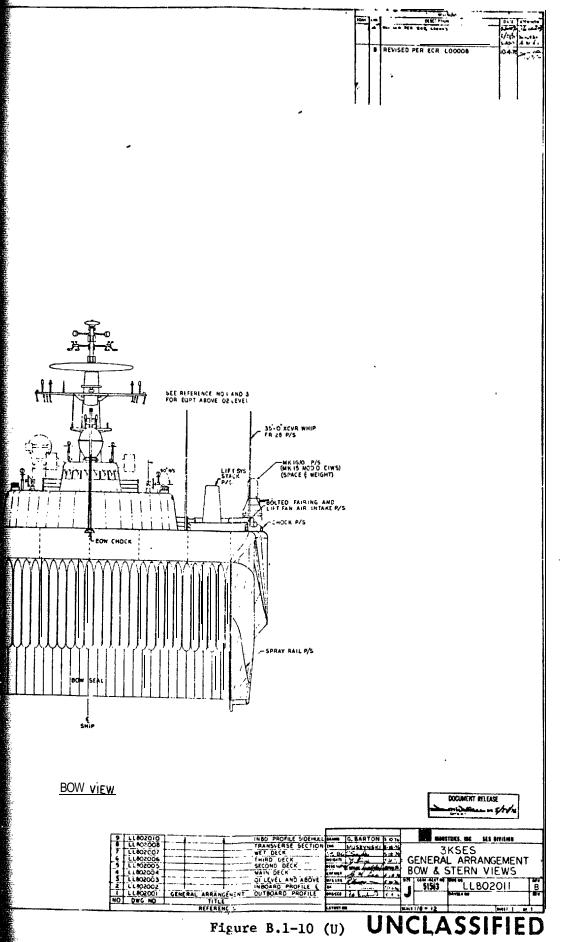




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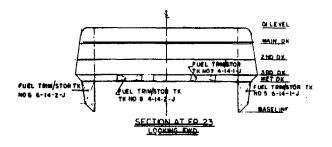


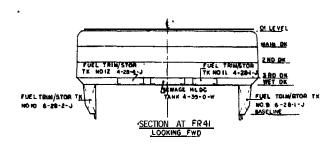
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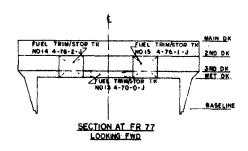
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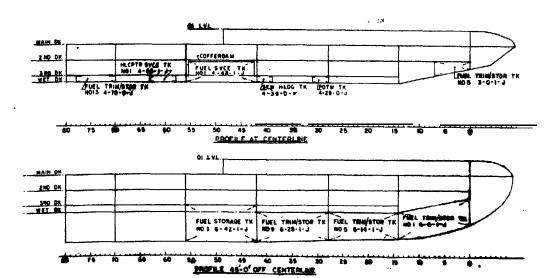
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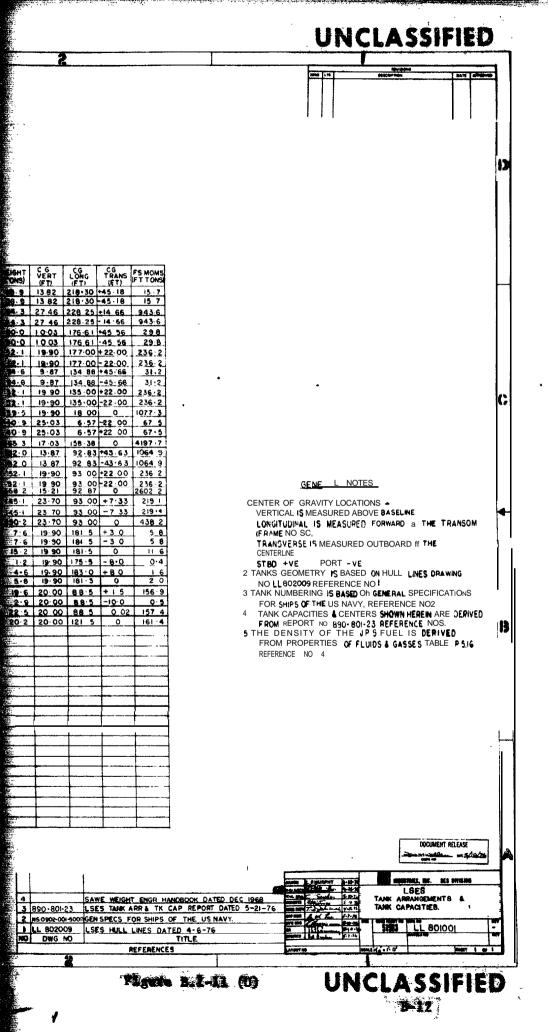
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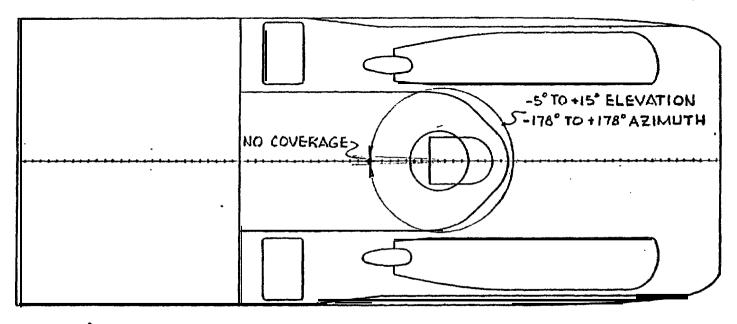
- (U) B.2 WEAPON AND SENSOR COVERAGE DIAGRAMS
- (U) This section of Appendix B contains the weapon and sensor coverage diagrams for the ANVCE near term SES Point Design. These diagrams are:

<u>Figure</u>	<u>Title</u>
B.2-1.	APS-125 Air Surveillance Coverage is Unobstructed
B. 2-2	Elevation Coverage from the High APA-171 Antenna Position Extends the Radar Horizon for "Pop-Up" Threat Detections
B.2-3	Surface Search Coverage with the AN/SPS-55 is Extensive for Maneuvering, Piloting, and SUW
B. 2-4	Elevation Coverage for Surface Search is not Obstructed Along Critical Azimuth Bearings
B. 2-5	Collision Avoidance Coverage is Unobstructed for Critical Sectors and Ship Maneuvering Options
B. 2-6	Elevation Coverage is Unobstructed for Debris Detection Close to the Ship's Bow, Dead Ahead
B.2-7	Fire Control System Surveillance and CW Illumination Coverage is Extensive
B.2-8	Elevation Coverage Provides Full Capability for AAW and ASMD Fire Control
B.2-9	The Mk 54 Mod 0 Antenna Site Augments Mk 92 FCS Coverage
B. 2-10	Full STIR Elevation Coverage for Sea Skimmer and Zenith Threats
B.2-11	The CIWS Weapons Groups Provide Complete 360-Degree ASMD Azimuth <i>Coverage</i>
B. 2-12	Full CIWS Weapons Group Elevation Limits are Only Reduced for a Small Sector Dead Ahead

- (U) B.2-13 The Forward Location of the IR Sensor in the EW Suite Covers Critical Threat Approach Corridors
 - **B.2-14** The Forward IR Sensor has an Elevation Coverage Providing an Unrestricted Field-of-View
- (U) These diagrams each have descriptive titles that emphasize the features of the weapon and sensor coverage inherent in the near term SES.

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AN/APA-171 (APS-125)



356° COVERAGE

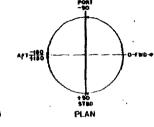
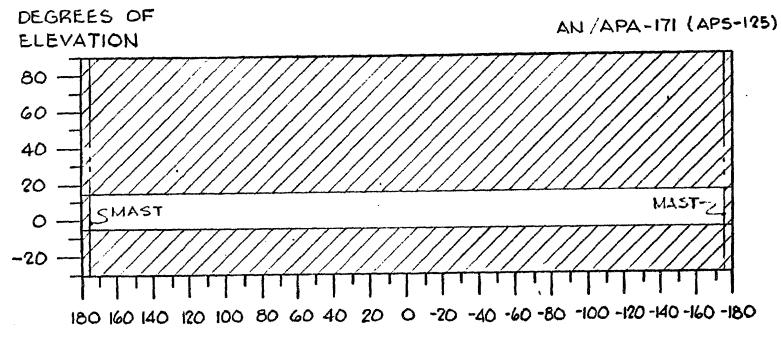


Figure B.2-1.(U): APS-125 Air Surveillance Coverage is Unobstructed (U)



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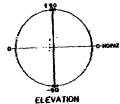
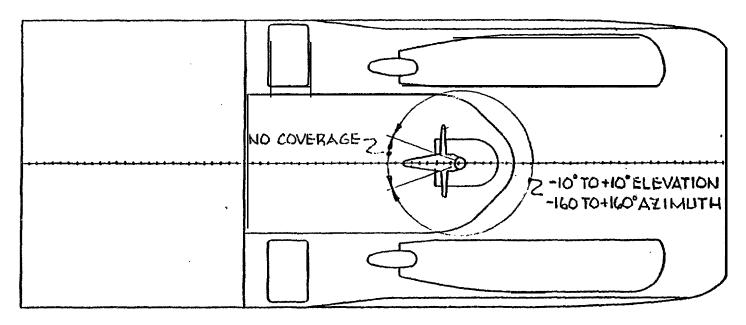


Figure B.2-2 (U): Elevation Coverage from the High APA-171 Antenna Position Extends the Radar Horizon for "Pop-Up" Threat Detections (U)





320° COVERAGE

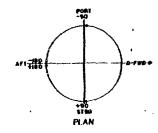


Figure B.2-3 U): Surface Search Coverage with the AN/SPS-55 is Extensive for Maneuvering, Piloting and SUW (U)

ELEVATION

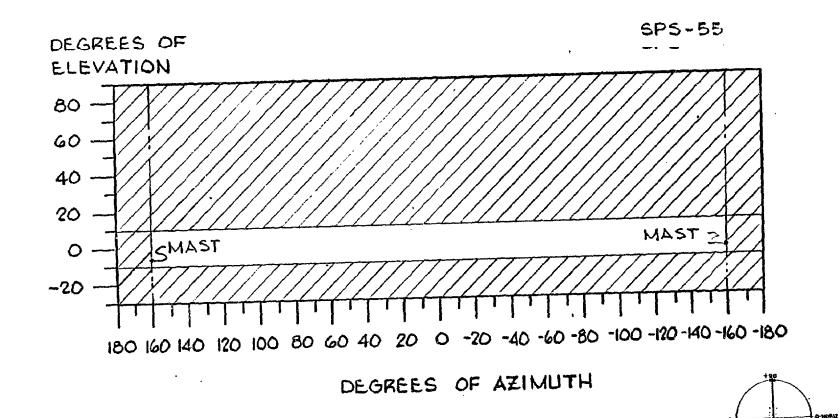


Figure B.2-4 (U): **Elevation** Coverage for Surface Search is not (U)

AS-279/AP (APS-116(M) CAS)

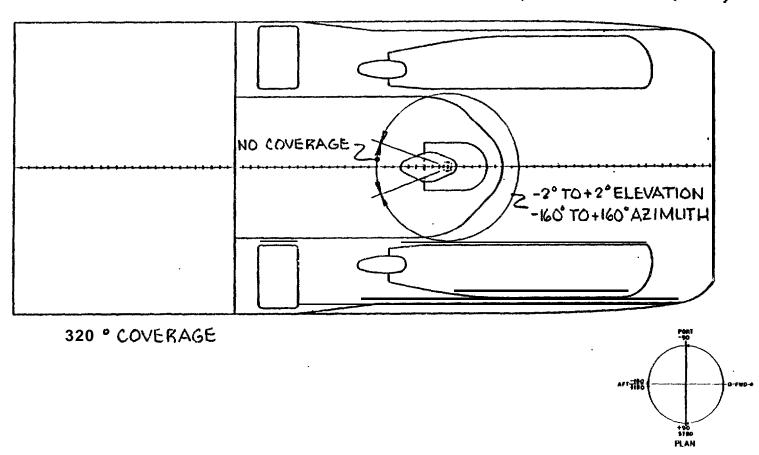


Figure 3.2-5 (U): Collision Avoidance Coverage is Unobstructed for Critical Sectors and Ship Maneuvering Options(U)

ELEVATION

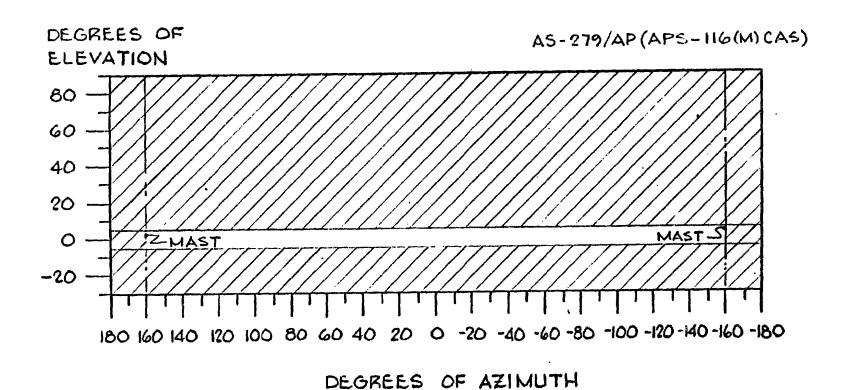
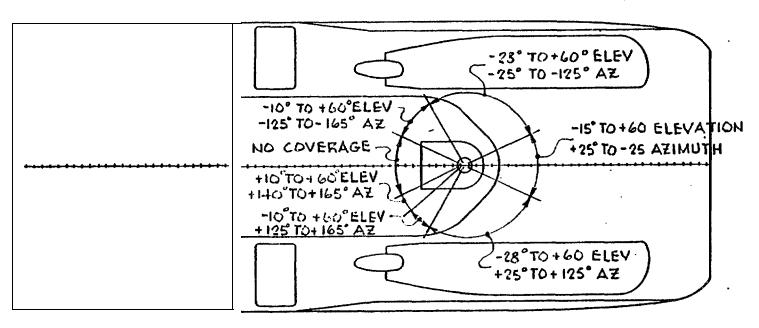


Figure B.2-6 (U): Elevation Coverage is Unobstructed for Debris
Detection Close to the Ship's Bow, Dead Ahead (U)

MK-53/1 (MK 92 FCS)



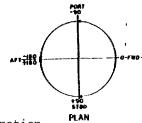


Figure B.2-7 (U): Fire Control System Surveillance and CW Illumination Coverage is Extensive (U)

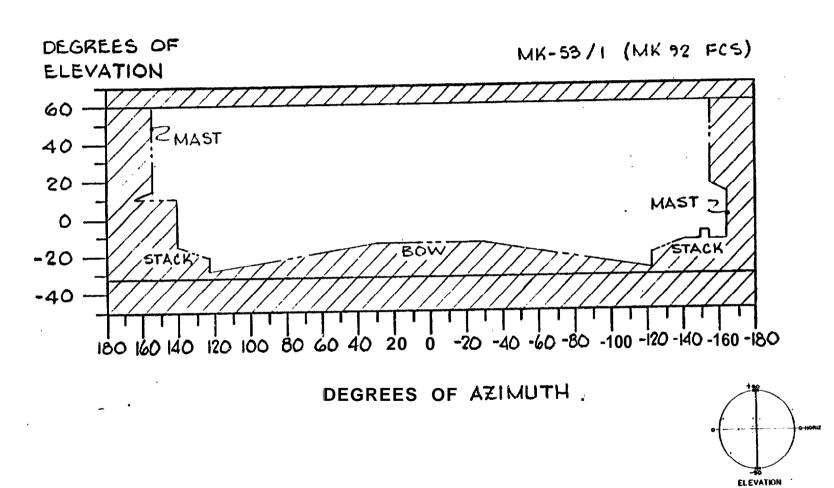


Figure B.2-8 (U): Elevation Coverage Provides Full Capability for AAW and ASMD Fire Control(U)

MK 54/0 (STIR)

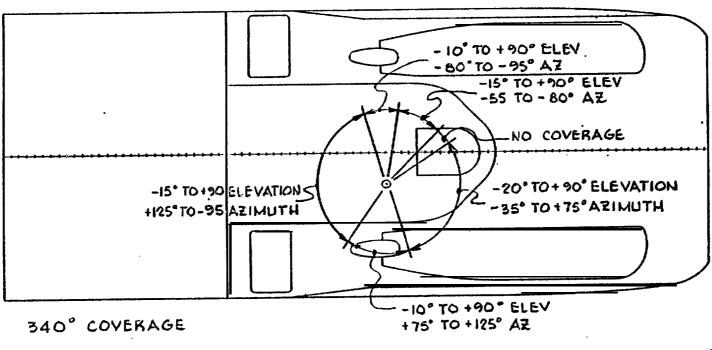
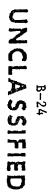


Figure B.2-9 (U): The MK 54 MOD 0 Antenna Site Augments MK92 FCS Coverage (U)



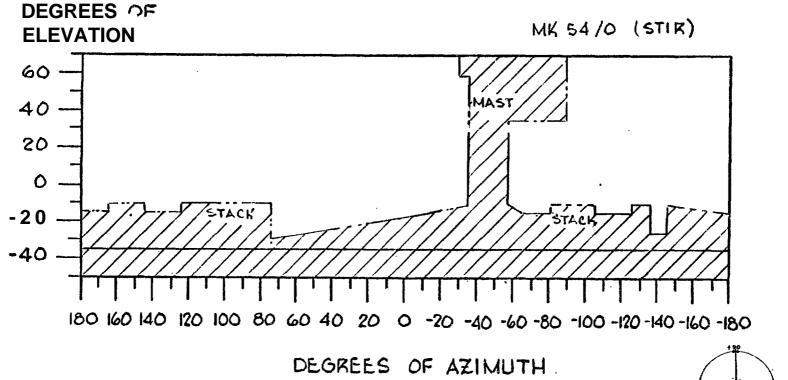


Figure B.2-10 (U): Full STIR Elevation Coverage for Sea Skimmer and Zenith Threats (U)

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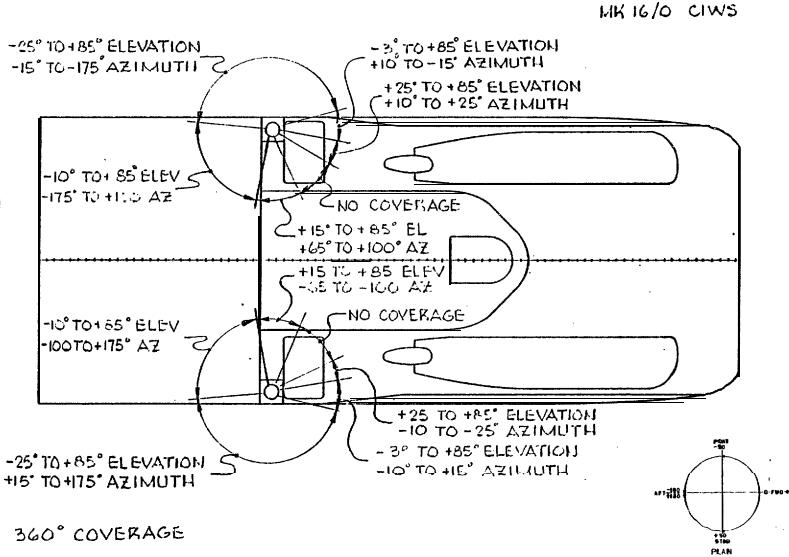
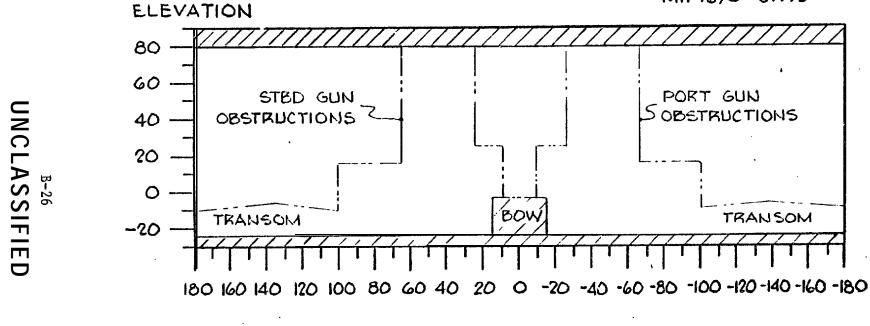
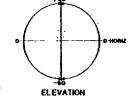


Figure B-2-11 (U): The CIWS Weapons Groups Provide Complete 360-Degree ASMD Azimuth Coverage (U)



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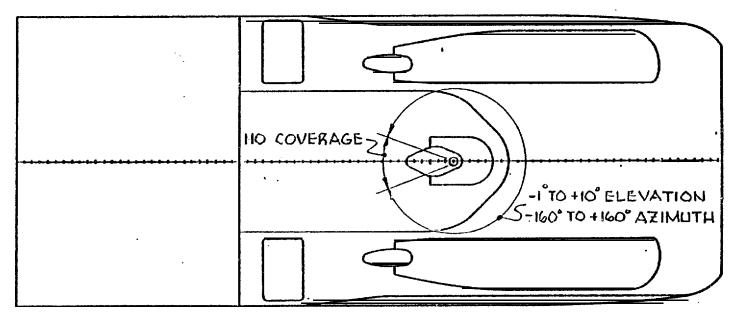


MK 16/0 CIWS

Figure B.2-12 (U): Full CIWS Weapons Group Elevation Limits are Only Reduced in Depression for a Small Sector Dead Ahead (U)

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IR SENSOR AN/SLQ-31 (V2)



320" COVERAGE

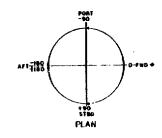
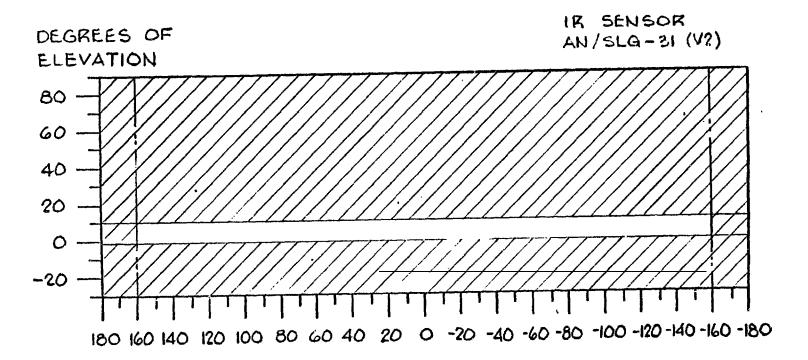


Figure **B.2-13 (U):** The Forward Location of the IR Sensor in the EW Suite Covers Critical Threat Approach Corridors (U)



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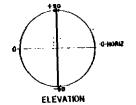


Figure B.2-14 (U): The Forward IR Sensor has an-Elevation Coverage Providing an Unrestricted Sensor Field of View (U)

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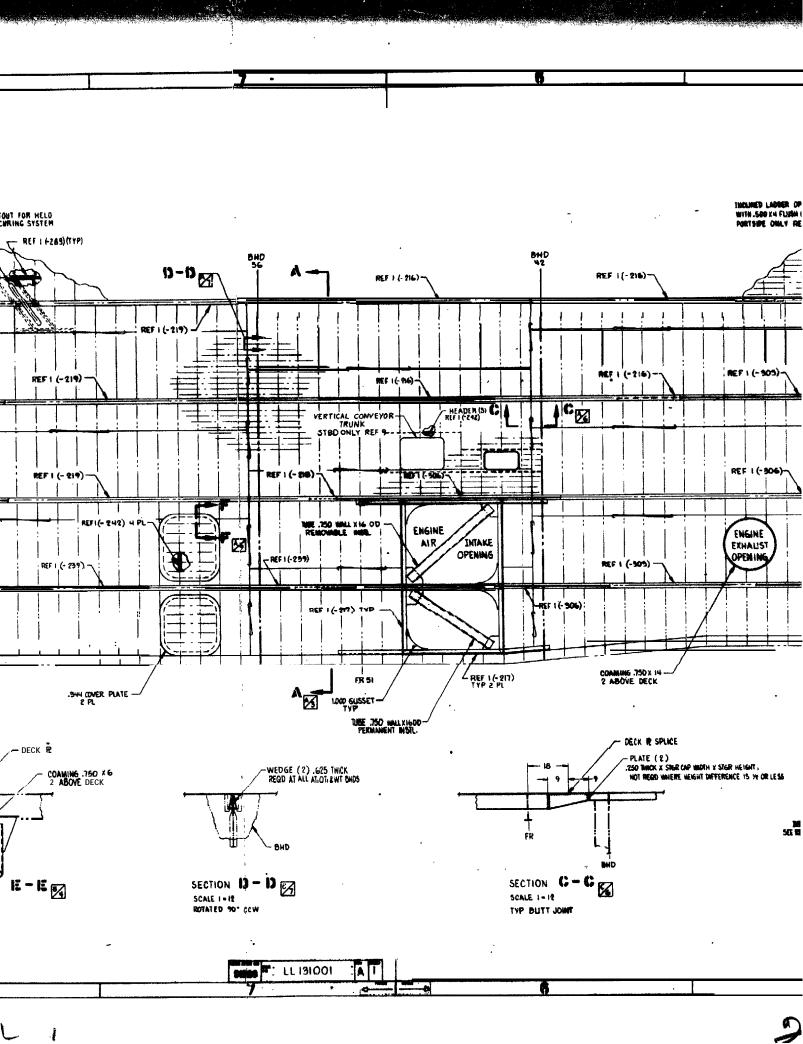
- (U) B.3 STRUCTURE **DRAWINGS**
- (U) This section of Appendix B contains the structure drawings for the ANVCE near term SES Point Design. These drawings are:

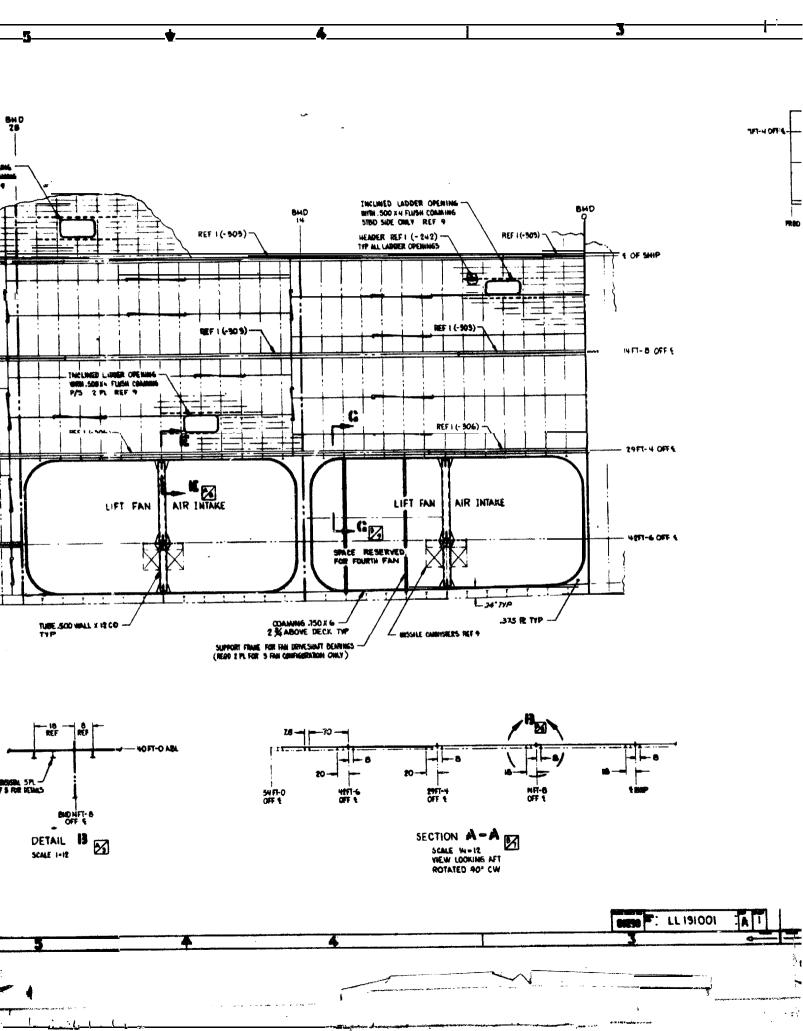
<u>Figure</u>	<u>Title</u>
B.3-1	Deck Plating - Main Deck
B.3-2	Bulkhead - Long, CL
B.3-3	Transverse Bulkheads
B.3-4	Transverse Frame
B.3-5	Bow Plating and Framing
B.3-6	Superstructure
B.3-7	Extrusions - Structural
B.3-8	Tabulation - Plating/Tee

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PLATING/STRINGER DIAGRAM FOR DETAILS OF R & STER SEE REF 2

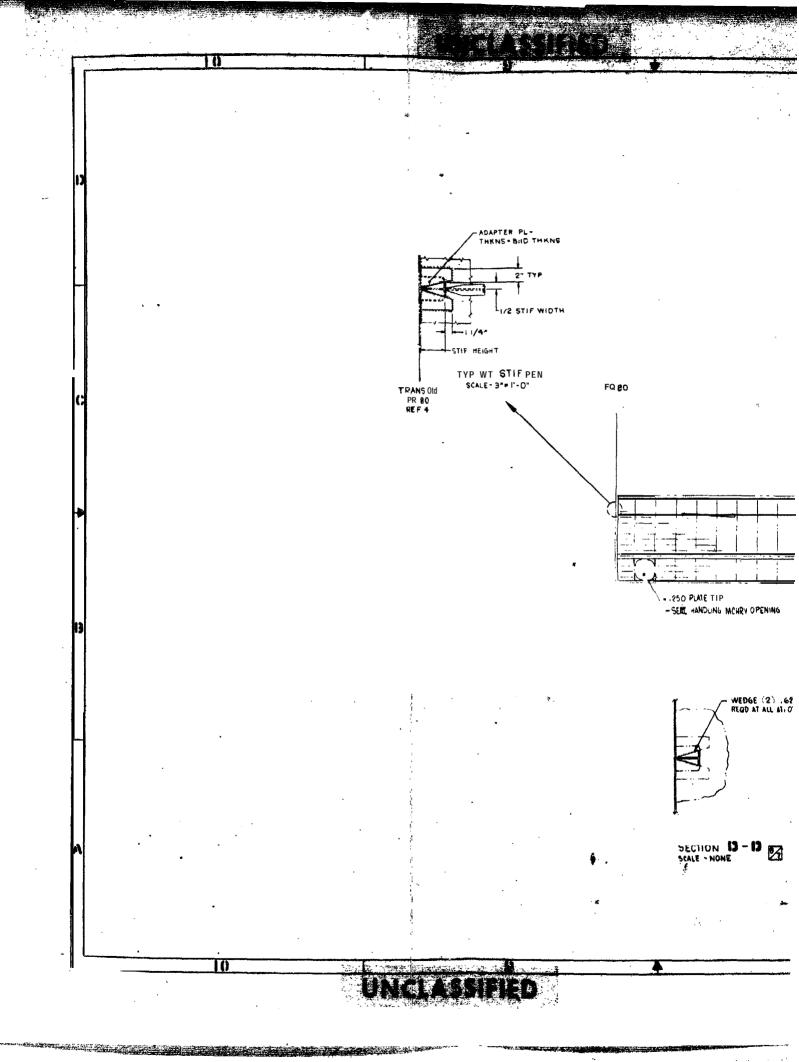
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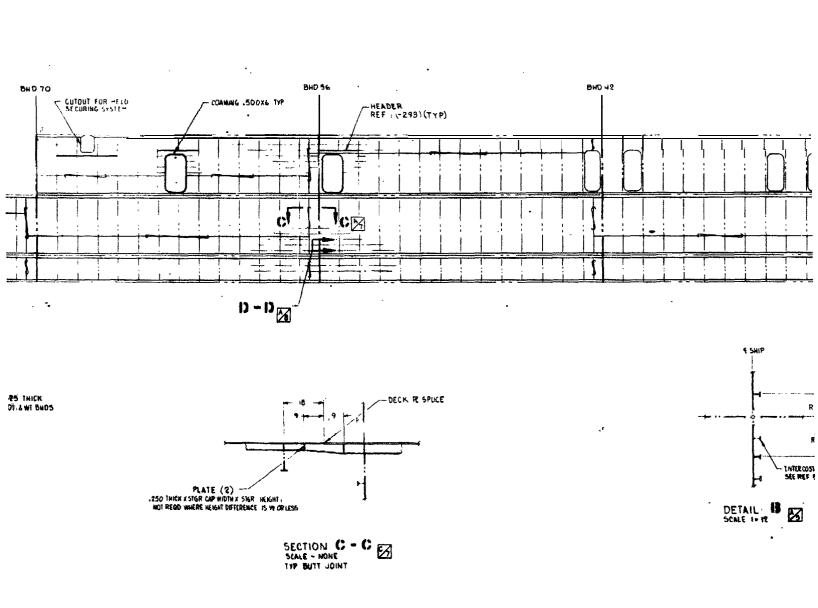
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- 9. FOR LOCATION PROPORTION AND CLOSURE REQUIREMENTS OF ALL OPENINGS SEE REF 9
- & FILLET WELDS AROUND ENDS OF WELDED MEMBERS TO FORM CLOSED LOOP.
- I ABBREVIATIONS PER MIL-STD-12.
- 6. SEE DIAGRAM (BONE D 2) FOR PLATING AND STRINGER COMBINATIONS.
- S. STARBOARD SIDE SHOWN PORT SIDE OPPOSITE EXCEPT AS NOTED.
- 4 FRAME SPACING 36 & EXCEPT AS NOTED.
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- 2. FABRICATE WELD AND INSPECT PER NAVSHIPS 0900 - 060- 4010.(SEE 2K SES PROGRAM SHEETIVES 500.12, 500.16, 500.19, 500.20)
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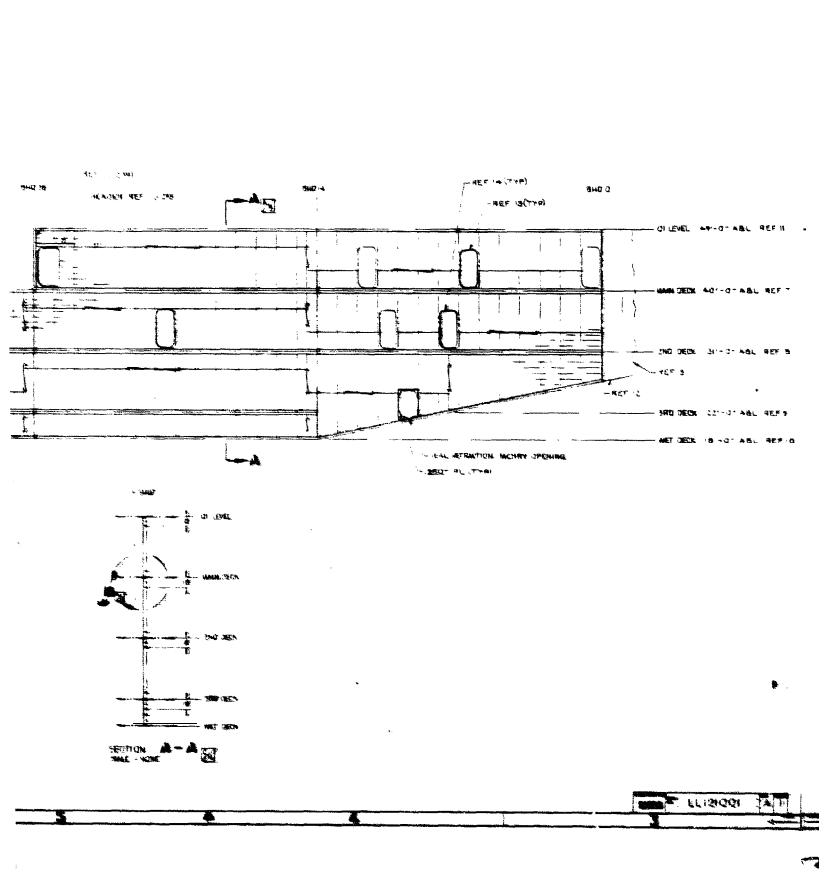
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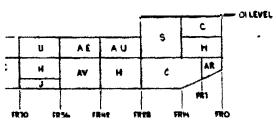


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- 9. FOR LOCATION PROPORTION AND CLOSURE REQUIREMENTS OF ALL OPENINGS SEE REF 138M
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- 1. ABBREVIATIONS PER MIL-STD-12.
- G. SEE DIAGRAM (ZONE, DZ.) FOR PLATING AND STRINGER COMBINATIONS.
- 5. STARBOARD SIDE SHOWN PORT SIDE OPPOSITE EXCEPT AS NOTED.
- 4. FRAME SPACING 56M EXCEPT AS NOTED.
- S. GENERAL STIFFENER SPACING ID . EXCEPT AS NOTED.
- 2. FABRICATE WELD AND INSPECT PER MAYSHIPS 0900-060-4010. (SEE '2KSES PROSPRAM DIRECTIVES 500.12.500.16.500.19.500.20)

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14 LLISGOD DECK PLATING- SED DECK
15 LLISGOD DECK PLATING- SED DECK
16 LLISGOD DECK PLATING- MAIN DECK
16 LLISGOD BUCKHEADS-TRANSVERSE
16 LLIIGOD SHELL PLATING
16 LLIIGOD BOW
17 LLIIGOD BOW
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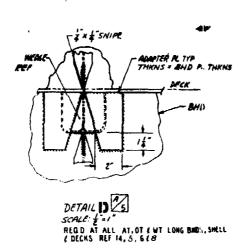
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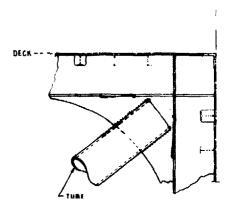
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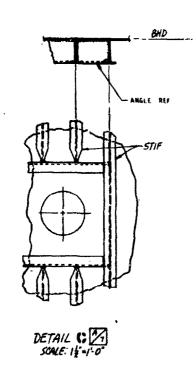
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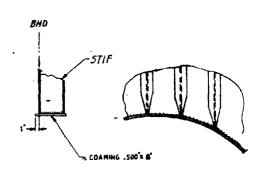




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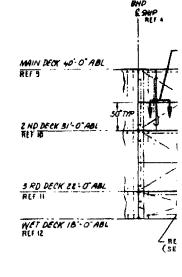
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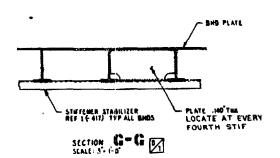


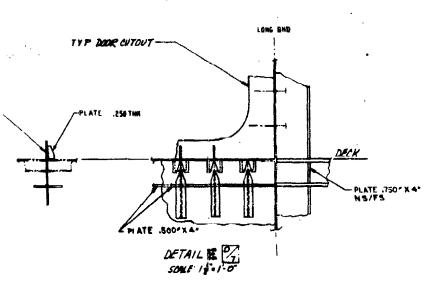


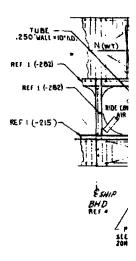
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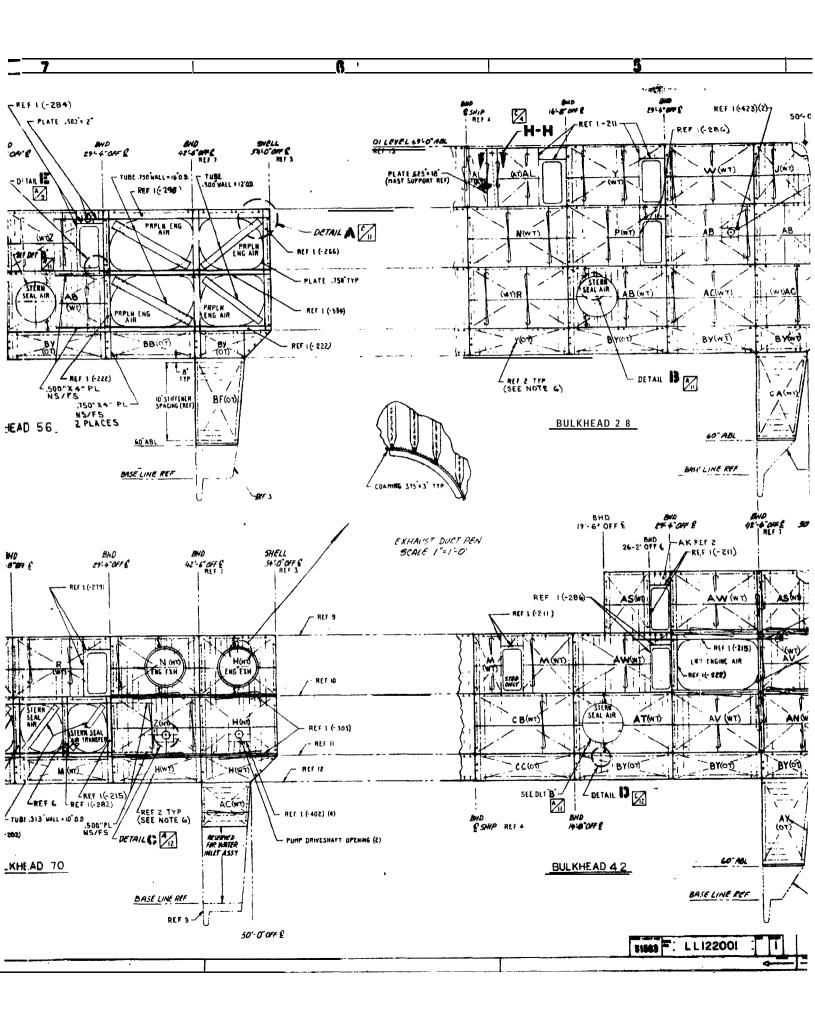


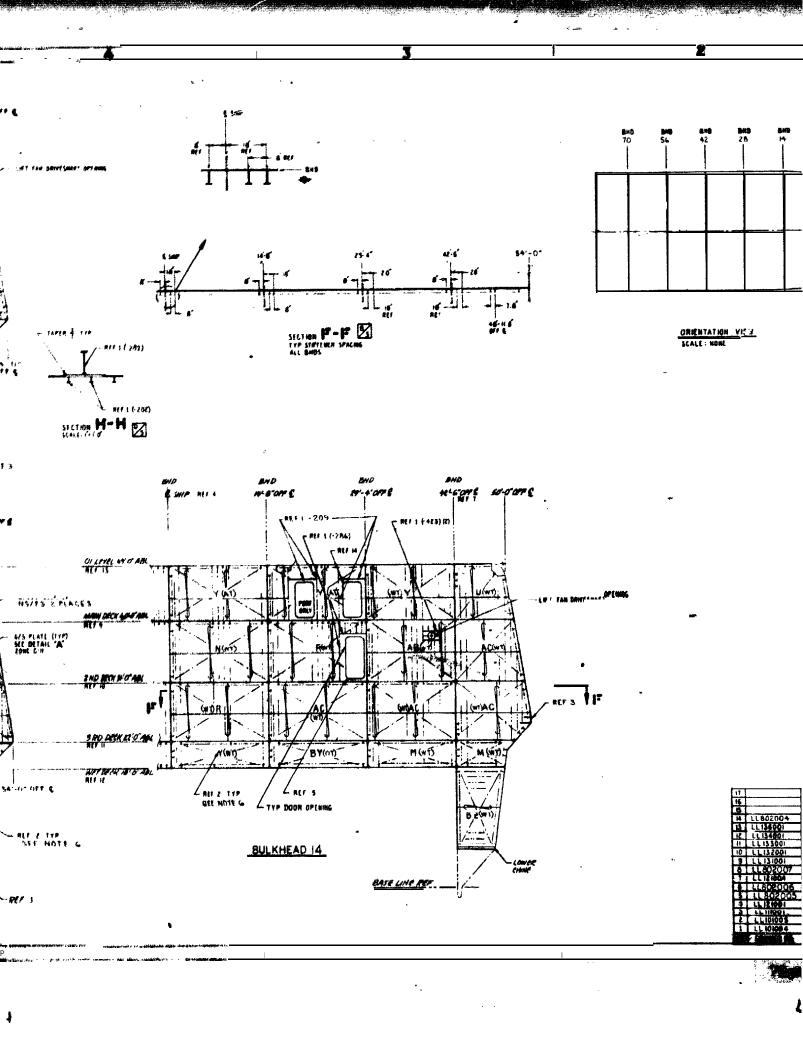


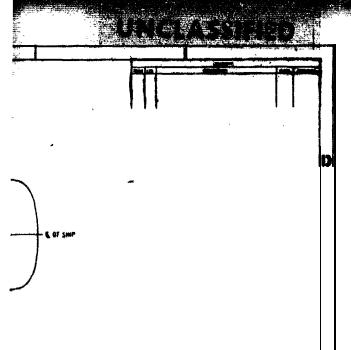


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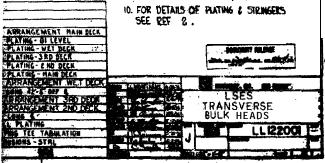






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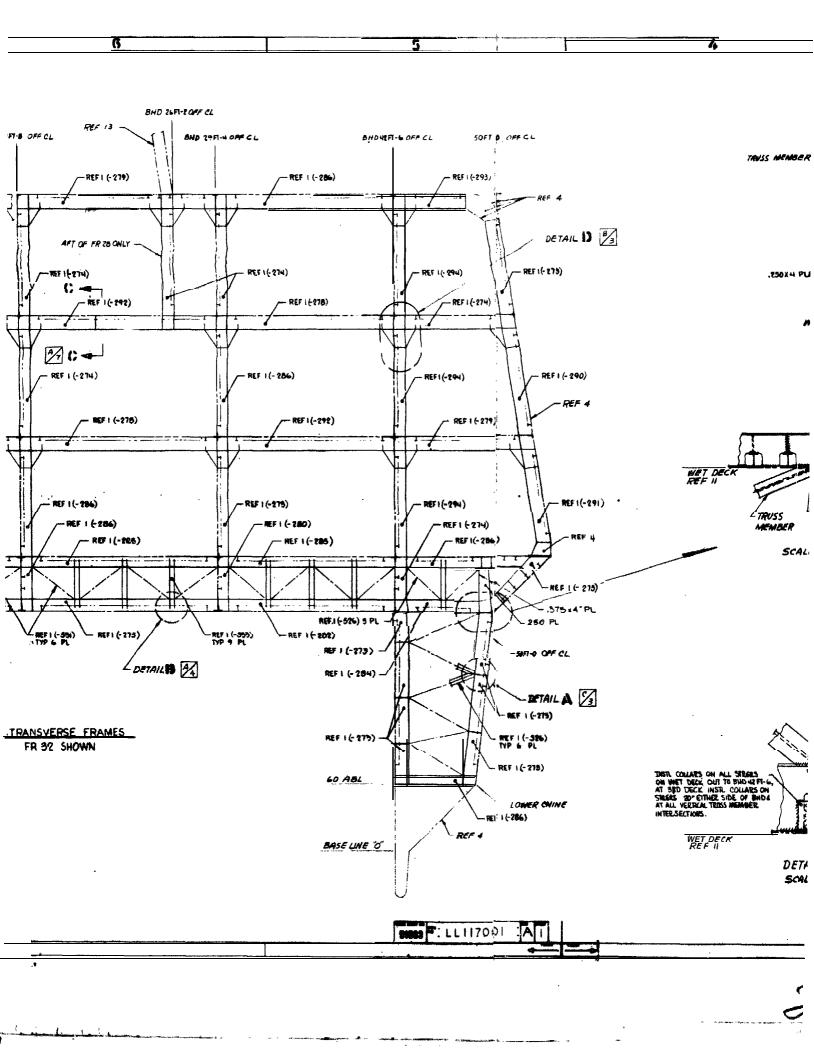
- I. MATERIAL: (SHEET & PLATE) 5456 AL MY CONDHIIGHT FOR THICKNESSES .IBB AND ABOVE. COND H323 FOR THICK-NESSES BELOVY. IBB, PER FED SPEC QQ-A-250/20. (EXTRUSIONS) 5456 AL ALY COND HIII PER FED SPEC QQ-A-2007
- 2. FABRICATE, WELD AND INSPECT PER NAVSHIPS 0900-060-4010 (SEE 2KSES PROGRAM DIRECTIVES 500.12, 500.16,500.19 & 500.20).
- 3 GENERAL STIFFENER SPACING
- 4 FRAME SPACING 36 EXCEPT AS NOTED.
- 5 STARBOARD SIDE SHOWN- PORT
- 6, S E E REF Z TABULATION CODE IDENT ON F/D FOR PLATING & STRINGER COMBINATIONS,
- 7. ASSREVIATIONS PER MIL-STD-12.
- & FILLET WELDS AROUND ENDS OF WELDED MEMBER TO FORM CLOSED LOOP.
- 9. FOR LOCATION, PROPORTION AND CLOSURE REQUIREMENTS OF ALL OPENINGS SEE REF 14,15,16/17.

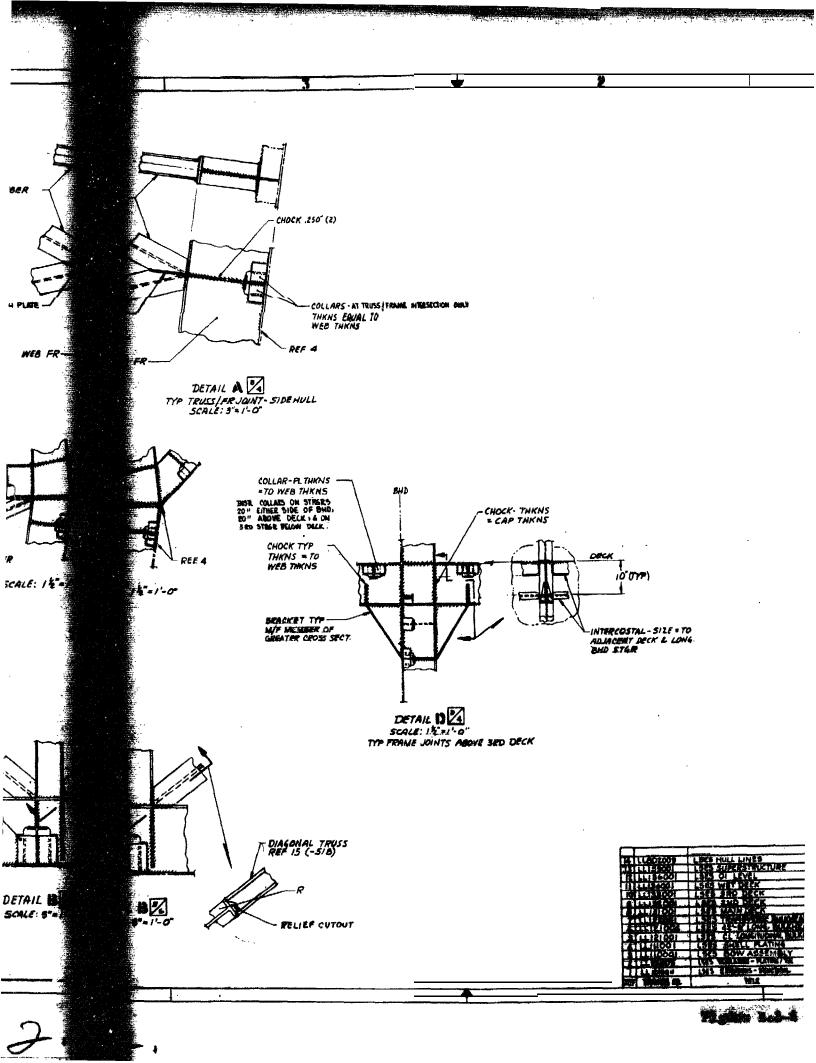


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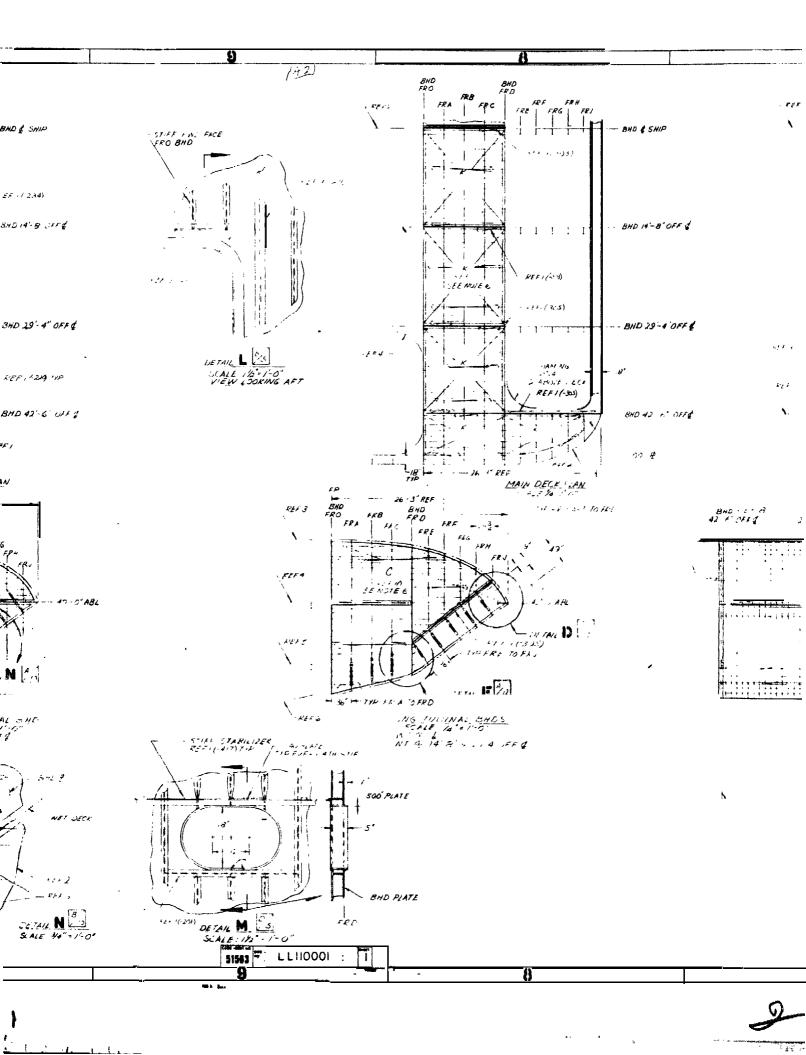


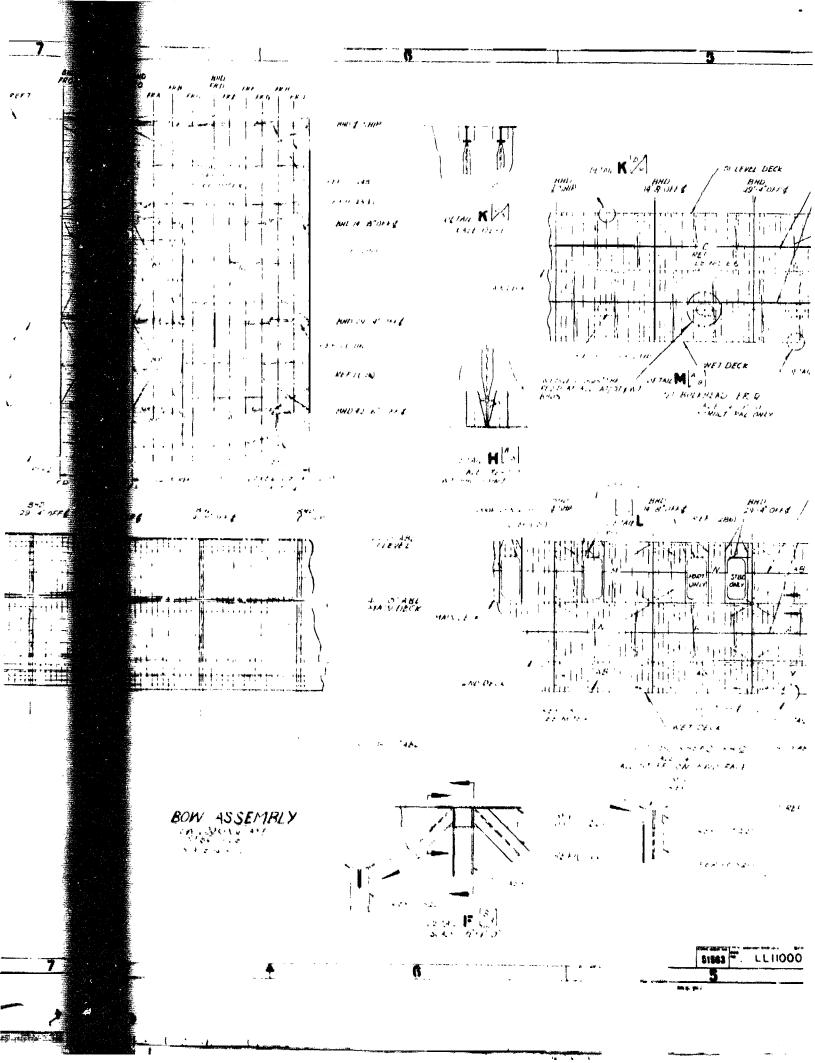


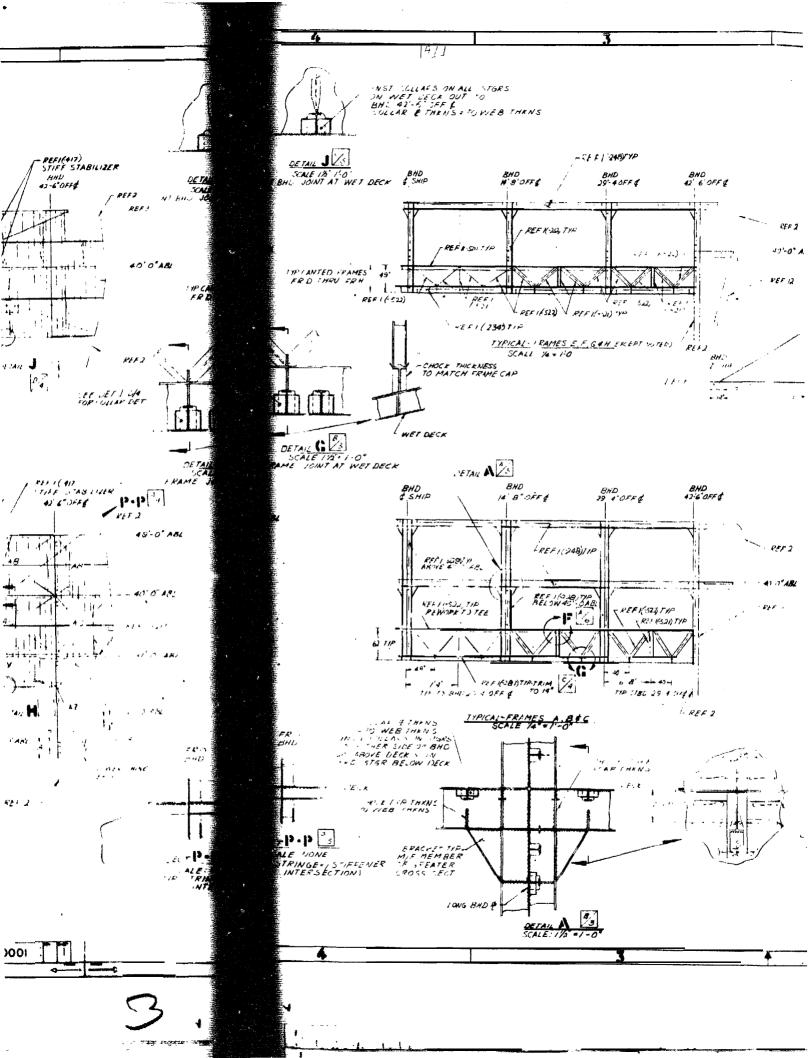
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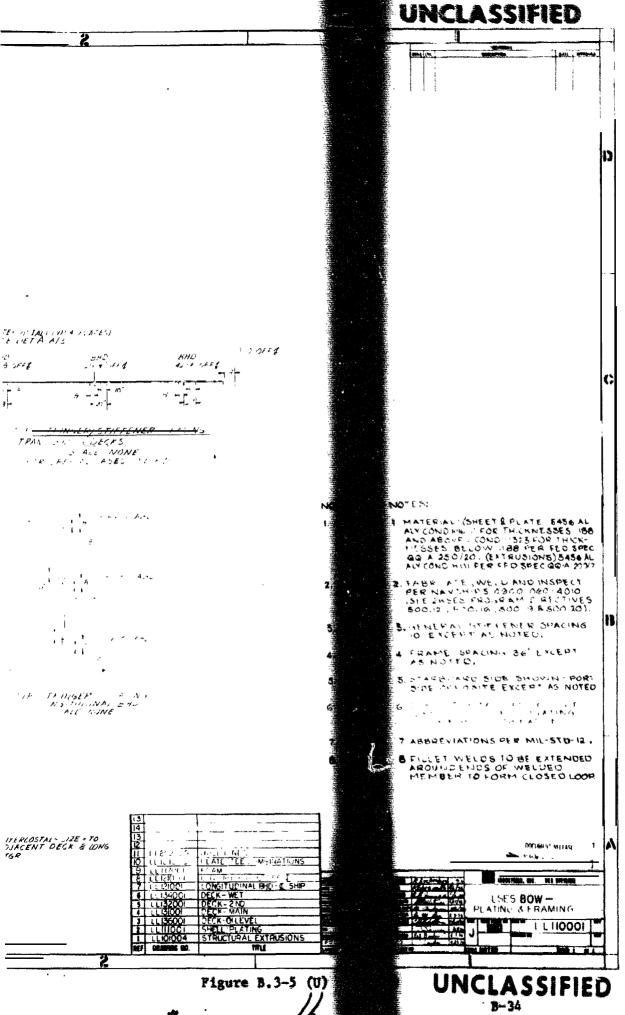


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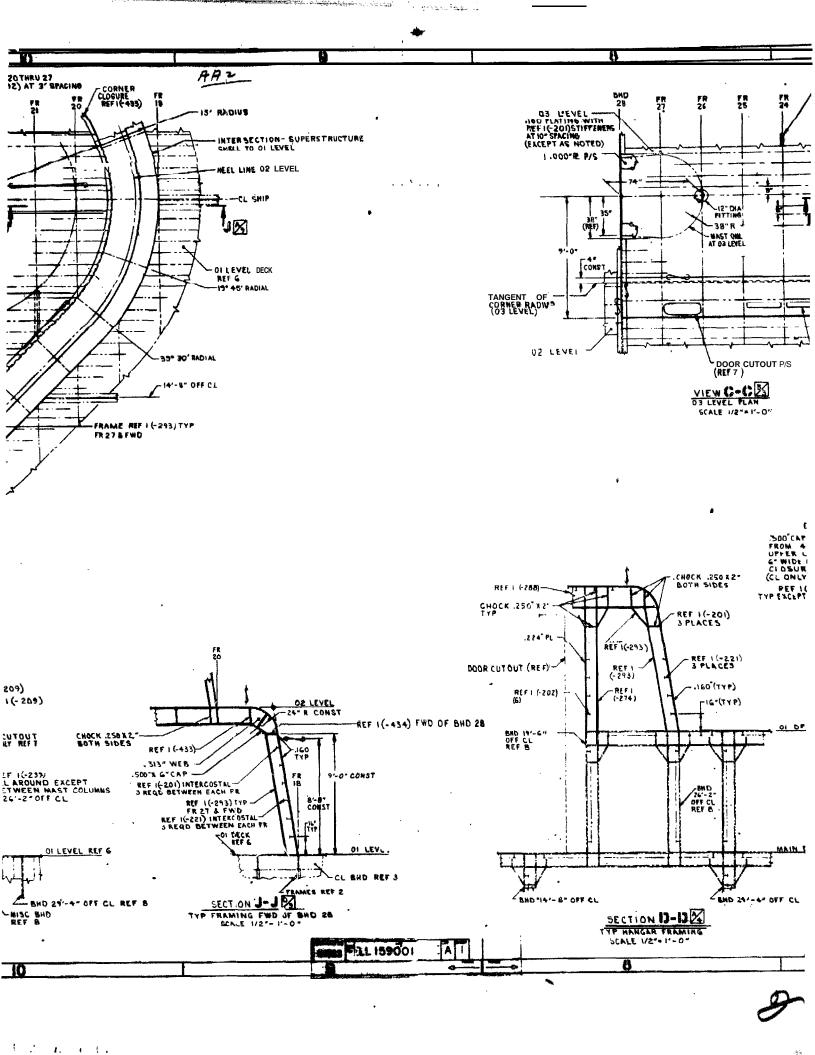


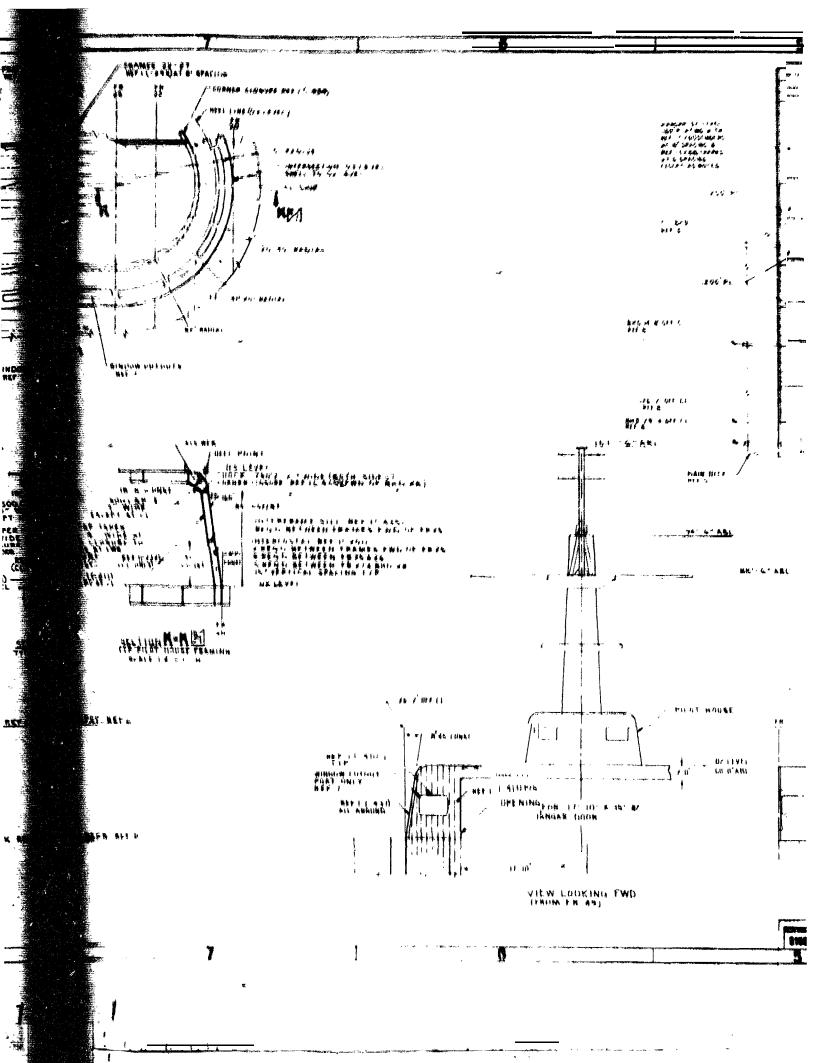


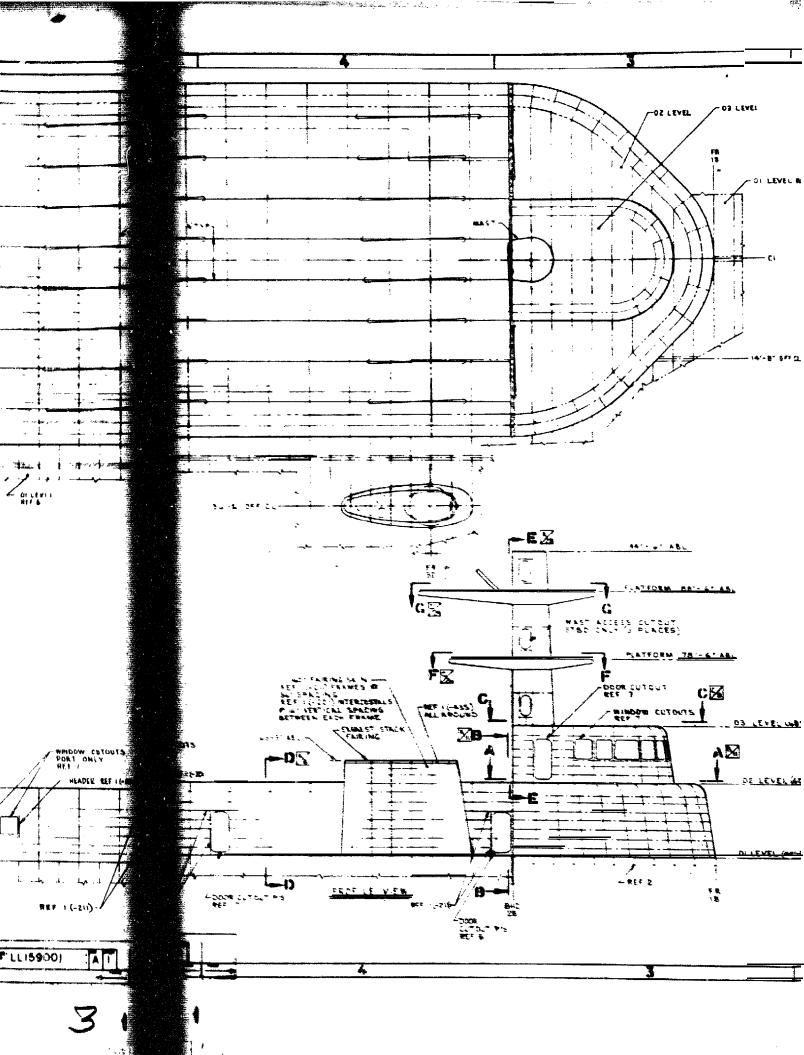
UNCLASSIFIED HEF I(292)(2) MAST STIFFENER-WAST FWD COLUMN MAST SKIN .160 (BEF) MAST SKIN 13 MAST COLUMN TO STANCHION ATTACH FITTING REF 1(-303) OB LEVEL REF 1 (-239) 1.000 PL INTERCOSTAL -CL INTERCOSTAL REF I(-293) STANCHONAL ATTACH BITTING ALD "SIX"000,1 STANCHION ATTACH FITTING STANCHION AIC "8 XULAW "002. CHIN- HANGA REF I (- 201) INTERCOSTAL (5) 02 LEVEL CL INTERCOSTAL REF 1(-303) REF 1(-292) PILOT HOUSE TO 02 LEVEL REF 1 (-239) CL BHD -.160"PL WITH REF I(-201) STIF @ 10" STACING & (-292) 141-8" OFF CL BOOK CUTOUT -REFI-293 QI LEVEL 3' SPACING -M/F REF 1 OL LEVEL ₹_{REF} ∠ REF 6.-A#CONST REF 4 TANGENT O F CORNET SECTION M-HS VIEW A-A DE LEVEL PLAN REF 1 (-239) 03 LEVEL STIFFENER STABILIZER REF : (-417) M/FREF 1 (-402) ZVE" X 2" -WINDS W CUTOUT REF 7 REF 1(-295) P/S OI TO 03 LEVEL \ PEFI (-431) A L L AROUND EXCEPT BETWEEN MAST COLUMNS STIFFENERS REF 14-201) EXCEPT REF 1(-246) DETWEEN MAST COLUMNS .425 X 13" R. P/S .160 PLATING (TYP) FACEPT AS NOTED .425 X16" R P/5 .IGO" PLATING (TY P) REF 1 (-209) STIFFENER STABILIZER ... REF I (-417) (TYP) DOOR CUTOUT REF 7 SECTION 13-13 LA LOOKING FWD AT BHD 28

(NOTE: STIFFENERS ABOVE OI LEVEL ARE ON FWD SIDE OF BHD)

SCALE 1/2"=1"-0" CL BHD REF 3 -BHO IN'- 8" OFF CL TRANSVERSE BHD







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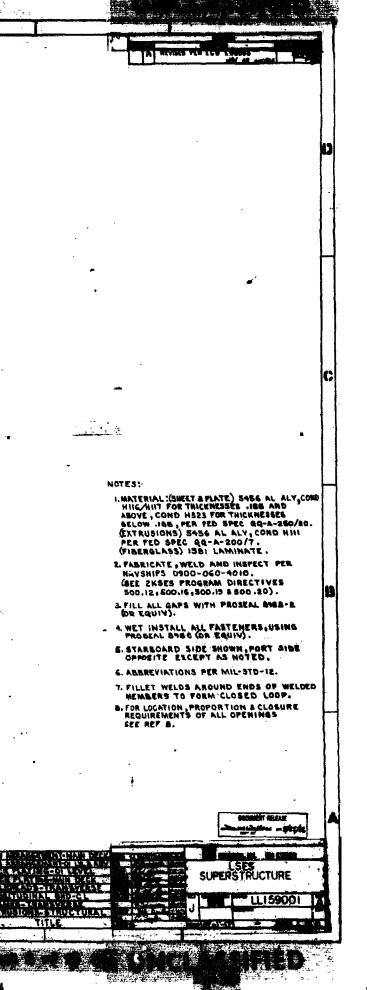
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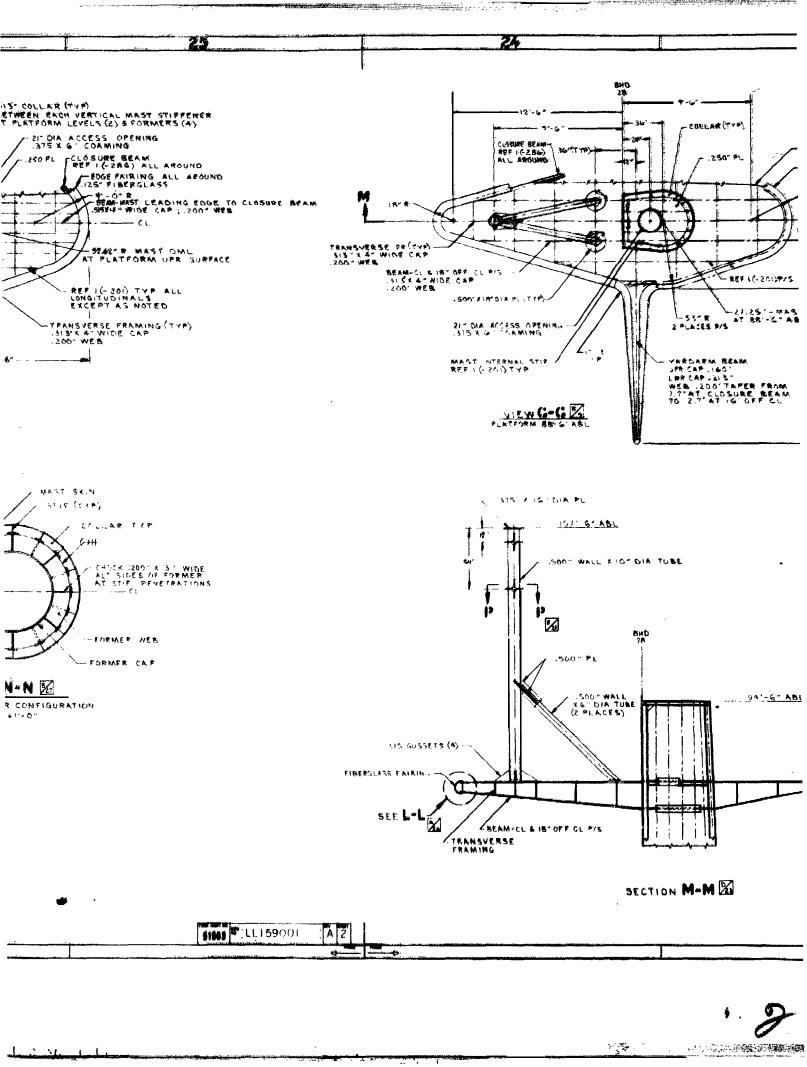
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CLOSURE 4 (TYP)

SECTION L-L M

.125" FIBERGLASS ALL AROUND 8" REF

VIEW LOOKING FWD AT BHD 28 NOTE: ALL STIF ON FWD SIDE



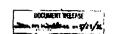
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COULAR .500 PLATE

SUBSET ...00 PLATE

SECTION 13-13

BB'-6" ABL



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Pigure B,3-6 (Sheet 2 of 2) (U)

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Figure 8.3-7 (Sheet 1 of 5) (U)

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Figure B.3-7 (Sheet 2 of 5) (U)

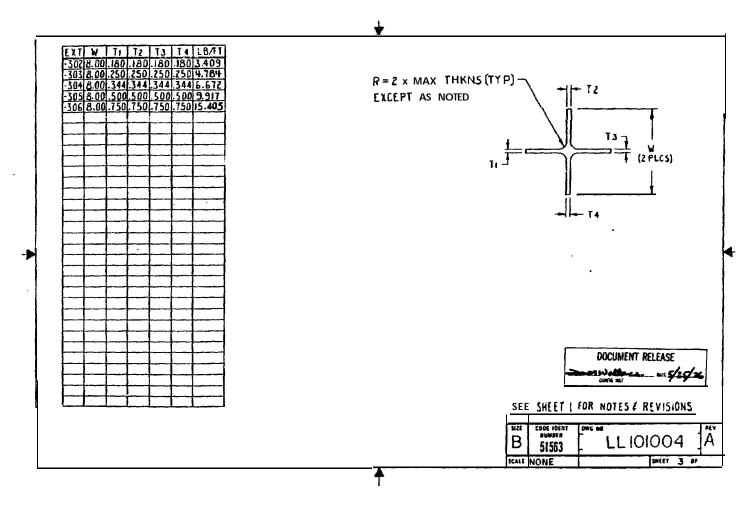


Figure **B.3-7** (Sheet 3 of 5) (U)

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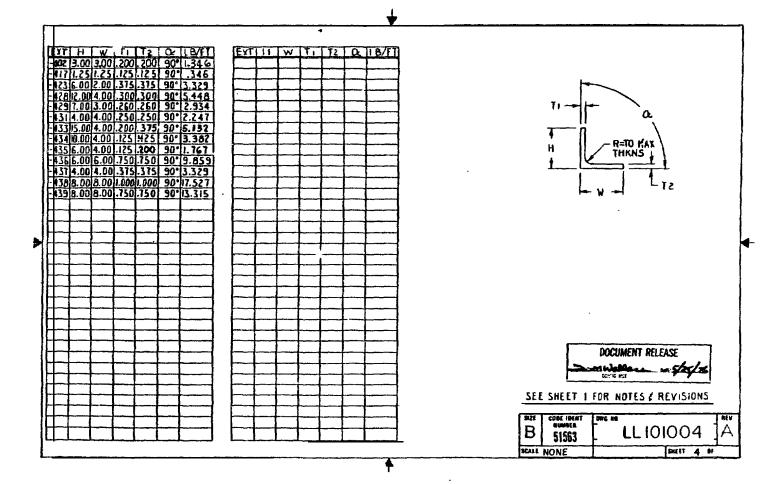


Figure B.3-7 (Sheet 4 of 5) (U)

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SEE SHEET I FOR NOTES & REVISIONS

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Figure **B.3-7** (Sheet 5 of 5) **(U)**

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521 | 6.00 | 6.00 | 6.00 | 250 | 250 | 250 | 5.055 |
522 | 6.00 | 6.00 | 6.00 | 300 | 300 | 125 | 5.228 |
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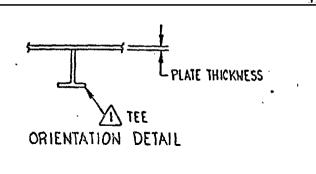
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Figure **B.3-8** (Sheet 1 of 2) (U)



SUPERSTRUCTURE

IDECK PLATING - OI LEVEL

DECK PLATING - WET DECK

DECK PLATING - 3RD DECK

DECK PLATING - 2ND DECK

DECK PLATING - MAIN DECK

BULKHEAD - LONG. , 42 FT-6 OFF &

BULKHEADS - TRANSVERSE

BULKHEAD-LONG., CL

EXTRUSIONS - STRUCTURAL

TITLE .

SHELL PLATING

BOW

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LL 134001

LL 133001

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LL 121001

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Figure B.3-8 (Sheet 2 of 2) (U) UNCLASSIFIED

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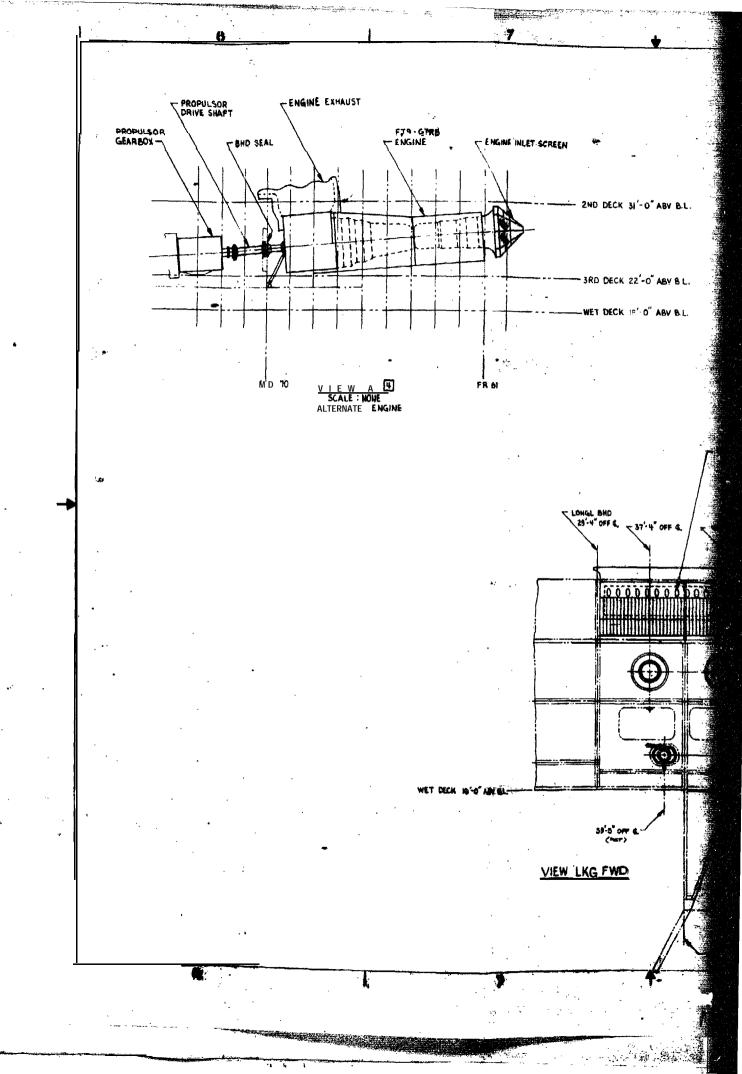
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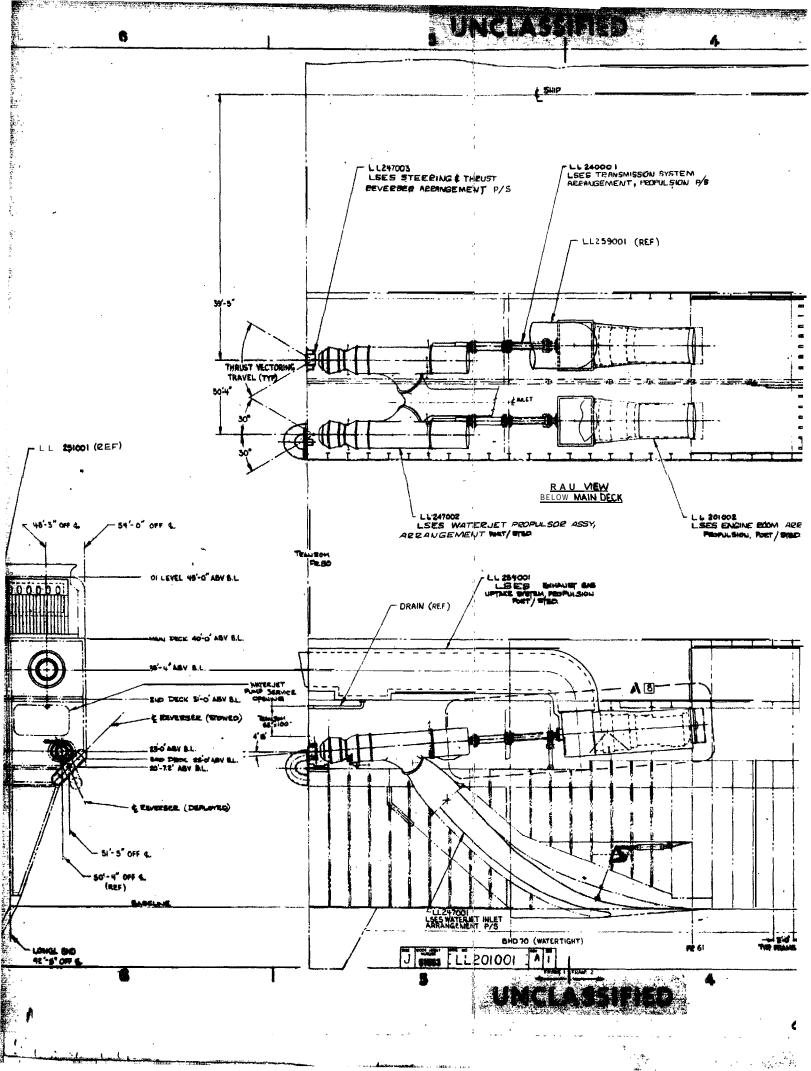
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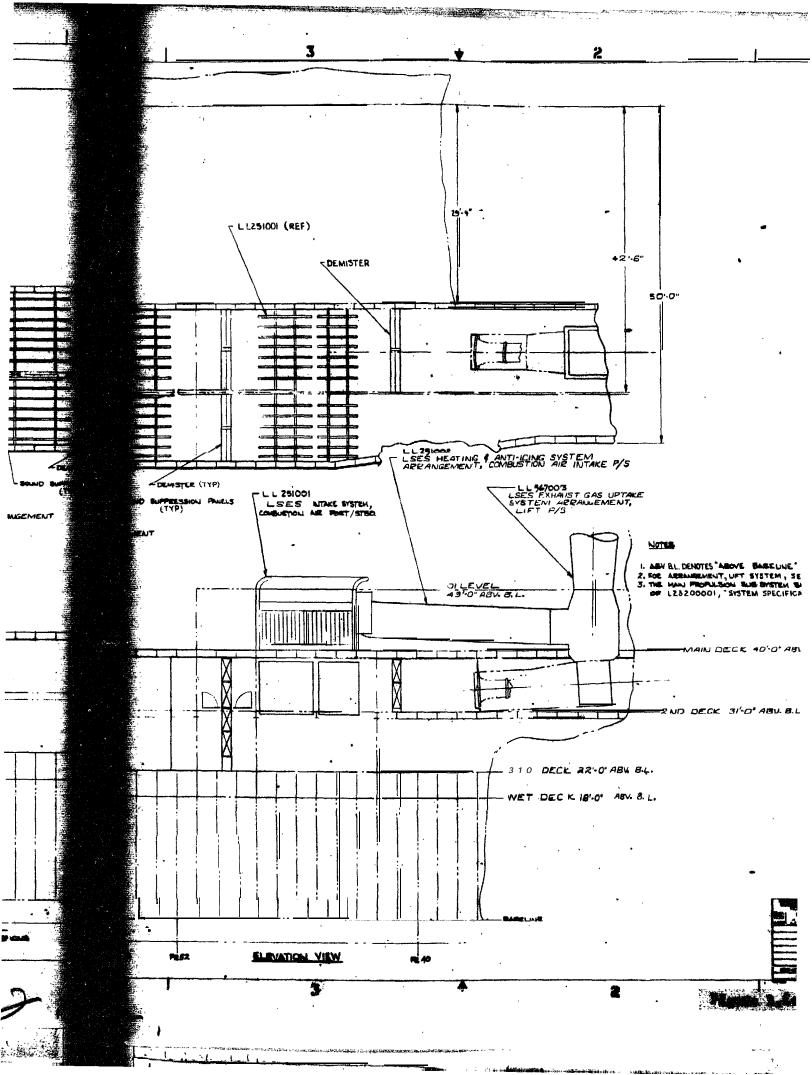
- (U) B.4 PROPULSION SYSTEM DRAWINGS
- (U) This section of Appendix B contains the following propulsion system drawings:

Figure <u>Title</u>

- **B.4-1** Main Propulsion Machinery Arrangement, P/S
- B.4-2 Waterjet Inlet Arrangement, Port and Starboard
- (U) The remainder of the detailed propulsion system description for the ANVCE near term SES is contained in Section 2.3.2.



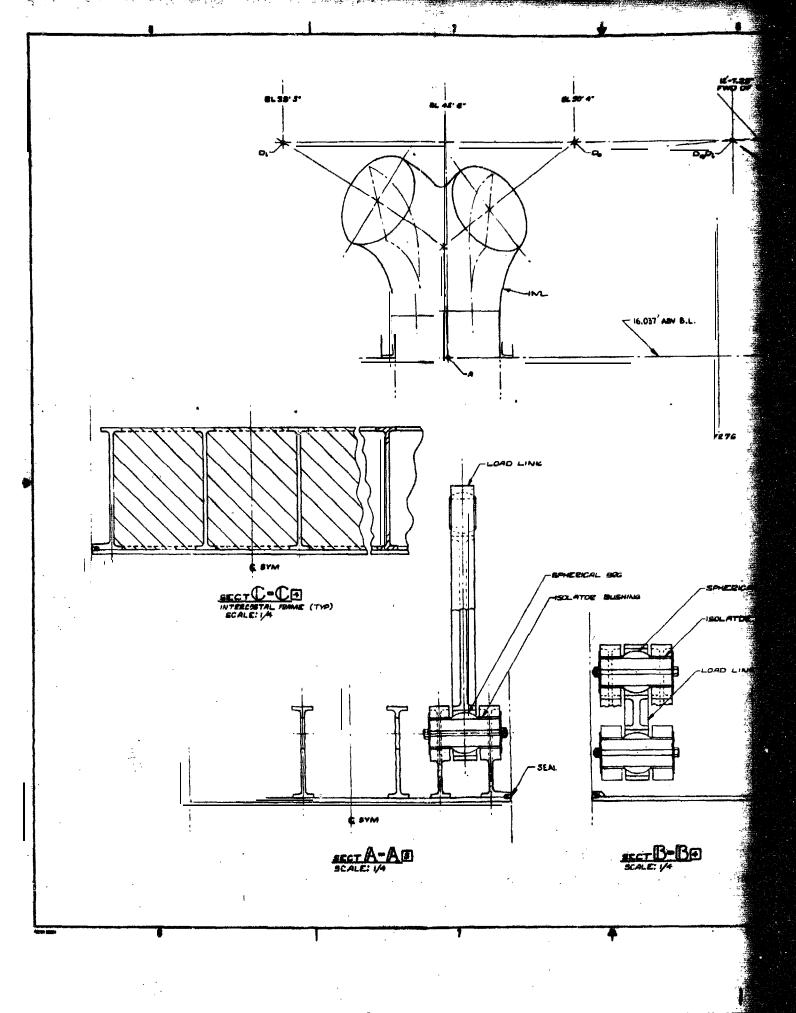




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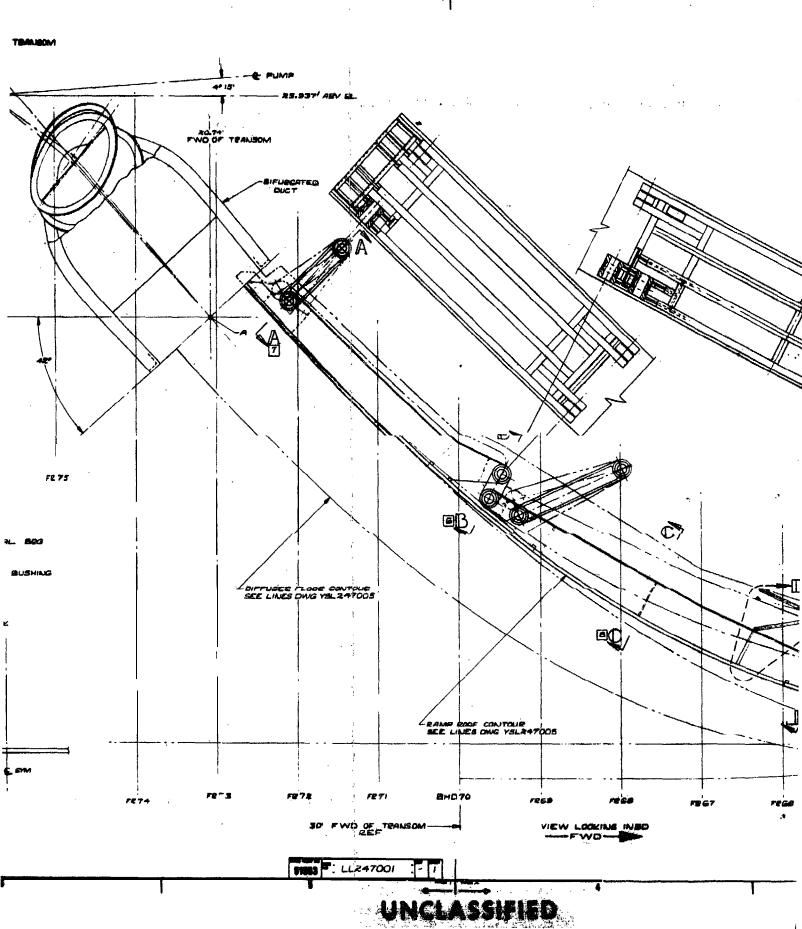
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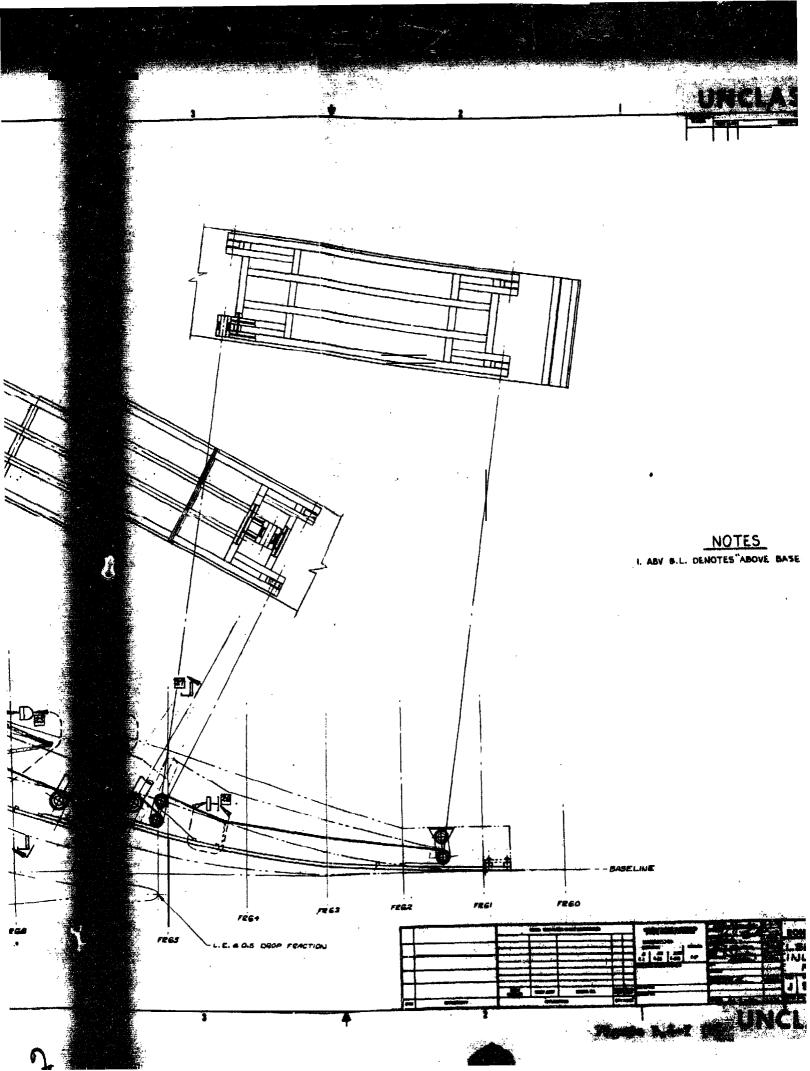
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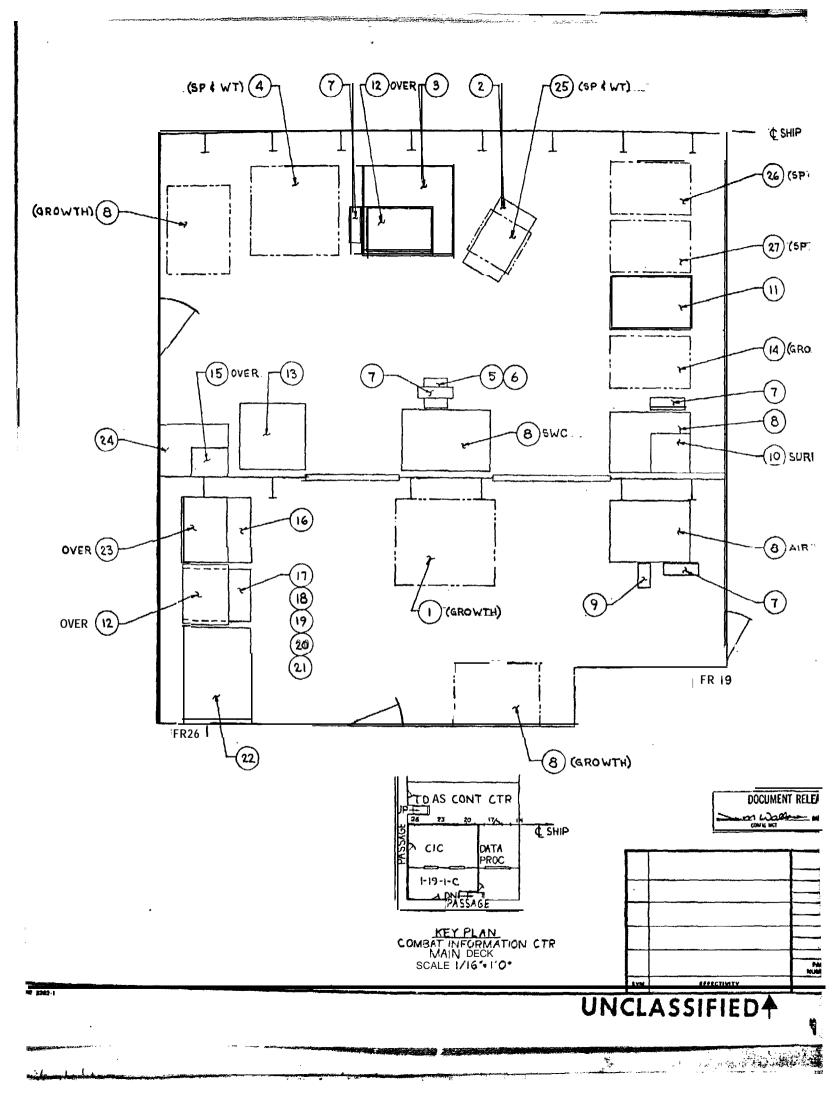
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- (U) B.5 c^3 arrangement drawings and block diagrams
- (U) This section of Appendix B contains the command, control, and communications (C³) arrangement drawings and block diagrams for the near term ANVCE SES. They are:

<u>Figure</u>	<u>Title</u>
B.5-1	Combat Information Center, Main Deck
B.5-2	Communication Center, Main Deck
B.5-3	Radio Transmitter Room, Main Deck
B.5-4	Data Processing Room, Main Deck
B.5-5	Helicopter Control Station, 01 Level
B.5-6	Command and Surveillance Block Diagram
B. 5-7	IC Voice System Matrix



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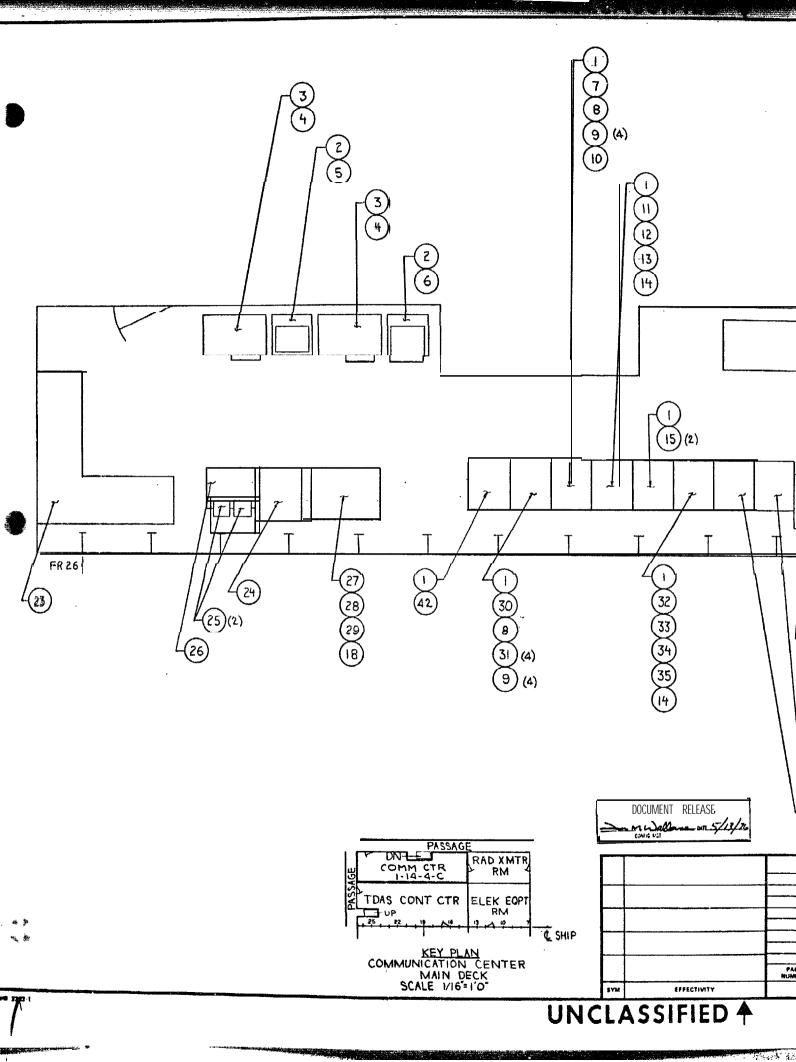
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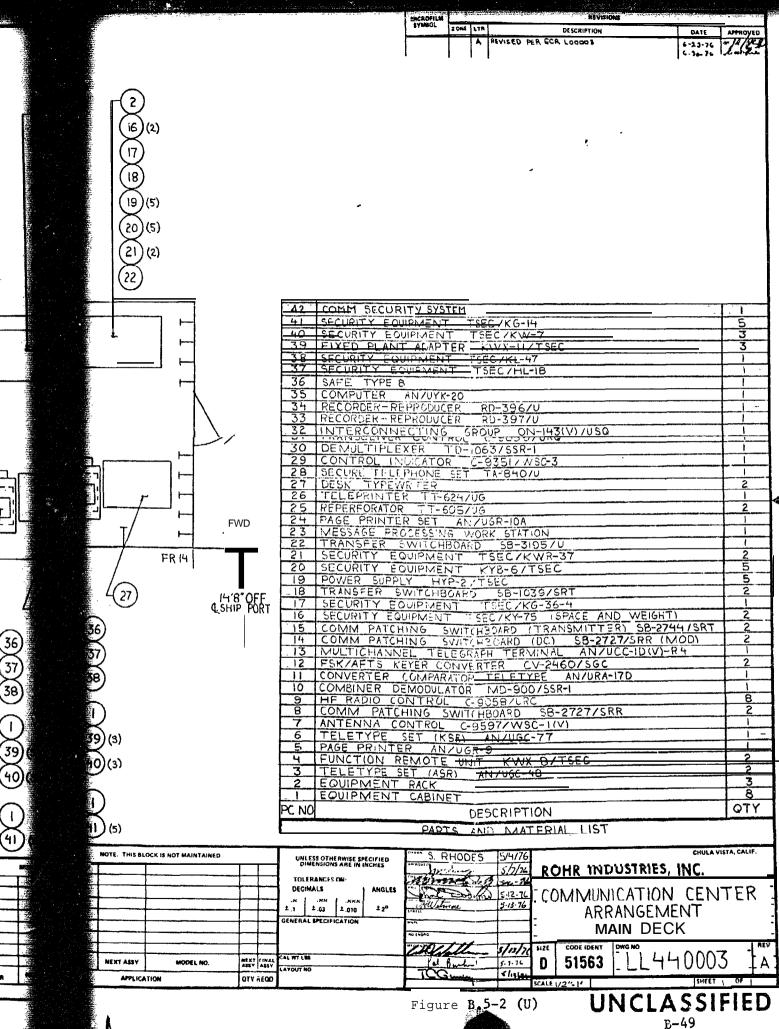
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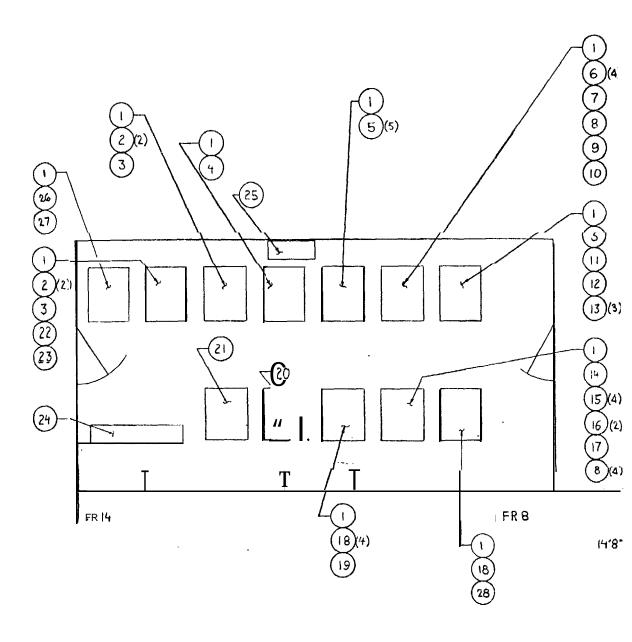
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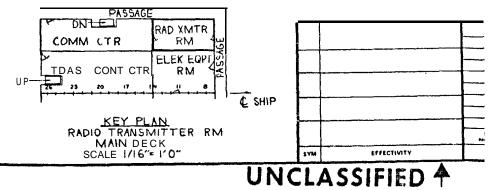
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UHE NTDS TRANSCRIVER HE NTDS TRANSCRIVER 27 HF NTDS TRANSCEIVER

26 LF/MF RECEIVER

25 DUMMY LCAD DA: 242 A/U

24 PATCH PAREL MATPIA SE-3721/ 5KR-3

23 TEST SET TS-3337/ 5KR-3

22 PATCH PAHEL MATPIA SE-3720/5KR-3

21 TELLIM: TPY PECEIVER P-1039/5KR-3

20 DEMODILATOR MD-8: 72-8-3

19 UHF LOCAL CONTROL UNIT C-9059/URC

18 UHF TANSCEIST TY/VAC 02

17 FREQUENCY STANDARD AN/URO-23

16 RF DISTRIBUTION - WPLIFIER AM-2123A/U

15 BANDPASS FILTER F-332///

14 UHF MULTICOUPLER TD-1046/URC

13 HF RECEIVER AN/UPR-67

12 FILTER LP-101C

11 RECEIVER MULTIPLEXER 512J-2

10 RF SVITCHING UNIT 5A-2000/WSC-3

9 ANTENNA COUPLER SMITCH SA-1712/56

7 UHF LOG/SATCOM TPANSCEIVER RT-1107

6 ANTENNA COUPLER CONTROL C-3698/

5 COUPLER ADAPTER MX-4845/SR

4 RF SWITCHING UNIT SA-1070/UR

3 HF RADIO SET CONTROL C-9058/URC

2 HF RADIO SET CONTROL C-9058/URC

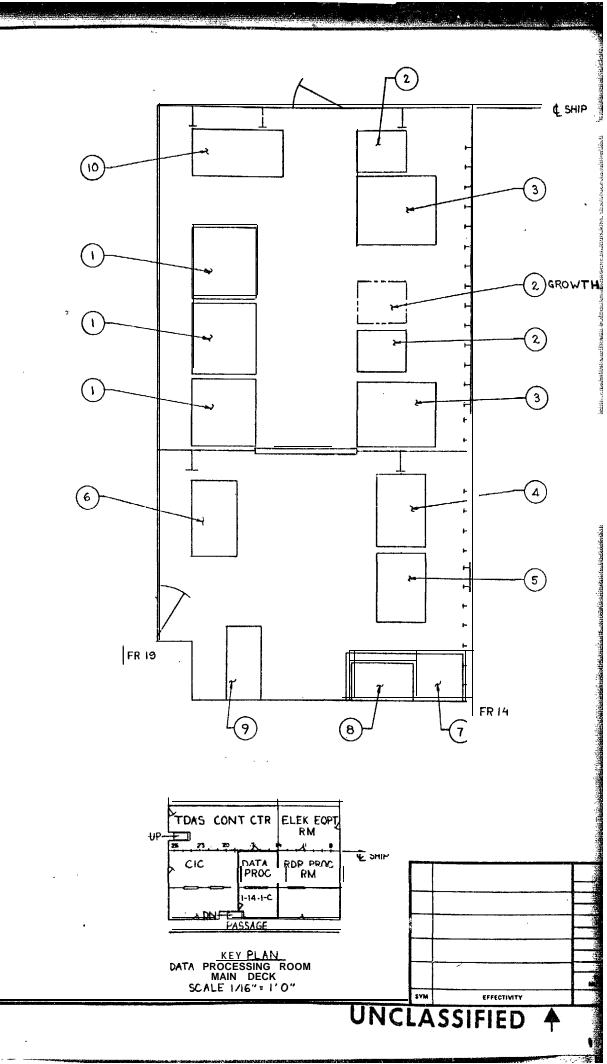
2 HF RADIO SET AN/URC-81

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Figure B.5-3 ('')

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2 ONE LIR DESCRIPTION A REVISED PER SCR LOGGOS DESCRIPTION

FWD -

10	MONITOR CONTROL CONSOLE 0J-200/UYA-A(V)	
9	COMM PATCH PANEL SB-2781/UYA-41V)	
8	TEST SET TS-2460/INA-4	
	(AOUSK - KEVICE) = -	
6	RADAR DATA DISTRIBUTION SWITCHBOARD SB-2780/UYA-41V)	
5	CENTRAL FOURMENT GROUP OU- 91(V)3/UYA-4(V)	-} }
4	SIGNAL DATA CONVERTER CV-2953/UYK-/IV)	
_ 3	INPUT/OUTPUT CONSOLE OJ-172/UYK-7(V)	
2	COMPUTER ANXIVEZIVI (INCLUDING GROWTH)	
1	COMMAND AND SURVEILLANCE SYSTEM SWITCHBOARDS	
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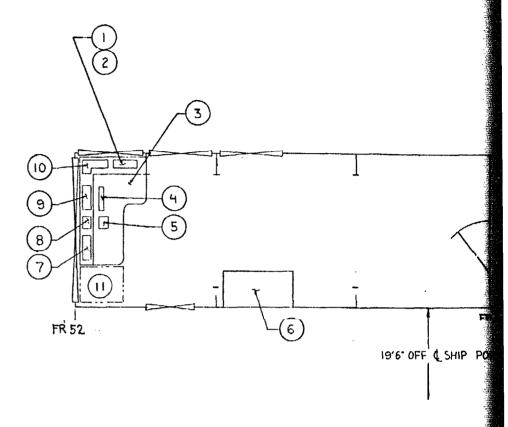
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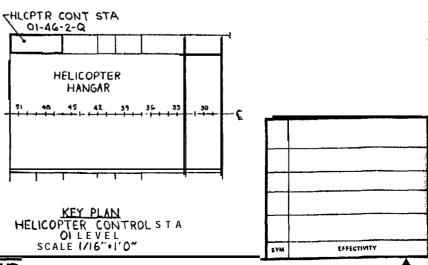
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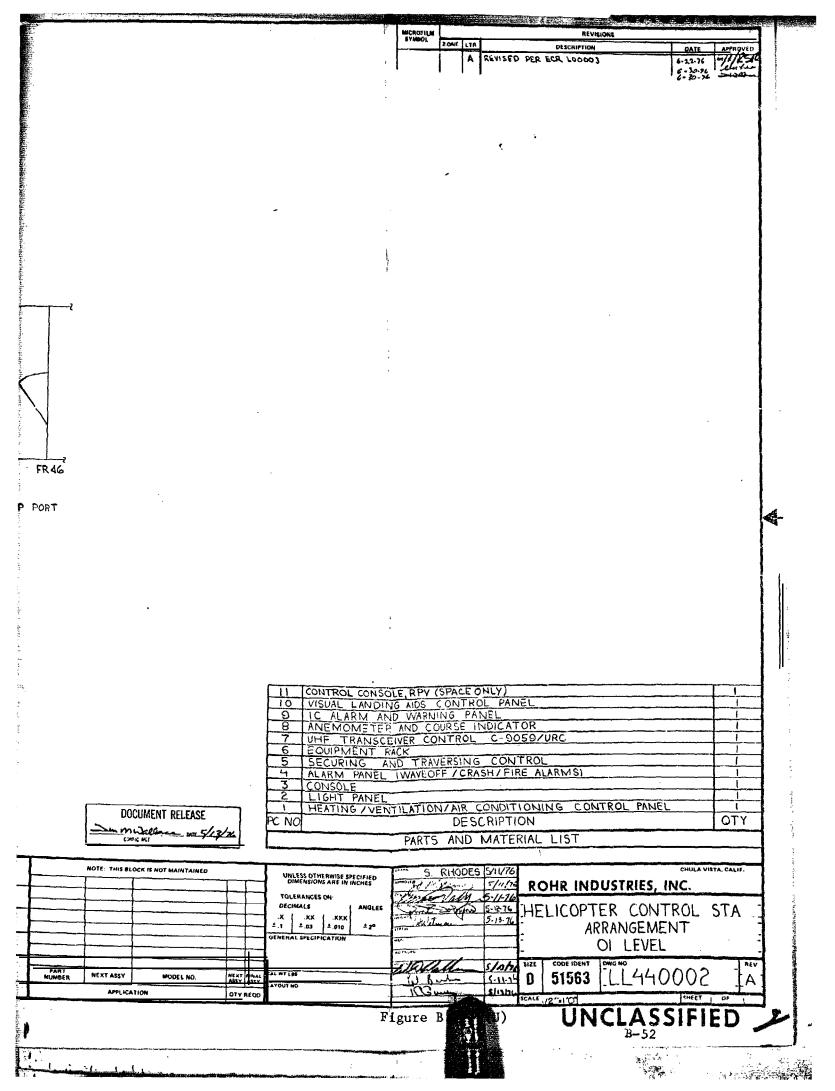
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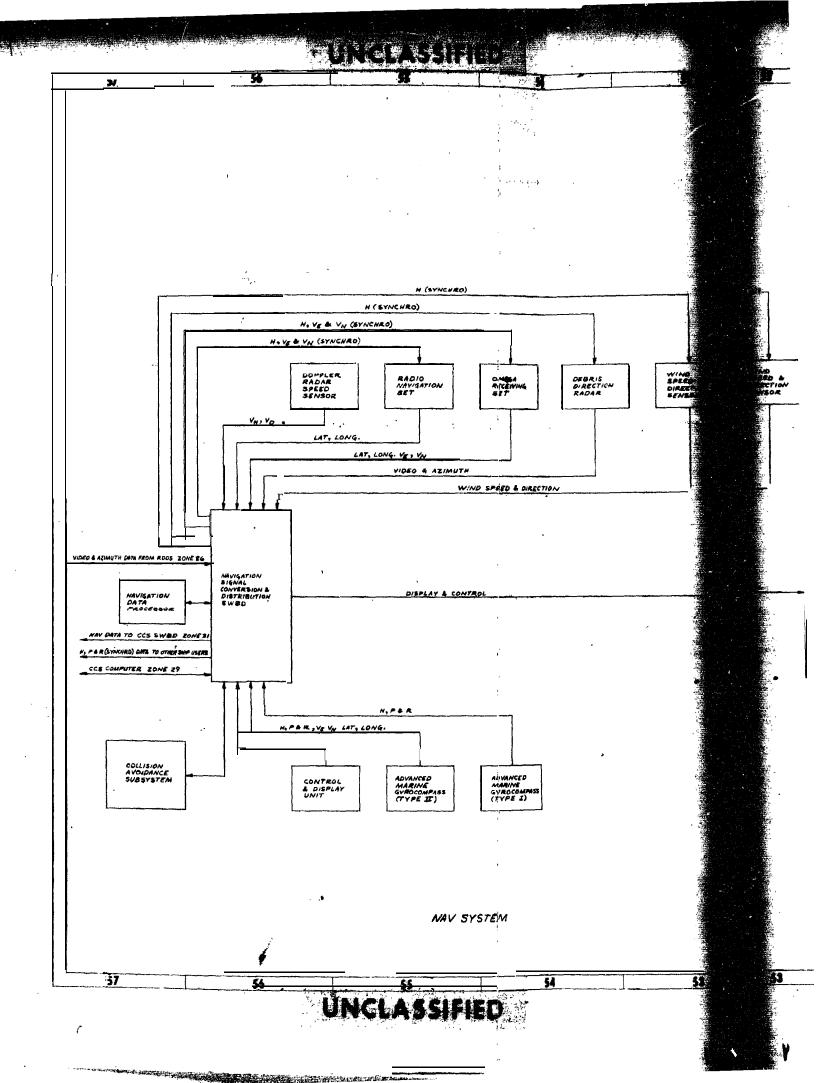
Figure B.5-4 (U)

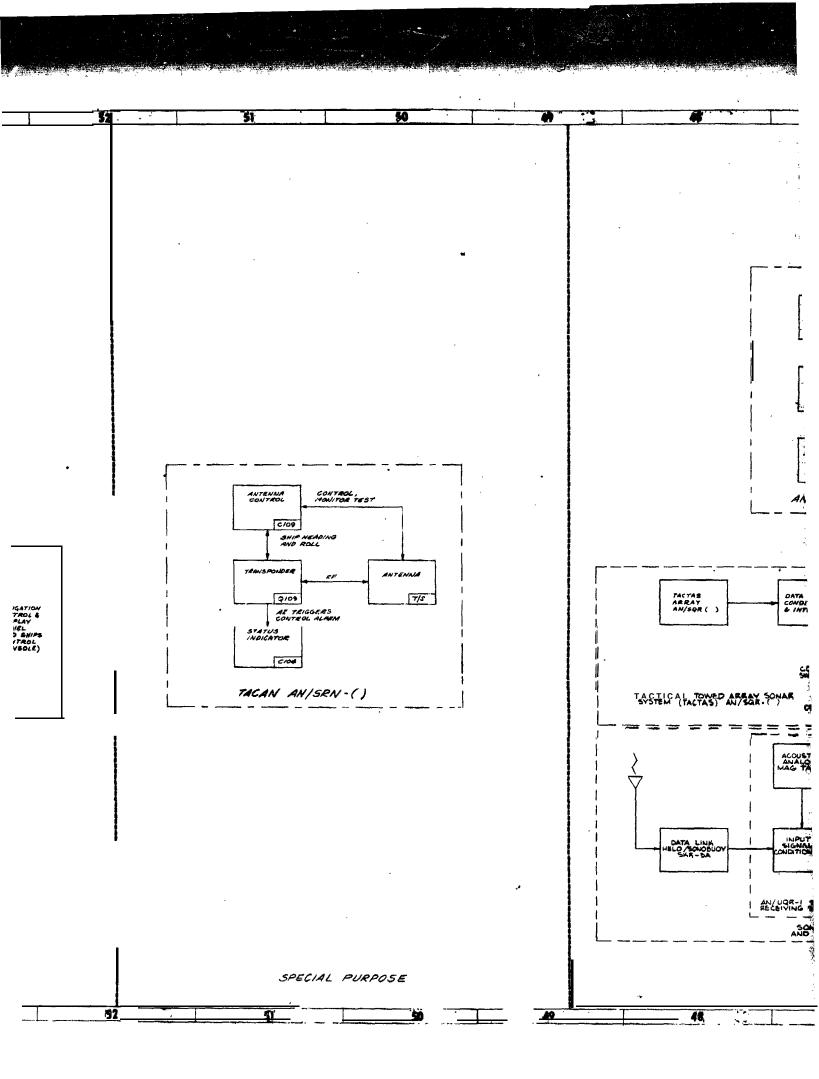


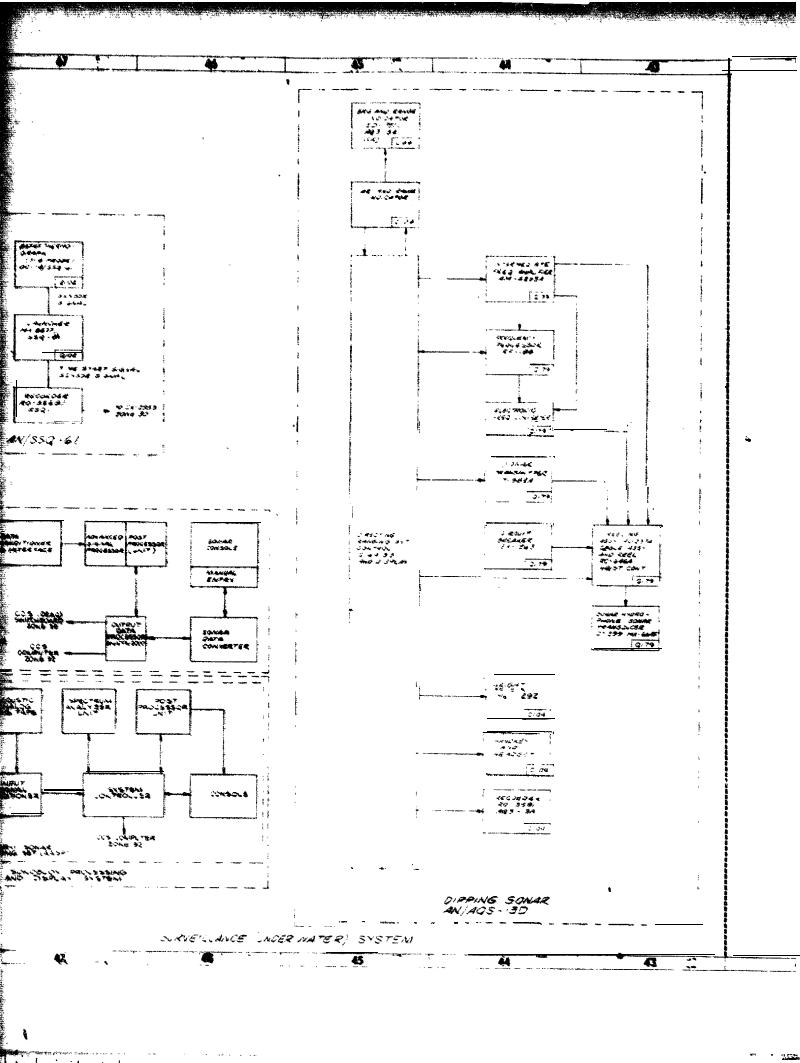


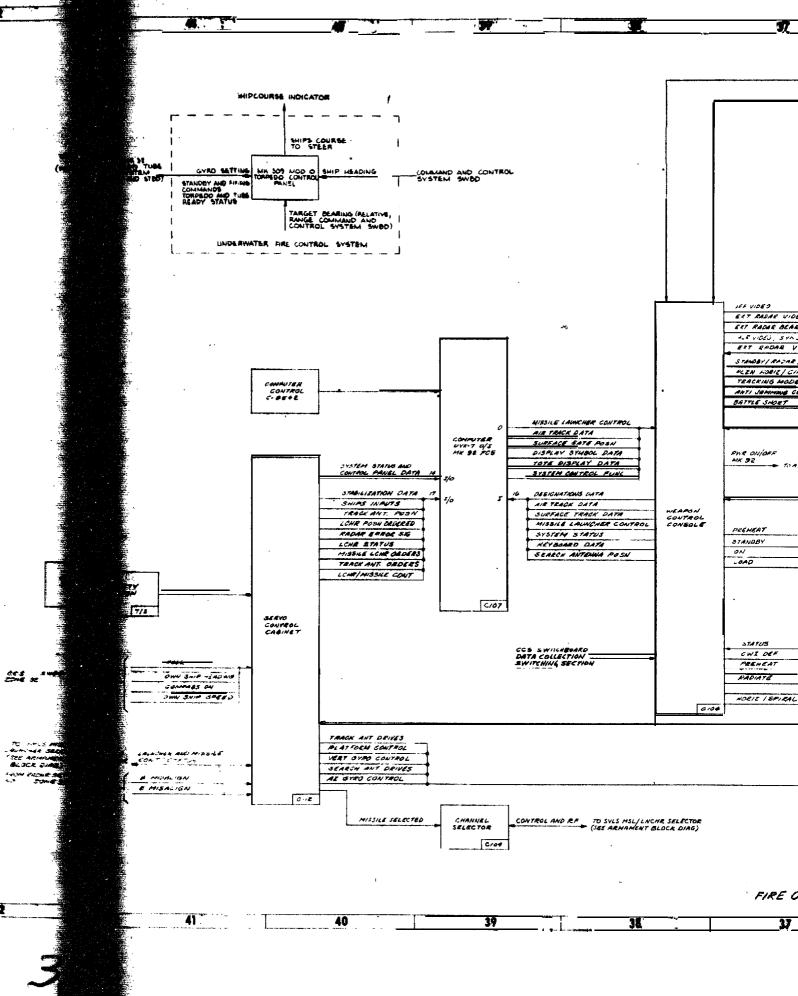




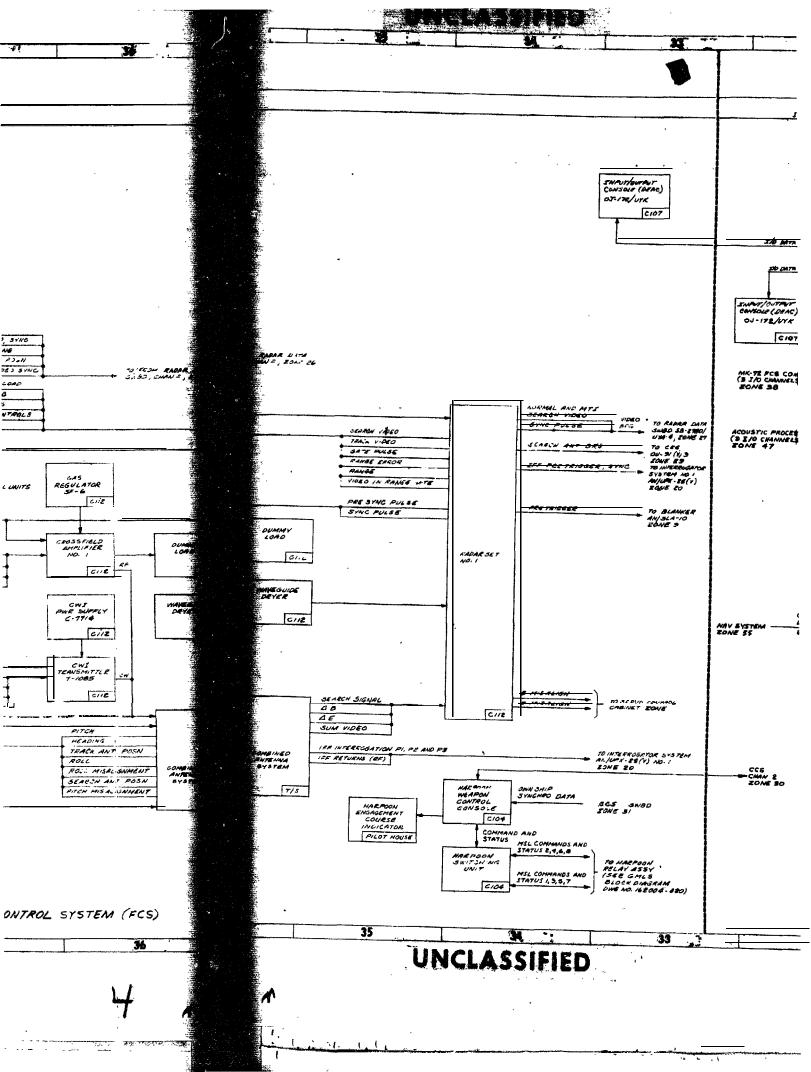


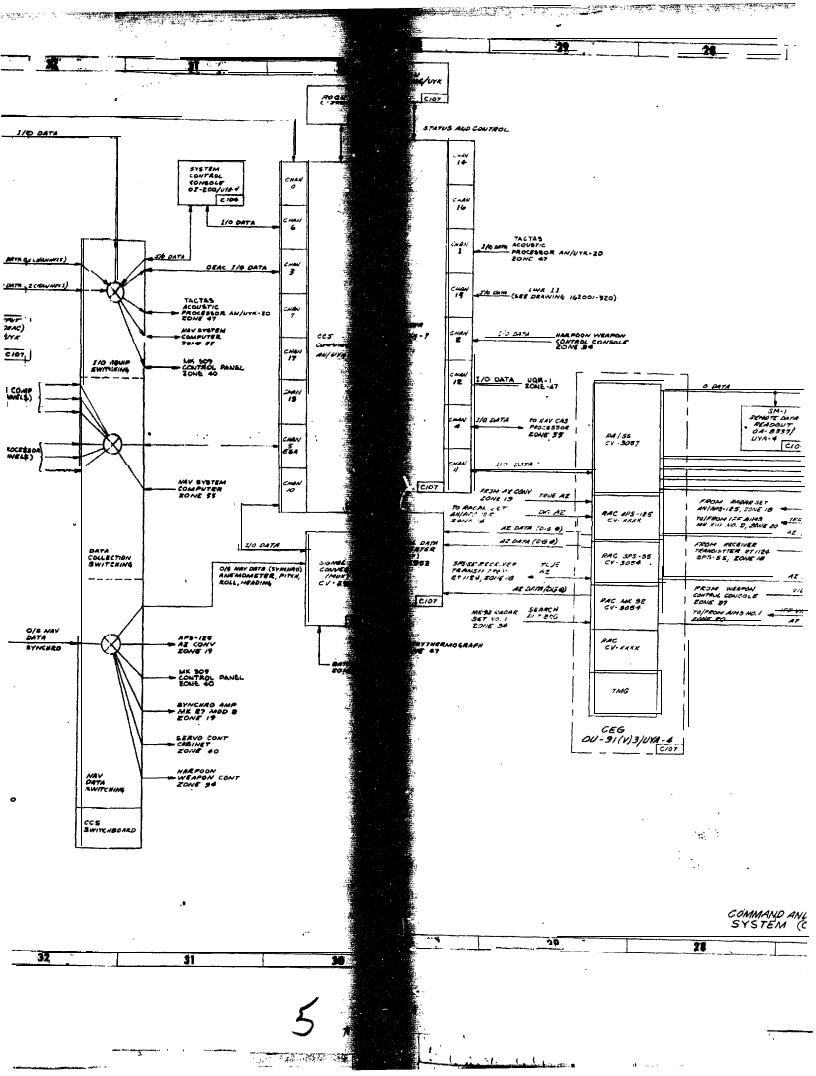


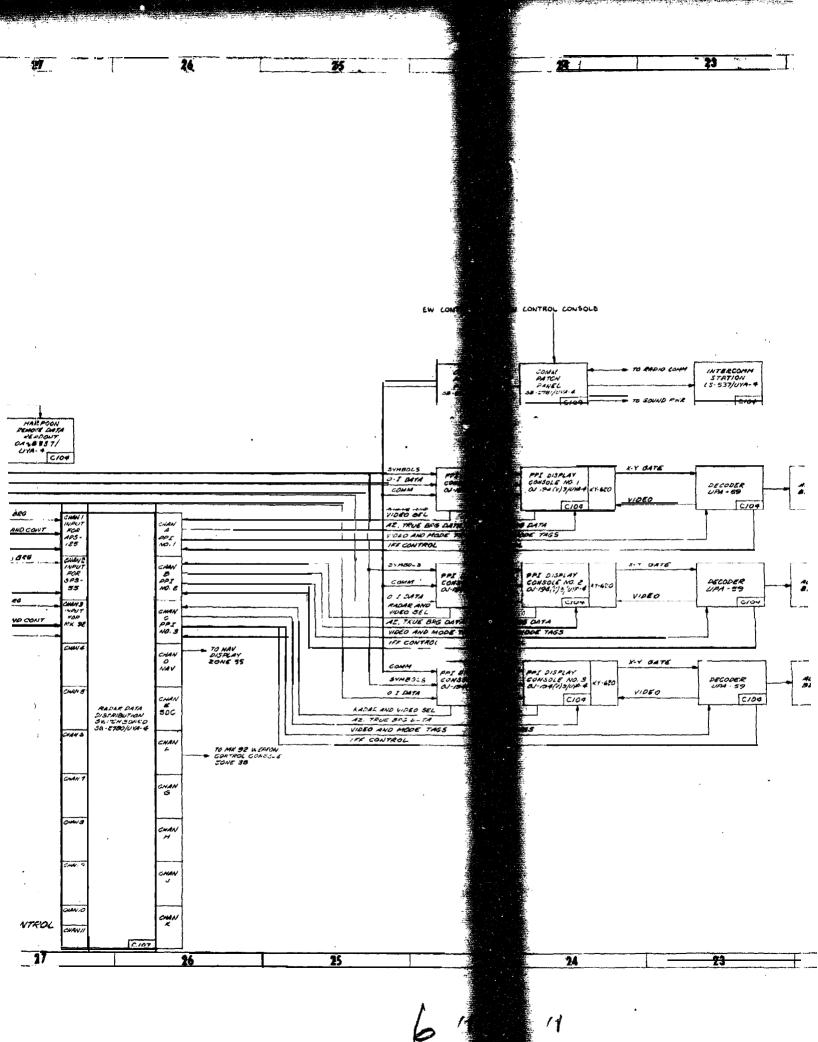


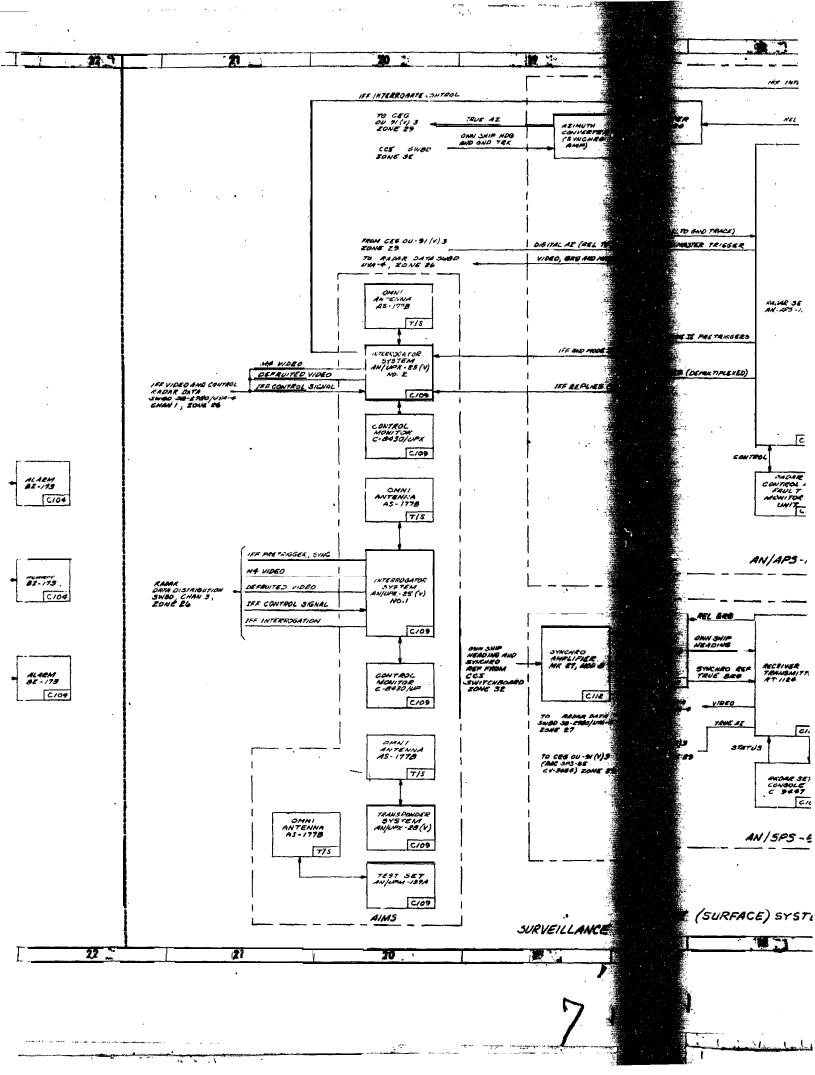


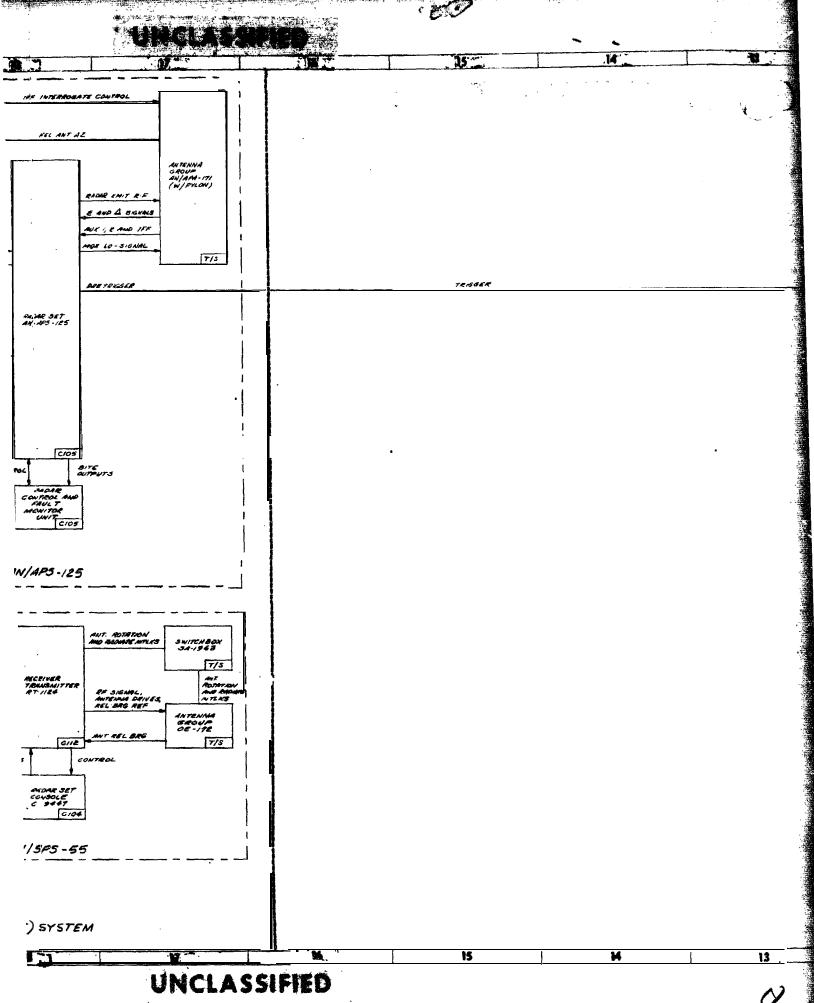
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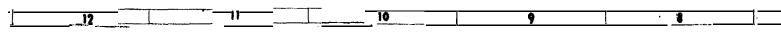
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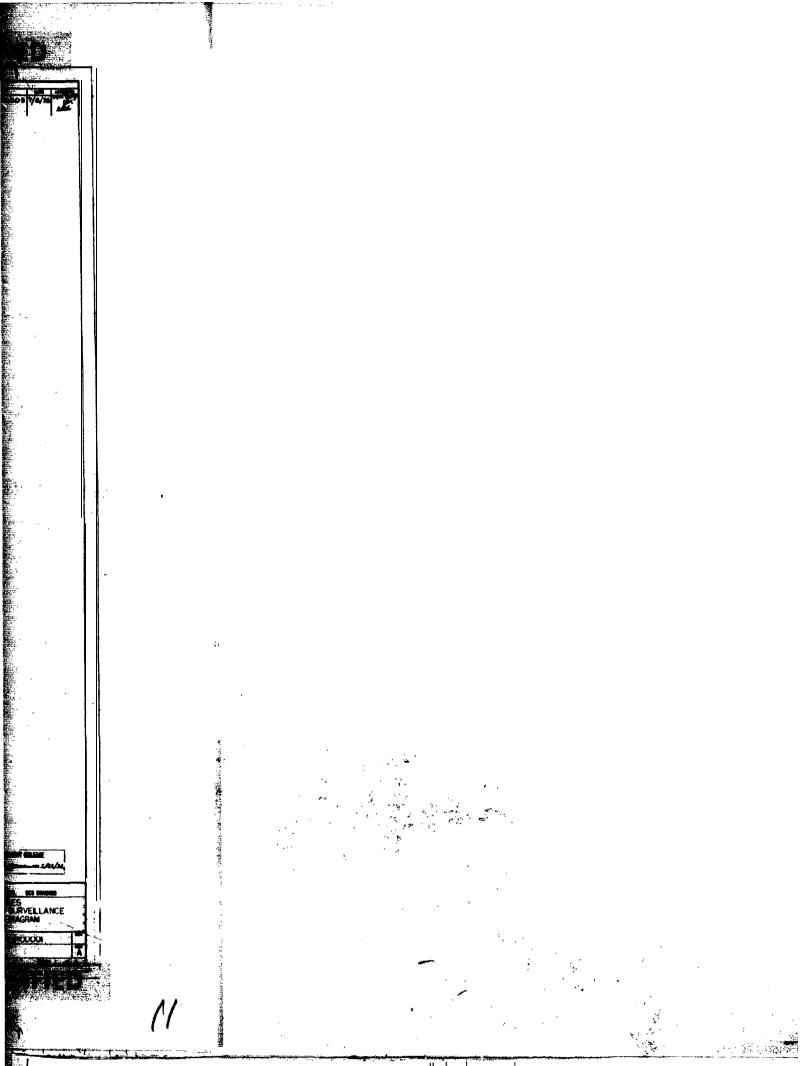
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10	MONITOR CONTROL CONSOLE 0J-200/UYA-4(V)	-
_9	COMM PATCH PANEL 58-2781/UYA-4(V)	
8	TEST SET TS-2460/UYA-4	
-7	WORK BENCH	
5	RADAR DATA DISTRIBUTION SWITCHBOARD SB-2780/UYA-4(V) CENTRAL EQUIPMENT GROUP OU-9(V)3/UYA-4(V)	<u> </u>
4	SIGNAL DATA CONVERTER CV-2953/UYK-7(V)	i
- 3	INPUT/OUTPUT CONSOLE OJ-178/UYK-7(V)	2
2	COMPUTER ANZUYK-7(V) (INCLUDING GROWTH) COMMAND AND SURVEILLANCE SYSTEM SWITCHBOARDS	3
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NOTE: THIS BLOCK IS NOT MAINTAINED S. RHODES UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES mg. ANGLES .XX. XX. .XXX 2.010 ±.010

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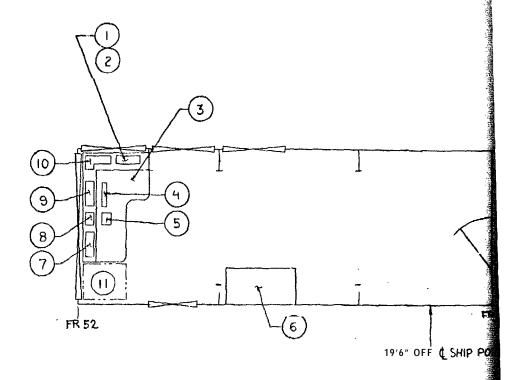
DATA PROCESSING ROOM ARRANGEMENT MAIN DECK

CODE IDENT 51563

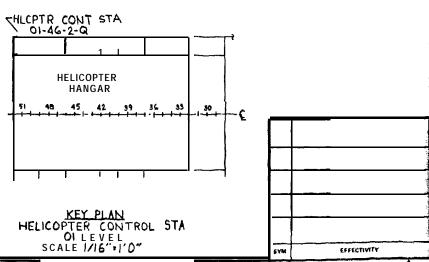
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Figure **B.5-4** (U)





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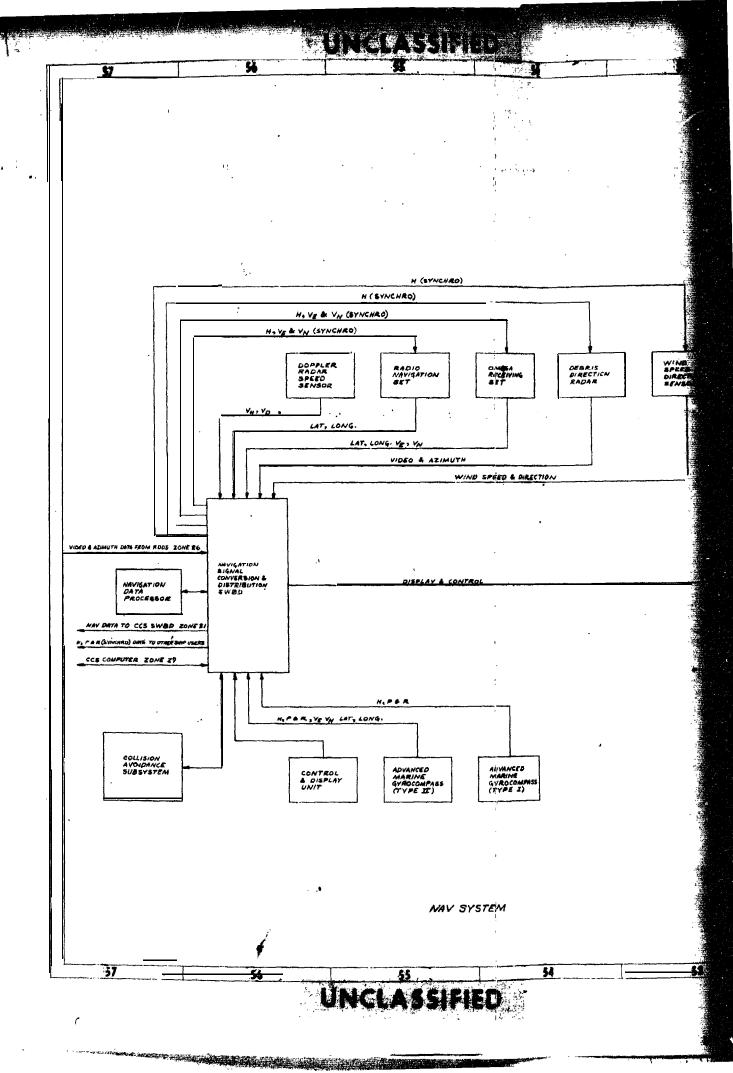


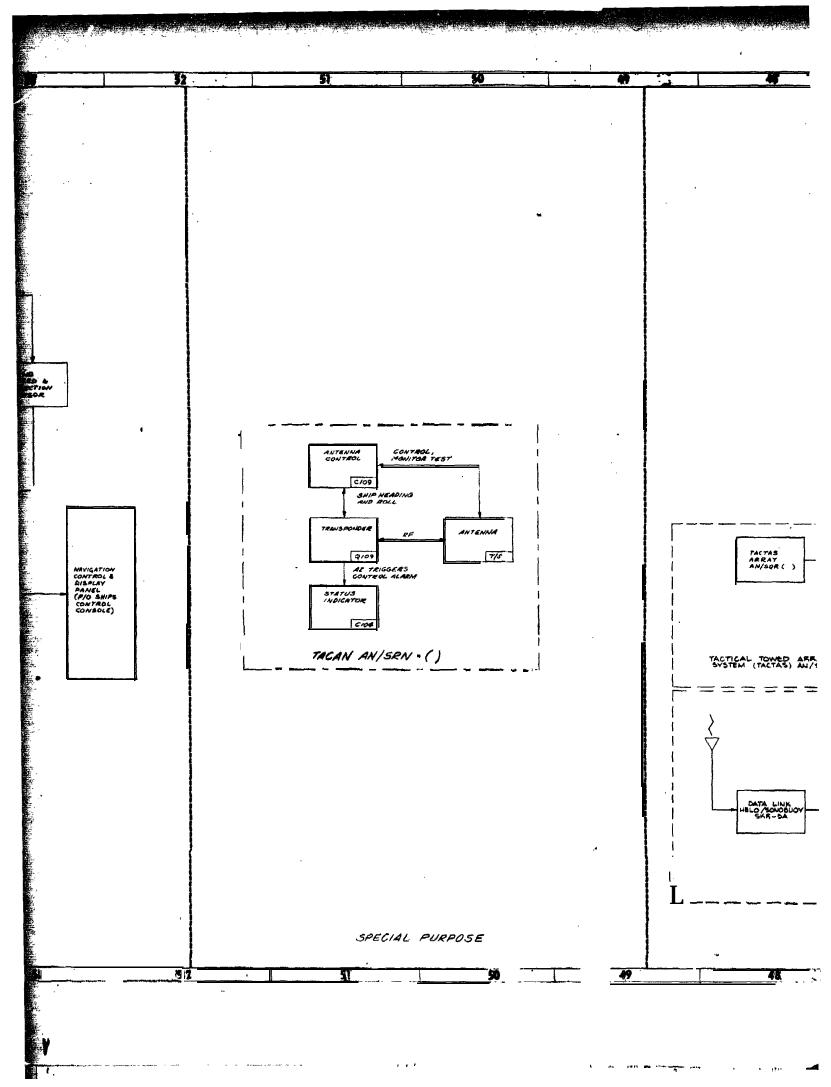
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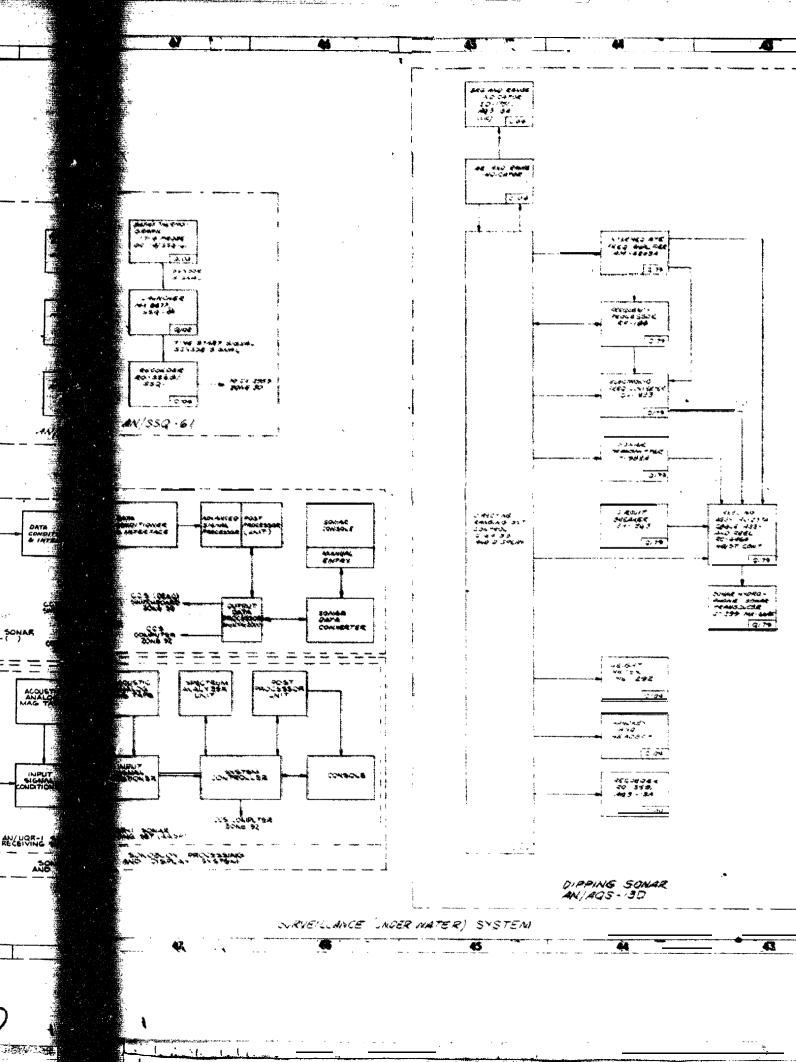
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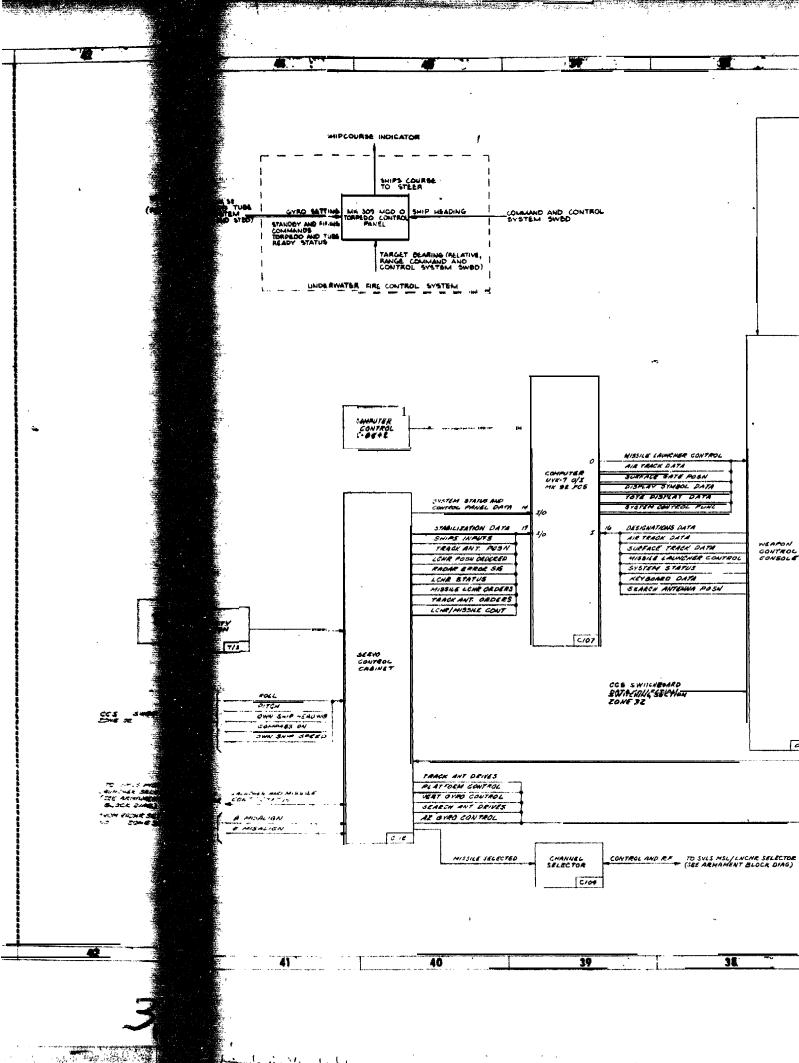
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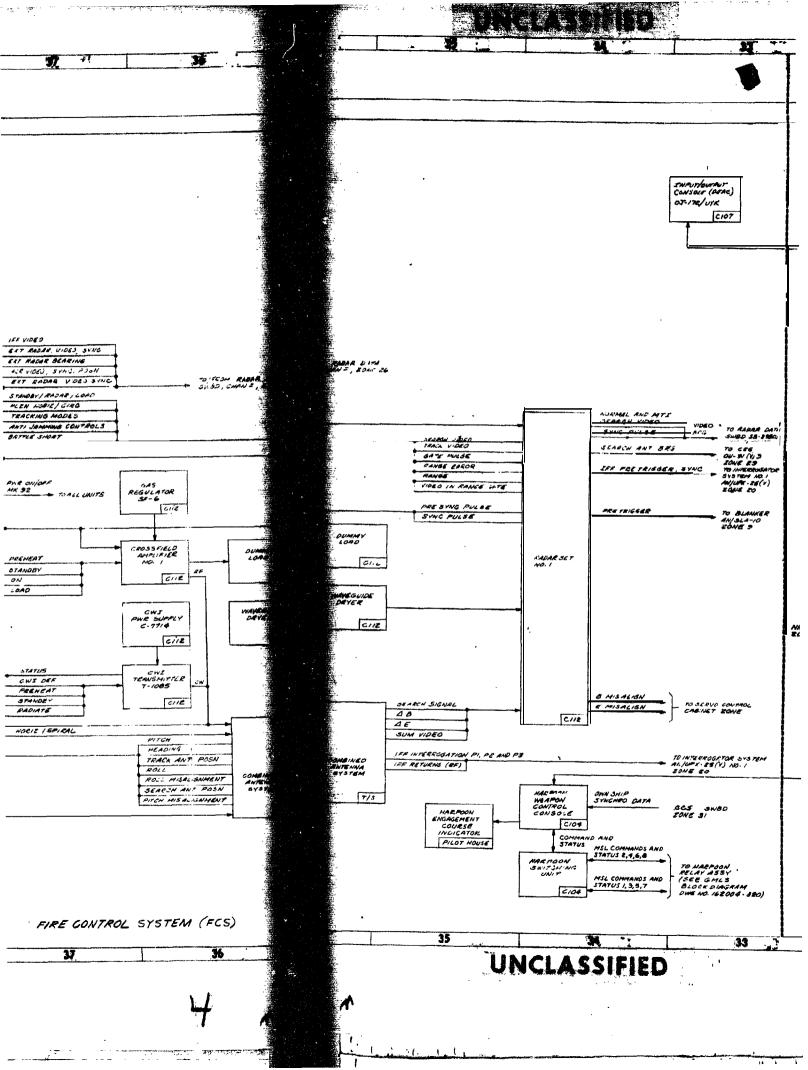
ZONE LIA -12-76 my//254 DESCRIPTION A REVISED PER ECR LOCOCS FR 46 PORT CONTROL CONSOLE RPY (SPACE ONLY)
VISUAL LANDING AIDS CONTROL PANEL
IC ALARM AND WARNING PANEL
ANEMOMETER AND COURSE INDICATOR UHF TRANSCEIVER CONTROL C-9059/URC EQUIPMENT FACK SECURING AND TRAVERSING CONTROL ALARM PANEL (WAVEOFF / CRASH / FIRE ALARMS) CONSOLE LIGHT PANEL HEATING / VENTILATION / AIR CONDITIONING CONTROL PANEL DOCUMENT RELEASE QTY PC NO DESCRIPTION mulation our 5/13/2 PARTS AND MATERIAL LIST NOTE: THIS BLOCK IS NOT MAINTAINED S. RHODES S/11/76 UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES ROHR INDUSTRIES, INC. The wall TOLERANCES ON: DECIMALS 5-12-76 Justina 5-13-76 HELICOPTER CONTROL STA XXX. XX. XX. XX. ±.010 **ARRANGEMENT** GENERAL SPECIFICATION OI LEVEL state 51563 CAL MY LES NEXT ASSY D MODEL NO. MEXT FINAL 5-11-24 APPLICATION بطديلك QTY REOD SHEET | DF Figure B B-52

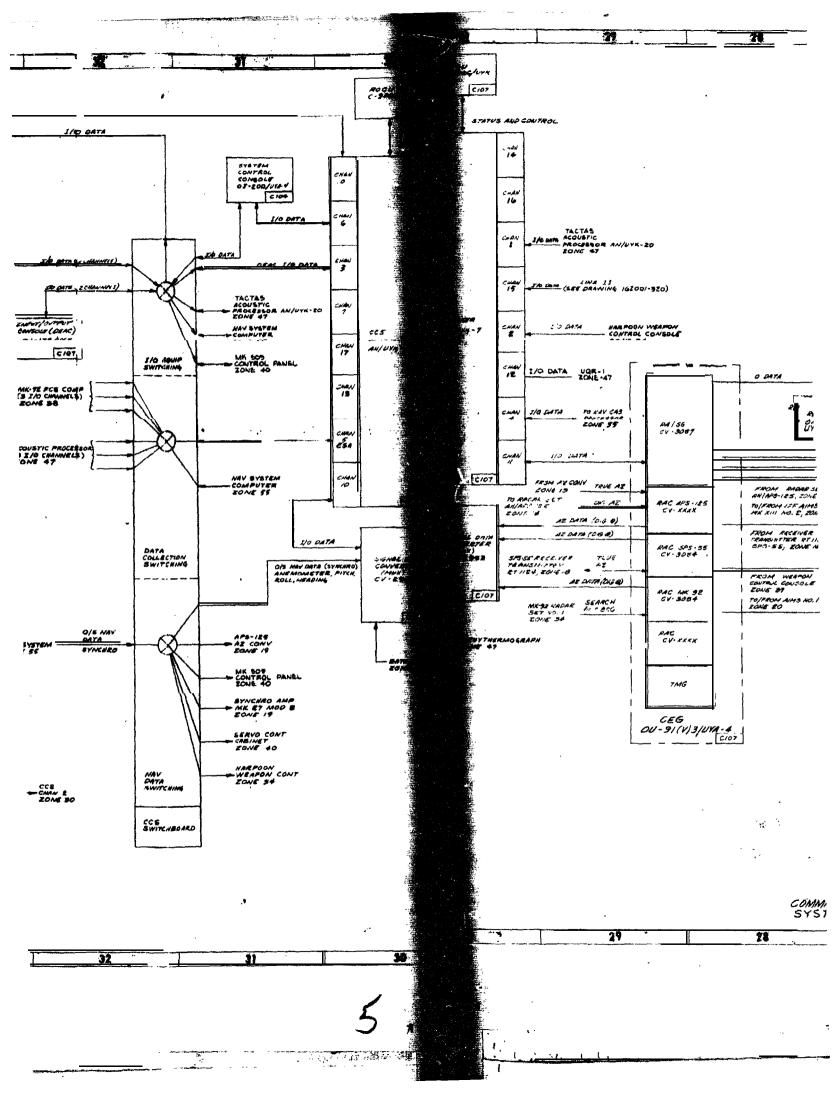


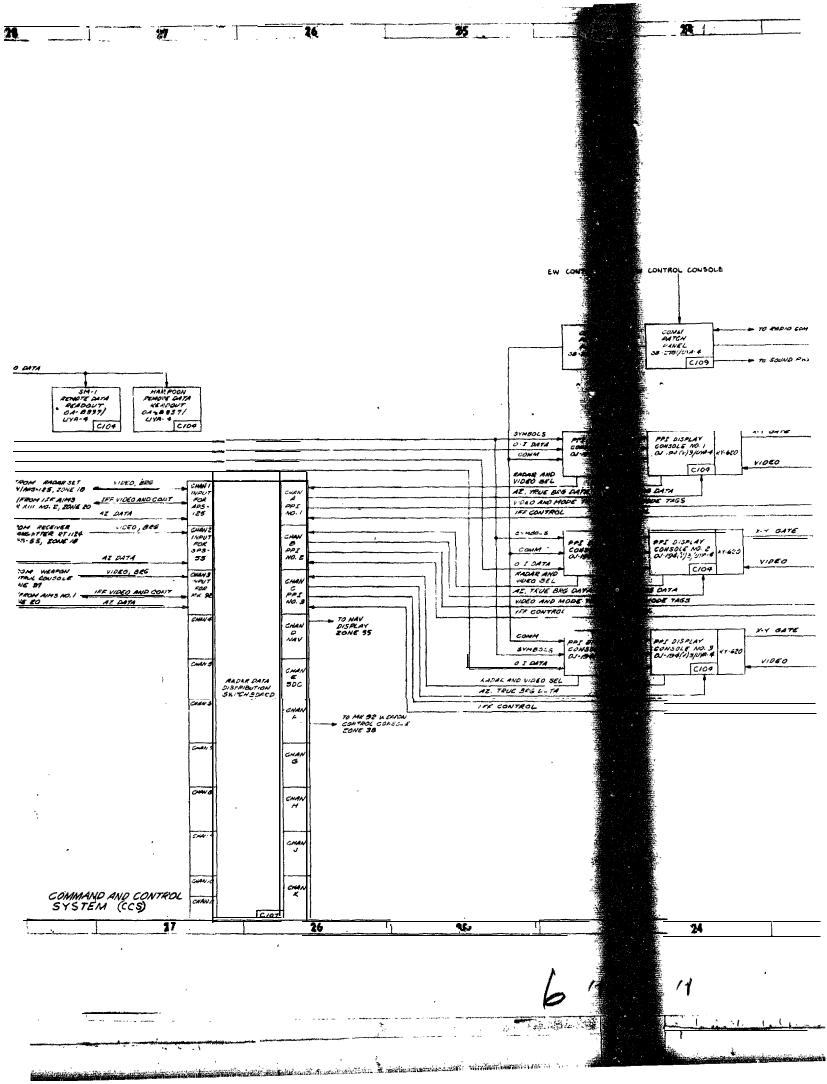


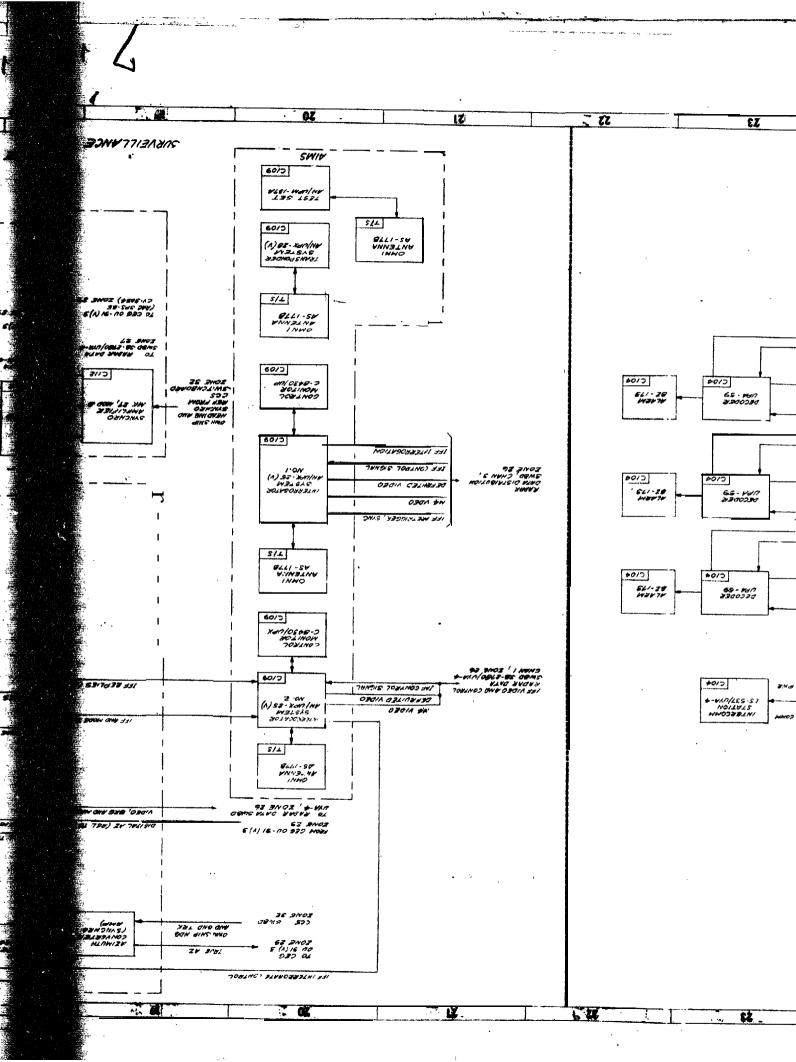


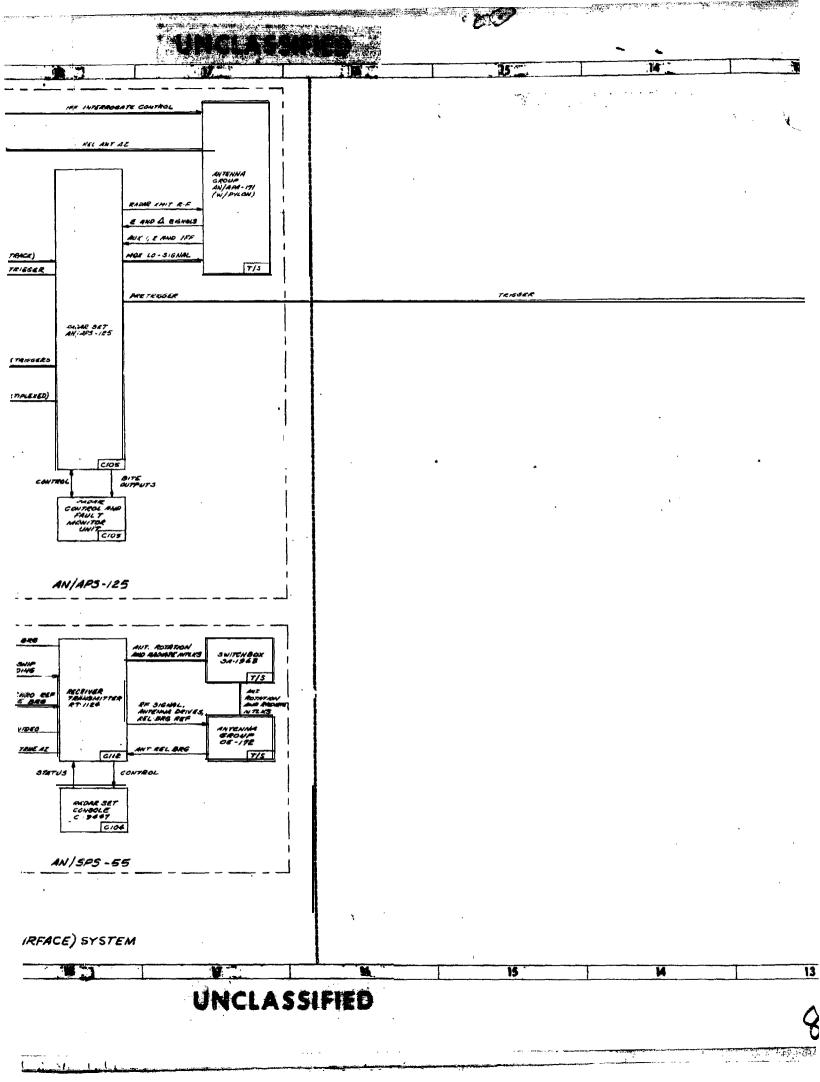












SPB-55 MK-92 SEARCH MK-92 TRACK DEBAS DETECTION RADAR OWN SHIP AN ANA TO INTERFACE UNIT IR SENSOR COOLING I/O DATA SIGNAL PROCESSOR DIGITAL DATA AN/UYK-20(V) SUPERHET RECEIVERS MECEIVER/PROCESSOR ANTENNAS OISPLAY AND 1 CONTROL SWITCHES RBOC CHAFF LAUNCHER CONTROL MAST

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DESIGN -TO PRICE ELECTRONIC WARFARE. SYSTEM A4/5LQ-3 (V)

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FIRE CONTROL SYSTEM (FCS)	39 THRU 42
SURVEILLANCE (WOERWATER) SYSTEM	43 THEU 49
SPECIAL PURPOSE SYSTEM	50 THRU EI
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Figure B.5-/ (U)

- (U) B.6 HULL INSULATION, SHEATHING, AND DECK COVERING SYSTEMS
- (U) This section of Appendix B consists of 29 Sheets of Rohr Drawing No.

 LL635001, "Hull Insulation, Sheathing and Deck Covering Systems".

 This drawing describes the cited systems covered in the text in Section 2.3.6.

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GENERAL NOTES

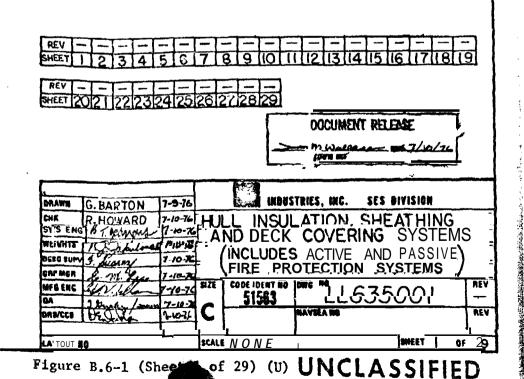
INSULATION

- A. ALL INSULATION MATERIALS INSTALLED SHALL COMPLY WITH APPLICABLE GOVERNMENT SPECIFICATIONS OR SHALL BE EQUIVALENT TO PRODUCTS IDENTIFIED.
- B. IN ADDITION TO THE ABOVE, ALL FIRE, THERMAL AND ACOUSTICAL INSULATION INSTALLED SHALL SATISFY THE REQUIREMENTS OF USCG INCOMBUSTIBILITY TEST 164.009.
- C . FIRE PROTECTIVE INSULATION SHALL BE ALUMINA/SILICA FELT OF FOUR-POUND DENSITY PER CUBIC FOOT (FIBRAFRAX OR EQUAL).
- D. THERMAL INSULATION SHALL BE FACED FIBROUS GLASS CONFORMING TO MIL-1-742. ALTERNATELY, UNFACED FIBROUS GLASS MIL-1-742 SHEATHED WITH.032 INCH THICK ALUMINUM SHALL BE USED...
- E. ACOUSTICAL INSULATION SHALL BE ONE-INCH THICK PERFO-RATED HARD SURFACE FIBROUS GLASS ACOUSTIC BOARD CON-FORMING TO MIL-A-2364 AND SHALL SATISFY THE RE-QUIREMENTS OF THE USCG INCOMBUSTIBILITY TEST. ALTER-NATELY ONE-INCH THICK, SOUND-ABSORBING, FIBROUS GLASS FELT MIL-I-Z2023, TYPE II AND SHEATHED WITH 1032 INCH PER-FORATED ALUMINUM, MINIMUM 10% FREE AREA SHALL BE USED.
- F. CLIPS ANGLE SUPPORTS & STAND-OFFS SHALL BE BONDED TO STRUCTURE TO ACCOMMODATE STANDARD PANEL FASTENER PATTERN (SEE SHEETS 26.27 8.28).
- G. PANEL BUTT & CORNER JOINTS SHALL BE SEALED BY COM-PRESSING 6 LB/FT³ REFRACTORY FIBROUS FELT STRIPS INTO GAPS & COVERING WITH CRES FLASHINGS
- H. DECKS REQUIRING FIRE PROTECTION (GROUP I & GROUP II)
 SHALL HAVE .25 INCH THICK CERAMIC FIREM
 MOIST FELT INSULATION BONDED TO THEM, USING A CERAMIC
 CEMENT. THICKNESS DEFENDS ON FIRE LOADING.
- I. WHERE REQUIRED AS SHOWN ON INSULATION ARRANGEMENT PLANS, FIRE, THERMAL AND ACOUSTICAL INSULATION SHALL BE COMBINED IN VARIOUS THICKNESSES AND FABRICATED INTO S-FOOT WIDE PANELS
- J. THE INSULATION PANELS SHALL BE OF A SANDWICH TYPE CON-STRUCTION CONSISTING OF AN ALUMINUM BACKING SHEET.516 INCH THICK, COMBINATIONS OF FIRE, THERMAL AND ACOUSTICAL INSULATION IN THICKNESSES INDICATED IN LEGEND AND A FACE SHEET AND EDGE CLOSURES AS FOLLOWS:
 - IN MACHINERY SPACES (GROUP I FIRE HAZARD), THE FACE SHEET SHALL BE .012 INCH THICK CRES WITH CRES CHAN-NEL TYPE EDGE CLOSURES.
 - IN AREAS OTHER THAN MACHINERY SPACES REQUIRING THERMAL INSULATION OVER FIRE INSULATION. OR ONLY THERMAL INSULATION, THE FACE SHEET SHALL BE .032 INCH THICK ALUMINUM. ALTERNATELY, THERMAL INSULATION SHALL BE HARD FACED INSULATION BOARD CONFORMING TO MIL-1-742.
 - IN AREAS REQUIRING ACOUSTICAL INSULATION OVER FIRE INSULATION, THE INSULATION SHALL BE HARD SURFACE FIBROUS GLASS ACOUSTICAL BOARD CONFORMING TO MIL-A-23054. ALTERNATELY, THE INSULATION AND FACE SHEET SHALL BE MIL-I-22023, TYPE II, SHEATHED .032 INCH PERFORATED ALUMINUM.

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RESERVATIONS

I. DECK INSULATION IN WAY OF LANDING PLATFORM AND HANGAR PENDING DEVELOPMENT.



B-56



GENERAL NOTES (CONTD)

• PANEL LENGTHS 'SHALL BE AS FOLLOWS:

DECK	BULKHEADS	OVERHEAD
OI LEVEL	8'-7" L G	6-0"LG
MAIN DK	7′- 7″ LG	6'-0" LG
2 ND D K	7'-7" LG	6'- 0" LG
3RD DK	7-7" LG	6′-0″ LG

2. EXTENT OF INSULATION

- A INSULATION SHALL BE FITTED TO UNDERSIDE OF DECKS, BULK-HEADS AND STRUCTURAL MEMBERS AS INDICATED ON INSU-LATION ARRANGEMENT PLANS AND THE FOLLOWING NOTES.
- B. INSULATION ON THE WARM SIDE OF THE VERTICAL SURFACES BOUNDING UPTAKE ENCLOSURES, MAGAZINES, AND FIRE INSULA-TION SHALL EXTEND FROM THE DECK TO THE OVERHEAD. IN ALL OTHER AREAS, THERMAL INSULATION ON VERTICAL SUR-FACES SHALL EXTEND FROM SIX INCHES ABOVE THE DECK TO THE OVERHEAD.
- C. INSULATION SHALL NOT BE INSTALLED IN WAY OF SHOWER STALLS OR BUILT-IN FURNITURE, EXCEPT DOWN TO THE DECK AND ALONG THE DECK FOR A WIDTH OF NINE INCHES FROM THE WEATHER BOUNDARY OR TO THE BACK OF SUB-EASE, WHICH-EVER IS LESS.
- D. WHERE ONLY A PARTIAL AREA OF A BOUNDARY REQUIRES IN-SULATION, THE INSULATION SHALL BE INSTALLED SUCH AS TO EXTEND 12 INCHES BEYOND THE AREA REQUIRING INSULATION.
- E. BOUNDARIES ABUTTING INSULATED BOUNDARIES WHERE INSU-LATION IS NOT OTHERWISE REQUIRED, SHALL BE INSULATED FOR A DISTANCE OF 12 INCHES FROM SUCH INSULATED BOUNDARIES.

10	LSES PRESERVATIVE & COVERINGS SPEC	51563	L24630001
9	ACTIVE FIRE PROTECTION SYSTEM	51563	LB555001
8	LSES SYSTEM SPECIFICATION	51563	L01000001
7	AIR CONDITIONING, VENTILATION AND HEATING DESIGN CRITERIA MANUAL		0938 - 018 - 0010
	INLET INSTL-LIFT SYSTEM	51563	LS567020
	ARR-INTAKE SYS, COMBUSTION AIR P/S	51563	LL251001
	LSES GENERAL ARROMT - 3RD DECK	51563	LL802006
	LSES GENERAL ARRGMT - 2ND DECK	51563	L L802005
2	LSES GENERAL ARRGMT - MAIN DECK	51563	L L802004
	LSES GENERAL ARRGMT - OI LVL & ABOVE	51563	LL802003
NO	TITLE	CODE	MFR OR CONTRINO
	REFERENCES		

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METHODS REQUIRES

M. EXPOSED GORY I SI INSULATIO TRUSSES I 1-1/2 INCH

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POORS TO MAGAZINE AND MISSILE STORAGE SPACES SHALL BE INSULATED WITH ONE-INCH BOARD.

COUSTICAL TREATMENT. WHEREVER ACOUSTICAL ABSORPIVE TREATMENT IS REQUIRED FOR NOISE REDUCTION IN ROOMS,
OORS TO THESE SPACES SHALL BE COVERED ONLY IF THE ADNTIONAL AREA IS NECESSARY TO OBTAIN THE DESIRED
REDUCTION.

ME SURFACE OF THE ACOUSTICAL ABSORPTIVE TREATMENT #ALL BE PAISTED AS SHOWN IN PAINTING SCHEDULE.

WHERE ACOUSTICAL ABSORPTIVE TREATMENT IS REQUIRED ON VERTICAL SURFACES, TREATMENT SHALL BE ELIMINATED BEHIND ITATUS BOARDS AND LARGE EQUIPMENT WHICH HIDES THE BULKHEADS AND WHERE SOLID SHEATHING IS REQUIRED.

MERE ALUMINUM SHEATHING IS USED, THE SHEATHING MAY E-FORMED INTO PANS AND SECURED TO THE STRUCTURE WITH HROUGH CONNECTIONS.

MILESS OTHERWISE SPECIFIED, IF ACOUSTICAL TREATMENT IS LEQUIRED FOR ANY AREA IN WHICH THERMAL INSULATION IS PECIFIED, ONLY THE ACOUSTICAL TREATMENT SHALL BE APLIED TO THE PLAIN SURFACES, AND THERMAL INSULATION, IF LEQUIRED, SHALL BE APPLIED TO BEAMS AND STIFFENERS.

MERE ACOUSTICAL ABSORPTIVE TREATMENT IS REQUIRED FOR IN OVERHEAD IN WHICH A DROPPED CEILING MUST BE IN-TALLED, ONE OF THE FOLLOWING METHODS SHALL BE USED:

- PERFORATED ALUMINUM SHEATHING SHALL BE INSTALLED AT THE DESIRED HEIGHT AND SOUND-ABSORBING FIBROUS GLASS FELT SHALL BE INSTALLED DIRECTLY ABOVE AND UPON IT.
- PERFORATED HARD-SURFACE FIBROUS GLASS ACOUSTICAL ABSORPTIVE BOARD SHALL BE INSTALLED AT THE DESIRED HEIGHT WITH NO SHEATHING REQUIRED.

ETHODS ALL ABOVE DO NOT ELIMINATE THERMAL INSULATION EQUIREMENTS FOR THE OVERHEAD.

(POSED STANCHIONS AND TRUSSES SHALL BE INSULATED IN ALL CATE-)RY I SPACES WITH A 3/4-INCH THICKNESS OF MOLDABLE FIBER MOIST FE ILT SULATION, THE MOIST FELT WILL BE BONDED TO THE STANCHIONS AND IUSSES WITH CERAMIC CEMENT. THE MOIST FELT SHALL BE OVERLAPPED 1/2 INCHES TO PREVENT A DIRECT PATH TO THE PROTECTED MEMBER.

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GENERAL NOT'S (CONT'D)

3. INSULATION PROTECTION

A INSULATION ON BULKHEADS OF COMMISSARY IN WAY OF HEAT-PRODUCING COMMISSARY EQUIPMENT AND VEGETABLE-PEELING MACHINES SHALL BE SHEATHED, INSULATION ON THE OTHER AREAS ADJACENT TO THESE FIXTURES WHICH MAY BECOME WET OR COATED WITH GREASE OR IN ANY AREA WHERE INSULATION IS SUBJECT TO DAMAGE OR EXPOSED TO HEAVY TRAFFIC. SHALL ALSO BE SHEATHED.

INSULATION BEHIND LAVATORIES, SERVICE SINKS. WATER CLOSETS, AND FOOD PREPARATION TABLES SHALL BE SHEATHED FROM THE DECK TO AT LEAST TWO FEET ABOVE THE WORKING SURFACE OF THE FIXTURES

- B. EXPOSED EDGES AROUND AIRPORTS, **DOORS** AND EXPOSED EDGES IN OTHER LOCATIONS WHERE SUCH EDGES ARE SUBJECT TO DAMAGE SHALL EE PROTECTED BY LIGHT 2 OR FLAT BARS.
- C. EXPOSED EDGES OF INSULATION NOT SUBJECT TO DAMAGE SHALL BE PROTECTED WITH FIBROUS GLASS TAPE.
- D. INSULATION IN PASSAGEWAYS AND OTHER AREAS SUBJECT TO HEAVY TRAFFIC, SHALL BE SHEATHED FROM THE DECK TO AT LEAST 36"ABOVE THE DECK. SHEATHING SHALL BE CRES.019
 AISI TY-304, FIN 4
- E. WHEN ATTACHING CRES SHEATHING IN WAY OF ALUMINUM STRUCTURE, ALL SUPPORT ANGLES AND COAMINGS TO BE INSU-LATED WITH DIELECTRIC TAPE.

4 INSTALLATION

A. INSTALLATION PROCEDURE SHALL BE IN ACCORDANCE WITH THIS PLAN.

REPAIR

A WHERE PRACTICABLE, DAMAGED FACED FIBROUS GLASS BOARD SHALL BE COVERED WITH GLASS CLOTH AND CEMENT.

6. ANTISWEAT TREATMENT

A. ANTISWEAT TREATMENT TO BE APPLIED IN ACCORDANCE WITH PAINT SCHEDULE.

VAPOR BARRIER

A VAPOR BARRIER COATING CONFORMING TO MIL-C-19993 OR EQUIV SHALL BE APPLIED TO THE EXPOSED SURFACE OF ALL INSULATION WITHIN LAUNDRIES, SCULLERIES, AND GALLEYS. ONE HUNDRED PERCENT COVERAGE SHALL BE PROVIDED BY ANY SINGLE COAT OF VAPOR BARRIER COATING SEE PAINT SCHEDULE FOR APPLICATION.

8. PAINTING

- A STRUCTURE BEHIND HULL INSULATION SHALL **NOT** BE PAINTED UNLESS IT IS A WEATHER BOUNDARY OR BETWEEN AIR CONDITIONED AND NON-AIR CONDITIONED SPACES. SEE PAINT SCHED-ULE FOR FINISH PAINTING OF INSULATION.
- B. FORWARD SIDE OF FIRE ZONE BULKHEADS SHALL RECEIVE TWO COATS OF THERMAL INSULATING (INTUMESCENT) PAINT, MIL SPEC MIL-C-46081.

DECK COVERING

- A. DECK COVERING AN RANGEMENT DRAW!
 TRATE THE INTEGRE WITH THE DECK CO.
 ING SPECIFICATION REGARDING DECK CO.
- a UNDERLAY PC 20 88
 DECK COVERING IS
 AND UNDERLAY PC
 DECK SEAMS ARE T
- C. DECK COVERING SHA ELECTRONIC EQUIPM PANELS SEE DWG L MOUNTED EQUIPMEN

10. ACTIVE FIRE PROTECTION

THE ACTIVE FIRE PARARANGEMENT DRAW
TRATE THE INTEGRA
PROTECTION SYSTEM
PLETE INFORMATION
SYSTEM.

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ING AND DECK INSULATION IS SHOWN ON THE AR-PRAWINGS. (SEE SHEETS 17 THROUGH 25) TO ILLUS-ITEGRATION OF THE PASSIVE FIRE PROTECTION CK COVERING, WHERE REQUIRED. SEE DECK COVER-ATION NO L2463001 FOR COMPLETE INFORMATION DIECK COVERING.

20 SHALL BE USED ONLY WHERE NO IS TO BE SLOPED FOR DRAINAGE AY PC 21 SHALL BE USED ONLY WHERE ARE TO BE FAIRED.

NG SHALL NOT BE INSTALLED UNDER FURNITURE SUB-BASES, EQUIPMENT, MECHANICAL EQUIPMENT, OR BEHIND BULKHEAD DWG LL802004 THRU LL802007 FOR EXTENT OF DECK DIPMENT, ETC.

ECTION SYSTEM

THE PROTECTION SYSTEM IS INDICATED ON THE DRAWINGS (SEE SHEETS 8 THROUGH 11) TO ILLUSTEGRATION OF THE ACTIVE AND PASSIVE FIRE TYSTEM. SEE DRAWING NO LL555001 & 2 FOR COMPATION REGARDING THE ACTIVE FIRE PROTECTION

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<u> </u>		MATER			NTIFICATION LIST		
TEM NO	DESCRIPTION			HT-LBS TOTAL		REF TO	SPECIFICATION
1	4 LBS/CU FT	47,906.16 SQ FI SQ	66 3 F T	7052			CARBORUNDUM FIBER-
2	I" FIRE INSUL 4 LBS/CU FT	15,836 .3 SQ FT SQ	33) FT	5000	CERAMIC FBRS FELT		CARBORUNDUM FIBER-
·3	! ACOUSTIC INSUL 2 LBS/CU FT	37,467 .I SQ FT SC	17 Q FT	6,369	FIBROUS GLASS FELT		MIL-1-22023 TYPE I CL 5 OR MIL-A-23054
4	I' THERMAL INSUL	SQ FTISQ		2,010	FACED FIBROUS GLASS BOARD	<u> </u>	MIL-1-742
5	2" THERMAL INSUL	<u> </u> SQ_FT SQ	.6I 3 FT	1,230	FACED FIBROUS GLASS BOARD		MIL-1-742
6	OI2" FACE SHEET	48,196 .4 SQ FT ISQ	48 32 FT	23,134	CRES SH		QG-A-766 TY PE 304 FINISH 2B
7.	OI6 BACK SHEET	18,95.1.3 S. FT(SO	23 2 F T	20,692	ALĮSH		00-A-250/8 5052-H32
8	.012" PANEL CLOSURE	82,535 .I	18 IN FT	1	1	'	QQ-A-766 TYPE 304 FINISH 2B
9	54 MIL ACOUSTICAL SHEATHING	3,307 .8 SQ FTSC	87 Q F T	, ,	LEAD VINYL		SOUND FAB OR EQUIV
10	1/2" THERMAL INSUL	2,144 .19 SQ FT SC	QFT	418	FACED FIBROUS GLASS BOARD	<u>'</u>	MIL-1-742
11	114" FIRE INSUL - DK 15 LBS/CU FT	17,302 .3 SQ FT SQ	31 3 F T	5764	LACOLANIC CLUCO		REFRACTORY PROD CO WRP X AQ OR EQUIV *
12	ADHESIVE	17,302 .1 SQ FT SC	QFT	1770	CERAMIC;	} [OR EQUIV
13		17,302 .0 SQ FTS0	08 Q F T	4 004	FIBROUS GLASS		MIL-C-9084 TYPE II CLASS 2
14	RESIN	17,302 .1 SQ FTSG	10 Q FT	4 700	EPOXY RM TEMP CURE		SHELL EPON 934 OR EQUIV *
15	DECK TILE	ISQ FT SG		9,701	VINYL ASBESTOS		MIL-T- 8830
16	RUG	3,732 .5 SQ FTSG	50	1066	BETA FIBROUS GLASS	0	AROLINA NA RROW FABRIC CO OR EQUIV
17							
18	SLIP RESISTANT COVERING	22,624 .2 SQ FT SQ	FT	1 '	EPOXY PLUS AGGREGATE	1 1	MIL-D-23003 TYPE II
19	EPOXY COVERING	1,504 2. SQ FT SC	2.88 Q FT	4,332	IN MIDE I LOS MODINEDAS E		
20	UNDERLAY 18'TO	0 608 I. KISC FT SC	.75 * Q F T	1,064	LATEX MASTIC ** *MEAN THICKNESS 1/4*	NOTE II	MIL-D-3135 TYPE I
21	UNDERLAY FEATHEREDGE	7,795 I. SQ FT SQ	1.0 * Q FT	7,795	MEAN THICKNESS INC	111	MIL-D-3135 TTPE II
22	BOTTOM CHANNEL	6,922 .4 LN FT LN	463 NFT	3,205	ALUMINUM 11/2 24 x.090		QQ-A-200/B 6061-T6
1	·	41,440 .18	80 NFT	7.450	ALUM IX25X1" E .062 COMPCOR TYPE C-1 THK		QQ-A-200/8 6061-T6
24	SUPPORT CLIP	TLN FT I I	06 NET	1297	ALUMINUM I"x.090		QQ-A-200/5 5086-H I I I
25	INSULATOR PAD		017	344	14 (85)(1) [1] 272 15 16		CARBORUNDUM FIBER- FRAX OR EQUIV
26	ANGLE 2 3 .062	L649 P 7 L.		1,384	COMPOOR		QQ-A-200/8 6061-T6
27	l"x l"x.0235	10 4 43 .1 LN FT L1	65 NET	1,723	.0235 CRES		QQ-A-766 TYPE 304 FINISH 2B
							

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		MAT	ERIAL		NTIFICATION LIST	,	
ITEM NO	DESCRIPTION	QTY	WEIG	HT LBS	MATERIAL	REF TO	
28	SEALING STRIP 2 1/2 W	43,351 LN FT	.205 LNFT	8,887	.0235 CRES		QQ-A-766 TYPE 3 FINISH 2B
29	RIVET, BLIND	20,238			1/8" DIA CRES LE TO SUIT		;
30	ANGLE 4" 4" × .062THK	6,992 LN FT	.32 LNFT	2,215	ALUMINUM COMPCOR		QQ -A -200/8 6061-T6
31							
32	SPACER 3/8 DIA x 2" LG	20,238	,125	2,530	CERAMIC		
33							
34	CHANNEL I'x 2'x I'	6,922 LN FT	.329 LNFT		0235 THK CRES		QQ-A-766 TYPE 3 FINISH 2B
35	COAMING	6,922 LN FT	.588 LNFT	4,070	4"x 1/8" AL FLAT BAR		QQ-A-200/5 5086-HIII
36							
37	RIVET BLIND	13,834	0625	865	% DIA CRES LG TO SUIT		
38					·		e e e e e e e e e e e e e e e e e e e
39	DECK INSULATION	6,922 LN FT		2,146	CERAMIC FIBER MOIST FELT 1/4"THK × 3-1/4" W		REFRACTORY PROD WRP-X-AQ OR EQU
40							
41	ADHESIVE (RUG)	3,732 SQ FT	.10 SQ FT	373			3-M CO BLUEGLUE OR EQU
42	ADHESIVE (TILE)	9,701 SQ FT	.10 SQ FT	882			MIL-A-21016
43	.032 FACE SHEET (PERFORATED)	2.9,518 SQ FT	.384 SQ FT	11,335	ALUMINUM SHEET		QQ-A-250/8 5052~H32
44	ISOLATION STRIP	28,900		462	RUBBER 1/8 THK		1x 2/2
45		6,922 LN FT	ILVIETI	914	1 1/6 THK		34 WIDE
46	1/2 FIRE INSULATION 4 LBS/CU FT	43,351 SQ FT	.034 SQ FT	1,474	CERAMIC FIBROUS .		CARBORUNDUM FIB FRAX OR EQUAL
47	.032 FACE SHEET	12,254 SQ FT	.452 SQFT	5,539	ALUMINUM SHEET		QQ-A-250/8 5052-H32
48							
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FIRE HAZARD CATEGORIZATION

SPACES ARE CATEGORIZED THRU 4 ACCORDING TO THE ANTICIPATED FIRE LOADING AND POTENTIAL FIRE HAZARD OF THE SPACE

FIRE LOADING IS DEFINED BY THE WEIGHT OF COMBUSTIBLES PER SQUARE FOOT (LBS/SQ FT), SEE TABLE BELOW,

GROUP	DEGREE OF HAZARD	FIRE LOADING
	HIGH (MCHRY SPACES)	IO LBS/SQ FT
2	MODERATE TO HIGH	5-10 LBS/SQ FT
3	MODERATE	3-5 LBS/SQ FT'
4	Low	O-3 LBS/SQ FT

A=ACOUSTICAL CATEGORY - SEE SHEET 7

(A) = THERMAL CATEGORY -SEE REF NO 7

ITEM. NO	DECK COVERING	SS
15	DECK TILE	#
16	RUG	
18	SLIP RESISTANT COVERING	1
19	EPOXY	********

INCLUDING ACOUSTICAL AND

	# tot lites honoo	-
ITEM	PASSIVE	FIRE P
NO	THICKNESS	MATERIA
	1/2*	FIRE
2	1"	INSULA
-	•	ACOUST
3		INSULA
4	1, 8HD	THERM
5	2" OVHD	INSULA
10	II 2 BHO	THERM
4	1 ⁵ 0VHD	INSULAT
		SHEATHI
1&3	** 1/2" F + 1"A	FIRE A
2&3	"F+1"A(MACH SP)	ACOUST
10&3	1/2 T+1 A (BHD)	THERMAL
4&3	1 T+1 A (OVHD)	ACOUST
3&4	** IT+IA(BHD)	THERMAL
3&5	** 2 T+1 A(OVHD)	ACOUST
9&3	٠, ١, ١, ١,	LEAD VIN
70.3	16 LV+ 1A	& ACOUST
1,349	1/2F+1/16 LV+ 1 A	FIRE LEAD VIN
	***	AND
2,349	1 F+1/16 LV+1 A	ACOUS
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MD THERMAL INSULATION
THERMAL, LV=LEAD VINYL

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RMAL LATION	†#####		I8 [®] DRY CHEM EXTINGUISHER	•
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SPACE/COMPARTMENT AIRBORNE NOISE CATEGORIE

Q LEVEL.		01 LEVEL		MAIN OECK		2ND DECK		JAD DECK	
SPACE/COMP	CATE- GORY	SPACE/COMP	CATE- GORY	SPACE/COMP	CATE	SPACE/COMP	ATE-	SPACE/COMP 6	
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PACES ANERE IT IS ESSENTIAL TO MAINTAIN HET CONDITIONS.

HIGH PASSE LEVEL AREAS WHERE VOICE COM-S NOT PASORTANT; WHERE EAR PROTECTION ED, AND PREVENTION OF HEARING LOSS IS THE EDERATION.

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> * Prospective Category Change Upon Space Assignment

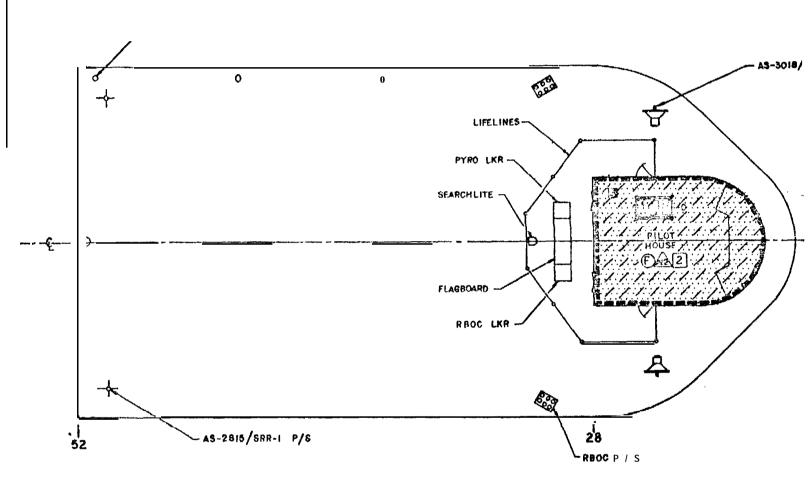
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02 LEVEL
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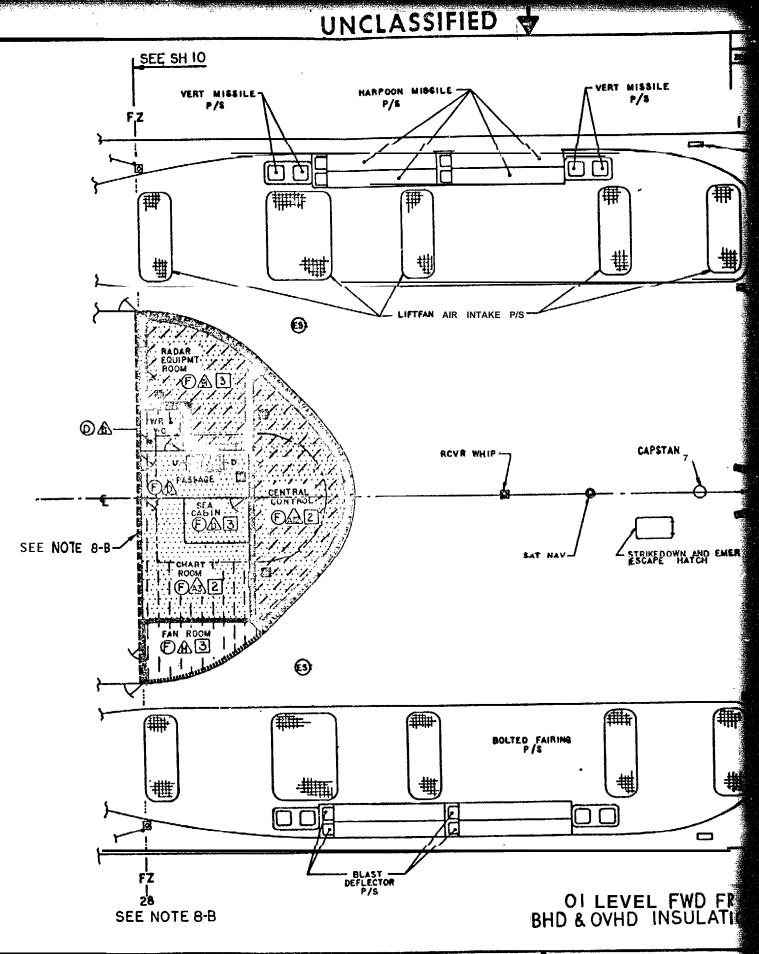
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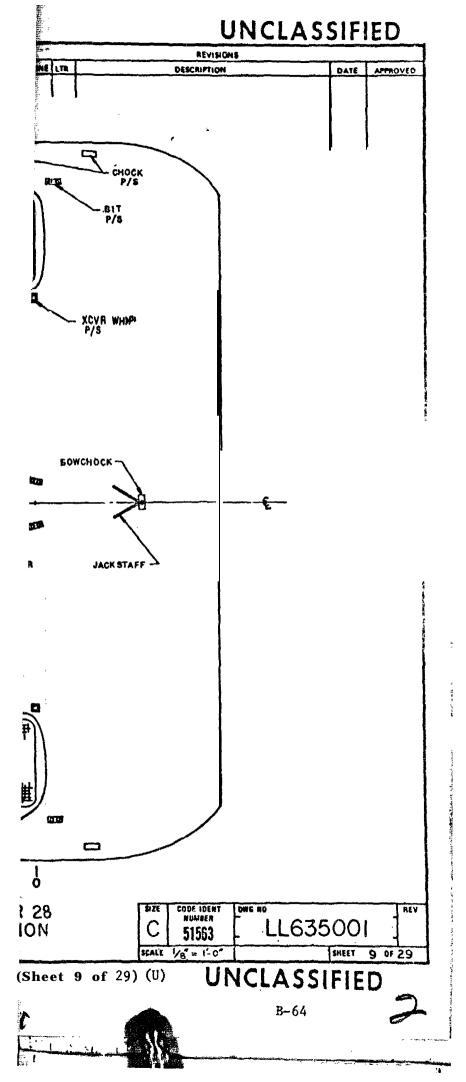
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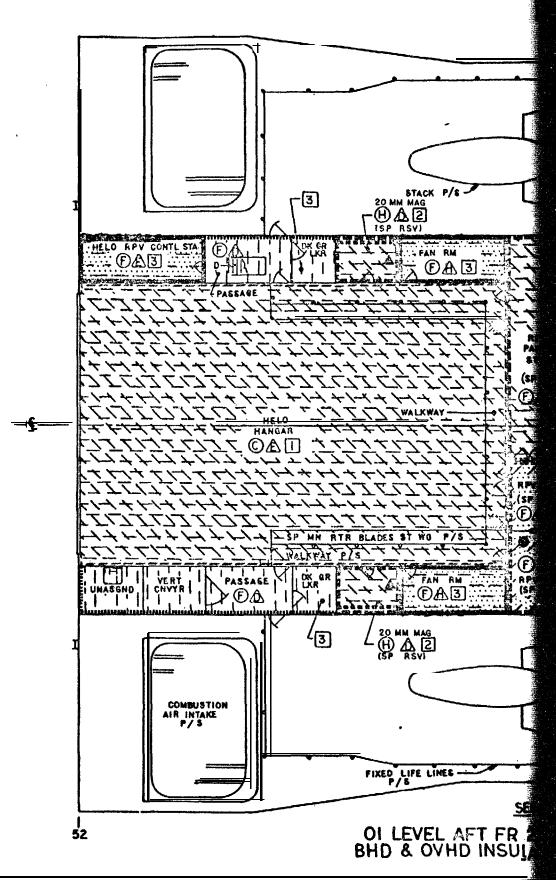
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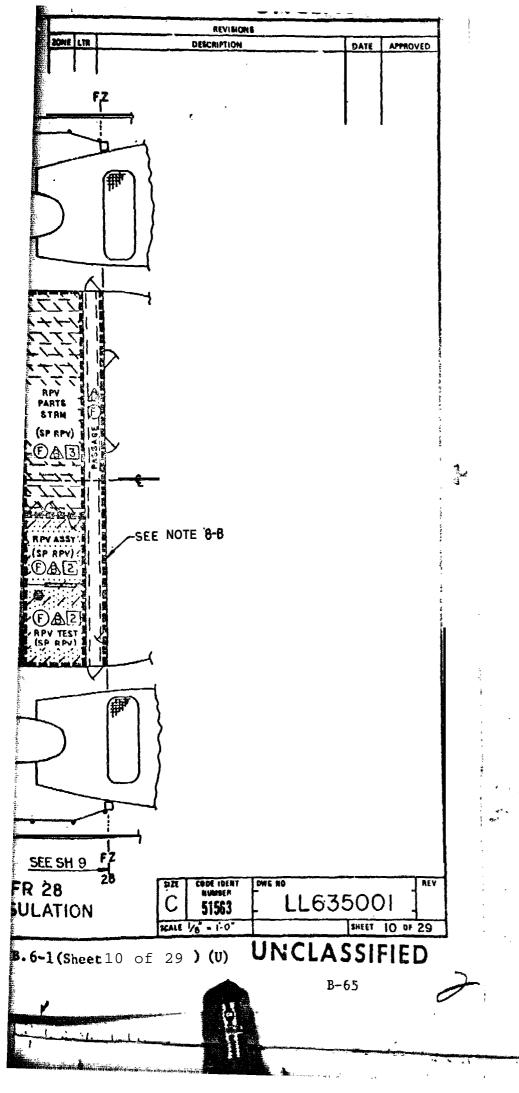


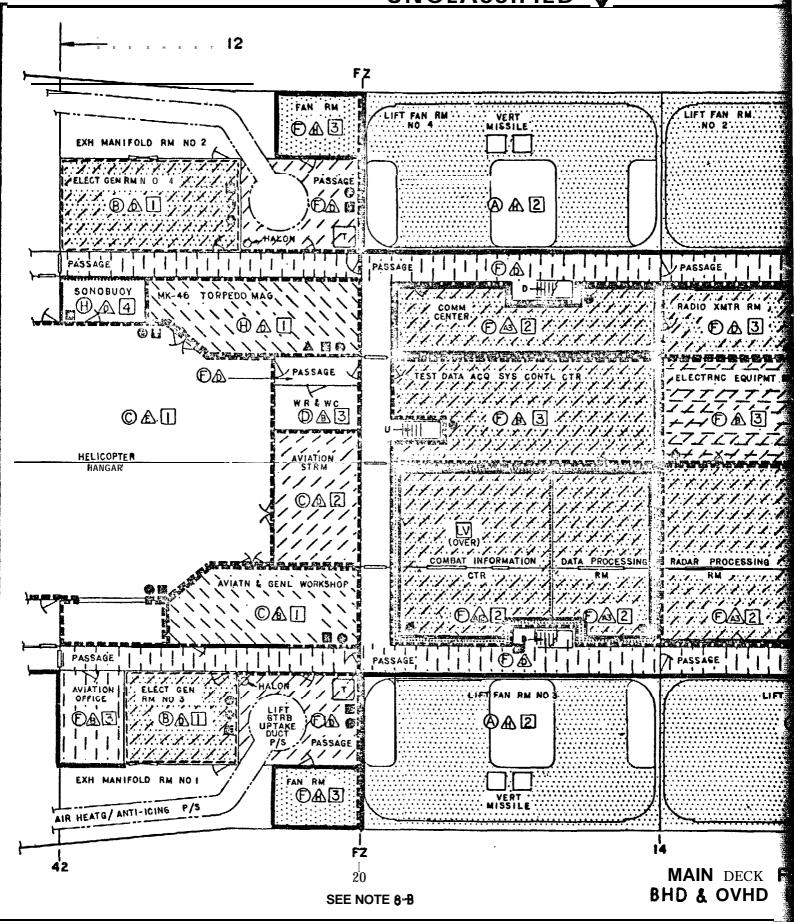




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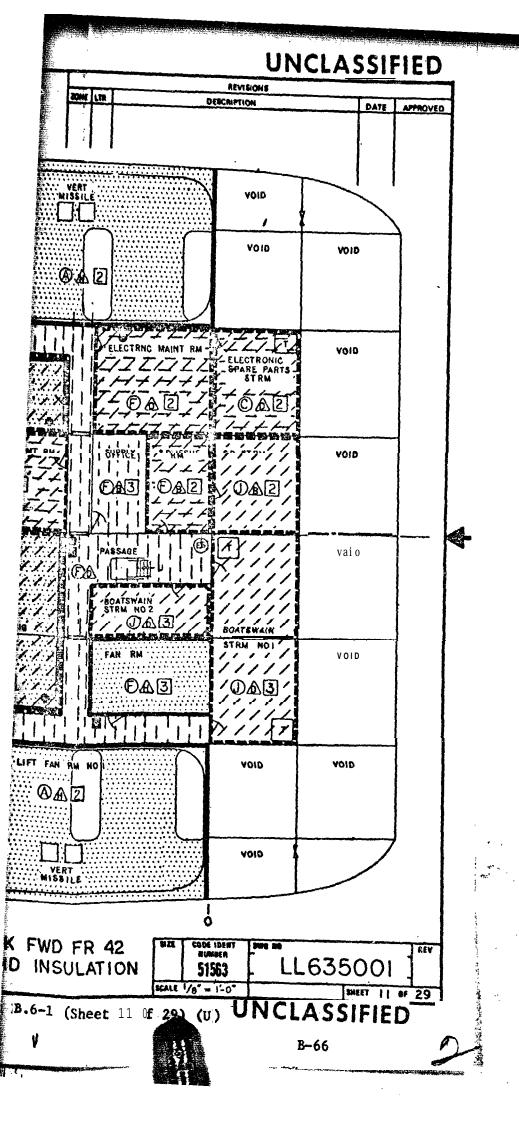
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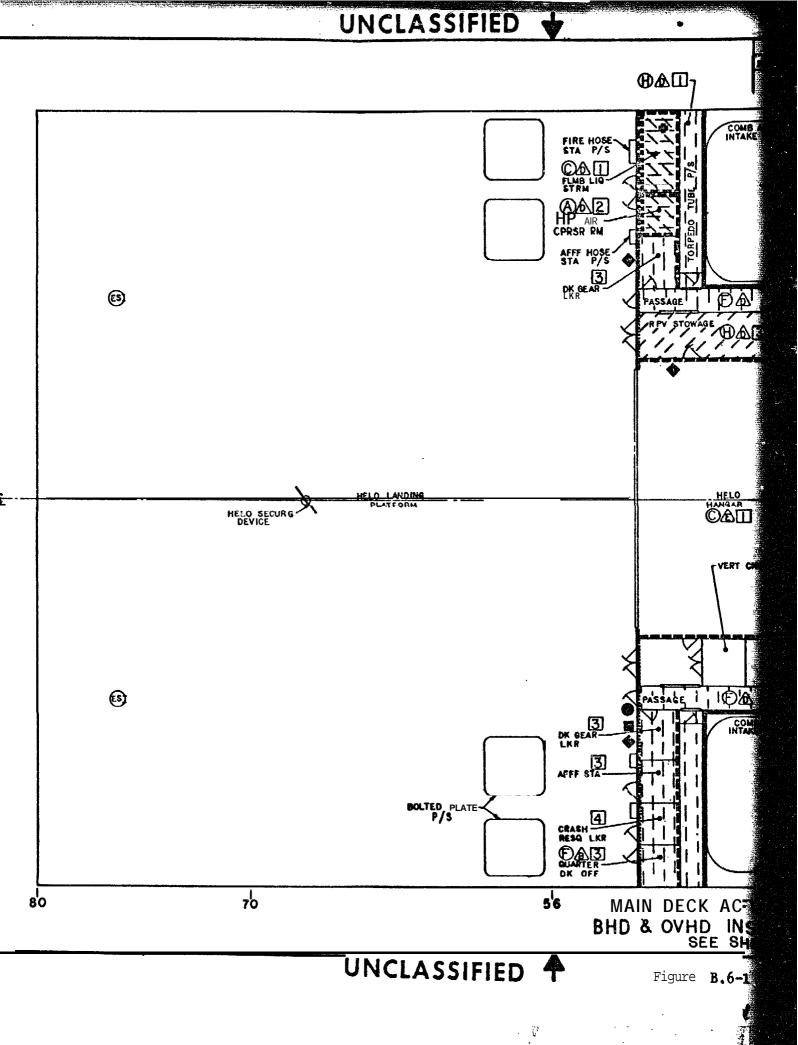


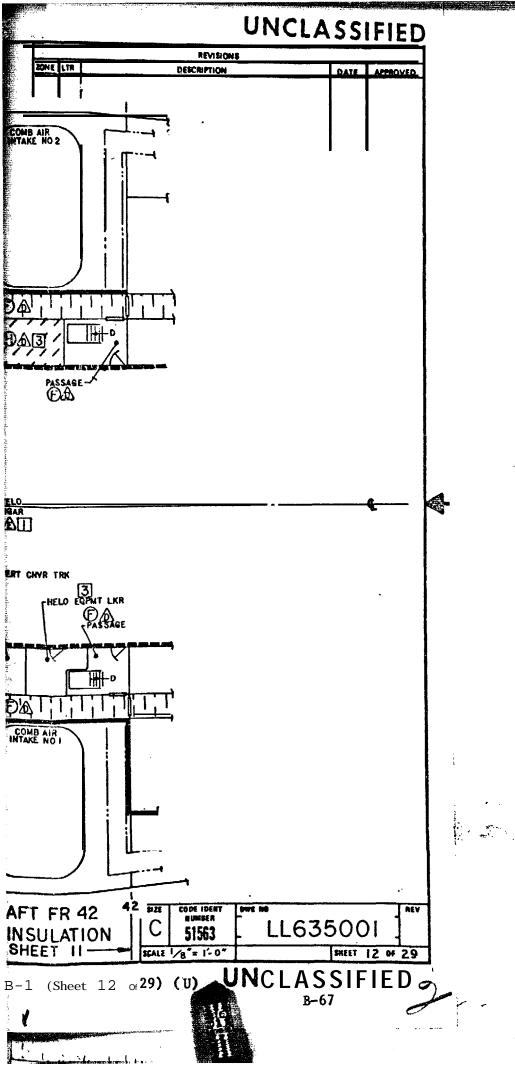


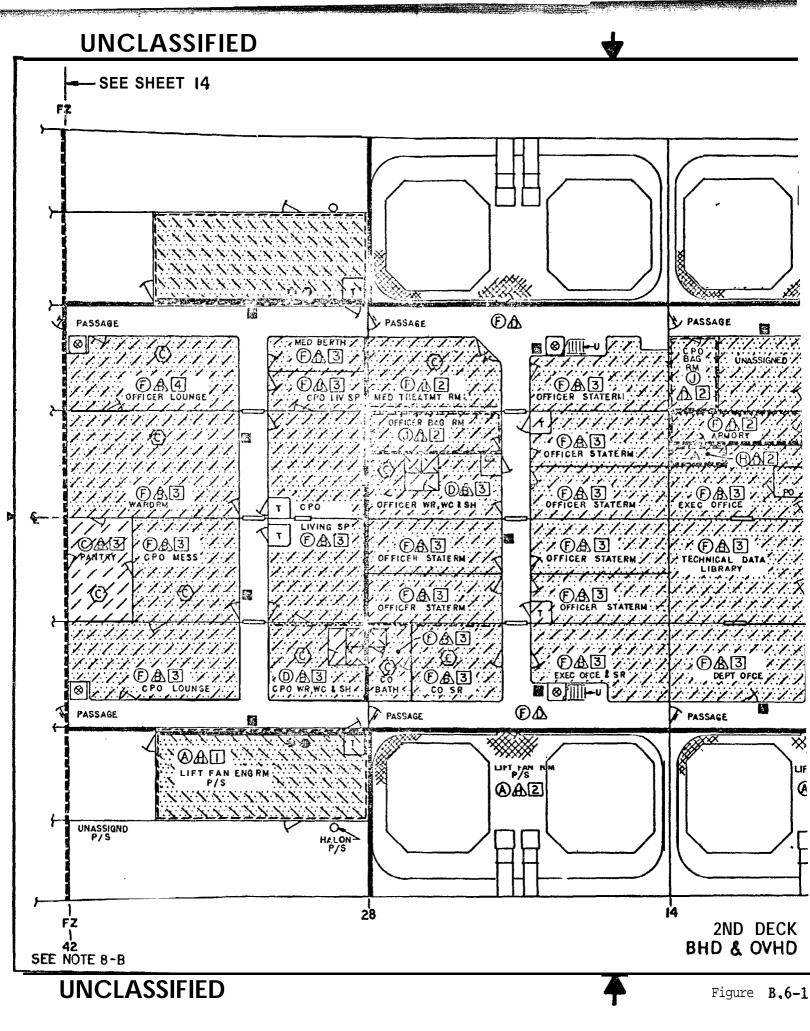
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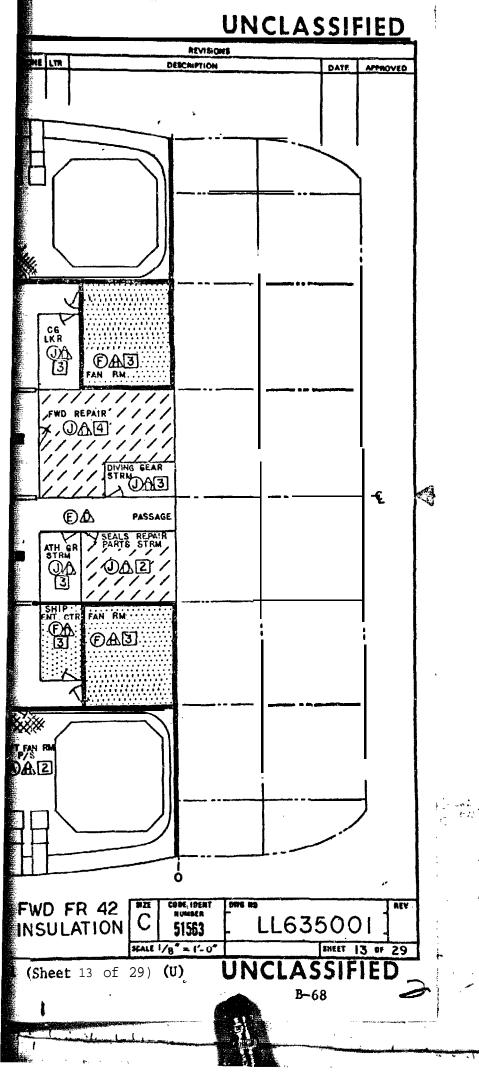
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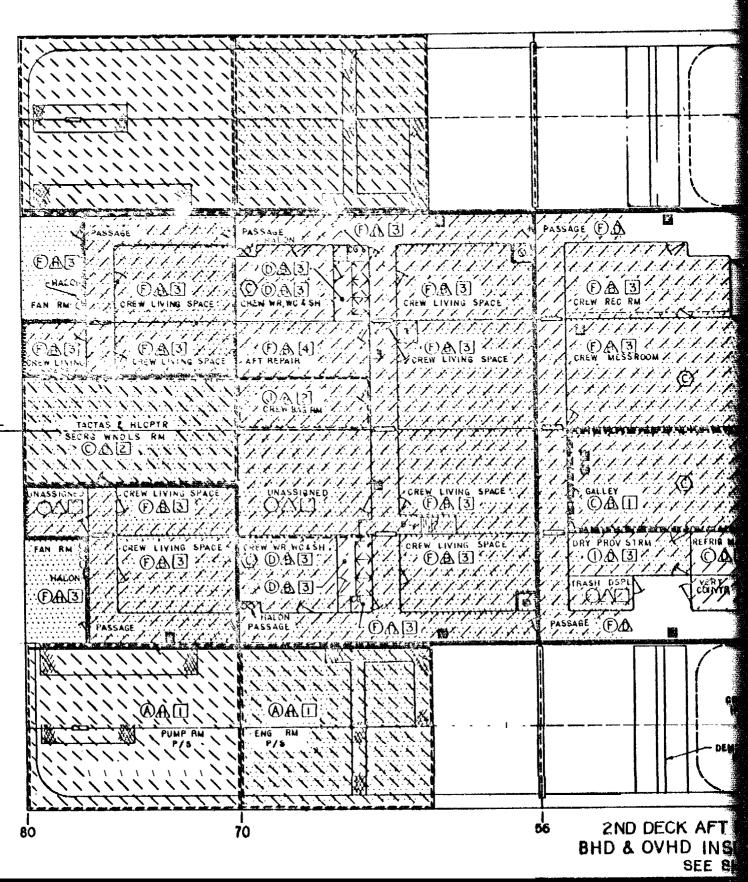


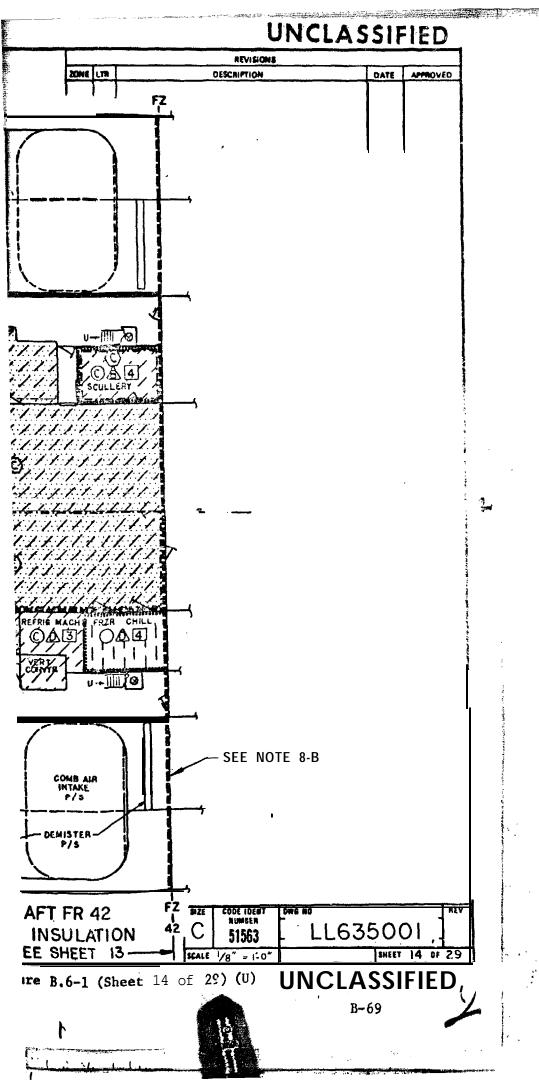


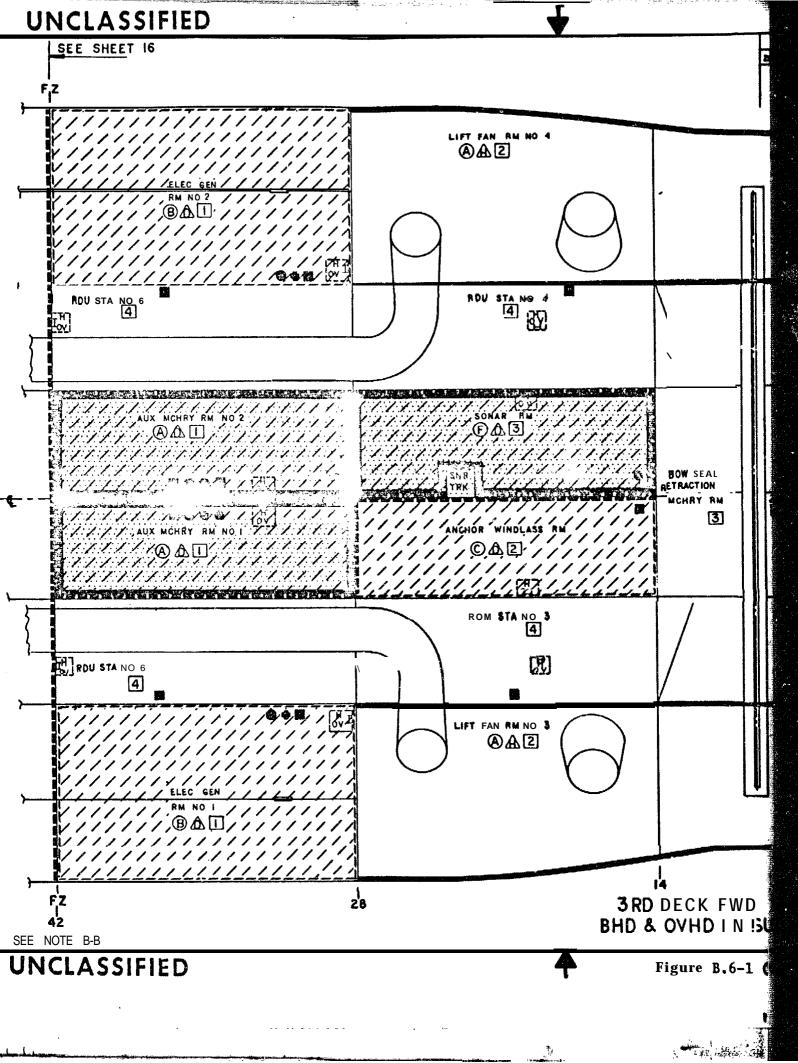


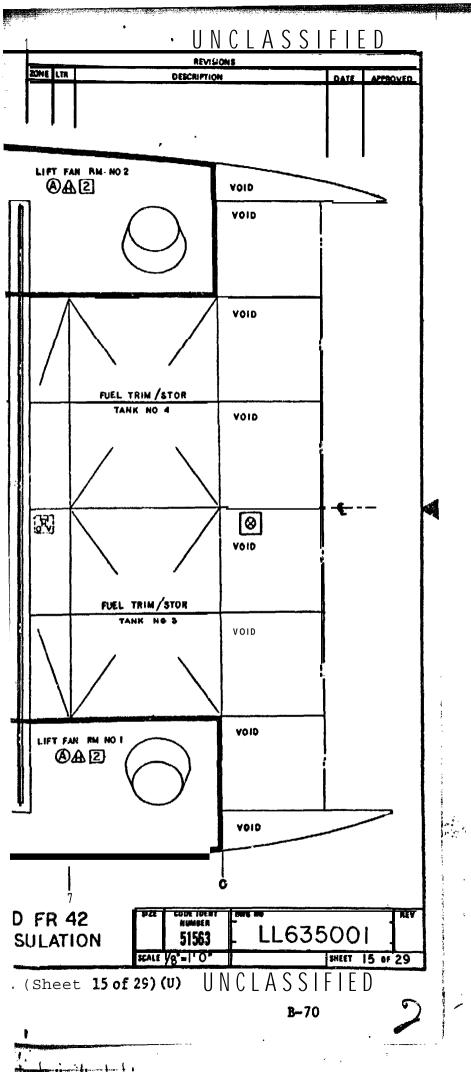












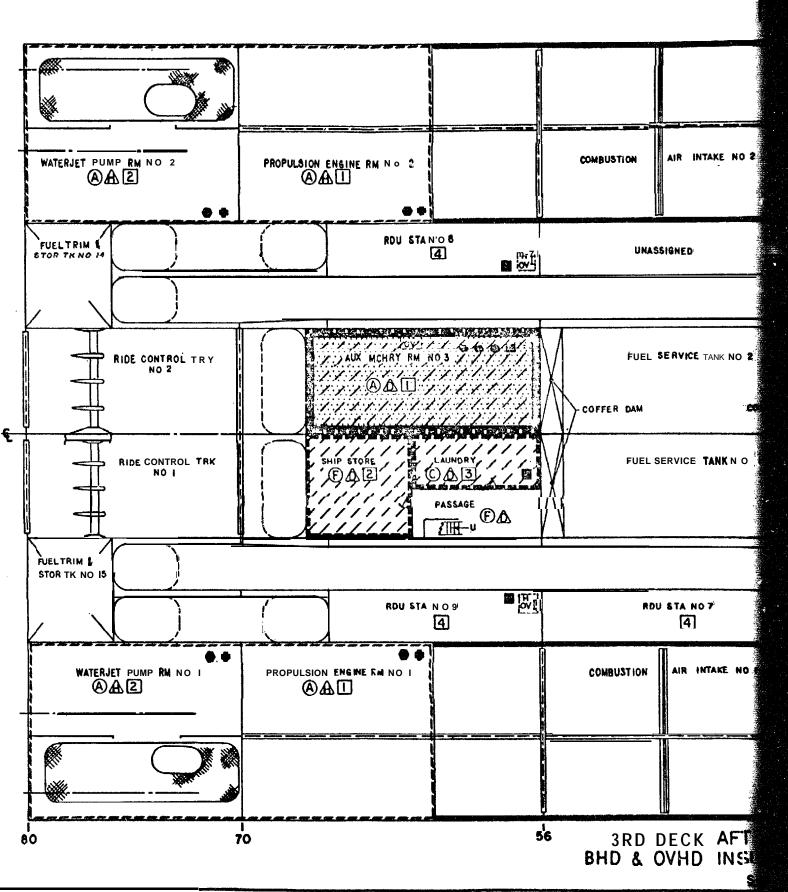
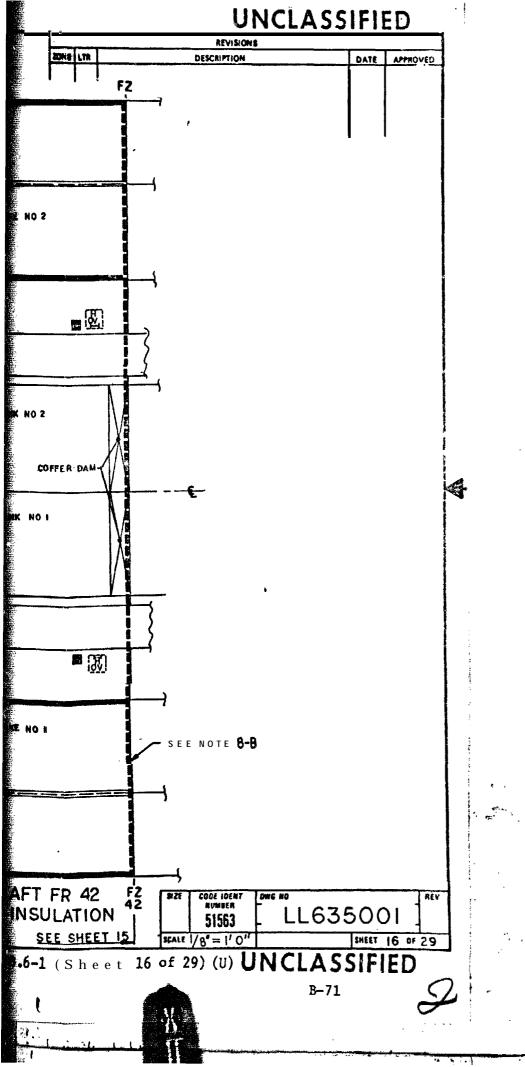
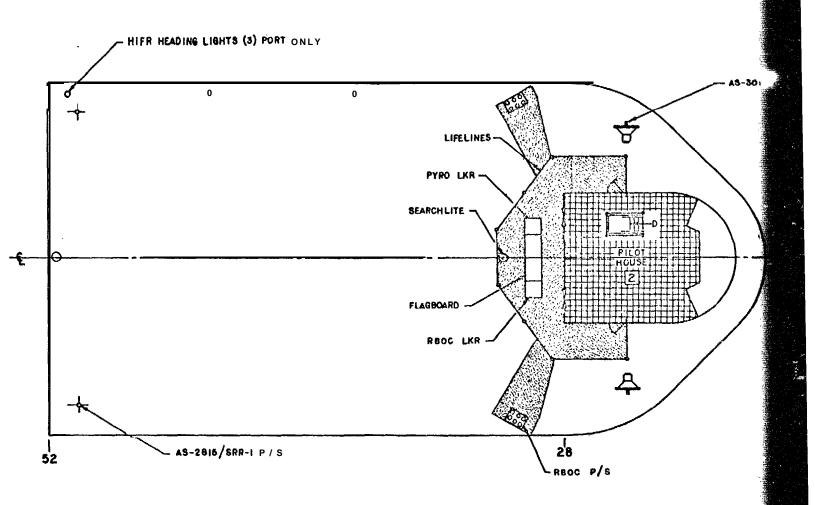


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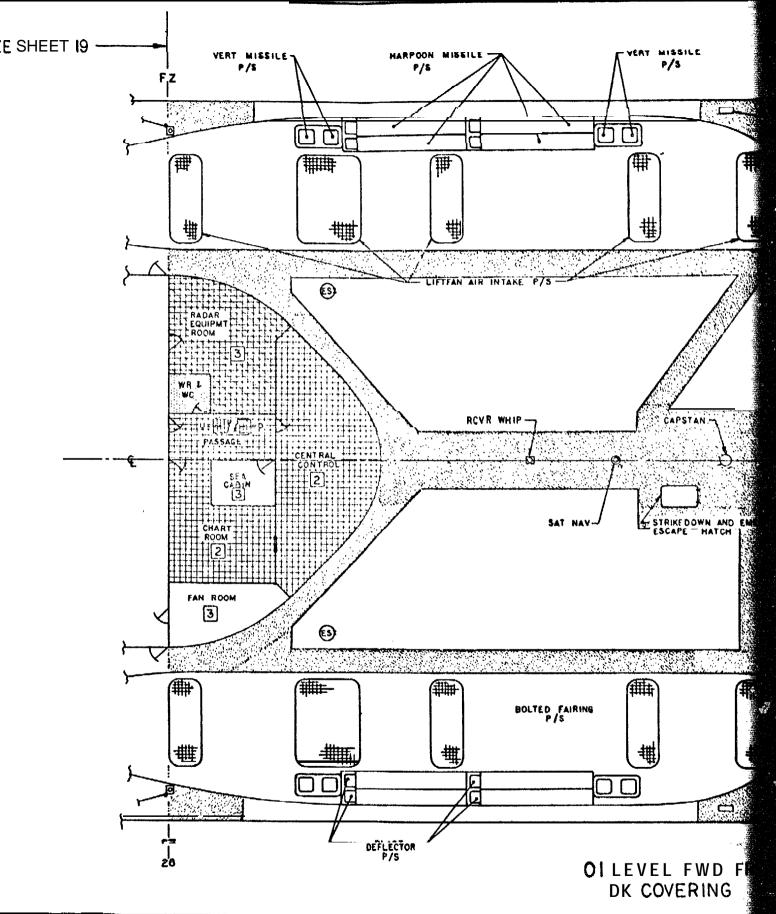
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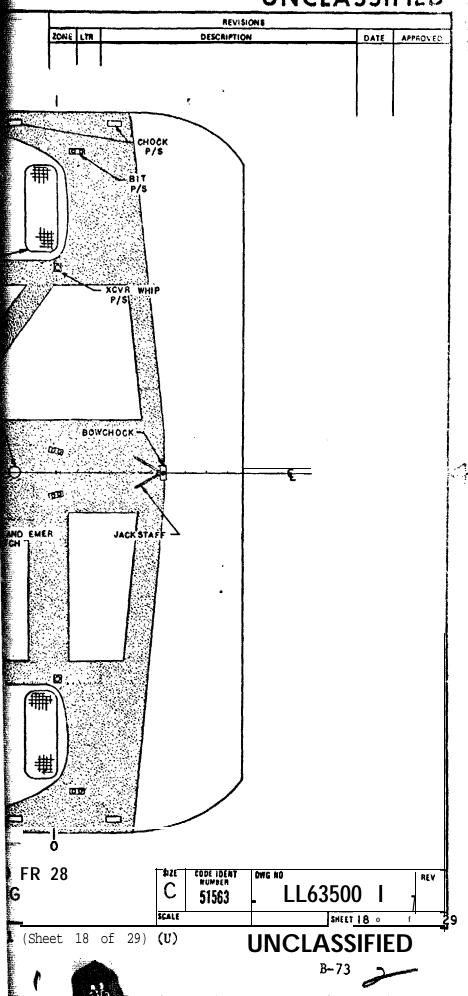
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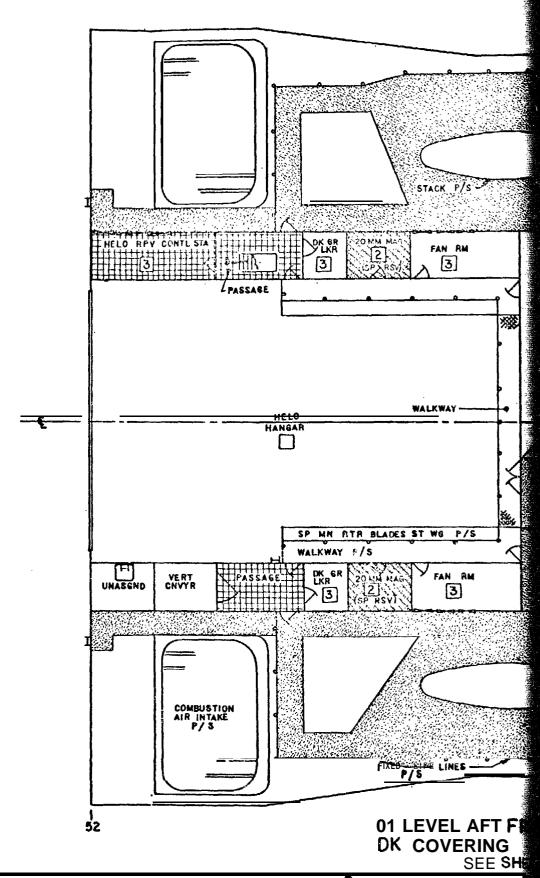
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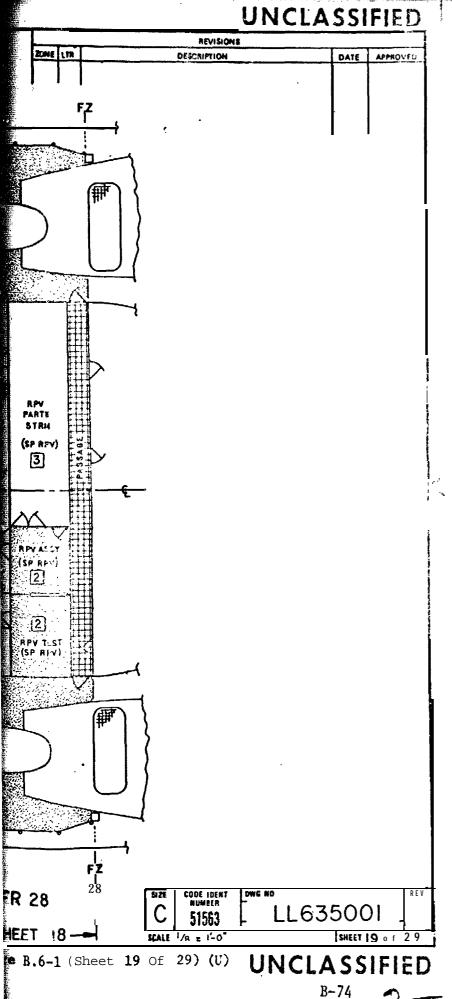


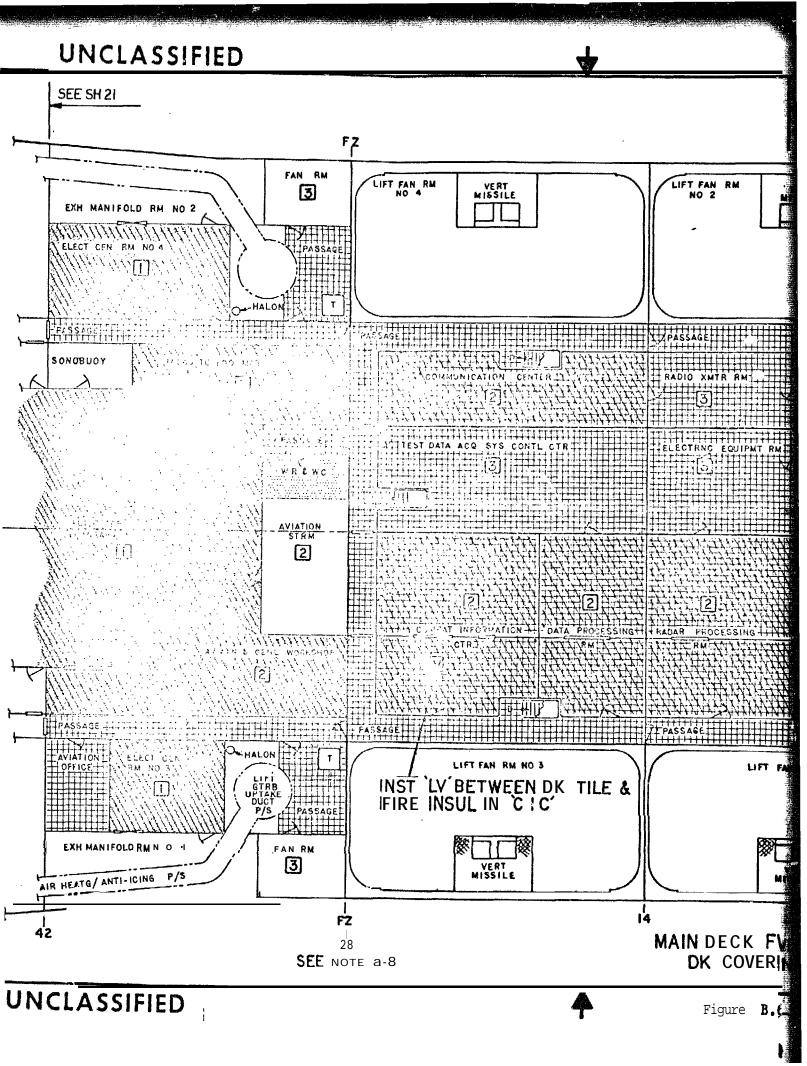


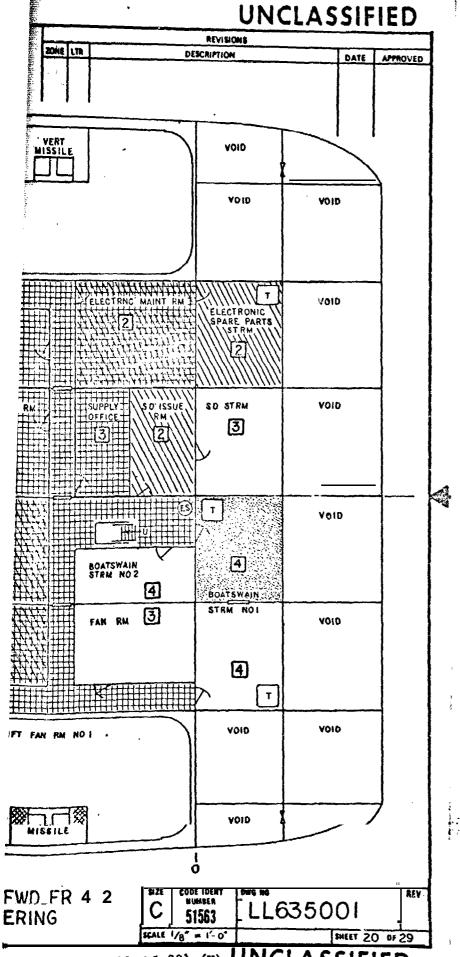


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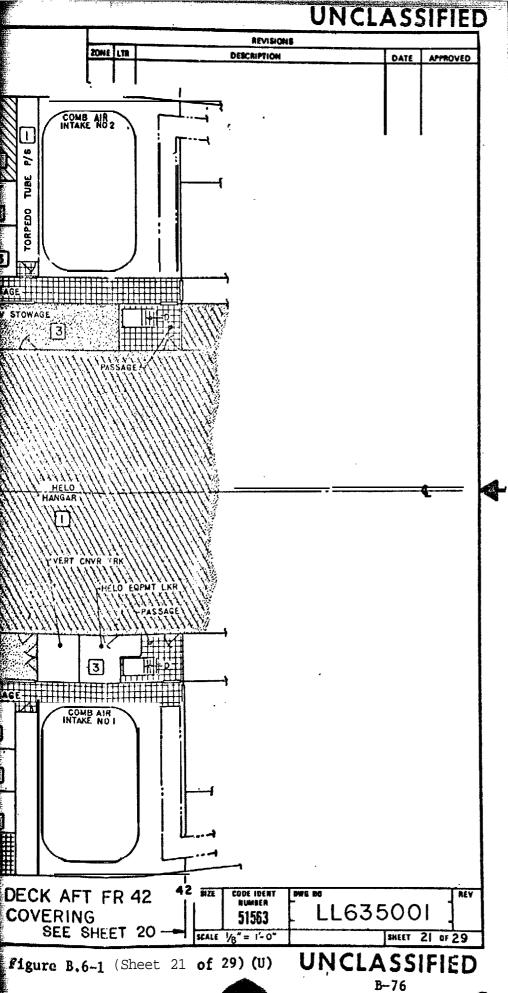


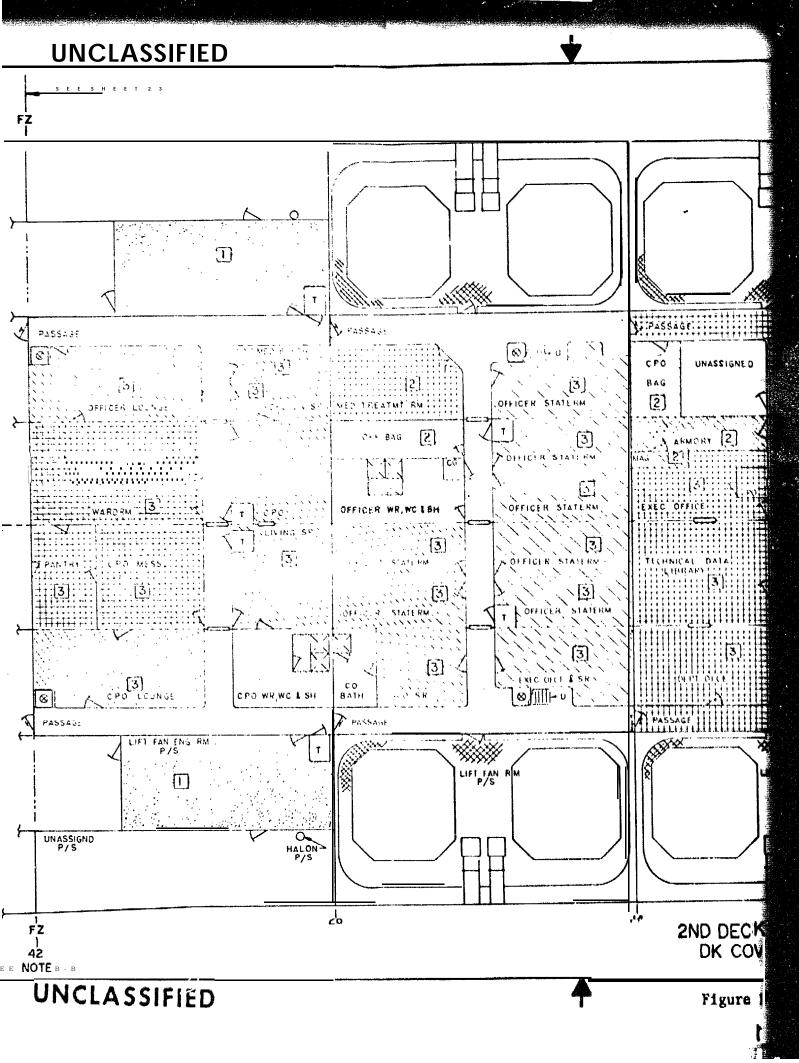


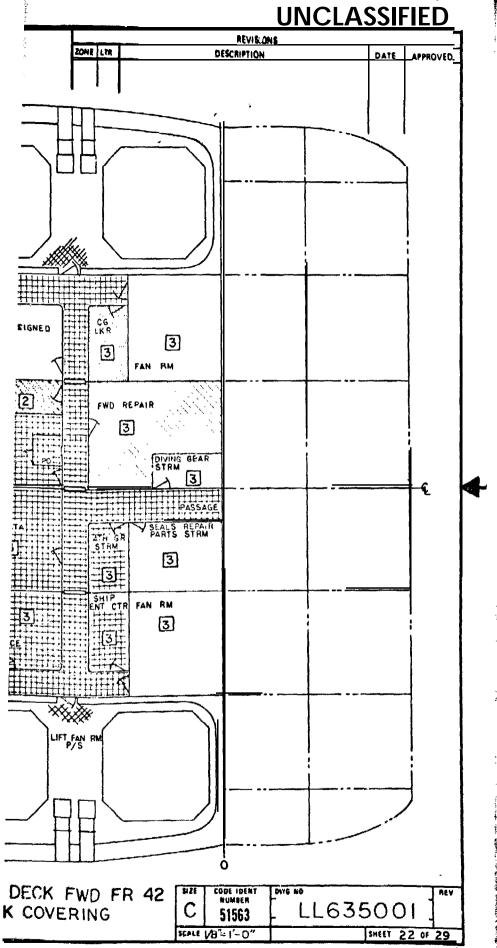
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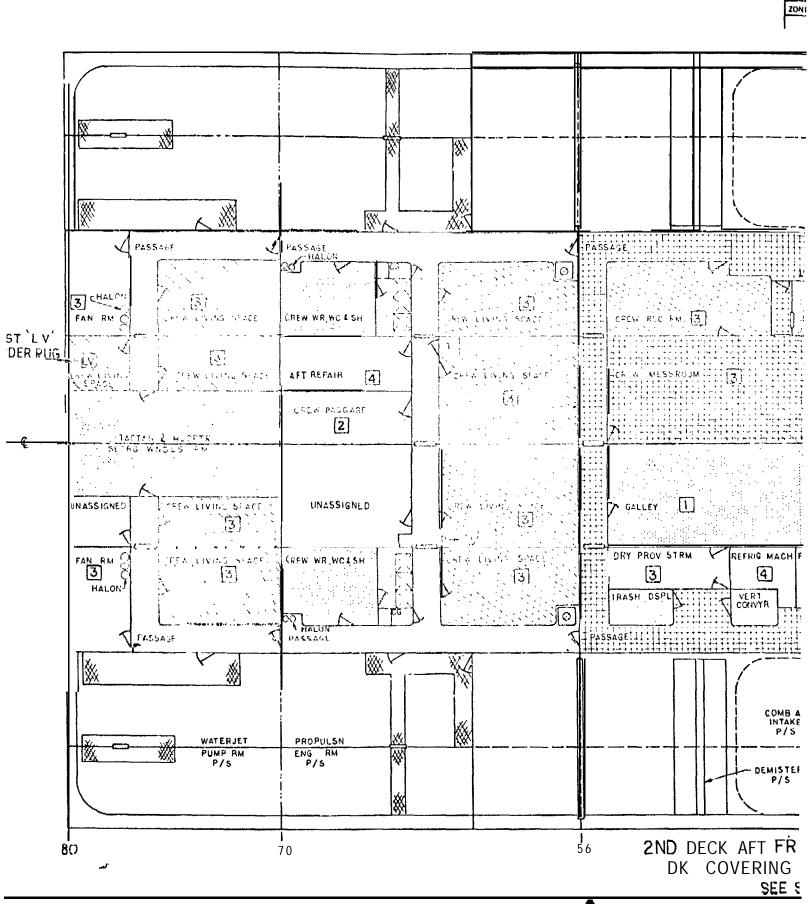
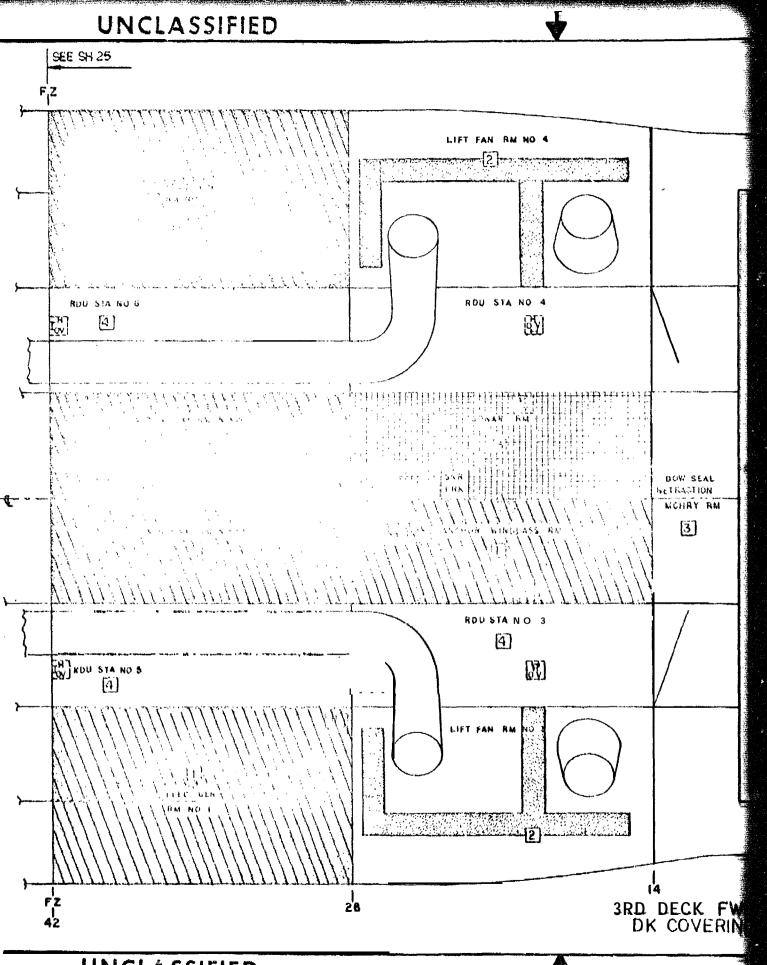
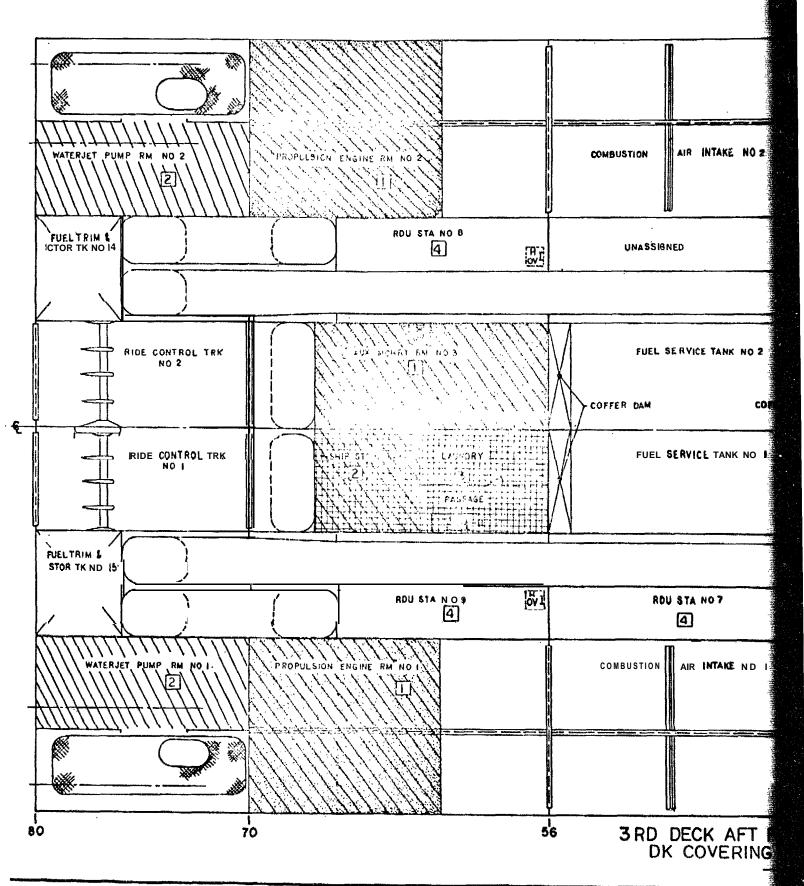


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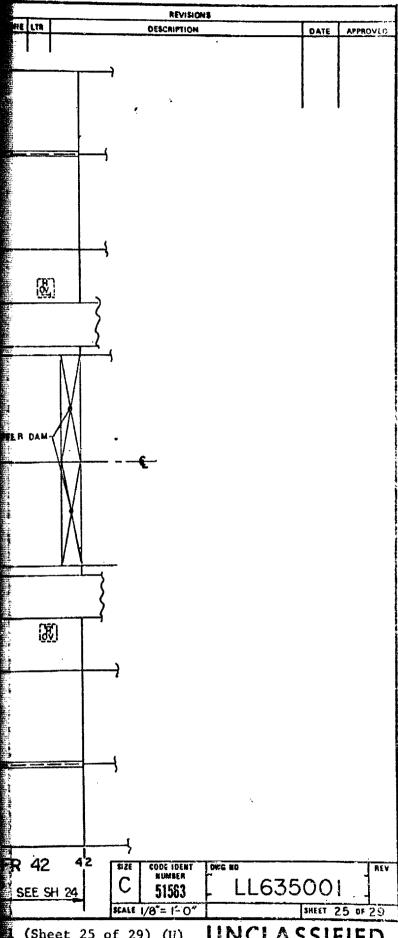
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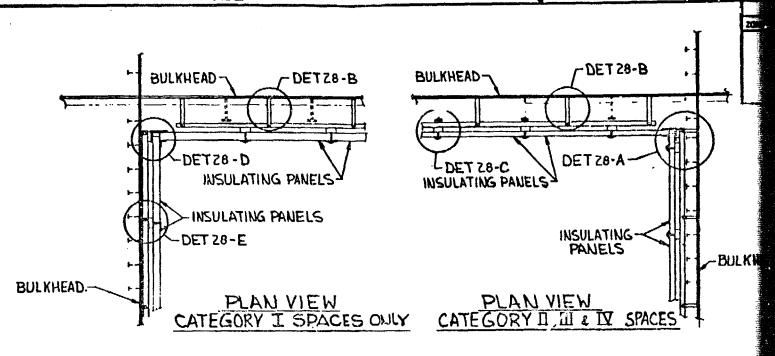
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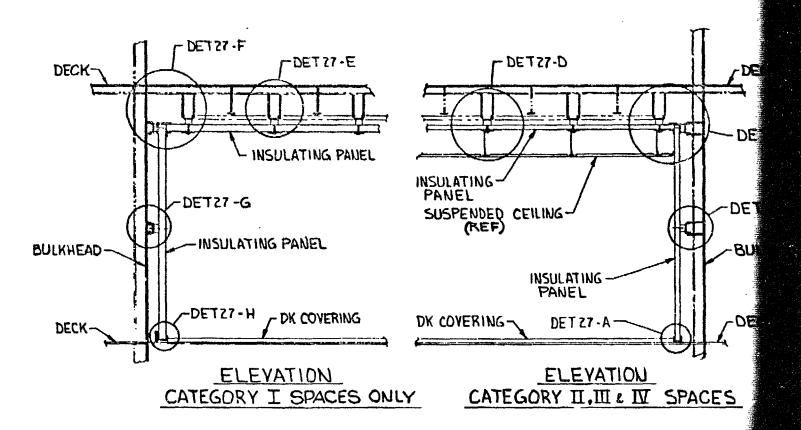
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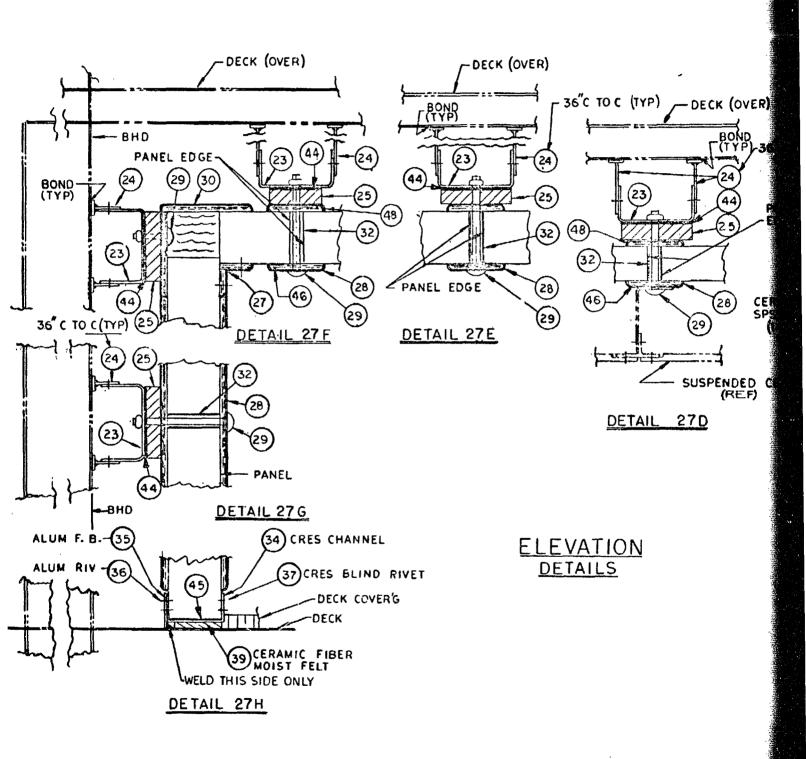
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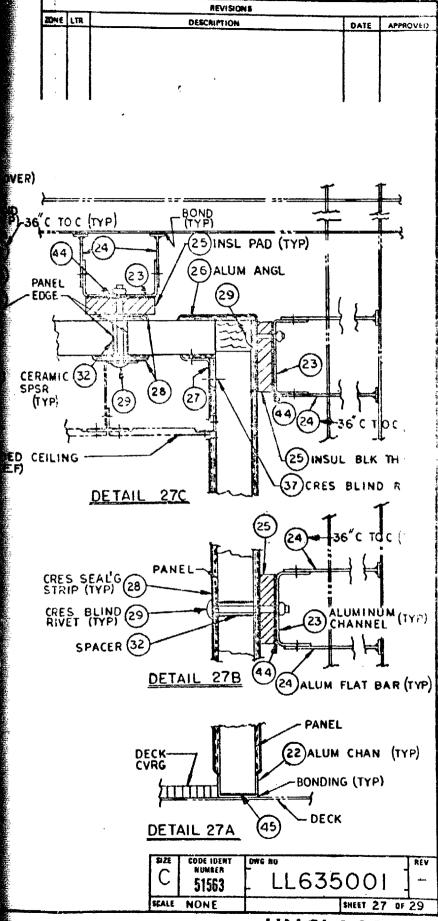
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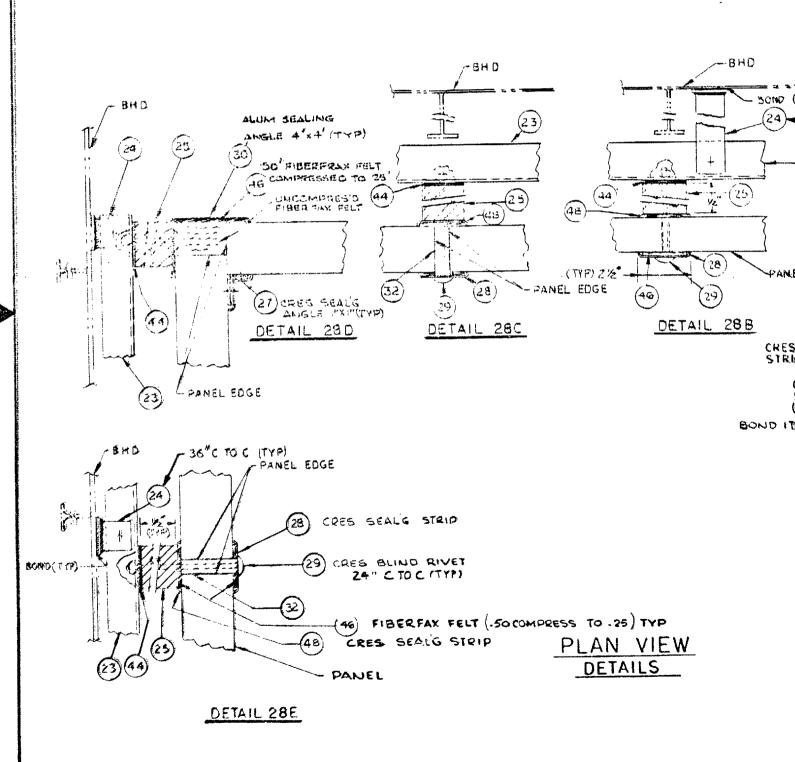




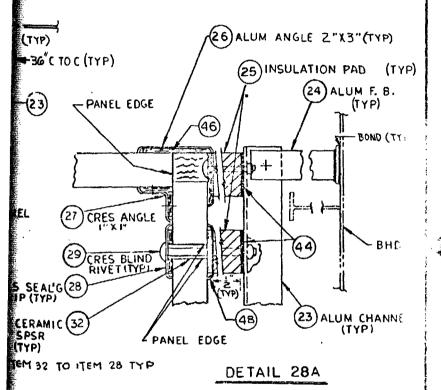




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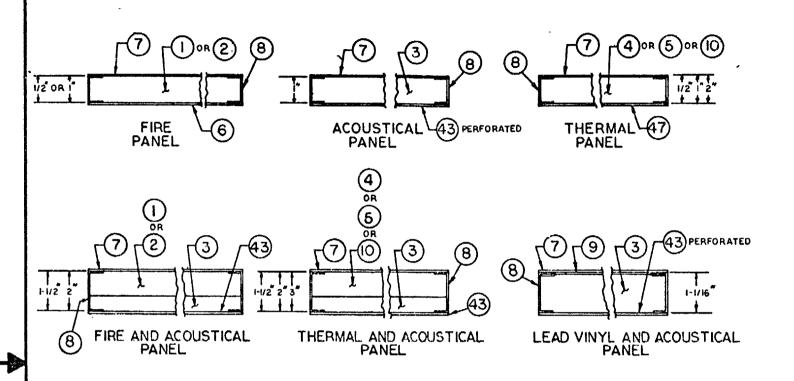


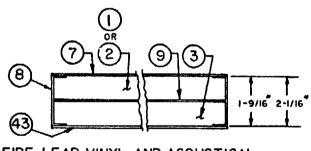
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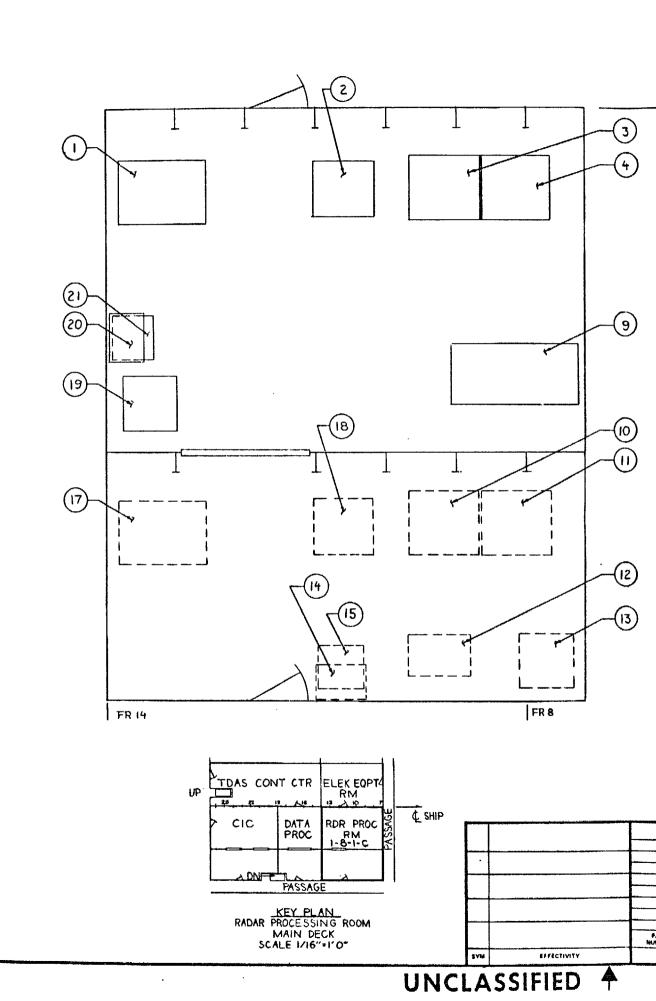
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- (U) B.7 COMBAT SYSTEMS ARRANGEMENT DRAWINGS AND BLOCK DIAGRAMS
- (U) This section of Appendix B contains the combat systems arrangement drawings and block diagrams for the near term ANVCE SES. These arrangements are:

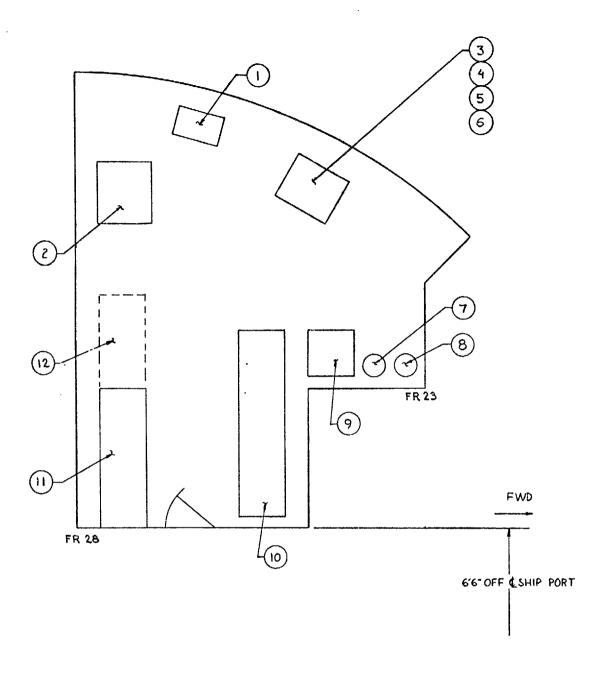
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B.7-1	Radar Processing Room, Main Deck
B.7-2	Radar Equipment Room, 01 Level
B.7-3	Electronic Equipment Room, Main Deck
B.7-4	Mk 46 Torpedo Magazine
B.7-5	Armament System Functional Block Diagram

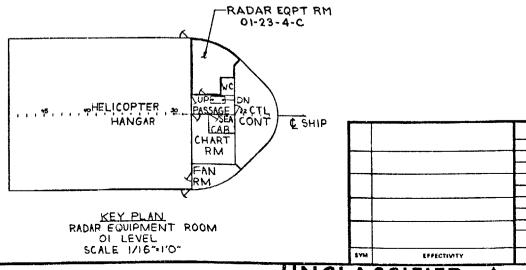


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MICROFILM REVISIONS

ZONE LTR DESCRIPTION DATE APPROVED

A REVISED PER ECR LOGOG 3 4-13-76 -74/1/2000

4-10-76 -10-76

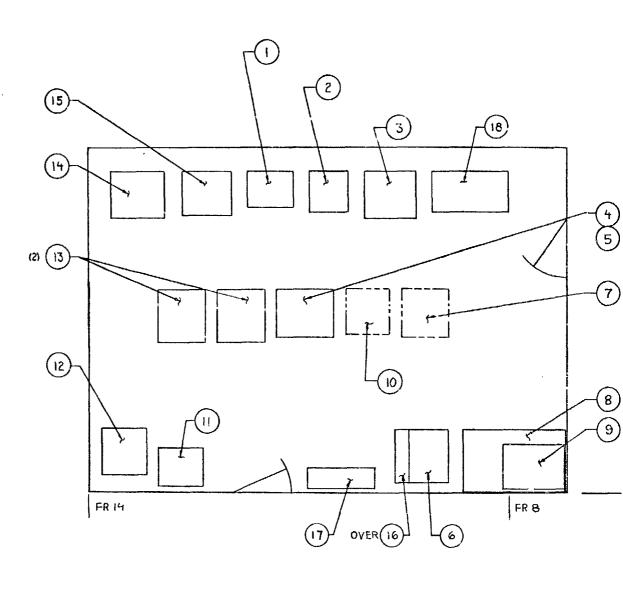
12 AN/SLQ-31 (GROWTH)
11 COMPUTER/SIGNAL PROCESSOR (AN/SLQ-3) V-2)
10 RADAR RACK /AN/APS-125)
1 9 DETECTOR PROCESSOR / DIGITAL SIGNAL ANALYZER (AN/APS-125)
1 8 HEAT EXCHÂNGER (AN/APS-125)
1 7 NITROGEN TANK (AN/APS-125)
1 6 SYNCHRONIZER-EXCITER SN-460/APS-116
1 7 RECEIVER-PULSE COMPRESSOR R-1747/APS-116
1 7 POWER SUPPLY PP-6633/APS-116
2 RADAR TRANSMITTER T-1203/APS-116
2 RADAR RECEIVER TRANSMITTER RT-1124/SPS-55
1 SYNCHRO AMPLIFIER MK-27
PC NO DESCRIPTION QTY

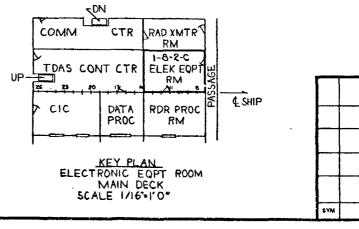
		NOTE THIS BL	OCK IS NOT MAINTAINED			UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES	S RHODES	4127176	CHULA VISTA, CALIF.
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4				士		DECIMALS ANGLES	111111111	<u>5-L-76</u> 5-12-76	
				+-		.X .XX .XXX ±.1 ±.83 ±.010 ±2°	The same of the sa	5-13-76	RADAR EQUIPMENT ROOM ARRANGEMENT
				1		GENERAL SPECIFICATION			OI LEVEL
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1	PART NUMBER	HEXY ASSY	MODEL NO.	MEKT	FINAL	CAL WY LOS	CREATE TO THE PARTY OF THE PART	\$/13/X \$17:76	D 51563 1 45200 A
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Figure B.7-2 (U)

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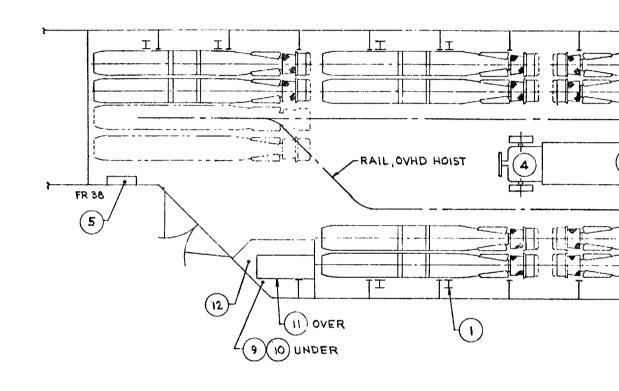
ROM 3282-1

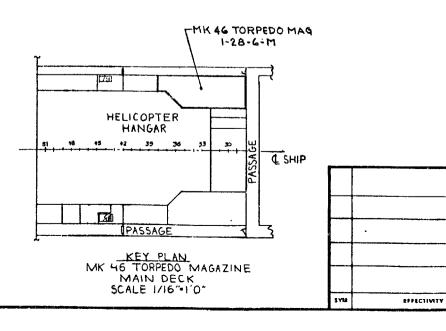
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- & SHIP FWD	17 MK 4 CHANNE	ING SET-REMOTE UNIT		
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	13 IFF INTERROS	SATOR ANJUPX-25(V)		2
	1. 10 1 MK 140 CONSO	UNA CUNTROL UNIT LE (SPACE & WEIGHT)		
	9 IFF TEST SI B WORK BENCH			
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	4 CONTROL MO	NITOR C-8430ZUPX		
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PART NEXT ABBY MODEL NO. MEXT PINAL MARKET PARTY	CAL NT LIN	10/2 5/11/2 Vat Ember 5.7-76		1001 - REV
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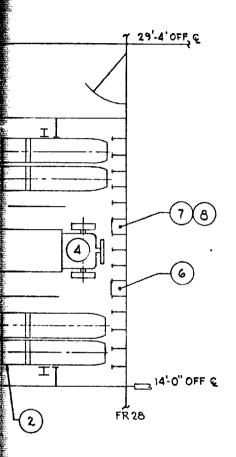
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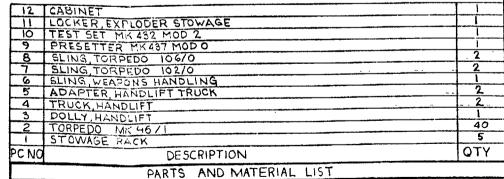
ZONE LTR DESCRIPTION DATE APPROVED

A REVISED PER ECR LOCOCC C-33-76

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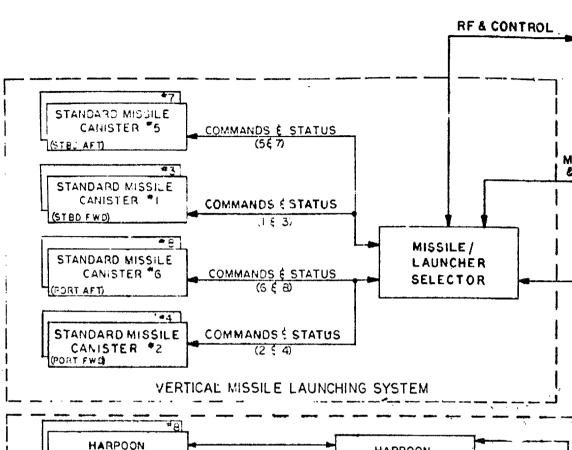


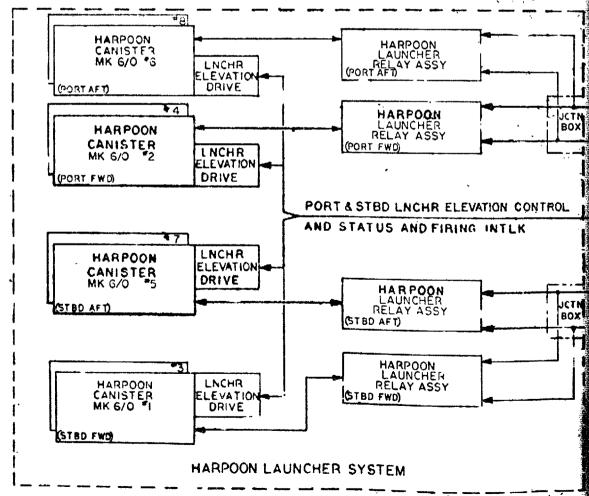
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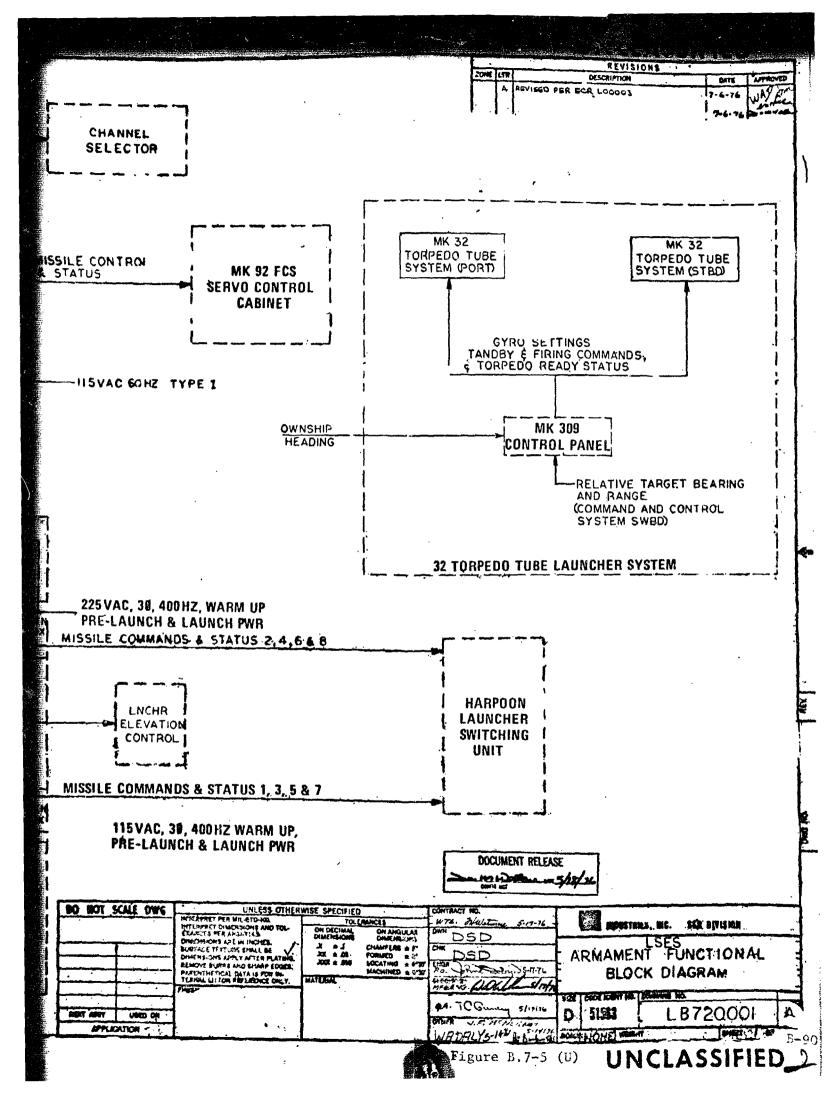


m wallen 5/13/76 COMP IC MICT CHULA VISTA, CALIF. NOTE: THIS BLOCK IS NOT MAINTAINED RHODES 5/7/76 UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES 5/7/00 ROHR INDUSTRIES, INC. (pose of) TOLERANCES ON -27 The state of the s DECIMALS. TORPEDO MAGAZINE ANGLES 5-12-76 MK 46 5-13.76 ±.010 Mustrum ARRANGEMENT GENERAL SPECIFICATION MAIN DECK WALK s/ish SIZE CODE IDENT LL 783001 CAL MY LES 51563 NEXT ASSY MEXT FINAL 5.1.16 100 mm יטני/ז APPLICATION OTY REOD

Figure B.7-4 (U)







APPENDIX C

This appendix contains equipment data sheets for the near term SES ${\ensuremath{\text{C}}}^3$ system and combat system elements.

Items marked with a \diamondsuit are part of the C^3 system and those marked with a \diamondsuit have navigation and IC related functions. All other elements not marked are part of the combat system.

Weights, size and service requirements are shown for unit elements. The tables show the quantity required for the near term SES.

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				ELE	CTRI	CAL	POWE	R, W						(
INSTALLATION DATA														
EQUIPMENT	S NO.	aUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		Ш	LOCATION	WEIGHT	неіснт	WIDTH	ОЕРТН
	SWBS	aU/	115	115	450	115	115	OC	ТҮРЕ	700	1bs (N)	in (mm)	in (mm)	in (mm)
Display System AN/UYA-4(V)	411 A	1												
PPI Display Cons. OJ-194(U)3/UYA-41(V)	Δ1	3					950			CIC	676 3507	50 1270	32 813	45 1143
Redar Date Dist. Swbd SB-2780/UYA-41(V)	Δ2	1					1600			DPC	640 2847	72 1829	37 940	23 584
Central Eqt Grp OV-91(V)3/UYA-41(V)	Δ3	ſ					1350			DPC	709 3154	72 1329	37 940	23 584
Signal Gen SG-10S1/UYA-4(V)	A4	1								DPC	25 111			
Remote Data R/O OA-8337(V)2/UYA-4(V)	45	2					156			CIC	125	11 275	33 8:5	23 554
Intercom Station LS 537A/UYA-4(V)	A6	3								CIC DPC PH	27	9	19	15 381
Comm P.P. SB-2781/UYA-4(V)	A7	ı								DPC	328 :459	72 1829	37 940	12
Monitor Control Cons OJ-200/UYA-4(V)	8A	1					500			DPC	440	50 1210	24 610	47
OPS Summary Cons OJ-197/UYA-41(V)							1730			CIC		49	52	41
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HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	•
w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
320	630	ΔW	1.7 107.3		40.5						18" (457 mm) Rear Clearance Two Additional Consoles are Growth Item DW Deminwater
310	610	DW	1.5 94.6		40.5	4.9 33.8					18" (457 mm) Rear Clearance
320	430	DW	1.4 88.3	150 1034	40.5	9.7					18" (457 mm) Rear Clearance
150								-			
19											
											18" (457 mm) Rear Clearance
											'18" (457 mm) Rear Clearance
490	1240	DW	3.2 201.9	150 1034	40.5	8.0 55.2					*Growth Item Only

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INSTALLATION DATA											1	—		A House Street
EQUIPMENT	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ň	LOCATION	WEIGHT	неіснт	WIDTH	DEPTH
	SWE	σn	115	115	450	115	115	DC	TYPE	707	lbs (N)	in (mm)	in (mm)	in (mm
Computer System AN/UYK-7(V)	412 A	1												4-718-48497
Computer AN/UYK-7(V)	ΑI	1					2760			DPC	535 2380	41 1041	20 508	23 584
Computer Cont C-8542/UYK-7(V)	A٤	1					3			DPC	72 2	152	7	7
Test Set TS-2942/UYK-7(V)	A 3	1					16			DPC	35 156	19 483	19	6 152
Digital/Analog SWBDS	413													
Combat Syst SWBD	A Al	3								DPC	1000 4448			
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нтаэа	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	нхор	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in (mm)	W	W		gpm {cm ³ /s}	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
23 584	2290											One additional comp is growth item
7 179 6	3											
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	INSTALLATION DATA							2				E	Ŀ		
	EQUIPMENT	SWBS NO.	DUANTITY	115V-10-60Hz	.JV-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		<u>а</u>	LOCATION	WEIGHT	HEIGHT	WIDTH	DEPTH
L			OO	115	1.	450	115	115	oa	TYPE	Ρ̈́	lbs (N)	in (mm)	in (mm)	in (mm
】	Interface Equipment	414 A													
L	I/O Cons Deac OJ-172/UYK-7(V)	Αl	,	486				1900			DPC	970 4315	63 1600		34 864
L	Signal Data Comv CV-2953/UYK	A 2	ı	500				2000			DPC	625 2780	72 1829	37 740	23 584
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	Data Terminal Set AN/USQ-63	415 A													, i
	Electrical Equipment Dwr	ΑI	ı				160				EER	11	9 229	19 483	21
	Digital-Anal Conv CV-2969A(P)/U	A2	1									32	8 203	7	19
	Comp Adapter MX-9222/U	АЗ	ı									12 53	8 203	3	20
	Fault Isol Cont C-9061/U	A4	1									3	4	6	102
	Address Cont. Ind C-9062/U	A5	J								CIC	30	11 279	19	12
	Data Term Set Cont C-9063/USQ-59	AG	1				230				CIC	40	11 279	19 483	12 305
	Equipment Rack	A7	ı								CIÇ	40	25 635	23	14. 354
	Cont SWBD SB-3372	A8	,								EER	160 712			
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	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	нхор	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in mm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi ′kPa)	ft ³ /min m ³ /s	NOTES
34 864	2386									:		
2 3 584	2000											18" (457 mm) Rear Clearance
			!									
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21 141												Holis A2, A3, & A4 and fits into std 19" (482 mm) rack
141 19 183												
20												
4												
12												
12												
14												Holds A5 & A6
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A	t 0/UYA-4		1	183				600			arg				32 813	19 483	American provided
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HEAT DISSIPAT (AIR)	HEAT DISSIPAT (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	нхор	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
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				ELE	CTRI	CAL	POWE	R, W						
INSTALLATION DATA					-						IT .	T		
EQUIPMENT	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ш	LOCATION	WEIGHT	неіснт	WIDTH	e projektjip (g., daga bleho
Edon MEIV		aUA	115\	115\	450\	115	115	DC	TYPE	001	lbs (N)	in (mm)	in (mm)	3
Electrical NAV aids	422 A													A Section States of
Masthead Light	AI	1_	50							MAST	9			A SERVER COM
Side Lights	42	2	50							PILOT HOUSE	9 40			and the debit of the second
Stern Light	Δ3	-1	50							AFT	44			Color Section 18
Towing Lights	Δ4	2	<u>50</u>							MAST	12 53			7
Task Lights	45	Ģ	450	 						MAST	13 58			100
Wake Lights	ΔCo	2	50							MAST	20 89			
Anchor Lights	47	2	50							MAST	36		-	
Speed Light	Δ8	1	135							MAST	20 89 26			
Blinking Light	49	2	180							AFT	116			
Search Lights	A 10	2	1,000							02 LVL	512			
Supply Control & Telltale Panel	١١٨	1			ļ					PILOT HOUSE	5G 249			
Supply & Control Panel	A 12	1_1_								PILOT HOUSE	34			
Dimmer Panels	A 13	2			ļ					PILOT HOUSE	33 147			
Range Light	Δ 14	1	50							MAST	9/40			
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ОЕРІН	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	HXDP	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in nm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
												20 DT Light-White No range it unless 2nd mast installed
								-				10 pt. lights - green • s red • port
												12 pt light - white
												portable
					T-2-1000							Red - white - red - 32 pt Two sets for full coverage
								~				Gred Bulbs Includes A/C warning It 3 White
		To The Health State.								-		
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INSTALLATION DATA														
EQUIPMENT	SWBS NO.	DUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ТУРЕ	LOCATION	WEIGHT	неіснт	WIDTH	
		O	118	115	45(118	11	၁၀	<u>}</u>	2	lbs (N)	in (mm)	in (mm)	in (mi
Helo Landing Lights	422 B													
Edge Lights (Red) 256.1 256.7	Ві	17	38							LANDING PLATE	80			•
Lineup Lights (white) 260.1 356.9	62	20	45								16			
Touchdown Lt (Green) 256.1 256.7	83	-	38								18 80			WHAT WAS
Vertrep Lineup Lights (White) 262.1 257.3	B4	15	90								14	9 229	6	20.7
Vertical Dicopline Lights (Red) 243.1	B 5	12	100								3 13			41,000,00
Extended lineup Lights (White)	B6	12	45								3			
Deck Surface Flooding Lights (Red or White)	87	8	150								15 67			alimeds to the
Maint Flood Light (Red)	B8	ı	300								15			
Hier Headine Lights (Red)	89	3	50							03 FAF	5 22			A Company
Signal Wands	810	2									1 4			
Wave off Lights (Red)	ВП	2	100							02 FAF	5 22			
Overhead Flood Lights (Red)	B!2	B	300								15			
Landing Signal Light Kit	813	ı					•				5 22			
Homing Beacon (White) 258	814	3	150								12 53			
Rotary Beacon Signal System (Red Green Amber)	815	3	150								12 53			Sections
DECK STATUS LIGHT (Red, Green, Amber)	816	,	150								12			
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WIDTH	рертн	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	ТҮРЕ	FLOW	PRESSURE	TEMP IN	НХDР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in (mm)	in (mm)	w	w		gpm (cm ³ /s)	psi) (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
तिस्ताति । जन्म											·		At least 4 per edge 120/12 XFMR 38W ea. 12V
ed with the season of the seas													Turn on only 6 at a time
													120/6.5 XFMR 6.5V 45W
6	3 76	ļ!											120/12 XFMR 38W ea. 12V
													120/6.5 XFMR 6.5V 45W ea.
A distance of the state of the													115/12 XFMR 100W 13V
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A STATE OF THE PARTY OF THE PAR													
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				ELE	CTRI	CAL	POWI	ER, W						
INSTALLATION DATA										·				700000000000000000000000000000000000000
EQUIPMENT	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ш	LOCATION	WEIGHT	HEIGHT	WIDTH	ОЕРТН
	SWB	۵∪۵	115\	115\	450\	115\	115\	ည	TYPE	200	lbs (N)	in (mm)	in (mm)	in (mm)
Stabilized Glide Slope Ind.	422 c	1									500 2224			200
Hydraulic Pump Assembly	· 6 1	l			2100		,			HANGAR		28	20	17 432
Remote Panel Assembly	c 2	1								HELO CONTROL STATION		15 381	12 305	ھ)
Stable Platform Assembly	c 3	١								02 LVL		25 635	25	28
Glide Slope Indic Assembly	c 4	1								os rar		13	13	2 C
Transformer Assembly	c s	1								HANGAR		14 356	11 279	7 178
Electronics Encl Assembly	c 6	l	960							HANGAR		30 762	24	10 25

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				-	-	-	10.00	-	Series Series	THE RESERVE TO A SECOND PORTION.	L			

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	TED	TED		COOL	ING	WATI	ER	н	'DR	A	NR.	
DEPTH	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	HXDP	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in	101	1Ar		gpm	psi	°c	psig	psi	gpm	psi	1t3/mir	
nm)	W	W		(cm ³ /s)	(kPa)	C	(kPa)	(kPa)	cm ³ /s	(kPa)	m ³ /s	NOTES
7										······································		Close to Glide Scope Indic & Platform
32		-					·					Self Contained Hydraulics 2.7A
52 28												
5 C			- 									,
<u>600</u> 7												W/in 8' of Platform Base (914 mm)
78 0 54												10760 W total Helo Hq pwr 60 hz
27										***		
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				ELE	CTRI	CAL	POWE	R, W							3
INSTALLATION DATA											_	J			DISSIPATED
	NO.	ITITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		:	LOCATION	WEIGHT	HEIGHT	WIDTH		HEAT D
EQUIPMENT	SWBS	QUANTITY	115V.	115V-	450V-	115V.	115V.	၁၀	TYPE	/7007	lbs (N)	in (mm)	in (mm)	in (mm)	4 C
Receiver R-1843/WRN-5	423 Al	1	400								150 667	13 330	19 482	30 762	40
Pre Amp AM-6603/WRN-5	A2	ı								•	13 58	13 330	10 254	5 127	A TANK TANK
Remote Display IP-1154/U	A3	-	20								5 22	5 127	5 127	11 279	2
Headset H3/ARR-3	44	1									14				
Antenna CA-3086	A5	ı								03 FAF	20 89	46 1168		DIA DIA	Application of the second of t
Omega AN/SPN-17	ಶ														
Receiver/Computer OR-133 (V)/URN	ВІ	1								CHART ROOM	68 302	12 305	14 356	24 607	100
Antenna AS-2960/SRN-17	B2	ı									9 40	126 3200	152	6	
Control Indicator C-9462/SRN-17	В3	ı									17 76	14 356	10 254	9 229	
Interconnecting GP ON-128/WRN	в4	ı				415					33 147	10 254	9	11 279	31
Test Set TS-3389/URN	B <i>5</i>	1									40	14 356	19	11	0.0
Echo Sounding System AN/UQN-4	424 A														
Indicator Display ID-1566/UQN-4	AI	3	100								21	7 179	10 254	17	
Transmitter/Receiver RT-888/UQN-4	AZ	,	230								195	29	24	20	
Transducer ,	A3	,									131	8 203	15	DIA	
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	ED	ED	(COOL	ING V	VATE	R	НΥ	DR	Al	R	
DEPTH	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	нхор	PRESSURE	FLOW .	PRESSURE	VOLUME/FLOW	
in (mm)	w	w		gpm (cm ³ /s)	psi (KPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
30 762	400											
5												
11	20											·
279	20											·
DIA DIA				-								Studs out to 13" (330 mm) diameter 38.25 (972 mm) max base dia
24								-				Powered via ON-128
607				 							-	
152 9 229				 	-						-	Powered via ON-128
11 279	222											
11	330	 									T	Powered via ON-128 420W 60Hz 415W 400Hz Total
279	85		-	 	 	<u> </u>			┨──		-	
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432	30	 	-	-	-	 	 	+-	-	-	-	
508	75	 	-			-	 -	-	-	-	+	530 W 60Hz
DIA			 	-	-	-		-	 	-	-	
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			org and company of the same		makat kanan na marana				No ≈ P de de or	Min the Manager automore into a substitute que page alter de l	· · · · · · · · · · · · · · · · · · ·	بديدي ونسد	• \ ~ ***	
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	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХБР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in pm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
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				ELE	CTRIC	CAL	POWE	R, W						
INSTALLATION DATA											F	Ŀ		-
EQUIPMENT	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		Ä	LOCATION	WEIGHT	неіснт	WIDTH	DEPTH
		00	115	115	450	115	115	20	TYPE	007	lbs (N)	in (mm)	in (mm)	in (ma
Integrated C/D Unit	426 11									CR	133			200
Radar Beacon (Space & Weight)	12									CR	40 178			
NAV/CAS Control/Display Panel D/p Ship Cont Console	13										100			
Ship Water Speed Sensor SR301(M)	14													
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	TED	TED	(COOL	ING V	NATE	R	НΥ	DR	A	IR	·
ОЕРТН	HEAT DISSIPA' (AIR)	HEAT DISSIPAT (WATER)	ТУРЕ	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in (mm)	w	w		gpm (em ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
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	in	in	in	HEAT DISSIPAT (AIR) HEAT DISSIPAT (WATER)	in (mm) W W (cm ³ /s)	in (mm) W W (cm ³ /s) (kPa)	in (mm) W W (cm ³ /s) (kPa) oc	in (mm) W W W (cm ³ /s) (kPa) OC (kPa)	in (mm) W W W Rem3/s) psi oc psig (kPa) psi (kPa)	in (mm) W W W icm³/s) (kPa) PSig Psig Psig Cm³/s	in (mm) W W W P Psi (cm ³ /s) (kPa) Psi (kPa)	Name

Inertial Nav System Gyro Type II 427					FIF	CTRI	CAI	PCWF	R.W		The supplies of the later than the l	-/		-,-,-		۵
EQUIPMENT 1	INSTALLATION					•	-, _			,						ATE
EQUIPMENT 1	DATA															SSIP
EQUIPMENT 1				N		2	42	7				HH	YH.	E	H	ă
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Control No. System Gyro Type II 427 1 1 1 1 1 1 1 1 1	EQUIPMENT	N S	FN	/-10-	/-30-	/-30	√-10·	7-30		ш	ATI					<u> </u>
Space & Wi.)		SWB	αUA	115\	115\	450\	115	1151	20	TYP	707	1		ł	i	w
System Center Equipment A	Inertial Nav System Gyro Type II (Space & Wt.)							500			CHART RM					300
System Center Equipment A																
System Canter Equipment A	Telephone Systems	:														
Dial Terminal Non W/T	System Center Equipment	۸,									SHIP ENTERTAINMEN	558	į	i	1	
Loud Speaker Ext A3 29 25 DIST 10 350 350 178 Handset A4 90 DIST 1 1 4 5 45 305 254 203 Handset A5 80 DIST 1 1 4 5 47 10 DIST 1 1 1 4 5 DIST 1 1 4 5 DIST 1 1 4 5 DIST 1 1 4 5 DIST 1 1 1 4 5 DIST 1 1 1 4 5 DIST 1 1 1 1 4 5 DIST 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	Ph. A. 10. 141/57	<u> </u>	-													
Loud Speaker Ext	Dial Terminal Non W/ f	A2	129								DIST	 				
Handset A4 90 DIST 1 1 2 2 7 102 127 Headset A5 80 DIST DIST	Loud Speaker Ext	A3	29	25							DIST		l .	1		
Speaker Phone (2-way)	Handset		90								DIST	1	11	4	5	
Speaker Phone (2-way) A6 IO DIST DIST Loudspeaker (1-way) A7 IO DIST DIST Extensions AB IO DIST DIST	Headset	AS	80								DIST					
Extensions A8 10 PIST	Speaker Phone (2-way)										DIST					
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(AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
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				ELE	CTRI	CAL	POW	ER, W	1 .					
INSTALLATION DATA													,	
EQUIPMENT	SWBS NO.	DUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		m m	LOCATION	WEIGHT	неіснт	WIDTH	DEPTH
	-	QU.	115	115	450	115	115	ည	TYPE	Γο	ibs (N)	in (mm)	in (mm)	in (mm)
Announcing Systems	433 A													
Ampt OSC AN/S1A-123	AI	_	860							SHIP ENTERTAINMENT ROOM	270	51 1295	20 508	13 330
**************************************	<u>^2</u>	2									15 67	12 305	10	6
***************************************	A3	90									7 21			
Loudspeaker LS-387/STC	Λ4	24									30 133			
	434 A													
T.V. Entertainment	ΔΙ											<u>-</u>		
Control Site 1-1	1.1	1	2000							SHIP ENTERTAINMENT ROOM	1300 5762	70 1778	44	23 584
Uniplexor Site 1 - 3	A1.2	ı									600	·	59	22
Color TV Revr	Δ1.3	5									90 400	22	25	21 533
VHF Dist Net	41.4	-									50			
Audio Ent.	A2													
	A2.1	1								SHIP ENTER TAINMENT ROOM	50 222			
	A2.2	}	1100								415 1846	66 1676	7 ¹ 533	14 356
Loudspeaker LS-444/WIH	Δ2.3	25									15	14 356	14	8 203
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רו - דר מי	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
n m)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
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					ELE	CTRIC	CALI	POWE	R, W						
ince s	INSTALLATION DATA											L	. _		
	EQUIPMENT	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz	DC	TYPE	LOCATION	€ ₹ WEIGHT	ੂੰ ₃ HEIGHT	∄ 5 WIDTH	a
\Diamond	Alarm, Safety & Warning System	436 A										(14)	(6)111/		=
\Diamond	Indic, Order & Metering System	437 A													
\Diamond	Integ Cont System	438 A													
\Diamond	Recording & TV System	439 A													
	An/UNQ-8 Recorder	AI										400			
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344															

UNCLASSIFIED HEAT DISSIPATED (AIR) COOLING WATER HEAT DISSIPATED (WATER) HYDR AIR VOLUME/FLOW PRESSURE PRESSURE PRESSURE TEMP IN FLOW FLOW HXDP TYPE gpm psi gpm ft3/mic psi (kPa) cm³/s (cm³/s) °c W W n) (kPa) (kPa) **NOTES** (kPa) m3/s **Ship Control System**





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	INSTALLATION DATA													
	EQUIPMENT	SWBS NO.	DUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ш	LOCATION	WEIGHT	HEIGHT	WIDTH
		SWB	οΩ	115\	115\	450\	115	115\	20	TYPE	00	lbs (N)	in (mm)	in (mn
>	Radio Systems HF XCVR GP	44! A									RADIO XMTR RM			
	HF Radio Set AN/URC-81	Αı	5				300	3000			RADIO XMTR BM	140		
	HF Radio Set Cont C-9058/URC	A2	в								2 RADIO XMTR RM 4 COMM	75 334		
	RF SW Unit SA-1070/UR	ДЗ	1								RADIO XMTR RM	130		
	XMTR Matrix Cont C-4787/SRA-34	Δ4	1								RADIO XMTR RM	60 267		
	XMTR Temp Alarm	ΔS	ì								COMM	2 9		
	Dummy Load DA-242A/U	A6	Į.								RADIO XMTR RM	35 156	11 279	178
	Coupler Adapter MX-4845/SR	A7	ł								RADIO XMTR RM	75 334		, <u>, , , , , , , , , , , , , , , , , , </u>
	Broad Band Antenna 2-4MH ₂	A8	,								OI LVL	120	35	
	Broadband Ant 4-10MHZ	A9	ı								ol LVL	120		4+
	Broadband Ant 10-30MHZ	Alb	1								OI LYL	120		ţ,
	Cabinet CY-()	ΔII	2								RADIO KMTR RM	275	(0)	•••
	Multicoupler 2-6MH ₂ CU-1179	A12	ı								TWIN KIM	75 334		
	Multicoupler 4-12MH ₂ CU-1180	AI3									RADIO XMTR RM	75 334		
	Multicoupler 10-30MHz	Δ14	,								ZMIJA AMI	75 334		
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	DEPTH	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	нхор	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
	in mm)	W	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES

		2300									Service and service		2 Local
		18											4 Remote
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			<u> </u>	ELE	CTRI	CAL	POWE	R, W							25-721-25-55-55
INSTALLATION DATA					·	2	2			٠		нТ	нт	Ŧ	T
EQUIPMENT	S NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ш		LOCATION	WEIGHT	неіснт	WIDTH	DEPTH
2001111211	SWBS	αΩ	115\	115\	450\	115	115	oa	TYPE			lbs (N)	in (mm)	in (mm)	in (mm)
HF Radio KCVR GP	В						2400			RAD XMTP					
HF RCVR AN/URR-67	BI	4	-									75 334			
RCVR Multiplxr Collins 512J2	B2	ł				·						39 175			
Term Box J-3152/SRC	B3	1										50 222			100 mg
Ant Whip AS-2537A/SR	B4	2									Y	120 534			in a local
Local Cont Unit C-9058/URC	85	5								I RAC	R RM	4			100
Filter LP-101C	B6	4								RA		275			
Eqpt CAB CY-() 651—8	87	1													
651-5	88	1									,				
															7
UHF Radio XCVR GP	С														HARAN CAMP
UHF XCVR AN/URC-82	c1	6					600			PADI	O R RM	50 222			
Local Cont Unit C-9059/URC	cz	9								1 R X 4 CON 1 CIC	MTR RM	75 334			
Remote XFR SWBD SB-1039/SRI	C3	1								RAD		19			
Bandpass Filter F-1332/UR	C4	6				150						24	18 457	4	21 53 3
UHF Multicplr TD-1046/URC	C 5	2									V	150			
UHF Antenna AS-1735/SRC	CG	2								BELC 108.5	W fr 33,1 m ATE	25	28 711	30 762	30 769
Eqpt CAB CY-11	c7	3								RAD	10	275 1223			
											1177				
										 					

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ASSAGA BALANTA REGIONALIA AND AND AND AND AND AND AND AND AND AN	DEPTH	HEAT DISSIPATED	HEAT DISSIPATED	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	·
in Mr		a) W	w		gpm (cm ³ /s	psi (kPa)	°c	psig (kPa)	Psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
State of the Experience		1000	1000	FW	94.6								
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				ELE	CTRIC	CAL	POWE	R, W						
INSTALLATION DATA	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz			LOCATION	WEIGHT	неіснт	WIDTH	
EQUIPMENT	SWBS	QUA	115V	115V	450V	115V	115V	၁၀	TYPE	7007	lbs (N)	in (mm)	in (mm)	
UHF LOS/SATCOM XCVR Group	D													
UHF LOS/SATCOM XCVR RT-1107/WSC-3	DI	1	1500							RADIO XMTR RM	148 658			
Control Indic. C-9351/WSC-3	02	1								Commo	3 13	نه 152	8 203	
XCVR SW SA-1712/UR	D3	1								RADIO XMTR REA	4			
RF SW Unit SA-2000/WSC	04	1			396					RAPIO XMTR RM	130 578			
Ant Control C-3597/WSC-1	ರಾ	1				Ď				RADIO XMTE RAN	18 80			
Ampl Filter AS-6691/WSC-1	06	2								OZ LVL	125 556	23 584	15 381	1.
Antenna AS-1018/URC (Mod)	07	١								931-6" PLTF 208.5 In	50 222			
Antenna AS-3018/WSC-1	D8	2	250							05 F^F	325 1445	54 1372	1270	1
Interconnect Group ON-143(V)/USQ	DG	١	265							COMMA	60 267			
Recorder - Repro RD-396/U	Dio	l	265							comm	41 182			
Recorder - Repro RD-397/U	DII	١								comm	41 182			
Control Panel	013	1								COMM	75 334			
Equipment Cabinet	E10	4								RTR	275 1223			
Computer AN/UYK-20	014	1	1000							comm	220 979	ł		
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W W Gm ³ /s) (kPa) C psig psi (kPa) m ³ /s (kPa) m ³ /s NOTES	
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				EĻE	CTRI	CAL	POWI	ER, W	1					
INSTALLATION DATA	ďO.	ITY	2-60Hz	0-60Hz	0.60Hz	115V-10-400Hz	115V-30-400Hz			NO!	WEIGHT	неіснт	width	рерти
EQUIPMENT	SWBS NO.	QUANTITY	1:5V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-1	115V-3	DC	TYPE	LOCATION	ibs (N)	in (mm)	in (mm)	in (mm)
UHF SATCOM BCST RCVRGP	E													
Comb - Modulator MD - 900/SSR-1	El	ı	¢5							comm	84 373			A center
RF AMP AM-6334-SSR-1	£2	4								02 LVL	12 53	8 203	12 305	9 22 9
Antenna AS-2815/SSR-1	E3	4								02 LVL	13 58	36 914	31	31
Demultiplexer TD-1063/SSR-1	E 4	,	24							comm	72 320			
Fault Alarm IC/BSIA	E5	1									8 36			
					<u> </u>									
			1		<u> </u>									
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				'		!								
UHF Tel Data Gp (Helo)	F													
Telem Rcvr R-1893/SKR-3A	Fi	1	1100							RADIO XMTR RM	465 2068	1	24	27 686
Ampl AM-6663/SKR-3A	F2	1								comm	2	2	5	3
Monitor Panel ID-1949/SKR-3A	F3	1								Comm	27	8	6	22
Demod MD-912/SKR-3A	F4	-								RADIO XMTR RM	350 1557	36	24	27
Test Set TS-3335/SKR-3A	F5		100	1						RADIO XMTR RM	15	19	12	1.5
RF AMPL AM 64 93/SKR3	FG	1								Mast	14	70.		1
Antenna AS-2743/SKR-3A	F7										3	19	3 76	
Test Antenna AS-2893/SKR-3A	F8										3	19	3 DIA	
Patch Panel Matrix SB-3721/SKR-3A	F9	,	1050							RIR	450	79	48 1219	8 203
Patch Panel Matrix SB-3720/SKR-3A	FIO	1								RTR	2002	11	10 254	10
	Acquire contract	And in contrast of	AND DESCRIPTION OF	AND DESCRIPTION OF	American V	A THE PERSON A	Acres and the second	Managery 19	Lane Market	A STREET, STRE	Water Vision	-	A-manny	-

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	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	ТҮРЕ	FLOW	PRESSURE	TEMP IN	НХБР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in im)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpin cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
												
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20 -												
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				ELE	CTRI	CAL	POWI	ER, W						S. Salar
INSTALLATION DATA														
EQUIPMENT	IS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ш	LOCATION	WEIGHT	HEIGHT	WIDTH	рертн
	SWBS	σΩ	115	115	450	115	115	ည္ထ	TYPE	700	lbs (N)	in (mm)	in (mm)	in: (ma
VHF Bridge-to-Bridge Group	G													
Transceiver, VHF AN/URC-80	GI	1				50				PILAT House	133	152	9 229	18 45
Control C-8980/URC	G2	1					**************************************			PILOT HOUSE	3			
Antenna AS-2809/SRC	43	2									7.			
Transceiver, VHF AN/URC-86	G 4	١				50				PILOT House	178			
Wideband Secure Voice														
Group Secure Voice	Н													
SW SA-2112(V)/STQ	Hi	1								COMM	267	12 305	19 483	
Ampi Speaker AN-4453/U	H2	٩								10 1 1 5 1 5 1 5 1 5 1 5 1 5 1 5 1 5 1 5	21 93	8 203	13 330	9 2 2
Sec Tel Set TA-840/U	нз	ی								SULTU CORT	4			0 d
Plain Cipher SW SA-1711/UR	Н4	5			,					comm	4			
Interconnect Box J-2910/UR	Н5	2								Comes	2			est collection
RCS Remote Channel Sel	H U	10				•				comm				ing the two or
Analog - Digital Model CV-3333	ΗΊ	ı												No. of Land
Signal Conditioning	1													A STATE OF THE PARTY IN
Telegraph Signal Converter CV-2460/SGC	<u> 31</u>	2	20 40							COMM	25 III			
Telegraph Conv/Comparator AN/URA-170	12	1	70							Comm	65 289			
Telegraph Mux Terminal AN/UCC-1D(V)	JЗ	1	154							Comno	144			
Telegraph MUX Terminal AN/UCC-1D T/R	J 4	١	154			•				comm	144			

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DEOTU	HEAT DISSIPA	(AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХБР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
ir (m)		N	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	1t ³ /mir m ³ /s	NOTES
18 45		0		-									

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365	>	4											
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INSTALLATION DATA			2	2	Z	Hz	Hz				GHT	SHT	표	E
EQUIPMENT	SWBS NO.	DUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		TYPE	LOCATION	WEIGHT	НЕІСНТ	WIDTH	. ОЕРТН
	MS	οC	1	11	45	-1	11	ည္ရ	7	70	lbs (N)	in (mm)	in (mm)	in (mm)
Miscellaneous	K	-11-0												
XMTR XFR SWBD SB-2744/SRT(Mod)	ĶΙ	3								COMM	74 329			
Rcvr XFR SWBD \$B-2727/SRR	K?	ž								соми	196			
Low Level DC XFR SWBD SB-2727/SRR/Mod)	K3 _	3								сомм	196			
Low Level Power Supply (±6VDC) OP-94S	K4	3								СОММ	75			
Low Level Power Dist PNI	K5	3								сомм	50 222			
XFR SWBD SB-3195/U	Ke									сомм	11 49			
XFR SWBD SB-1039	K٦	1								сомм	19			
Freq Std Amecon, CTFS	KB.	1				15				COMM	23			
RF Dist Amp 1 AM-2123A/U	K۶					100	-			COMM	16			
Alarm, Freq Std	K10	1			-		*			сомм	3			
Guard Rovr	KII	1								сомы	100			
Antenna CA-1128-1	K12	1									120		 	
Audio Ampl/Spkr AM-4453/U	K13	3								СОММ	21			
Time Stamp NA-18BG	KIY.									сомм	2 9			
Duplicator E-141U	KIS	1								сомм	60			
AM Ampl AM-6694/SR										COMM	30	 	 	
Altrodayon	KIL						 				133	-	-	1
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ОЕРТН	HE集T DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХБР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in (mm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
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INSTALLATION DATA						·								
EQUIPMENT	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		Ш	LOCATION	WEIGHT	неіснт	WIDTH	DEPTH
EGOIFMENT	SWB	auA	115\	115	450\	115\	115\	၁၀	TYPE	007	lbs (N)	in (mm)	in (mm)	in (mm)
Eqpt Cabinet	K17	10								сомм	175			
CW Keyer SB-315/U	K18	ı								сомм				
Radio Relay Cont C-4621/SR	K19	1								сомм	6 27			
Switch SA-734/SG	Keo	3								COMM	4			
Indicator ID-866/SG	K21	3												Alley Medical
Antenna AS-3025/SRG	K22	,												
														ALIEN DAME
					,									
MF RCVR GP	L													
LF/MF RCVR AN/WRR-3B	LI	1	60							RADIO KMTR RM	70 311			
Antenna Group AN/SQA-17D	L2	1									222			
Filter AN/SRA-12B	LB	1												
														##

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	DEPTH	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
	in (mm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
State of the state													
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		٠		ELE	CTRI	CAL	POWE	R, W	-					-
INSTALLATION DATA		-												
501115115	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz			LOCATION	WEIGHT	неіснт	WIDTH	ОЕРТН
EQUIPMENT	SWBS	QUA	115V	115V	450V	115V	115V	OC	TYPE	7007	lbs (N)	in (mm)	in (mm)	in (mm)
Underwater Systems	442													
UW Tel GP AN/WQL-2	<u>A</u>													
Revr - XMTR RT-876/WQC-2	Δı	-	3450							SONAR	345 1535			21 533
Comm Set C-7440/WQC-2	A 2	1								CIC	10 45	8 203	14 356	10 254
Comm Set C-7441/WQC-2	А3	-								PILOT HSE	6 27	8 203	9 229	8 203
XDCR TR-232/WQC-2	Δ4	4								SIDEWALL	440 1957	13 330	22 559	DIA
XDCR TR-233/WQC-2	A5	_								SIDE WALL	100 445	18 457	12 305	DIA DIA
										-				,
Visual & Audible Gp	443													
Signal Flags	A	1								02 LVL	150 667			
Whistle	В	1								PILOT HSE	50 222			
Signal Search Lights (12")	c	2								02 LYL	25 111			one growth and the
	4 / 4													
Teletype Eqt	445 A													
Teleprinter TT-624/UG	AI	1	350							Сомм	265			
Teletype Set AN/UGC-48A	A2	3	240								267	39 991	36 914	24 610
Teletype Set AN/UGC-77	Αz	,	80								52 231			
PAGE PRINTER AN/UGR-9	A4	2	120								40	10 254	17 432	12
PAGEPRINTER SET AN/UGR-10	A5	,	375								576 2562	72 1829	22 559	28
PERFORATOR TI-605/UG	A6	1	27								46 205	11	17	305
								-						7

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		ED	ED.	C	OOLI	NG V	VATE	R	HY	DR	Al	R	
WICH	рертн	HEAT DISSIPAT (AIR)	HEAT DISSIPATED (WATER)	ТҮРЕ	FLOW	PRESSURE	TEMP IN	HXDP	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
- 1	in mm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s		ft ³ /min m ³ /s	NOTES
	21												
4	33 10 254												Powered FM RT-876
,	8 203												Powered FM RT-876
2 1	DIA												
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	30 <u>5</u> 28	80		 	-		-			-		-	
4	711	264		-	-	-					-	-	
2	305	80		-	-	-	<u> </u>	 	-	-	 	-	
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INSTALLATION DATA														
EQUIPMENT	SWBS NO.	QUANTITY	115V-10·60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ш	LOCATION	WEIGHT	HEIGHT	WIDTH	DEPTH
	SWB	σΩ¢	115	115	450	115	115	၁၀	TYPE	רסנ	lbs (N)	in (mm)	ia (mm)	in (mm)
Security Equipment TSEC/KG-36-4	446 A1	1								comm	120			o de chailte de chai
TSEC/KY-8										comm	73 352			200
KYB-G/TSEC	Δ2	2								comm	36 160			,
HYP-2/TSEC P.S.	SA	2	130							COMM	37 165			
TSEC/KWR-37	Δ4	2	320							Comm	149 663			
TSEC/KG-14	Δ5	4	100							COMM	130 578			
TSEC/KW-7	46	4	85							Comm	74 329			
Plant Adapter KWX-11/TSEC	Δ7	4								comm	20			
Remote Fitn KWX-8/TSEC	84	3								conn	3	5	7 178	152
TSEC/KL-47	49	1	200							Comm	30			
TSEC/HL-1B	A10	1	350							Comm	50 222			
Socurity Equipment TSEC/KG-40	A 1)	1				60				درو	37 165	10 254	9 229	22 559
Remote Control KGX-40/TSEC	Δ12	1								CIC	4	5	G 152	8 203
Security Equipment OK-313(V)SC CSS	∆ 13	1								Comm				
Type 8 Safe	Δ14	1								Comm	175			
Int Comp KIR-1A/TSEC	Δ 15	2	·											
Trans Comp KIT-1A/TSEC	Δ16	1												
Crypto Key KIK-18/TSEC	۵17	1												

		Maritana and A	attende editing Chip.		Consider Consider	interestination and the	en de mariantes.		Charles and the second			Sparinger Sparinger	
King ayar dik		ED	ED	C	OOL	NG V	VATE	R	НΥ	DR	Αl	R	
WIDTH	рертн	HEAT DISSIPAT (AIR)	HEAT DISSIPATED (WATER)	ТҮРЕ	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in (mm)	in (mm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
		34											
		400											26.5 VDC FM HYP-2 80 Watts Ea.
										ļ			
		320											
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INSTALLATION DATA															DISSIPATED
	NO.	ΤΥ	-60Hz	-60Hz	-60Hz	115V-10-400Hz	115V-30-400Hz			NO:	WEIGHT	HEIGHT	WIDTH		HEAT D
EQUIPMENT	SWBS N	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10	115V-3(DC	TYPE	LOCATION	ibs (N)	in (mm)	in (mm)	in (mm)	V
Surveillance System (Surface)	450														
Surf Search Radar AN/SPS-55	451		,							0716 (1/05 5	195	27	78	1 છે.	
Antenna Assembly OE-172/SPS-55	1	1								83'-6" (25.5 k) PLTF 83'-6" (25.5 m)	867			457	
Antenna AS-2953/SPS-55	1. 1	1								PLTF 83'-6" (25,5m)	195 867 195				_
Antenna Pedestal AB-124/SPS-55	1.2	L								83-6 (25,5m	867				_
RADAR SET CONTROL C-9447/SPS-55	2	1								CIC	25		19 483		3
Rcvr - Transmitter 1124/SPS-55	3	1	1150								560 5491	73 1854	29 737	26 660	5
Safety SW SA-1963/SPS-55	4	1								83'-6' (255m) PLTF	5 22	7	7 178	5 127	
Sync. Amp MK 27-8	5	1								radar Egpi km	65 289	15 381	20 508	10 254	3
Air Search Radar AN/APA-171	452						28 Kv					JK 14	1		6
Antenna Group AN/APA-171	i	1								Pole MAST	3800 169k	¥ 3·3.	7.5) DIA	
Radar Set AN/APA-125	2	1								RADAR EQPT RM					
Radar Control Panel	3	1								cic	7.5 33	279	152	152	
Perf Monit AN/ASM-440	4	1								Radar Erpt Ran					
Liquid Cooling Hx	5	,								RODAR EAPT RM					
Electric Cooling Subsystem	6	1								RADAR EOPT RM					(A)
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INSTALLATION DATA														.	i
EQUIPMENT	s NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		Ш	LOCATION	WEIGHT	неіснт	WIDTH	DEPTH	
EUUIFWENT	SWBS	QUA	115V	115V	4501	115\	1151	oa	TYPE	700	lbs (N)	in (mm)	in (mm)	in (mm)	_
Ident Syst (IFF)	455														-
Interrogator AN/UPX-25(V)	1	5	850							EER		63 1600		686	•
Transponder AN/UPX-28(V)	2	1	300							EEK	99			19 483	7
Antenna AS-1778/UPX	3	4								93'-6" (28.5m) PLTF	7 31	20 508	7	7	_
Test Set AN/UPM-137A	4	1	285							EER	190 845	23 584	32 813	2 <i>Z</i> 559	2
Control Monitor C-8430/UPX	5	2	17 34							EEK	12 53	8 203	10 254	8 203	1
Decoder Group AN/UPA-59A (V)	G	3	75							CIC.	32 142				
Video Decoder KY-761(P)/UPA-59A(V)	ا،ی	3	75							CIC	32 142	12 305	6 152	17 432	
Intra-Target Data ind. ID-1844/UPA-59A(V)	6.2	3	75								32 142				
Alarm Monitor BZ-173/UPA-59A(V)	6.3	3	75							CIC	32 142	7 178	5 127	3 76	
Cabinet CY-6816/APX-72 -	7	,								EER		10 254	8 203	7178	
XMTR Set Cont C-6280/APX	7.1	4								EER	ş 36				
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MEAT DISSIPA (A(R)	HEAT DISSIPATED	1/16	110713	をなるのとであ	31.77.11	40,714		£.0#	PRESSURE	VOLUME/FLOW	
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INSTALLATION DATA														e de la constitución de la const	
	NO.	TY	-60Hz	-60Hz	-60Hz	-400Hz	-400Hz			NO	WEIGHT	HEIGHT	WIDTH	ОЕРТН	
EQUIPMENT	SWBS N	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz	DC	TYPE	LOCATION	ibs (N)	in (mm)	in (mm)	in (mm)	
Passive Sonar	462														
Tactical Towed Array Sonar	۵		·												
Winch, Array, Cable	۱۵	1								ETAS RM	15570 69.36		96 2438	80 2032	
Levelwind, Fairlead, Bellows	42	١								etas Rm	1300 5782	12 305	96 2438	24 610	
Winch, Hydraulic Power	A 3)			55 K						1700 7562		24	1524	
Winch Control	Δ4	}								ETAS CONTROL RM	1000 4448	48 1219	24	24	
Module Fill Station	45	1								etas Rm	300 1334	30 762	24	18 457	
Storage Trough	16	,					,			ETAS	2669		7	480	The state of
Transformer 60Hz	Δ7	3	50							ETAS RM	110	17 432	9 229	8 203	
Power Dist Cab	48	1	270							ETAS RN1	175 7 78	36 914	24	152	NAME OF THE OWNER, OF
Input Signal Conditioner (ISC)	Δ9	1								EEK	600 2669		26	23 584	
Advanced Signal Processor (ASP)	410	_								EEK	214 952	57 1448	21	18	
Acoustic Computer AN/UYQ-L	AII	ı								EEG	220 979	20 508	19 483	31 787	
Display Console AN/UYQ-L	Δ12	1				4300				CIC	950 4226	1549	29 737	40	
Data Convecter AN/UYQ-L	Δ13	1				4300				EEK	950 4226	72 1829	26	23 584	
Transformer 400 Hz	Δ14	3				218 654				TACTASS	30	152	8	7	Carlos Bearing
Acoustic Sensor Data Dist SWBD	415	1													The second
															S. Carlotte
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War and Mark	TED	C	OOL	NG V	VATE	R	НҮ	DR	Αl	R	•
	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХDР	PRESSURE	FLOW	PRESSURE	VOLUME/F! OW	
w	w		gpm (cm³/s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
2. 0 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1.		,									
											208√30' WYE Type I 60 Hz 3500 W
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				ELE	CTRI	CAL	POWE	R, W						egidzke i ûn die
INSTALLATION DATA	NO.	TITY	1'i5V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz	-		TION	WEIGHT	неіснт	WIDTH	DEРТН
EQUIPMENT	SWBS NO.	QUANTITY	115V-1	115V-3	450V-3	115V-	115V-:	DC	TYPE	LOCATION	lbs (N)	in (mm)	in (mm)	in (mm)
Active Passive Sonar	463													
Active Dipping Sonar 13-D-AN/AOS-12D	Δ													
Revr Indic Assy Revr R-1695/AQS-13B ndic IP-1045/AQS-13B	۵1	1				25				CIC	79 351	25 635	14 356	23 584
Multiplex Processor Assembly	A 2	1								SOHER	43 191	8 203	9 227	23 584
Bearing & Rge Indic Assembly ID-1751/AWS-13	ΔS	1								CIC	2			
Recorder Assembly RO-358/AQS-13A	44	1								CIG	33 147	17 432	9 227	13 330
Dome Control Assembly	45	1								SONAR	5 22	5 127	5 127	7 178
Launch, Reel & Cable Assembly	AG	1_					4500			50~AR	213 947	36 914	44	26
Transducer Assembly	۵7	1					1000			SONAR	201 894	1219	15 381	016
XDCR HSG & Funnel Assembly	84	1								SOLAR	41 182			
XDCR Cable Attitude Ind.	49	1												
APRAPS (Space & Weight)	В	1								50-AR)	9375 41·7 k			
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		TED	TED		COOL	ING	WATI	ER	НΥ	/DR	A	IR.	•
	H	DISSIPA (AIR)	HEAT DISSIPATED (WATER)			PRESSURE	Z		PRESSURE		PRESSURE	VOLUME/FLOW	
The state of the s	DEPTH	HEAT	HEAT	TYPE	FLOW	PRES	TEMP IN	НХОР	PRES	FLOW	PRES	VOLUM	
	in (mm)	w	w		gpm (cm ³ /s	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
	0.3	10.5											
	23 584	135											
	23 584												28vdc 10a
		9								-			Contains RS & Beam Forming Network
	13 330	108											
1	7 .178												
100	660	4500								-			B/U dc Motor for Winch 80a, 28vdc 12 757cm ^{3/5} gpm 2550 19.6MPa psig motor for winch
F	10												
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Cineciliention Sound AASP Sound Hear Sol AN/Hith (Milit)	4-4	•	āpau				-							
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DEPTH	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in imm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
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INSTALLATION DATA														
	NO.	TITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz			LOCATION	WEIGHT	неіснт	WIDTH	ОЕРТН
EQUIPMENT	SWBS	QUANTITY	115V-	115V-	450V-	115V-	115V-	DC	TYPE	LOCA	lbs (N)	in (mm)	in (mm)	in (mm)
PASSIVE ECM DESIGN-TO-PRICE WARFARE SYSTEM AN/SLO-3IV(2)	472 A													
RF ANTENNA/RECEIVER SUBSYSTEM (MID/HIGH BAND)	ΑI	l	1200				8600			DIST	3045 13.5	*		4000
RF ANTENNA/RECEIVER SUBSYSTEM (LOW BAND)	A2	ı												elain eigh
IR SENSOR SUBSYSTEM	A3	ı												
AUTO SIGNAL PROCESSOR	A4	1								RER				
COMPUTER AN/UYK-20	A5	1								REK				88 97 Page 18
GRAPHIC DISPLAY UNIT WITH OPERATOR CONTROL	ALO	1								CIC				
OFF BOARD DECOY LAUNCHER	A7_	2												
BLANKER AN/SLA-10	В	1	170							RER				
BLANKER - VIDEO MIXER MX-7544	BI													
BLANKING DISABLE CONTROL C-7132	52	1												
RADIO FREQUENCY SWITCH SA-1512 (60 MC)	63	1									ļ			
RADIO FREQUENCY SWITCH SA-1513 (160 MC)	В4	1												
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	4TEC	4 T E C		OOL	ING V	VAIE	H	HY	DK			
	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in mm)	w	w	· '	gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
mm)												
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INSTALLATION DATA															DISSIPATED
EQUIPMENT	3S NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ďπ	LOCATION	WEIGHT	неіснт	WIDTH		HEAT DI
	SWBS	o O	115	115	450	115	115	വ	TYPE	ĽŎ	lbs (N)	in (mm)	in (mm)	in (mm)	L
FIRE CONTROL SYSTEM (NON-SONAR DATA BASE)	482	2394													L
FCS MK92-3	Α	1		i!		I!									
COMBINED ANTENNA SYSTEM MK53/I	Αı	ı	300		40cc						1650 7339	131 3327	96 2438	AIQ AIQ	
MAN ALOFT SWITCH MKI42/0	A2	ı									5 22	-	7 178	5 203	Ī
WAVE GUIDE DRYER MKI3 MOD 0	A 3	,	1350							ВРВ	175	25 635	78 711	28	,
RADAR XMTR T-1085/SPG-51D	A4		600					50 V 30 V	D¢ AC	RPR	1150	78 1981	45	33 838	
POWER SUPPLY CONTROL C-7714/SPG-5ID	A5	,	800				1	50V 130V	DC	RPR	925 4114	78 1981	31 787	28	4
RADIO FREQUENCY AMPLIFIER (CAS) MKI67/0	A6	1	250				4200			RPR	940 4181	1781 66 1676	36	32 813	1
RADAR RECEIVER - XMTR (CAS) MK69/I	A7	ı	300				5000				1100	66	36	32 813	Ī
RADAR TEST SET MK574/0	A8		400							RPR	100	18	22	12 305	,
SERVO CONT' CAB' MKI60/I	A9		700				1800			RPR	2000	77	66	29	İ
WEAPON CONTROL CONSOLE (CAS) MKI06/I	A10	,	900				1800				1200 5338	1	40	45	Ī
GAS REGULATOR MK2I/0	AII	,	700				1000			RPR	77 342	22	51	1143 16 406	T
WAVE GUIDE SW NAVIGATION SWITCH	۸.۵										75	9	7	15	
AND DUMMY LOAD MK143/0 CHANNEL SET SWITCH MK4/I	A13		800							RPR	33 <i>4</i> 85	36	26	14	-
COMPUTER AN/UYK-7(V)	A14						2700			DPC	378 535	914	20	23	T
COMPUTER CONTROL C-8542/UYK-7(V)							2700			DPC	2380	6	7	7	F
TEST SET TS-2940/UYK-7(V)	A15						3				35	152	178	178	-
I/O COMS (DEAC) OJ-172/UYK-7(V)	A 17	4	486				1900			DPC	470	483 63	30	34	t
U3-172/UTN-7(V)	~ '	-	400							DPC	4315	1600	764	864	13

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	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	HXDP .	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW		
m)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES	٩.
in the second second												7	
										3.0	-510		
A A 3 03										20.7			
03												,	
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3 38	1200	23500	рW	5.5 3470	115 793	30							
8 	4400										.015	CROSS FIELD AMPLIFIER	
8 11 2 13		4000	cw	3.1 195.6		10				3.0 20.7	1 1		
13		3200	cw	2.5 157.7	150	10				3.0 20.7	1 1	RADAR CABINET	
_	140							ļ					
-	3600												
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16 06													-
S 81		1000	200	63.1	1034	10				<u> </u>			-
14 56	640	,					<u> </u>						4
23 84	2290	,						<u> </u>					7
78	3												
ر. 52	160						ļ						-
84 64	2386	2											
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				ELE	CTRI	CAL	POWE	ER, W					-1-1	
INSTALLATION														
DATA						N.I	N			•	L.	17	_	,
		>	30Hz	30Hz	30Hz	100t	100t			Z	WEIGHT	неіснт	МІ РТН	DEPTH
EQUIPMENT	S NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ш	LOCATION	>	-1-	×.	. (ب)
	SWBS	OU/	115	115	450	115	115	DC	TYPE	700	lbs (N)	in (mm)	in (mm)	in (mm
STIR (SPACE & WEIGHT)	В													
RADAR ANTENNA MK54/0	B)	1					360				4060 18100			
SAFETY SWITCH MKI42/0	82										5 22			e e e e e e e e e e e e e e e e e e e
WAVEGUIDE DRYER MKI3/0	В3		1000								165 734			200
WAVEGUIDE SWITCH ASSEMBLY MKI43/0	B4	١									25 111			1 1 1 1 1 1 1 1 1 1 1
RADIO FREQUENCY AMPL. MKI68/0	B5	١	900				4200				1100			100
RADAR XMTR T-1085/SPG-51D	ВС	1	600					500ck			1150 5115			PROPERTY AND PROPERTY OF THE P
POWER SUPPLY CONTROL C-7714/SPG-5ID	В7	ı	800					Sorde 130	H		925			Section 2
RADAR XMTR MK86/0	ಶಿಶಿ	1	300				3200				1100			
RADAR TEST SET MK574/0	B9	1	400								100			
ANTENNA CONTROL MKI6I/0	B10	1					1140		I		500			
STIR WCC MKI07	Вп	-	900				1800				1250 5560			*
GAS REG MK21/0	B 12	1									100			
,														, contraction
HARPOON FCS	c													
HARPOON WCC	CI	1					29∞	28 vdc		cic	600		20 508	39
LAUNCHER SWU	cz	١	150				·			EER	220	15	30	18
MISSILE SIM AN/DSM-13(XN-I)	c 3	1								EER	294	160	22	10
HARPOON ENGAGEMENT COURSE INDICATOR	c4	1								PH	20			
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WIDTH	рертн	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	ТУРЕ	FLOW	PRESSURE	TEMP IN	НХДР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in (mm)	in (mm)	w	w]	gpm (cm ³ /s)	psi) (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s		ft ³ /min m ³ /s	
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				ELE	CTRI	CAL	POWE						LICAT DISCIDATED		
INSTALLATION DATA															200
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		>	30Hz	0Hz	30Hz	HOOt	HOOt			20	WEIGHT	неіснт	WIDTH	DEPTH	4
FOLUDIATION	NO.	I I	-10-6	-30-6	-30-6	-10-	-30-		•••	LOCATION	×	-T	5		3
EQUIPMENT	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz	DC	TYPE	LOC	lbs	in	in	in	
				•	,				-		(N)	(mm)	(mm)	(mm)	_
FCS (SONAR DATA BASE)	483										2				
TORPEDO FC PANEL MK309	Δ	1	1760								300 1334				ı.
MK48 WFCS (SPACE & WEIGHT)	В	1													
MK8I CONSOLE	B)	1													
FIRING PANEL	B2	١													
MK82 CONSOLE	83	١													
MKI40 CONSOLE	В4	1	·												
MK22 WEAPON SIM.	85														

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	рертн	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХDР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
m)	in (mm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
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		1700											
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				ELE	CTRI	CAL	POW	R, W						
INSTALLATION DATA						ZĮ	z				нТ	нт	H	arrays also arrays
EQUIPMENT	S NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		ш	LOCATION	WEIGHT	невент	WIDTH	3 Free DEPTH Contrasts depart on the Contrast of the Contrast
	SWBS	γno	115	115	450	115	115	20	TYPE	700	lbs (N)	in (mm)	in (mm)	in (mn
ELECTRONIC TEST MONITORING & CHECKOUT EQUIPMENT	491										200 891			
MARTIN MARTIN STATE AND AND AND AND AND AND AND AND AND AND														
FLIGHT CONTROL & INSTALLATION LANDING SYSTEM	492													
AN/SRN-() TACAN	۵	1												Hackback
ANTENNA	۵I	1									100	10 254	1320	
TRANSPONDER	Δ2	1	2000								5e o 2224			٥ <u>ن</u>
ANTENNA CONTROL	43	1									100	21 534	19 483	ž Š
STATUS INDICATOR	4	4									46	8 203	10 254	
														Short selection of the
COMMAN & SURVEILLANCE OPERATING FLUIDS	498										25			
		·												Hereital History
COMMAND & SURVEILLANCE REPAIR PARTS & TOOLS	499										4448			7700000
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			TED	TE0		COOL	ING	WATI	ER	Н	/DR		AIR.	·
		DEPTH	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in bn		in mm)	W	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kDa)	1t ³ /mi m ³ /s	
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INSTALLATION														Ì
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		<u>></u>	30H2	30H2	30H2	1001	400			Z	WEIGHT	HEIGHT	WIDTH	DEPTH
50,000	N	LI LI	-10-6	-30-6	-30-(.10-	-30-			ATI(5		\$	
EQUIPMENT	SWBS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz	DC	TYPE	LOCATION	lbs	in	in	in
	S		1	1	4	-	-	<u> </u>	- 		(N)	(mm)	(mm)	(mm)
LAUNCHING DEVICES	721													
VERTICAL MISSILE Launcher	A													
LAUNCHER CANISTER	Δ1	కి								VERT MISL TRUNK	1150 5115	i	1	30
BLAST DEFLECTOR										VERT MISL	850	1	1.0 1	762
SEAST DEFLECTOR	A 2	පි								Rooms ELEV EAPT	3781	38	18	30
MISSILE/LAUNCHER SEL.	Δ3	1	250							Room	890	1	457	762
HARPOON CANISTER LAUNCHER	υ						11,220							
CANISTER MK6/0	ಕಾ ।	පි								OILYL	443			,
HARNESS ASSEMBLY	₽2	8								OILVL	100			20,000
LAUNCHER RELAY ASSEMBLY	ರಿಕಿ	4								OILVL	48			
SUPPORT ASSEMBLY MKI40/0	84	4								OILVL	625			1,500
HOIST/ROTATION BEAM ASSEMBLY	B 5	١												100
TOOL KIT	B6	1												
														411,000
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DEPTH	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	HXDP	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in (mm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
उठ												
767												
	235	-										
762												
												MISSILE WARMUP - 910 WATTS PER MISSILE MISSILE ELECTRIC POWER 985 WATTS PER MISSILE
		-										
												8000 WATTS 60 Hz II,220 WATTS 400 Hz
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	-			_			-					
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				ELE	CTRI	CAL	POWE	R, W							TED
INSTALLATION DATA	NO.	TITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz			LOCATION	WEIGHT	неіснт	WIDTH	ОЕРТН	HEAT DISSIPATED
EQUIPMENT	SWBS NO.	QUANTITY	115V-	115V.	450V-:	115V-	115V-	DC	TYPE	LOCA	lbs (N)	in (mm)	in (mm)	in (mm)	w
GUNS	711	_													
CLOSE-IN-WEAPON SYSTEM	Δ	2	_												
LAUNCHING DEVICES	72I														
HARPOON CANISTER LAUNCHER	Δ						11,220								
LAUNCHER CANISTER MK6/0	ΔΙ	ප								OI LVL	443 1970				
MISSILE CANISTER LAUNCHER MKI40/0	42	4								OI LVL	673 2993				-
HOIST ROTATION BEAM ASSEMBLY	A 3	1									25 111				
TOOL KIT	44	1									25				
HARNESS ASSEMBLY	A5	8								OILVL	100 445				
VERTICAL MISSILE LAUNCHER	В														est fait the second second
LAUNCH CANISTER	ВІ	8								LIFT FAN	1150 5115	•	31 787	30 762	Assessed Self-
BLAST DEFLECTOR	82	8								LIET FAN	850				Spirotest conta
MISSILE/LAUNCHER SET	В3	1	250							EER	200 890		18 457	30 762	235
															E. Shalenania
			 												24 mm 45 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55 mm 55
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Martin French	ED	(COOL	NG V	VATE	R	НΥ	DR	Al	R	
Springer de la Company de La C	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	нхор	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
mer.	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpin cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
1											
											MISSILE WARMUP 910 WATTS PER MISSILE
											MISSILE ELECTRIC POWER 785 WATTS PER MISSILE
											HEATER POWER 60 \(\Omega\) 400 Hz
											INCLUDE 4 LAUNCHER RELAY ASSEMBLIES
	è										
									1		
	سيسط	سسبال	اسالناكش	عراب عراب	يرادينه أي	المستحداث	decom	-tra-		سياس	

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\				ELE	CTRIC	CAL	POW	ER, W						State Section 1
INSTALLATION DATA														September 200
						7 1	12				H	H	Ŧ	
	.	<u>}</u>	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz			NO O	WEIGHT	HEIGHT	WIDTH	6
EQUIPMENT	SWBS NO.	QUANTITY	٧٠١٥-	V-30-	v-30-	V-10	V-30		Μ̈́	LOCATION				
	SWE	7No	115	115	450'	115	115	2	TYPE	LO(lbs (N)	in (mm)	in (mm)	(eq
TORPEDO TUBES	751													
TORPEDO TUBE MK32	Δ	2		5700 11400						TOPSIDE	2300 10.2 KA			20 mm
TORPEDO TUBE MK25 (SPACE & WEIGHT)	В	2												
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	WIDTH	рертн	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	HXDP	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
	in (mm)	in (mm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	Psi (kPa)	gpm cm ³ /s		ft [©] , mir m ³ /s	NOTES
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				ELE	CTRI	CAL	POWE	ĒR, W]	1	
INSTALLATION DATA												
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EQUIPMENT	S NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		l w	LOCATION	WEIGHT	неіснт
EGOIFMENT	SWBS	aUA	1150	115	450v	115	115\	DC	TYPE	7007	lbs (N)	in (mm)
TORPEDO ACCESSORIES	761 13							'		1		
PRESETTER MK437/0 TEST SET	13-1	1								HELO	25	
TORPEDO STARBOARD IN CONTAINER MK3I/0	13.2	2									30 133	
SUSPENSION BAND IN CARTON (SET)	13.3	2									14 62	
EXHAUST VALVE ASSEMBLY LD620105 TW88	13.4	2									4	
ARMING WIRING MK2/I	13.5										17	
NOSE GUARD	13.6	S									9 40	
PROP GUARD	13.7	S									2 9	
MK46 RECORD BOOK	13.8	20									1 4	
700/S AND ACCEL. SET MK46	13.9	1									25	
SMALL ARMS & PYRO STOWAGE	763											
PYRO MAGAZINES	Ĺ	3										
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i)	in (mm)	w	W		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s	psi (kPa)	ft ³ /min m ³ /s	NOTES
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INSTALLATION DATA												1 _		
EQUIPMENT	SS NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		E	LOCATION	WEIGHT	неіснт	WIDTH	DEPTH
	SWBS	on/	115	115)	450	115	115	20	TYPE	ο̈́	fbs (N)	in (mm)	in (mm)	in (mm)
AIRCRAFT WEAPONS HANDLING	782													
OVERHEAD ELECTRIC HOIST/TRACK	1	1									400			
MK45/I HANDLIFT TRUCK	2	2								TORP RM	128 569			
MK24/0 HANDLING DOLLY	3	1								MK 4C TORP MAG		•	22 559	9 4 23 88
MK28/I HANDLIFT TRUCK ADAPTOR	4	2								MK 46 TORP MAG	22 98	10 254	406	4 102
MK99/0 WEAPONS HANDLING SLING	5	1								MK 46 TORP MAG	120			Right Walder
TORPEDO SLING MKI02/0	Co	2								MK AG TORP MACT	18			with the section of t
TORPEDO SLING MKI06/0	7	2								TORIS MAG	4			
AIRCRAFT WEAPONS STOWAGE	783													
STOWAGE CHOCKS		40 5E75								TORP	120 534	ž .		
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MIGIW	DЕРТН	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	
in in)	in (mm)	w	w		gpm (cm ³ /s)	psi (kPa)	°c	psig (kPa)	psi (kPa)	gpm cm ³ /s		ft ³ /min m ³ /s	
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9 37	22 559								'				
59	94 2388 4												
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INSTALLATION DATA											J	L		
SOLUBNISHT	S NO.	QUANTITY	115V-10-60Hz	115V-30-60Hz	450V-30-60Hz	115V-10-400Hz	115V-30-400Hz		848	LOCATION	WEIGHT	неіснт	WIDTH	рертн
EQUIPMENT	SWBS	ana	115V	115V	450V	115	115	ည	TYPE	гос	lbs (N)	in (mm)	in (mm)	in (mm)
SHIP AMMUNITION	F 2)													
HARPOON MISSILE RGM-84-I	Δ	8								OI LVL	1530 6800			
STANDARD MISSILE RIM-66B	В	පි									1373 6100			
SMALL ARMS AMMUNITION	c	-								LCKR	935 4700			
BT PROBES DC-14/SSQ-56	۵	సెంం								TOCTAS EQPT RM	2			
TORPEDO MK46-I	E	g									53 <i>c</i> 2357	5		
TORPEDO MK48	П	ر									3600			
CLOSE-IN-WEAPONS SYSTEM AMMUNITION	ß										4000 17800			
ORDNANCE DELIVERY SYSTEM AMMUNITION	F22													
TORPEDO MK46-I	A	34									530 2357			
DICASS AN/SSQ-62	В	50									39			
DIFAR AN/SSQ-53	c	80								·	129			
,														
ORDNANCE DELIVERY SYSTEM	F 23													
HELO SH-3H	A	2									15000 66720			
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of the state of th	HEAT DISSIPATED (AIR)	HEAT DISSIPATED (WATER)	TYPE	FLOW	PRESSURE	TEMP IN	НХОР	PRESSURE	FLOW	PRESSURE	VOLUME/FLOW	·
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